

Designation: D 5302 – 01a

Standard Test Method for Evaluation of Automotive Engine Oils for Inhibition of Deposit Formation and Wear in a Spark-Ignition Internal Combustion Engine Fueled with Gasoline and Operated Under Low-Temperature, Light-Duty Conditions¹

This standard is issued under the fixed designation D 5302; the number immediately following the designation indicates the year of original adoption or, in the case of revision, the year of last revision. A number in parentheses indicates the year of last reapproval. A superscript epsilon (ϵ) indicates an editorial change since the last revision or reapproval.

1. Scope

1.1 This test method is commonly referred to as the Sequence VE test² and has been correlated with vehicles used in stop-and-go service prior to 1988, particularly with regard to sludge and varnish formation and valve train wear.³ It is one of the test methods required to evaluate oils intended to satisfy the API SJ performance category.

1.2 The values stated in either inch-pound units or SI units are to be regarded separately as the standard. Within the text, the SI units are shown in parentheses when combined with inch-pound units. Some of the figures and forms have identical numerical designations, but with the letter M following the numerical designation: these are alternative figures and forms that contain SI units.

1.3 This standard does not purport to address all of the safety concerns, if any, associated with its use. It is the responsibility of the user of this standard to establish appropriate safety and health practices and determine the applicability of regulatory limitations prior to use. Specific precautionary statements are given in 7.7, 7.8.9.6, 7.8.13, 7.10.2, 7.10.3.2(c), 8.3.4.2, 8.4.4.3, 8.4.5.2, 9.3.4.4, 9.6.1.2, 12.1.1.5, 12.1.4.5, 12.2.1, 12.2.1.5, Fig. A3.6, and Annex A8.

1.4 Table of Contents:

	Section
Scope	1
Referenced Documents	2
Terminology	3
Summary of Test Method	4

¹ This test method is under the jurisdiction of ASTM Committee D02 on Petroleum Products and Lubricants and is the direct responsibility of Subcommittee D02.B0.01 on Passenger Car Engine Oils.

Current edition approved Nov. 10, 2001. Published January 2002. Originally published as D 5302 – 95. Last previous edition D 5302 – 01.

² Until the next revision of this test method, the ASTM Test Monitoring Center will update changes in the test method by means of information letters. Information letters may be obtained from the ASTM Test Monitoring Center, 6555 Penn Avenue, Pittsburgh, PA 15206-4489. Attention: Administrator. This edition incorporates revisions in all Information Letters through No. 01-1.

³ ASTM Research Report RR:D02:1226. Sequence VE Report to the Subcommittee D02.B Oil Classification Panel, contains the field-laboratory correlation information and the initial laboratory test precision data.

Significance and Use	5
Apparatus—General Description	6
Apparatus—The Test Engine	7
Sequence VE Test Engine Parts Kit	7.1
Required New Engine Parts	7.2
Reusable Engine Parts	7.3
Specially Fabricated Engine Parts	7.4
Special Engine Measurement and Assembly Equipment	7.5
Miscellaneous Engine Components-Preparation	7.6
Solvents and Cleansers Required	7.7
Assembling the Test Engine—Preparations	7.8
Assembling the Test Engine—Installations	7.9
Engine Installation on the Test Stand	7.10
Engine Fluids—Supply/Discharge Systems	8
Intake Air	8.1
Fuel	8.2
Engine Oil	8.3
Coolants	8.4
Measurement Instrumentation	9
Temperatures	9.1
Pressures	9.2
Flow Rates	9.3
Fuel Consumption	9.4
Speed and Load	9.5
Exhaust Gas	9.6
Miscellaneous Laboratory Equipment	10
Test Stand Calibration	11
Verification	11.1
Unacceptable Calibration Tests	11.2
Test Stand Modifications	11.3
Reference Oil Accountability	11.4
Test Numbering System	11.5
Procedure	12
Pre-Test Procedure	12.1
Engine Operating Procedure	12.2
Periodic Measurements and Functions	12.3
Special Maintenance Procedures	12.4
Diagnostic Data Review	12.5
End of Test Procedure	12.6
Interpretation of Results	13
Parts Rating Area—Environment	13.1
Sludge Ratings	13.2
Varnish Ratings	13.3
Clogging	13.4
Sticking	13.5
Wear Measurements	13.6
Assessment of Test Validity	14
Average Exhaust Gas NO _x Levels	14.2
Used Oil Analyses	14.3
Blowby Flow Rate	14.4
Intake Manifold Vacuum	14.5
Fuel Consumption Rate	14.6 14.7
Oil Consumption	14.7

Copyright © ASTM International, 100 Barr Harbor Drive, PO Box C700, West Conshohocken, PA 19428-2959, United States.

Report	15
Report Format	15.1
Special Forms for Automated Data Acquisition	15.2
Standard Report	15.3
Calibration Test Report	15.4
Precision and Bias	16
Precision	16.1
Bias	16.2
ANNEXES	
Special Service Tools for the Test Engine	A1
External Oil Heat Exchanger Cleaning Technique	A2
Detailed Specifications and Photographs of Apparatus	A3
Engine Part Number Listing	A4
Operational Data Log Sheets	A5
Rating Worksheets	A6
Final Report Forms and Photographs	A7
Safety Precautions	A8
Automatic Data Acquisition	A9
Oakite 811 Monitoring Program	A10
Test Precision—Reference Oils	A11
Control Chart Technique for Severity Adjustment	A12
Statistical Equations for Mean and Standard Deviation	A13
Data Dictionary	A14
One-half Millimetre Oversize Piston Modifications	A15
Required Inspections and Operations for Reuse of Cylinder Heads APPENDIXES	A16
Engine Measurement Sheets	X1
Sources of Materials and Information	X2
Analysis of Phillips "J" Fuel (Batch 26)	X3
Theoretical Exhaust Gas Relationships-Phillips "J" Fuel	X4
Description of Scott Quarterly Gas Audit Service	X5

2. Referenced Documents

2.1 ASTM Standards:

D 86 Test Method for Distillation of Petroleum Products⁴

D 287 Test Method for API Gravity of Crude Petroleum and Petroleum Products (Hydrometer Method)⁴

D 323 Test Method for Vapor Pressure of Petroleum Products (Reid Method)⁴

- D 381 Test Method for Existent Gum in Fuels by Jet Evaporation⁴
- D 445 Test Method for Kinematic Viscosity of Transparent and Opaque Liquids (the Calculation of Dynamic Viscosity)⁴
- D 525 Test Method for Oxidation Stability of Gasoline (Induction Period Method)⁴
- D 873 Test Method for Oxidation Stability of Aviation Fuels (Potential Residue Method)⁴

D 893 Test Method for Insolubles in Used Lubricating Oils⁴

- D 1266 Test Method for Sulfur in Petroleum Products (Lamp Method)⁴
- D 3237 Test Method for Lead in Gasoline by Atomic Absorption Spectrometry⁵
- D 3525 Test Method for Gasoline Diluent in Used Gasoline Engine Oils by Gas Chromatography⁵
- D 3606 Test Method for the Determination of Benzene and Toluene in Finished Motor and Aviation Gasoline by Gas Chromatography⁵
- D 4057 Practice for Manual Sampling of Petroleum and Petroleum Products⁵
- D 4175 Terminology Relating to Petroleum, Petroleum Products, and Lubricants⁵
- D 4485 Specification for Performance of Engine Oils⁵

- D 5134 Test Method for Detailed Analysis of Petroleum Naphthas Through n-Nonane by Capillary Gas Chromatography⁵
- G 40 Terminology Relating to Wear and Erosion⁶
- 2.2 SAE Standards:
- SAE J254 Instrumentation and Techniques for Exhaust Gas Emissions Measurement⁷
- 2.3 ANSI Standard:
- MC96.1 Temperature Measurement—Thermocouples⁸
- 2.4 Coordinating Research Council:
- CRC Sludge Rating Manual (CRC Manual No. 12)⁹
- CRC Varnish Rating Manual (CRC Manual No. 14)⁹
- CRC Techniques for Valve Rating (CRC Manual No. 16)⁹

3. Terminology

3.1 Definitions:

3.1.1 *air-fuel ratio*, *n*—*in internal combustion engines*, the mass ratio of air-to-fuel in the mixture being induced into the combustion chambers.

3.1.1.1 *Discussion*—In this test method, air-fuel ratio is controlled indirectly by exhaust gas analysis for CO and O_2 contents. During Stages I and II, O_2 is the primary determinant, while CO is the primary determinant in Stage III.

3.1.2 *blowby*, *n*—*in internal combustion engines*, the combustion products and unburned air/fuel mixture that enter the crankcase.

3.1.3 cold-stuck piston ring, n—in internal combustion engines, one that is stuck when the piston and ring are at room temperature, but inspection shows that it was free during engine operation.

3.1.3.1 *Discussion*—A cold-stuck piston ring cannot be moved with moderate finger pressure. It is characterized by a polished face over its entire circumference, indicating essentially no blowby passed over the outside of the ring during engine operation.

3.1.4 *debris*, *n*—*in internal combustion engines*, solid contaminant materials unintentionally introduced into the engine or resulting from wear.

3.1.4.1 *Discussion*—Examples include such things as gasket material, silicone sealer, towel threads, and metal particles. **D 5862**

3.1.5 free piston ring, n—in internal combustion engines, a piston ring that will fall in its groove under its own weight when the piston, with the ring in a horizontal plane, is turned 90° (putting the ring in a vertical plane).

3.1.5.1 *Discussion*—A slight touch to overcome static friction is permissible. **D 5862**

3.1.6 hot-stuck piston ring, n—in internal combustion engines, a piston ring that is stuck when the piston and ring are at room temperature, and inspection shows that it was stuck during engine operation.

⁴ Annual Book of ASTM Standards, Vol 05.01.

⁵ Annual Book of ASTM Standards, Vol 05.02.

⁶ Annual Book of ASTM Standards, Vol 03.02.

⁷ Available from Society of Automotive Engineers, 400 Commonwealth Drive, Warrendale, PA 15096.

⁸ Available from American National Standards Institute, 11 W. 42nd St. 13th Floor, New York, NY 10036.

⁹ Available from Coordinating Research Council, Inc., 219 Perimeter Ctr. Parkway, Atlanta, GA 30346.

3.1.6.1 *Discussion*—The portion of the ring that is stuck cannot be moved with moderate finger pressure. A hot-stuck ring is characterized by varnish or carbon across some portion of its face, indicating that portion of the ring was not contacting the cylinder wall during engine operation.

3.1.7 *knock*, *n*—*in a spark ignition engine*, abnormal combustion, often producing audible sound, caused by autoignition of the air/fuel mixture. **D 4175**

3.1.8 scoring, n—in tribology, a severe form of wear characterized by the formation of extensive grooves and scratches in the direction of sliding. **G 40**

3.1.9 *scuff, scuffing, n—in lubrication,* damage caused by instantaneous localized welding between surfaces in relative motion that does not result in immobilization of the parts. **D 4863**

3.1.10 *sludge*, *n*—*in internal combustion engines*, a deposit, principally composed of insoluble resins and oxidation products from fuel combustion and the lubricant, that does not drain from engine parts but can be removed by wiping with a cloth.

3.1.11 tight piston ring, n—in internal combustion engines, a piston ring that will not fall in its groove under its own weight when the piston, with the ring in a horizontal plane, is turned 90° (putting the ring in a vertical plane); by subsequent application of moderate finger pressure, the ring will be displaced. **D 5862**

3.1.12 *varnish*, *n*—*in internal combustion engines*, a hard, dry, generally lustrous deposit that can be removed by solvents but not by wiping with a cloth.

3.1.13 *wear*, *n*—the loss of material from, or relocation of material on, a surface. **D 5844**

3.1.13.1 *Discussion*—Wear generally occurs between two surfaces moving relative to each other, and is the result of mechanical or chemical action or by a combination of mechanical and chemical actions.

3.2 Definitions of Terms Specific to This Standard:

3.2.1 *clogging*, *n*—the restriction of a flow path due to the accumulation of debris along the flow path boundaries.

3.2.2 closed-loop lambda control system, n— one that functions by feeding back a direct measurement of lambda through a fully closed loop, and uses the measurement to alter the fuel input by changing the fuel injector pulse width.

3.2.3 enrichment, n— in internal combustion engine operation, a fuel consumption rate in excess of that which would achieve a stoichiometric fuel-to-air ratio.

3.2.3.1 *Discussion*—Enrichment is usually indicated by elevated CO levels and can also be detected with an extended range air/fuel ratio sensor.

3.2.4 *lambda*, *n*—the ratio of actual air mass induced, during engine operation, divided by the theoretical air mass requirement at the stoichiometric air-fuel ratio for the given fuel.

3.2.5 open loop lambda control system, n— one that utilizes manual adjustment of the fuel injector pulse width to alter lambda based on manual measurement of exhaust gas CO and O_2 content.

3.2.6 *low-temperature, light-duty conditions, n*—indicative of engine oil and coolant temperatures that average below normal warmed-up temperatures, and engine speeds and power outputs that average below those encountered in typical highway driving.

3.2.7 *ramping*, n—the prescribed rate of change of a variable when one set of operating conditions is changed to another set of operating conditions.

4. Summary of Test Method

4.1 Each test engine is assembled with many new parts and essentially all aspects of assembly are specified in detail.

4.2 The 2.3-L internal combustion engine is installed on a test stand, which is equipped to control speed and load and various other operating parameters.

4.3 The engine is operated for a total of 288 h, consisting of 72 repetitive cycles of 4 h each. Each cycle consists of three stages.

4.4 While the operating conditions are varied within each cycle, overall they can be characterized as a mixture of low-temperature and moderate-temperature, light-duty operating conditions.

4.5 To accelerate deposit formation and engine wear, the level of oxides of nitrogen in the blowby and the rate of blowby into the crankcase are significantly increased, the fresh air breathing of the crankcase is significantly decreased, and the transitions between two of the stages of operation are deliberately enriched, in comparison with normal operation.

4.6 The test engine oil's performance is determined at the end of the test by dismantling the engine and measuring the deposition and wear.

5. Significance and Use

5.1 This test method is used to evaluate an automotive engine oil's control of engine deposits and wear under operating conditions deliberately selected to accelerate deposit formation and wear. This test method was correlated with field service data, determined from side-by-side comparisons of two or more oils in taxi fleets and delivery van services, as examples. The same field service oils were then used in developing the operating conditions of this test procedure.

5.2 This test method, along with other test procedures, defines the minimum performance level of the API Categories SH and SJ (detailed information about these categories is included in Specification D 4485). This test method is also incorporated in U.S. Army and automobile manufacturers' factory-fill specifications.

5.3 The basic engine used in this test method is representative of many that are in modern automobiles, but not all. This factor, along with the accelerated operating conditions, should be considered when extrapolating test results.

6. Apparatus

6.1 General Description:

6.1.1 The test engine is a 2.3-L, spark ignition, four-stroke, four-cylinder in-line engine. Features of this engine include an

overhead camshaft, a cross-flow fast-burn cylinder head design, and electronic port fuel ignition. It is based on the Ford Motor Company's 2.3-L EFI Ranger¹⁰ truck engine.

6.1.2 Use an engine test stand equipped to control engine speed and load, air-fuel ratio, various temperatures, and other parameters.

6.1.3 Use appropriate air conditioning apparatus to control the temperature and humidity of the intake air.

6.1.4 Use an appropriate fuel supply system.

7. The Test Engine

7.1 Sequence VE Test Engine Parts Kit— The kit is available from the Ford Motor Company and contains all the necessary consumable hardware for four tests. Tests starting after May 31, 1995, shall use 1992 or later test engine parts kits with the dual plug cylinder head. A complete list of parts included in the kit is shown in Table A4.1.

7.1.1 The following required parts are to be obtained from the kit (unless substitutions for those parts are specifically approved by the Sequence VE Operations and Hardware Subpanel (contact the ASTM Test Monitoring Center (TMC) for guidance): cylinder block, cylinder head, pistons, piston rings, camshaft, rocker arms, and connecting-rod bearings.

7.1.1.1 Tests started on or after November 16, 1999 may use substitute non-kit parts (other than those listed in 7.1.1) obtained from Ford dealers, provided that the part numbers are the same as those in the kit. Where parts substitutions are made, maintain samples of the substituted part for possible comparison. Identify all substituted parts with the part numbers on the Supplemental Operational Data Form of the test report (Fig. A7.5). Cordinate the use of substituted parts with the TMC.

7.1.2 Premeasured and calibrated Sequence VE engine parts are available from the supplier listed in X2.1.5.

7.2 *Required New Engine Parts*—Install the following new parts in each new test engine assembly: cylinder head (may be reused for tests starting on or before May 20, 1999) cylinder head bolts (torque-to-yield), camshaft, camshaft bearings, camshaft drive belt, rocker arms, hydraulic lifters, intake and exhaust valves, valve stem seals, pistons, piston rings, wrist pins, connecting-rod bearings, main bearings, oil pump, oil filter, PCV valve, spark plugs, and gaskets and seals.

7.3 *Reusable Engine Parts*—The following parts can be reused: cylinder block (can be used for approximately two tests, depending on bore wear); valve springs (can be reused as long as they meet the specifications detailed in Annex A3); cylinder heads (for tests started on or after May 20, 1999, provided an acceptable reference oil test, incorporating reused cylinder heads and meeting the requirements in Annex A16, has been completed); auxiliary shaft and bearings, connecting rod, front seal housing, fuel management wiring harness, crankshaft ignition trigger, intake manifold, throttle body, camshaft drive parts, water pump drive parts, crankshaft, fuel injectors, ignition module, ignition wires, oil pump screen and pick up tube, timing belt sprockets, and water pump (all of

these can be used in numerous engine assemblies as long as they remain serviceable).

7.4 *Specially Fabricated Engine Parts*— The following subsections detail the specially fabricated engine parts required in this test method:

7.4.1 Intake Air Horn (see Fig. 1 and Fig. A3.4) :

7.4.2 *Camshaft Baffle* (see Fig. A3.24)— This is fabricated for attachment to the cam bearing pedestals. The clearance between the edges of the baffle and the rocker arm cover (RAC) permits a limited splash flow of oil to the top of the baffle and the RAC. Therefore, the dimensional accuracy of the baffle is important to minimize the influence on test severity. The camshaft baffle is available from the supplier listed in X2.1.13.

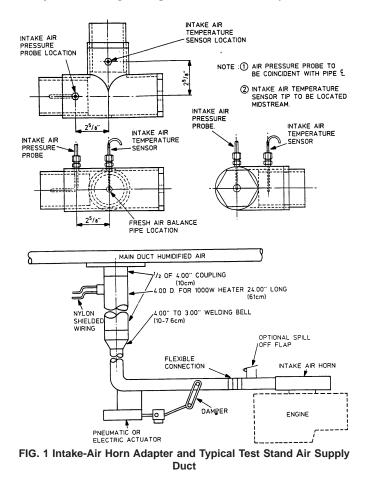
7.4.3 *Crankcase Oil Fill Tube* (see Fig. A3.27)—This design is not a requirement, but if an alternative part is used, install a suitable shut-off valve between the fill tube and the engine.

7.4.4 *Dipstick Tube*—Replace the existing dipstick tube with a ¹/₈-in. NPT pipe nipple. The nipple should be approximately 6-in. (150-mm) long and is threaded into the port on the left side of the crankcase, as shown in Fig. A3.13.

7.4.5 *Dipstick*—Use the special dipstick shown in Fig. A3.28 to measure the oil level in the modified oil pan.

7.4.6 *Oil Pan*—Obtain the special oil pan shown in Fig. A3.25 from the supplier listed in X2.1.13.

7.4.7 *Flywheel Timing Indicator*—Fabricate a bracket with an adjustable timing index pointer for use at the flywheel. Mark



¹⁰ Ford Ranger is a product of the Ford Motor Co., Dearborn, MI 48121.

the flywheel with graduations at top dead center (TDC) and from 26 to 30° in 1° increments before top dead center (BTDC) and after top dead center (ATDC). The suggested mounting location is between two of the flywheel housing bolts on the upper left side of the crankcase. Since ignition timing is measured with an accuracy of 1° , configure the index pointer to minimize parallax error.

7.4.8 *Rocker Arm Cover*—The RAC is fabricated from stainless steel and incorporates a water jacket. The RAC supplier is listed in X2.1.13.

7.5 Special Engine Measurement and Assembly Equipment—Items routinely used in the laboratory and work-shop are not included.

7.5.1 *Camshaft and Rocker Arm Hardness Tester and Fixtures*—Use a hardness tester to measure the camshaft lobe and rocker arm hardness in the range from 50 to 60 HRC (Hardness—Rockwell C). The tester and fixtures shall allow accurate and repeatable results at the specified locations on these parts (see 7.8.6.3 and 7.8.7).

NOTE 1-Repeatable hardness measurements are difficult to achieve without properly designed fixtures.

7.5.2 *Camshaft Lobe Measurement Equipment*—A snap gage dial indicator is recommended for measurement of camshaft lobe heel-to-toe distance.

7.5.3 Camshaft Lobe Orifice Flow Rate Device:

7.5.3.1 Determine camshaft lobe orifice plugging with the device detailed in Fig. A3.32. The device measures restriction of air flow relative to a clean lobe by means of a probe placed in the external opening to each lobe orifice. The device shall be capable of controlling static air pressure at 10.0 in. water (2.49 kPa).

7.5.3.2 Calibrate the flowmeter with a reference (clean) cam lobe each time a camshaft is rated for plugging. Since the measurement is a relative flow rate reduction, repeatability, rather than absolute accuracy, is the primary concern (see Fig. A3.32 and 13.4.4).

7.5.4 Rocker Arm and Connecting-Rod Bearing Weighing Equipment—A precision weight scale with an accuracy of 0.25 % and a resolution of 0.1 mg is necessary. Rocker arm weights are approximately 105 g, while individual bearing insert weights are approximately 20 g.

7.5.5 Unassembled Valve Spring Calibration Device—Use this device to screen valve springs before assembly in the cylinder head (see 7.8.8.3). The spring loading is measured at a compressed height of 1.16 in. (29.5 mm). The tester shall have an accuracy of 2 % and a resolution of 1 lbf (4.45 N).

7.5.6 Assembled Valve Spring Calibration Device:

7.5.6.1 This device is used to measure valve spring loading after the springs have been assembled in the cylinder head. Use the procedure specified in 7.8.8.5 in conjunction with the device. A suitable fixture may be purchased from the supplier detailed in X2.1.13 and X2.1.5.

7.5.6.2 Calibrate the valve spring force measurement device at least once every three months. The calibration technique is left to the discretion of the testing laboratory.

7.5.7 Valve Stem and Guide Measurement Equipment—Any of the following methods is acceptable for measuring valve stem-to-guide clearance (see 7.8.6.1). One method is specified

in Footnote 14¹¹ utilizing a special tool. Other commercially available automotive service equipment can also be utilized. Alternatively, air gaging equipment can be used to measure the guide diameter, and a micrometer can be used to measure the valve stem diameter.

NOTE 2—Accurate measurement of stem-to-guide clearance is important since this parameter can affect oil consumption.

7.5.8 *Connecting-Rod Heater*—The piston pins are fixed to the connecting-rods with an interference fit. A connecting-rod heater¹² is required to facilitate the installation of the piston pins and prevent piston distortion (see 7.8.13). This type of heater minimizes the heat exposure of the rods and consequently minimizes the safety hazard to personnel.

7.5.9 *Cylinder Block Stress Plates*—Install two stress plates on the cylinder block during honing and measurement operations to minimize cylinder bore out-of-round and taper. Fabricate one plate in accordance with the details shown in Fig. A3.36. The second stress plate can be purchased from the supplier detailed in X2.1.28. Install the thinner of the two plates on top.

7.5.10 *Cylinder Block Honing Machine*— Use a Sunnen CK-10 or CV-616 honing machine¹³ for cylinder bore resizing and finishing. The Sunnen honing lubricant or Mobil VAC-MUL 3-D lubricant,¹⁴ a 30-tooth ratchet, and EHU-525 and JHU-625 (or JHU-820) honing stones are required in this application. See Table 1.

7.5.11 Cylinder Bore Surface Finish Analyzer:

7.5.11.1 The analyzer is required to measure the cylinder bore surface finish after completion of the honing operation. The range of the analyzer shall cover a minimum from 8 to 20 μ in. (0.0002 to 0.0005 mm). An instrument with a mechanically driven profilometer stylus is recommended since repeatable measurements are difficult to achieve with a hand-held instrument.

7.5.11.2 Calibrate the surface finish analyzer with a commercially recognized surface finish calibration standard. The

¹³ The Sunnen CK-10 and CF-616 honing machines and the Sunnen honing lubricant are products of Sunnen Inc., 7910 Manchester, St. Louis, MO 63143.

¹⁴ Mobil VACMUL 3-D lubricant is a product of Mobil Oil Corp., 3225 Gallows, Fairfax, VA 22037.

TABLE 1 Honing Machine Set Up

Parameter	Honing Ma	Honing Machine Model Number		
Falameter	CK-10	CV-616		
Spindle speed, r/min	155	170		
Stroke rate, s/min	46	57		
eed ratchet	3 or 4	2 or 3		
Position	30 tooth gear			
ubricant flow rate	50 % of maximum	50 % of maximum		
Desired Microfinish	Honing Stone Select	Honing Stone Selection		
to 14 µin. AA	EHU-525 rough			
0.20 to 0.36 µm)	JHU-820 finish	JHU-820 finish		
5 to 20 µin. AA	EHU-525 rough	EHU-525 rough		
(0.38 to 0.51 µm)	JHU-625 finish	JHU-625 finish		

¹¹ Ford Shop Manual-Ranger/Bronco II, FPS 1987.

 $^{^{12}}$ The Sunnen Model CRH-50 Rod Heater, or equivalent, is suggested for this purpose. Information on availability is given in X2.24 .

frequency of calibration shall comply with the manufacturer's specification for the particular analyzer used.

7.5.12 Cylinder Bore Measurement Ladder— The bore measurement ladder is required to ensure precise location of the cylinder bore measurement at the top, middle, and bottom of the piston ring travel. Fabricate the bore ladder in accordance with the details shown in Fig. A3.38.

7.5.13 *Piston Ring Positioner*—Use the piston ring positioner to locate the piston rings 2.0 in. (51 mm) from the upper surface of the stress plate. This allows the compression rings to be positioned in a standard location in the cylinder bore before measurement. Fabricate the positioner in accordance with the details shown in Fig. A3.37.

7.5.14 *Piston Ring Grinder*—A precision ring grinder is required for adjusting ring gaps before assembly. A suitable ring grinder is available from the supplier shown in X2.1.21.

7.5.15 Oil Pump Calibration Device:

7.5.15.1 Calibrate the engine oil pump by precisely adjusting the pressure relief valve, utilizing a calibration device similar to the one detailed in Fig. A3.30 and Fig. A3.31. The device drives the pump at a constant speed and circulates Mobil EF-411 oil,¹⁵ while the oil temperature and flow rate are controlled. The pump discharge pressure is adjusted by means of the pressure relief valve. The oil flow rate sensor and readout shall have an accuracy of 2 % of full scale and a resolution of 0.1 gpm. The pressure measurement system shall have an accuracy of 1.0 % of full scale and a resolution of 0.5 psig (3.4 kPa).

7.5.15.2 Calibrate the oil pump calibration device at least once every three months. The calibration technique is left to the discretion of the testing laboratory.

7.5.16 PCV Valve Flow Rate Device:

7.5.16.1 Use this device to verify the flow rate of the PCV valves before the test and to measure the degree of clogging after the test is completed. Fabricate the device in accordance with the details shown in Fig. A3.34. The device shall have an accuracy of 5 % of full scale and a resolution of 0.1 cfm (0.047 L/s) (see 7.6.9).

7.5.16.2 Calibrate the flow rate device once every six months against a standard traceable to national standards. Perform calibration checks biweekly, or more frequently with modified PCV housings that contain fixed orifices (see Fig. A3.35).

7.5.17 *Probe for Engine Timing Calibration*—Use the cylinder probe to calibrate the crankshaft pulley and the flywheel at TDC of the Number 1 piston.

7.5.18 *Engine Service Tools*—A complete list of special tools for the test engine is shown in Annex A1. The tools are available from a Ford dealer or from the supplier listed in X2.1.27. These are designed to aid in performing several service items, in addition to the following specific service items that require special tools to perform the functions indicated (if not self-explanatory):

7.5.18.1 Camshaft Belt Tension Adjusting Tool, (Part No. T74P-6254-A).

7.5.18.2 Valve Spring Compressor Lever, necessary for rocker arm installation. (Part No. T74P-6565-A.)

7.5.18.3 *Engine Plug Replacer*, necessary for freeze plug installation. (Part No. T74P-6015-A.)

7.5.18.4 *Camshaft and Auxiliary Shaft Seal Replacer*, (Part No. T74P-6150-A).

7.5.18.5 *Fuel Injector Test Rig*—A suitable device capable of accurate, repeatable flow measurement of port fuel injectors is required. This device shall be capable of performing necessary port fuel injector evaluations, as outlined in 7.6.11. Since no suitable commercially available apparatus has been identified, design of the test rig is left up to the laboratory.

7.6 Miscellaneous Engine Components—Preparation:

7.6.1 Engine Buildup and Measurement Area— Environment—The ambient atmosphere of the engine buildup and measurement areas shall be reasonably free of contaminants. A relatively constant temperature (within $\pm 3^{\circ}$ C) is necessary to ensure acceptable repeatability in the measurement of parts dimensions. The relative humidity should be maintained at less than 75 % to prevent moisture forming on cold engine parts that are brought into the buildup or measurement areas.

7.6.2 *Oil Pump*:

7.6.2.1 Calibrate the oil pump to help provide uniform oil flow rates from test-to-test. The oil pump is calibrated with an oil pump calibration device (see Figs. A3.30 and A3.31, and 7.5.15.1). The oil pump relief adjuster assembly replaces the blanking plug and is used to position the relief valve spring during calibration and operation (see Fig. A3.29).

7.6.2.2 The oil pressure relief valve assembly is located behind the blanking plug. The plug is difficult to remove, and it may be necessary to apply heat to the oil pump casing to break down the thread locking adhesive. After the blanking plug has been removed, install the adjuster assembly.

7.6.2.3 Mount the oil pump on the calibration device. Fill the calibration device with EF-411 oil and circulate and heat the oil until it reaches $125 \pm 5^{\circ}$ F (51.7 $\pm 2.8^{\circ}$ C). Adjust the oil pump speed to 1250 ± 10 rpm (130 ± 1 rad/s), and set the flow rate at 6.6 \pm 0.1 gal/min (0.420 \pm 0.006 L/s). Adjust the relief valve spring tension until the pump output pressure registers 60 \pm 1 psig (414 \pm 7 kPa).

7.6.2.4 If oil pressure fluctuations are observed during calibration, disassemble the relief valve and correct the problem. Significant air entrainment in the oil in the calibration device indicates unacceptable leakage on the suction side of the pump.

7.6.3 *Oil Pump Pickup Tube and Screen*— Clean the oil pickup tube and screen assembly, and inspect the assembly for defects.

7.6.4 *Throttle Body*:

7.6.4.1 The required throttle body modifications are detailed in Fig. A3.19. Remove all tubes and plug the holes with pipe plugs and a suitable thread sealer. Remove and discard the outer and inner throttle body shaft seals. Use of the throttle position sensor and the multiplier linkage is optional.

7.6.4.2 Disassemble and thoroughly clean the throttle body before each test. Remove the butterfly from the throttle shaft, and remove the shaft from the throttle body. Discard the rubber

¹⁵ Mobil EF-411 oil is a product of Mobil Oil Corp., 3225 Gallows, Fairfax, VA 22037.

throttle body shaft seals, and soak all parts in a commercially available carburetor cleaner. After the parts have been thoroughly cleaned and air-dried, install new throttle body shaft seals. The seals should be lightly lubricated on all surfaces with EF-411 oil to facilitate installation. Do not use any other lubricant on the seals. Reassemble the throttle body.

7.6.4.3 There is no specific life for the throttle body. However, the clearance between the bore and the butterfly will eventually increase and render the body unserviceable. When the clearance becomes too great to allow control of speed, load, and air-fuel ratio during Stage 3, discard the throttle body.

7.6.5 Intake Manifold:

7.6.5.1 The intake manifold is comprised of three parts: upper intake manifold, lower intake manifold, and intake manifold spacer. The spacer is necessary to allow the upper intake manifold to clear the RAC (see Fig. A3.20). The spacer can be purchased separately or as part of a kit of modified parts available from the supplier detailed in X2.1.13. Modify the intake manifold, as detailed in Fig. A3.21. Block the EGR valve port by means of a cover plate or pipe plug. Locate the PCV valve vacuum connection at the lower port on the upper intake manifold, as detailed in Fig. A3.21. Locate the intake manifold absolute pressure (MAP) sensor connections at the upper port on the upper intake manifold, as detailed in Fig. A3.21.

7.6.5.2 Disassemble and thoroughly clean the intake manifold before each test by soaking in a commercially available carburetor cleaner. This is followed by a hot (> 60° C) water rinse and forced air-drying. The intake manifold can be used in repeated builds.

7.6.6 Rocker Arm Cover:

7.6.6.1 Before each test, flush the RAC coolant jacket with a mixture of hot (> 60° C) water and tri-sodium phosphate detergent. After flushing, inspect the coolant jacket. If a deposit or film is present, clean the RAC coolant jacket with a commercially available de-scaling cleaner, neutralizer, and inhibitor (see 8.4.4.1). Examples of acceptable cleaners are detailed in 7.7.

7.6.6.2 Inspect the appearance of the interior surface of the RAC. If the before test rating is less than ten on the CRC varnish rating scale (Manual 14), polish the interior surface lightly with Number 0 fine steel wool to achieve a fine dull shine. Rinse the cover with aliphatic naphtha and allow to air-dry before use.

7.6.7 *Camshaft Baffle*—Polish the camshaft baffle with Number 0 fine steel wool to achieve a fine finish. Rinse with aliphatic naphtha and allow to air-dry before use.

7.6.8 *Oil Pan*—Inspect the appearance of the tin plating on the internal surface of the pan. Polish lightly with Number 0 fine steel wool to achieve a dull shine. Rinse with aliphatic naphtha and allow to air-dry before use. Replate the pan when the finish becomes unserviceable (see 7.4.6).

7.6.9 *PCV Valve*—Measure and record the flow rates of the PCV valves with the calibrated flow device described in 7.5.16 and Fig. A3.34. Measure the flow rate at 8 and 18-in. Hg (27 and 61-kPa) vacuum. Because of the hysteresis in the PCV valve spring, make the vacuum adjustments in one direction

only. Measure the flow rate twice and average the readings. Reject any PCV valve that does not exhibit an average flow rate between 2.40 and 2.90 cfm (1.13 and 1.37 L/s) at 8 and 18-in. Hg vacuum.

7.6.10 *Water Pump Drive System*—Use only the pulleys provided in the Sequence VE Test Stand Set-Up Kit (see Table A4.2) to ensure that the water pump rotates at the proper speed. 7.6.11 *Fuel Injectors*:

7.6.11.1 Prior to engine installation, evaluate all injectors (new and used) for spray pattern and flow rate using a suitable apparatus as identified in 7.5.18.5. The evaluation procedure is outlined in this section. Injectors may be cleaned and reused if the criteria outlined in this section are satisfied.

7.6.11.2 Flush new injectors for 30 s to remove any assembly residue before flow testing.

7.6.11.3 Using a rig as described in 7.5.18.5, place the injector(s) in the rig and turn on the pressure source. After the pressure source is turned on, the test fluid (see 7.7.1) will start to flow through the injector(s). Maintain the test fluid pressure supplied to the injector(s) at 39 ± 0.5 psi (269 ± 3.4 kPa) during the entire test. The maintenance of this pressure is critical because a small change in pressure will have a dramatic effect on the flow rate and spray pattern. Once pressure is set, zero the volume measuring device.

7.6.11.4 Apply pressure to the closed injector(s) for at least 30 s. The injector(s) shall not leak or drip. Replace any injector that leaks or drips.

7.6.11.5 Flow-test each injector for a 60-s period. While the injector is flowing, make a visual observation of the spray pattern quality. The spray pattern should be typical for the make and model of the injector.

7.6.11.6 The total flow for each injector after the 60 s test shall be between 3.7 oz (109 cm³) and 4.5 oz (134 cm³) at 39 \pm 0.5 psi (269 \pm 3.4 kPa) of test fluid pressure. Discard any injector that flows above or below this range.

7.7 Solvents and Cleaners Required— No substitutions for the following are allowed: (Warning—Use adequate safety provisions with all solvents and cleaners.)

7.7.1 *Aliphatic Naphtha*, Stoddard solvent¹⁶ or equivalent is satisfactory.

7.7.2 Ethyl Acetate.

7.7.3 Organic Solvent, Penmul L460¹⁷ or Oakite 811.¹⁸

7.7.3.1 Implement the specified gas chromatographic monitoring program (see Annex A10) when using Oakite 811 to prevent gelatinous-type coatings on water-rinsed parts.

7.7.4 Pentane.

7.7.5 *Tri-Sodium Phosphate Detergent*, cylinder block and RAC cleaning detergent.

7.7.6 Formulation No. 7 Cooling System Cleaner —Heavy-Duty,¹⁹ cooling system flushing agent to engine cooling system cleanser consists of the following:

¹⁶ Stoddard Solvent is a product of UNOCAL Chemicals Division. 7010 Mykawa St., Houston, TX 77033.

¹⁷ Penetone L460 is a product of Penetone Corp., P.O. Box 22006, Los Angeles, CA 90022.

 ¹⁸ Oakite 811 is a product of Oakite Products, 10100 Hirsch, Houston, TX 77016.
 ¹⁹ Formulation No. 7 Cooling System Cleaner—Heavy-Duty is a product of Armor All. Department 2224, SCF, Pasadena, CA 91051.

7.7.6.1 Oxalic Acid Dihydrate Tech.

7.7.6.2 Alkylated Naphthalene, Sodium Salt— Petro Dispersant 425 (soap).

7.7.6.3 Soda Ash Light—(Neutralization).

7.7.7 *Dearsol 134 Acidic Cleaner with Inhibitor*, (or equivalent). RAC cooling system cleaner.

7.7.8 *Commercial solvent*, designated for cleaning carbure-tors.

7.8 Assembling the Test Engine—Preparations—Functions that are to be performed in a specific manner or at a specific time in the assembly process are noted. Any assembly instructions not detailed as follows should be completed in accordance with the instructions in Footnote 14.

7.8.1 *Parts Selection*—Instructions concerning the use of new or used parts are detailed in 7.1.1, 7.2, and 7.3.

7.8.2 Engine Measurement Records—Record the engine measurements on data sheets equivalent to those shown in Appendix X1.

7.8.3 *Buildup Lubrication*—Lubricate all engine parts with EF-411 oil during assembly.

7.8.4 Sealing Compounds—No specific compounds required.

NOTE 3—Silicone-based sealers should be used with care since they can elevate the indicated silicon content of the used oil. Also use tape sealers with care because tape fragments can plug oil orifices in the oiling system.

7.8.5 *Gaskets and Seals*—Use new gaskets and seals at all locations during each engine assembly.

7.8.6 Cylinder Head:

7.8.6.1 *Cleaning*—Rifle-brush cylinder head guides with a valve guide brush and aliphatic naphtha. Remove the end plugs of the camshaft and rifle-brush the central gallery with a valve guide brush and aliphatic naphtha. Rinse with aliphatic naphtha and air-dry with compressed air. Reinstall the camshaft end plugs after the air-drying.

7.8.6.2 *Valve Guides*—Measure and record the valve stemto-guide clearance for each valve (see 7.5.7). The clearances shall be within the following specifications: exhaust at 0.0019 to 0.0032 in. (0.048 to 0.081 mm) and intake at 0.0014 to 0.0027 in. (0.036 to 0.069 mm). The clearances can be adjusted by selectively fitting valves or reaming the valve guides.

7.8.6.3 *Camshaft Bearings*—Inspect the bearings for any anomalies and ensure the bearing holes are properly aligned with the oil gallery in the Number 2 and 3 pedestals. Tap the 0.237-in. (6.02-mm) holes in the top of the pedestals to receive the cam baffle hold-down bolts (see Fig. A3.24). While Fig. A3.24 shows that $\frac{5}{16}$ -in. (8–mm) machine screws are satisfactory for this purpose, experience has shown that 7-mm machine screws may be a preferred alternative.

7.8.6.4 *Camshaft:* Install 1992 VF kit cams, part no. E59E-6251-DA, serial numbers 920160 through 922882 (available October 31, 1993) for all reference oil tests and associated non-reference oil tests starting on or after October 1, 1994.

(a) Lobe Height and Lift Measurement—Measure and record the heel-to-toe dimension of each camshaft lobe at the maximum lift point and at a plane perpendicular to the maximum lift point. Subtract the value taken perpendicular to the maximum lift point from the heal-to-toe value. This difference is the camshaft lift. Reject any cam that exhibits a lift value that does not fall between 0.2367 and 0.2391 in. (6.012 and 6.073 mm). Note that the *before test* heal-to-toe dimension is also used as the baseline measurement for determining wear (see 13.6.1). Record the actual values for both measurements in the appropriate spaces of the form in Fig. X1.3.

(b) Lobe Hardness Measurement—Measure and record the lobe hardness 180° from the maximum lift point and approximately 0.05 in. (1.3 mm) from the forward edge of each lobe. Include the individual cam lobe hardness measurements on the hardness measurement data sheet (see Fig. X1.4).

(c) Oil Groove Measurements—Measure and record the oil groove depth and width on the Number 2 and 3 journals. Calculate the nominal depth and width using the following equations:

nominal groove depth =
$$\frac{G+D}{2}$$
 (1)

where:

G =maximum groove depth, and

D =minimum groove depth.

nominal groove width
$$=\frac{E+R}{2}$$
 (2)

where:

E = maximum groove width, and

R =minimum groove width.

The nominal oil groove depth shall be between 0.039 and 0.047 in. (0.99 and 1.19 mm), and the nominal oil groove width shall be between 0.052 and 0.063 in. (1.32 and 1.60 mm). The grooves can be machined to bring the dimensions within the specifications. Reject any camshaft that cannot be modified to achieve these specifications.

7.8.6.5 *Lobe Hole Measurements*—Measure and record the lobe hole diameter on each lobe. The lobe hole diameters shall be between 0.047 and 0.055 in. (1.19 and 1.40 mm). The holes can be drilled to bring the diameters within the specification. Reject any camshaft that cannot be modified to achieve this specification.

7.8.6.6 Journal Through-Hole Measurements— Measure and record the journal through-hole diameter of the Number 2 and 3 journals. The journal through-holes shall be between 0.116 and 0.124 in. (2.95 and 3.15 mm). The holes can be drilled to bring the diameters within the specification. Reject any camshaft that cannot be modified to achieve this specification.

NOTE 4—The axis of the journal through-hole is not perpendicular to the centerline of the camshaft.

7.8.7 *Rocker Arms*—Clean each rocker arm with ethyl acetate and allow the rocker arm to air-dry before taking any measurements. Measure and record the hardness and weight of each rocker arm. The hardness is measured on the cam lobe mating surface toward the fulcrum end of the rocker arm. The measurement shall be within 0.100 in. (2.5 mm) of the edge of the camshaft mating surface and near the centerline of the rocker arm. Reject any rocker arm that exhibits a hardness less than 57 HRC. A special holding fixture for the rocker arm is required for hardness measurements (see 7.5.1).

7.8.8 Valve Springs:

7.8.8.1 *Free Length*—The valve spring free length should be between 1.9 and 2.0 in. (48.3 to 50.8 mm).

7.8.8.2 *Out-of-Square*—The valve spring out-of-squareness should be 0.075 in. (1.91 mm) maximum.

7.8.8.3 *Load*—If the springs are within the free length and out-of-square specifications, then measure the load in the unassembled valve spring calibration device (see 7.5.5). This should be 167 ± 8 lbf at 1.16 ± 0.03 -in. (740 ± 20 N at 29.5 ± 0.8 -mm) deflection.

7.8.8.4 *Installation*—Lubricate each valve seal and valve stem with EF-411 oil. Install the valve seal over the end of the valve stem with a plastic installation cap in place. Carefully seat the seals fully on the guides. Install prescreened valve springs and retainers. When installing the valve springs and retainers, do not compress the springs excessively. Excessive spring compression can damage the valve seals. Measure and record the assembled height of the valve springs in accordance with the procedure described in Footnote 15. The assembled height shall be between 1.53 and 1.59 in. (38.8 and 40.4 mm).

7.8.8.5 *Calibration*:

(a) Check calibration of fixture load cell. Verify within procedure limits. Make calibration adjustments as required to achieve + 0.4 lb (+ 1.8 N) accuracy.

(b) Check head support fixture for correct setup for the VE cylinder head.

(c) Place VE cylinder head in holding fixture with intake valve springs accessible.

(d) Position air cylinder/load cell to allow compressing intake valve spring No. 1 (far left).

(e) Position dial indicator with plunger on rocker cover gasket rail (see Fig. A3.47).

(f) With zero air pressure to air cylinder, position the dial indicator to achieve exactly 0.100-in. (2.54-mm) deflection preload against the rocker cover rail. This allows determination of positive or negative displacement of the valve.

(g) Actuate the air cylinder in a rapid consistent manner to compress the intake valve spring.

(h) Adjust air regulator to achieve exactly 0.400-in. (10.16-mm) valve spring compression. This is indicated as 0.500 in. (12.70 mm) on the dial indicator.

(i) Discharge the air cylinder in a consistent manner and allow intake value to close to the fully seated position. The dial indicator should read 0.100 in. (2.54 mm) (zero value displacement).

(*j*) If the dial indicator did not accurately repeat 0.100 + 0.001-in. (2.54 + 0.02-mm) reading, conduct steps (*f*) through (*i*) and adjust the dial indicator as necessary to achieve accurate repeatability of indicated seated and displaced valve positions (0.100 in. and 0.500 in.).

(k) If the dial indicator accurately repeated the desired value, actuate and discharge the air cylinder three additional times and verify repeatability of the indicated value positions.

(l) Open air value to compress value spring to the displaced position.

(m) Record indicated force (lb).

(n) Position air cylinder/load cell to check remaining three intake valve springs. Conduct steps (e) through (m) for each remaining intake valve spring.

(*o*) Position cylinder head in fixture with exhaust valve springs accessible. Conduct steps (e) through (m) for each exhaust valve spring.

(*p*) Replace any springs requiring applied force less than 159 lb (707 N) or more than 175 lb (778.4 N). Recheck spring calibration and record compressed force of any replaced springs (see steps d through m).

7.8.9 Short (Cylinder) Block—Use the 1992 VF cylinder block (1992 VF cylinder block, part number F37E-6010-AB, is available for use October 31, 1993) for all reference oil tests and associated non-reference oil tests starting on or after October 1, 1994.

7.8.9.1 *Initial Preparation*—Disassemble the short block and remove all gallery plugs and freeze plugs. Reinstall the main bearing caps and torque them to 85 ± 5 lbf·ft (115 ± 7 N·m). Inspect the coolant jacket to ensure the coolant passages are reasonably free of casting slag. Remove the external oil separator on the left side of the crankcase. Plug the hole with a ¹/₂-in. NPT pipe plug. Remove the internal sheet metal baffle located inside the block, underneath the oil separator passage. Tap the dipstick tube hole to accept a ¹/₈-in. NPT pipe nipple (see 7.4.4). Tap the monolithic timing port to accept a ³/₈-in. NPT pipe (see 7.4.3).

7.8.9.2 *Cleaning*—Submerge the cylinder block in agitated organic solvent (see 7.7.3) until clean (approximately 1 h). Rinse the parts thoroughly with hot water (> 60° C). Spray rinse with aliphatic naphtha and use compressed air to force airdrying. If the cylinder block is going to sit idle for an excessive period of time before honing, spray all parts with aliphatic naphtha containing 10 % EF-411 oil by volume. Perform final cylinder block cleaning and preparation after the block is honed and the pistons and rings are fitted.

7.8.9.3 Honing:

(a) Install a used head gasket between the block and the stress plate. Install the stress plates (see 7.5.9), main bearing caps, and water pump for honing and measurement of the cylinder bores and ring gaps. Use torque-to-yield bolts when installing the stress plates. Torque the head bolts in accordance with the sequence described in Footnote 14 in two stages, 55 ± 5 lbf·ft (75 ± 7 N·m) and an additional 90 to 100° clockwise rotation.

(b) Install the block in a honing machine. (Sunnen CK-10 or CV-616 honing machines have been found suitable. However, any honing machine capable of producing the desired cross-hatch pattern and cylinder microfinish can be used.) The appropriate Sunnen honing machine setups and honing stone selections are shown in Table 1. These speeds and stroke rates provide a cross-hatch pattern with a 30 or 40° maximum included angle. The selection of stones is optional as shown in Table 1.

(c) Experience has shown that two complete strokes with the rough stones are required to break up the bore glaze and prevent the finer stones from loading up with debris. Ten to fifteen strokes are required to obtain the desired cylinder

microfinish. The honing lubricant shall not contain any excessive amount of honing debris.

(d) If the cylinder block has been used in a previous test, make sure all worn areas are eliminated during honing. If any worn areas are still present after honing, re-hone the cylinder block to the next larger size or discard the cylinder block.

7.8.9.4 *Surface Finish*—Measure the finish of each cylinder bore using a 0.030-in. (0.76-mm) cutoff on the profilometer. Record the typical finish measured at the top, middle, and bottom elevations of the bores. Calibrate the profilometer frequently.

7.8.9.5 Bore Measurements:

(*a*) Measure the cylinder bores with the stress plate and the main bearing caps in place. Clean the bores with a dry rag until no further residue appears after wiping. The bores shall be clean and dry when they are measured. Use the bore ladder (see Fig. A3.38) and a bore gage micrometer to determine the diameter of each cylinder at the top, middle, and bottom of second compression ring travel in both the longitudinal and transverse directions.

(b) Record the bore diameters in 0.0001-in. increments over 3.7800 in. (0.003 mm over 96.010 mm). The cylinder bore out-of-roundness represents the difference between the longitudinal and transverse diameters. Calculate and record the out-of-round value for each second ring travel location. This value shall not exceed 0.0010 in. (0.025 mm) at any location.

(c) Calculate and record the cylinder bore taper value for each cylinder: this represents the difference between the largest and smallest diameter in a given cylinder bore. This value shall not exceed 0.0015 in. (0.038 mm) at any location. If the cylinder bore out-of-round, taper, and micro-finish are not within the specified tolerances, rehone the particular bore(s).

(d) After achieving the specified cylinder tolerances, calculate the average bore diameter for each cylinder using the middle and bottom transverse diameters. Record the average bore diameter and the calculated range of acceptable piston sizes that can be fitted in each bore. These data will be used when fitting pistons to specific cylinders.

NOTE 5—Final sizing of the cylinder bores depends on the pistons available for fitting.

7.8.9.6 *Final Preparations*—After honing, bore measurement, piston ring gap adjustments, and piston fitting, soak the block in agitated clean organic solvent (see 7.7.3) for 1 h. Rinse the block with a mixture of hot (> 60° C) water and tri-sodium phosphate detergent and flush out all oil and coolant passages until no residue appears. Final rinse with hot (60° C) water. Spray the block with aliphatic naphtha containing 10 % EF-411 oil by volume. Use compressed air to force air-drying. (**Warning**—Do not spray the aliphatic naphtha/EF-411 oil mixture into the coolant passages. Clear all passages with compressed air and wipe the cylinder bores with EF-411 oil.)

7.8.10 Pistons:

7.8.10.1 *Inspection of Tin Plating Quality*— Reject any pistons with tin plating that is peeling or flaking. Examine the skirt surfaces for discoloration. Remove any discoloration by

rubbing the piston with a Scotch Brite²⁰ 7445 pad. Reject any pistons from which discoloration cannot be removed.

7.8.10.2 *Oversizes*—Either of the piston oversizes included in the Sequence VE Test Engine Parts Kits, 0.2 or 0.5 mm, can be used. However, all pistons in a given engine shall be of the same nominal oversize. Tests starting after May 19, 1998, may use 0.5-mm pistons, modified in accordance with Annex A15 by C&F Tool and Die²¹ and inspected by Test Engineering, Inc.²² Use pistons modified in accordance with Annex A15 for all tests started on or after December 31, 1998, with 0.5-mm oversize pistons.

7.8.10.3 *Measurement*—Measure and record the piston diameter at the wrist pin centerline elevation, perpendicular to the wrist pin centerline, and across the extended tips of the piston skirts. Use the wrist pin centerline diameter to determine the fitted clearance. Piston taper is the difference between the perpendicular centerline diameter and the skirt diameter. Positive piston taper means the skirt diameter is greater than the perpendicular centerline diameter. The taper shall be positive and shall not exceed 0.0015 in. (0.038 mm).

7.8.10.4 *Selection*—Select each piston to provide a calculated 0.0014 to 0.0022-in. (0.036 to 0.056-mm) clearance to its bore. (Alternatively, the cylinder bores can be honed to fit the pistons on hand.) Install the pistons in their respective bores to verify each piston is properly sized.

NOTE 6—Threading a feeler strip between the piston and the cylinder wall is a simple method to verify the piston-to-cylinder wall clearance. Remove the pistons for installation on the connecting rods.

7.8.11 Piston Rings:

7.8.11.1 *Selection*—Select the proper oversize piston rings to correspond with the nominal piston oversize. As an example, a 0.15-mm oversize ring corresponds with a 0.20-mm oversize piston, and a 0.45-mm oversize ring corresponds with a 0.50-mm oversize piston.

7.8.11.2 *Ring Gap Adjustment*—Enlarge both compression ring gaps on each piston-and-ring assembly to obtain the specified blowby flow rate. Ring gaps ranging from 0.048 to 0.096 in. (1.2 to 2.4 mm) have been used successfully in various laboratories. Any combination (wider or narrower top ring gap or second ring gap) is acceptable. Enlargement of the oil ring gaps is not necessary to achieve satisfactory control of the blowby flow rate.

²⁰ The sole source of supply of Scotch Brite 7445 pads known to the committee at this time is 3M Abrasive Systems Division, 3M Center, Building 223-6N-01, St. Paul, MN 55144-1000 U.S.A. If you are aware of alternative suppliers, please provide this information to ASTM Headquarters. Your comments will receive careful consideration at a meeting of the responsible technical committee, which you may attend.

²¹ The sole source of machining services known to the committee at this time is C&F Tool and Die, 7206 Eckhart Rd., San Antonio, TX 78238. If you are aware of alternative services, please provide this information to ASTM Headquarters. Your comments will receive careful consideration at a meeting of the responsible technical committee, which you may attend.

²² The sole source of inspection services known to the committee at this time is Test Engineering, Inc., 12758 Cimarron Path, Ste. 102, San Antonio, TX 782249-3417. If you are aware of alternative services, please provide this information to ASTM Headquarters. Your comments will receive careful consideration at a meeting of the responsible technical committee, which you may attend.

(*a*) Position each ring in its respective cylinder with the ring positioner (see Fig. A3.37) so that the gaps are located at the center of the thrust side of the cylinder wall. Measure and record the ring gaps.

7.8.11.3 *Installation*—Remove burrs from the ring ends. Then, install the compression rings and the oil control rings on the pistons with the gaps located outside the skirt areas. The gaps should be staggered at approximately 180° intervals. The gaps should be located alternately toward the front and rear of the engine.

7.8.11.4 *Side Clearance*—Measurement of the ring side clearance is not required but can help prevent possible blowby flow rate control problems. The side clearance should be between 0.0010 and 0.0030 in. (0.025 and 0.075 mm).

7.8.12 *Connecting Rods*—Measure and record the big end oil orifice diameter. The orifice diameter shall be from 0.062 to 0.068 in. (1.57 to 1.72 mm). The orifice can be drilled to achieve the specified diameter.

7.8.13 *Wrist Pins*—Heat the small end of the connecting rod in a connecting rod heater (see 7.5.8). Insert the wrist pin through the piston and rod by hand and center the pin on the connecting rod. Commercial connecting rod heaters have a fixture to ensure precise centering of the pin. (**Warning**—In addition to other precautions, use extreme care when working around the connecting rod heater and handling the heated rods.)

7.8.14 *Connecting Rod Bearings*—Mark position numbers on the connecting rod bearings. Rinse the bearings in aliphatic naphtha, then in pentane, and air-dry. Weigh and record the weights of each bearing half.

7.8.15 *Crankshaft*—Inspect the bearing journals carefully. Polish the journal surfaces with Number 320 or 400 grit crocus cloth and EF-411 buildup oil. Measure and record the journal diameters. The maximum journal out-of-roundness shall be between 0.0000 and 0.0006 in. (0.000 and 0.015 mm). Spray the crankshaft with aliphatic naphtha containing 10 % EF-411 oil by volume.

7.8.16 Modify the crankshaft ignition trigger by drilling out the rivets, tapping the holes to accept a machine screw, and relocate the hall effect ring to obtain 28° BTDC, at 750 r/min.

7.8.17 Modify the cylinder head drain holes in accordance with Fig. 2.

7.9 Assembling the Test Engine—Installations:

7.9.1 Crankshaft and Miscellaneous Cylinder Block Components—Install all oil gallery plugs. Remove the main bearing caps and install the main bearings in the cylinder block. Install the crankshaft and install the front cover. Install the main bearings in the main bearing caps and torque the main bearing caps to 85 ± 5 lbf-ft (115 ± 7 N·m). Install the front cover. Measure and record the main bearing clearances. The desired clearances are 0.0015 to 0.0020 in. (0.038 to 0.051 mm). The allowable clearances are 0.0008 to 0.0026 in. (0.020 to 0.066 mm).

7.9.2 *Auxiliary Shaft Drive*—Install the auxiliary shaft and the auxiliary shaft drive sprocket. Install a petcock with a provision for a drain hose on the right side of the block at the existing drain plug location.

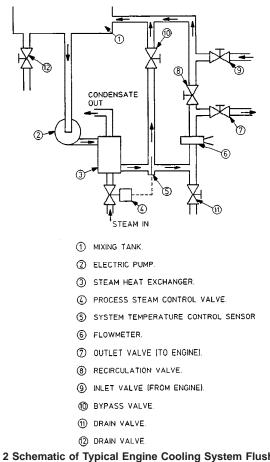


FIG. 2 Schematic of Typical Engine Cooling System Flushing Cart

7.9.3 *Pistons*—Install the pistons in the correct cylinder bores. Measure and record the rod bearing clearances. The desired clearances are 0.0015 to 0.0020 in. (0.038 to 0.051 mm). The allowable clearances are 0.0008 to 0.0026 in. (0.020 to 0.066 mm) for connecting rod and crankshaft main bearings.

7.9.4 *Oil System Components*—Install the oil pump, the oil pump pickup tube and screen assembly, and the oil pan.

7.9.5 Cylinder Head—Install the cylinder head on the cylinder block. Use new torque-to-yield cylinder head bolts. Do not use any sealing or anti-seizure compound on the cylinder head gasket. Lubricate the head bolts with EF-411 oil. Torque the head bolts in accordance with the sequence described in Footnote 14 in two stages: 55 ± 5 lbf·ft (75 ± 7 N·m) and an additional 90 to 100° clockwise rotation.

7.9.6 Camshaft and Related Components:

7.9.6.1 Install the camshaft.

7.9.6.2 Do not remove the oil in the new lifters. Fill the lifters with EF-411 using the lifter fill chamber, Part No. BX-390-1. This lifter fill chamber is available from the supplier shown in X2.1.13.

(a) Lifter Fill Chamber Operation—Install lifters in the lifter holding fixture in an upright position. Add sufficient quantities of EF-411 to cover lifters. One litre has proven satisfactory. Close chamber, start vacuum pump, and hold vacuum for 10 min. Maintain 15–20 in. Hg (381–508 mm) vacuum throughout filling. Release vacuum, open chamber,

raise holding fixture, and allow lifters to drain for 10 min. in upright position. Coat the lifter bodies with buildup oil prior to installation. Replace buildup oil in lifter chamber after filling three sets of lifters. Install the lifters in the cylinder head.

7.9.6.3 Install 1995 kit rocker arms, using Ford service tool, Part No. T74P-6565A. Install the camshaft drive sprocket, and measure and record the camshaft end-float.

7.9.6.4 Install the auxiliary shaft sprocket and align the cam drive. Torque the sprockets in accordance with the procedure noted in Footnote 14. Install a new camshaft drive belt. Tension the camshaft drive belt in accordance with the procedure noted in Footnote 14. Sealer may be required on the camshaft drive sprocket bolts if they have been used before.

7.9.6.5 Install the camshaft baffle and check to ensure the camshaft baffle does not come in contact with the RAC interior surfaces. The correct mounting orientation of the baffle is shown in Fig. A3.24.

7.9.7 Rocker Arm Cover-Install.

7.9.8 *Water Pump, Water Pump Drive, and Crankshaft Ignition-Trigger*—Install modified crankshaft trigger, the water pump, and the water pump and crankshaft pulleys (part numbers E9TZ6312A and E69Z8509D) or FZZZ6312A and F27A8509BA. Install the camshaft drive belt cover.

7.9.9 *Miscellaneous Parts*—Install the intake manifold and insert the spacer between the upper and lower half of the intake manifold. Install the freeze plugs.

NOTE 7—Alternatively, the intake manifold and freeze plugs can be installed after the engine is installed on the test stand.

7.10 *Engine Installation on the Test Stand*—Functions that are to be performed in a specific manner or at a specific time in the assembly process are noted.

7.10.1 Mounting the Engine on the Test Stand—Mount the engine on the test stand so that the flywheel friction face is 4.0 \pm 0.5° from vertical, with the front of the engine higher than the rear. The engine mounting system should be designed to minimize engine vibration at 750 and 2500 rpm (80 and 260 rad/s). Couple the engine directly to the dynamometer through a driveshaft. The engine cannot be used to drive any external engine accessory other than the water pump.

7.10.2 *Flywheel Timing Index Calibration*— If ignition timing is measured at the flywheel, calibrate the position of the flywheel index pointer to TDC of the number one cylinder (see 7.5.17) (**Warning**—A significant amount of backward crank-shaft rotation can cause the timing belt to jump time because of the arrangement of the belt tensioner.)

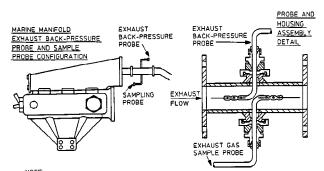
7.10.3 Exhaust Manifold:

7.10.3.1 *Installation*—Install the required water-cooled exhaust manifold shown in Fig. A3.8 and Fig. 3 (see supplier listed in X2.1.7).

7.10.3.2 Gas Sampling and Back-Pressure Measurement Fittings:

(*a*) The required fitting for exhaust gas sampling and back-pressure measurement is illustrated in Fig. 3 and detailed in A3.5.

(b) Install the exhaust gas sample and exhaust backpressure probes. Check all exhaust system connections to ensure they are secure.



NOTE: THE EXHAUST BACK-PRESSURE AND SAMPLE PROBES MAY BE POSITIONED AT ANY ANGLE AROUND THE CIRCUMFERENCE OF THE HOUSING, PROVIDED.-1. THE EXHAUST BACK-PRESSURE PROBE IS POSITIONED UPSTREAM AND THE EXHAUST GAS SAMPLE PROBE IS POSITIONED DOWNSTREAM 2. THE PROBE & IS COINCIDENT WITH THE PIPE &

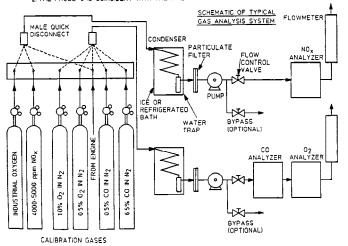


FIG. 3 Exhaust System and Exhaust Gas Analysis Apparatus

(c) The exhaust back pressure/exhaust gas sample probes can be used until they become unserviceable. If the existing probes are not cracked, brittle, or deformed, clean the outer surface and clear all port holes. Check the probes for possible internal obstruction and reinstall the probes in the exhaust pipe. Stainless steel probes are generally serviceable for several tests; mild steel probes tend to become brittle after one test. (Warning—Exhaust gas is noxious.) (Warning—Any leaks in the connections to the sample probe will result in erroneous O_2 readings and incorrect air-fuel ratio adjustment.)

7.10.4 Fuel Management System:

7.10.4.1 Fuel Rail Assembly:

(a) The fuel injectors may be used for several tests, provided the spray pattern and flow rate are checked (see 7.6.11). There is currently no specified life of the fuel injectors. Use only fuel injectors that have demonstrated flow rates of 3.7 to 4.5 oz (109 to 134 cm³) at 39 \pm 0.5 psi (269 \pm 3.4 kPa) for 60 s.

(b) Inspect the O-rings to ensure they are in good condition and will not allow air leaks. Install the fuel injectors into the fuel rail and into the lower intake manifold. Connect the wiring harness to the fuel injectors. The injectors should be paired so that Cylinders 1 and 4 and 2 and 3 are in parallel (see 7.10.4.2).

7.10.4.2 Injector Pulsing:

(*a*) The fuel injectors are pulsed in alternating pairs, the pairs consisting of Cylinders 1 and 4 and 2 and 3 respectively.

A schematic diagram of the necessary external connections to the injector wiring harness is shown in Fig. A3.44. Each injector pair is pulsed once per crankshaft revolution, without respect to crankshaft position. Typical pulse widths measured on the Ford EEC-IV system are described in the following section. A drawing showing what a pulse width represents is shown in Fig. A3.45. The pulse widths will vary slightly from engine-to-engine, although they should remain consistent for a given engine at a given fuel pressure. The measurement of fuel injector pulse widths is not required, although the measurement is a useful diagnostic tool.

(b) Typical fuel injector pulse widths for Stage 1 are 7.000 to 7.500 ms (approximately 38-psig [262-kPa] fuel pressure), for Stage 2 are 6.700 to 7.200 ms (approximately 38-psig [262-kPa] fuel pressure), and for Stage 3 are 2.500 to 3.500 ms (approximately 30-psig [207-kPa] fuel pressure). (The test stages are detailed in Table 2.)

7.10.4.3 Different Methods Currently Used:

(a) The fuel management controller hardware is not specified. The fuel management methods currently in use can be divided into five basic types of systems; modified Ford EEC-IV, open-loop analog, open-loop digital, closed-loop analog, and closed-loop digital. Process flow diagrams of typical open-loop and closed-loop digital systems are detailed in Fig. A3.22 and Fig. A3.23. Whatever fuel management system is used, there are specific ramping performance requirements (see 12.2.3.2). Use adequate precautions to ensure the fuel injectors cannot function while the engine is shut down. In addition, design the fuel management system to prevent excessive fuel input during engine startup.

(b) Exercise great care in the design, manufacture, and operation of the fuel management system to ensure that lambda is properly controlled throughout the test. It is recommended that the laboratory utilize the modified Ford EEC-IV system or an analog or digital system available from the suppliers detailed in X2.1.14.

7.10.5 Ignition System:

7.10.5.1 Components—Modifications to the ignition system are required. Modify the wiring harness by removing all circular connectors, except for the crankshaft ignition trigger connection. Figure A3.49 shows the required distributorless ignition system connections.

7.10.5.2 Spark Plugs-Install new Motorcraft AWSF-44 or AWSF-44C²³ spark plugs that have been gapped to between 0.042 and 0.046 in. (1.07 and 1.17 mm). Torque the spark plugs to between 5 and 10 lbf·ft (7 and 14 N·m). Install the spark plug wiring harness but do not install the distributor and ignition module at this time.

²³ Motorcraft AWSF-44 and AWSF-44C spark plugs are products of Ford Distribution Center, 11871 Middleton Belt, Levonia, MI 48150.

Controlled Stage				
Item	Controlled Parameter ^A			
	Falametei	1	2	3
Stage length	min	120	75	45
Engine speed	r/min	2500 ± 10	2500 ± 10	750 ± 25
Engine load	bhp	33.5 ± 0.5	33.5 ± 0.5	1.0 ± 0.5
Engine oil—Engine inlet temperature	°F	155 ± 2	210 ± 2	115 ± 2
Delta T (outlet-inlet)	°F	Record	Record	Record
Pump pressure	psig	Record	Record	Record
Engine gallery pressure	psig	Record	Record	Record
Delta P (pump-engine gall.)	psi	Record	15 max ^{<i>B</i>}	Record
Head gallery pressure	psig	Record	Record	Record
Delta P (engine gall-head)	psi	Record	Record	Record
Engine coolant—Engine out. temperature	°F	125 ± 2	185 ± 2	115 ± 2
Delta T (outinlet)	°F	Record	Record	Record
Flow rate	gal/min	15.0 ± 0.5	Record	
Pressure	psig	10 ± 1	10 ± 1	10 ± 1
Marine manifold outlet temperature	°F	Record	Record	Record
RAC coolant—Inlet temperature	°F	85 ± 2	185 ± 2	85 ± 2
Flow rate	gal/min	2.0 ± 2	Record	2.0 ± 2
Intake air-Temperature	°F	90 ± 2	90 ± 2	90 ± 5
Specific humidity	grains/lb	80 ± 5	80 ± 5	80 ± 5
Pressure	in. H ₂ O gage	0.2 ± 0.1	0.2 ± 0.1	0.2 ± 0.1
Engine breathing—Corrected blowby flow rate	cfm	2.00 ± 0.25		
Crankcase pressure	in. H ₂ O gage	Record	Record	Record
Intake manifold vacuum ^C	in. Hg gage	8 to 11	8 to 11	Record
Exhaust back pressure	in. Hg abs.	$30.6 \pm 0.3, -0.2$	$30.6 \pm 0.3, -0.2$	Record
Ignition—Timing	°BTDC	27 ± 1	27 ± 1	28 ± 1
Fueling—Consumption rate ^D	lb/h	Record	Record	Record
Pressure ^E	psig	Record	Record	Record
Exhaust gas analysis	0 ₂	1.1 ^F	1.1 ^{<i>F</i>}	0.7 max
	CO	0.4 max ^G	0.4 max ^G	6.5 ± 0.5
	NO _x .ppm		Record	

_. _. _ _ _

^A Maintain all parameters as close to midrange values as possible.

^B This specification only applies to Cycles 1 through 6.

^c Manifold vacuum is used as a diagnostics tool rather than a control parameter.

^D The fuel consumption rate shall be maintained below 18 lb/h at all times during the test.

^E Fuel pressure should be constant at a given operating condition and between 27 and 40 psig

F Target value is 1.1 %. Lower limit is 0.8 %. Upper limit 1.2 %.

^G The maximum individual reading should be below 0.7 %. The maximum average value should be below 0.4 %.

🕼 D 5302 – 01a

TABLE 2 (M) Sequence VE Operating Specifications (continued)

Parameter ^A		Stage			
	Parameter	1	2	3	
Time	Stage length, min	120	75	45	
Engine Loading	Engine speed, r/min	2500 ± 10	2500 ± 10	750 ± 25	
	Engine power, kW	25.00 ± 0.37	25.00 ± 0.37	0.75 ± 0.37	
Engine Oil	Engine inlet temperature, °C	68.3 ± 1.1	98.9 ± 1.1	46.1 ± 1.1	
	Delta T (outlet-inlet), °C	Record	Record	Record	
	Pump pressure, kPa	Record	Record	Record	
	Engine gallery pressure, kPa	Record	Record	Record	
	Delta P (pump-engine), kPa	Record	103.4 max ^{<i>B</i>}	Record	
	Head gallery pressure, kPa	Record	Record	Record	
	Delta P (engine-head), kPa	Record	Record	Record	
Engine Coolants	Engine outlet temperature, °C	51.7 ± 1.1	85.0 ± 1.1	46.1 ± 1.1	
	Delta T (outlet-inlet), °C	Record	Record	Record	
	Flow rate, L/min	56.8 ± 1.9	Record		
	Marine manifold outlet temperature, °C	Record	Record	Record	
RAC Coolant	Inlet temperature, °C	29.4 ± 1.1	85.0 ± 1.1	29.4 ± 1.1	
	Flow rate, L/min	7.6 ± 0.7	Record	7.6 ± 0.7	
Intake Air	Temperature, °C	32.2 ± 1.1	32.2 ± 1.1	32.2 ± 2.8	
	Specific humidity, g/kg	11.43 ± 0.71	11.43 ± 0.71	11.43 ± 0.71	
	Pressure, Pa	50 ± 25	50 ± 25	50 ± 25	
Engine Breathing	Corrected blowby flow rate, L/min	56.6 ± 7.1			
	Crankcase pressure, Pa	Record	Record	Record	
	Intake manifold vacuum ^C , kPa	24 to 34	24 to 34	Record	
	Exhaust backpressure, Abs kPa	104 to 0.68, + 1.66	104 to 0.068, + 1.00	Record	
Ignition and Fueling	Ignition timing, °BTDC	28 ± 1	28 ± 1	28 ± 1	
	Fuel consumption rate ^D , kg/h	Record	Record	Record	
	Fuel pressure ^E , kPa	Record	Record	Record	
Exhaust Gas Analysis	O ₂ , %	1.1 to 0.3, + 0.1	1.1 to 0.3, + 0.1	0.7 max	
	CO, %	0.4 max ^F	0.4 max ^F	6.5 ± 0.5	
	NOx, ppm		Record		

^A Maintain all parameters as close to midrange values as possible.

^B This specification only applies to Cycles 1 through 6.

^c Manifold vacuum is used as a diagnostics tool rather than a control parameter.

^D The fuel consumption rate must be maintained below 8.2 g·hr at all times during the test.

^E The maximum individual reading should be below 0.7 %. The maximum average should be below 0.4 %.

^F Fuel pressure should be constant at a given operating condition and between 186 and 276 kPa.

7.10.6 Crankcase Ventilation System:

7.10.6.1 *Oil Separator*—Cut the 1.25-in. (31.8-mm) inside diameter polyethylene hose to approximately 1.5-in. (38-mm) length and install the oil separator on the front of the RAC. Maintain the distance between the bottom of the separator adaptor (Item 4, Fig. A3.16) and the top of the RAC at 5.5 \pm 0.25 in. (139.7 \pm 6.4 mm) (see Fig. A3.15). The alignment of the oil separator with the RAC should provide a straight run with no offset (see Fig. A3.15 and Fig. A3.17). Maintain the separator assembly in a vertical position throughout the test. A bracket may be necessary to support the separator.

7.10.6.2 *PCV Valve*—Install clean hoses and a new, calibrated PCV valve as shown in Fig. A3.15 and Fig. A3.17. Use the elbow provided with the PCV valve to attach the 0.375-in. (9.53-mm) inside diameter hose. Attach a new 0.375-in. inside diameter hose to the three-way valve for each test. Maintain the PCV valve in a vertical position throughout the test.

7.10.6.3 *Air Horn Adapter*—Install a clean 0.625-in. (15.88-mm) inside diameter transparent hose to the air horn adapter (use new hose when the existing hose appears deteriorated).

7.10.6.4 *Three-Way Valve*—Install a clean three-way valve and attach the PCV valve hose. Install the remaining PCV valve hose between the three-way valve and the upper intake manifold (see Fig. A3.15). Do not allow the hose to flatten at the bend after installation.

7.10.6.5 *Inspection*—When the installation of the crankcase ventilation system is complete, inspect the system to ensure the

configuration will allow all oil condensate to drain to the elbow underneath the upper intake manifold. The pipe elbow leading to the intake manifold should be the lowest point of the system.

7.10.7 *Dipstick and Oil Fill Tubes*—Install the dipstick tube into the port on the left-hand side of the engine. Install the oil fill tube in the monolithic timing port on the left side of the engine (see 7.4.3 and 7.4.4).

7.10.8 *Intake Air Components*—Install the throttle body and intake-air horn.

7.10.9 *External Hose Replacement*—Inspect all external hoses used on the test stand and replace any hoses that have become unserviceable. Check for internal wall separations that could cause flow restrictions. Check all connections to ensure security.

8. Engine Fluids—Supply/Discharge Systems

8.1 Intake Air:

8.1.1 *Capacity*—The supply system shall be capable of delivering 80 scfm (38 L/s) of conditioned air, while maintaining the intake-air parameters detailed in Table 3. A typical intake air duct is shown in Fig. 1 (also see Fig. A3.3 and Fig. A3.4).

8.1.2 *Dew Point*—The dew point can be measured in the main system duct or at the test stand (X2.1.9 lists suppliers of suitable instruments). If the dew point is measured in the main system duct, verify the dew point periodically at the test stand. Maintain the duct surface temperature above the dew point

TABLE 3	Phillips	J Fuel	Analy	sis
---------	----------	--------	-------	-----

	1 ,		
Parameter	Tolerance (\pm)		
API gravity ^A	0.5		
RVP ^B	1.0 psig (6.9 kPa)		
Total sulfur ^C	0.01 g/gal (0.003 g/L)		
Existent gum ^D	1.0 mg/100 mL		
Distillation ^E	-		
IBP	16°F (8.8°C)		
10 %	10°F (5.6°C)		
50 %	11°F (6.1°C)		
90 %	8°F (4.4°C)		
EP	19°F (10.6°C)		
The following parameters are	e analyzed on an absolute basis:		
Color	clear		

Lead^F less than 0.01 g/gal (0.003 g/L) Oxidation stability^G ...

^A In accordance with Test Method D 287.

^B In accordance with Test Method D 323 or Automatic Reid Vapor Pressure Analyzer.

trace

^C In accordance with Test Method D 1266.

^D In accordance with Test Method D 381.

^E In accordance with Test Method D 86.

^F In accordance with Test Method D 3237.

^G In accordance with Test Method D 525.

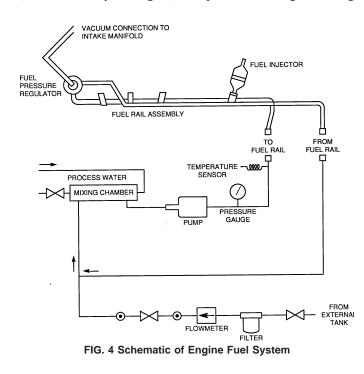
temperature at all points downstream of the humidity measurement point to prevent condensation and loss of humidity level.

8.1.3 *Filtration*—Since no air filtration is provided at the intake-air horn, the supply system shall provide either waterwashed or filtered air to the duct. The filtration system shall have sufficient flow capacity to maintain the specified intake-air pressure.

8.2 *Fuel*:

Water

8.2.1 *Description*—A schematic diagram of a typical fuel supply system is shown in Fig. 4. Supply an excess volume of fuel to the fuel rail at all times. Introduce make-up fuel (fuel used by the engine) into the loop from an external source. Mix the make-up fuel with fuel that is returned from the fuel rail (fuel not used by the engine). Pump the fuel through a mixing



chamber, or small heat exchanger, which is used to mix the two streams and provide fuel of consistent temperature to the engine. Deliver the fuel to a high-pressure pump, which boosts the pressure and supplies the fuel to the fuel rail.

8.2.2 *Controls*—Maintain the fuel temperature below the initial boiling point of Phillips J^{24} fuel. To ensure good atomization of the fuel, maintain the fuel pressure above 27 psig (186 kPa). In addition, the fuel pressure should be constant at all steady-state conditions to ensure good speed, power, and air-fuel ratio control.

8.2.3 *Fuel Volume Required*—Approximately 630 gal (2385 L) of Phillips J reference unleaded gasoline are required for each test.

8.2.4 *Fuel Batch Approval Process*—Each new batch of fuel is approved by the following process:

8.2.4.1 Before initial blending, each of the four fuel components is analyzed, and the data are compared with predetermined physical specifications. A small amount of fuel mixture is then blended, analyzed, and compared to predetermined specifications. The TMC determines the acceptability of the analytical data and authorizes blending of the entire batch for engine testing.

8.2.4.2 A sample of the fuel is shipped to two designated independent laboratories. A designed program involving more than one calibration test is completed using reference oils selected by the TMC. (The Sequence VE Reference Oils and Fuels Sub Panel, of Subcommittee B of ASTM Committee D-2, is involved in the design of the program.) The TMC reviews the test results and authorizes the fuel supplier to notify potential purchasers of the approval status of the fuel batch.

8.2.5 Fuel Batch Analysis:

8.2.5.1 Analyze each fuel shipment upon receipt from the supplier to determine the value of the parameters shown in Table 3 (except sulfur, oxidation stability, and distillation). Compare the results to the values obtained by the supplier on that particular batch. The results should be within the tolerances shown in parentheses beside each parameter. This provides a method to determine if the fuel batch is contained or has aged prematurely. If any results fall outside the tolerances shown in Table 3, the laboratory should contact the TMC for help in resolving the problem.

8.2.5.2 Analyze the contents of each fuel storage tank used for qualified Sequence VE tests every two months. Fuel in run tanks, those with a direct feed line to test engines, shall be analyzed monthly. It is recommended that laboratories take composite samples in accordance with Practice D 4057 as a guideline. Record on the fuel sample label the paragraph in Practice D 4057 that best describes the sampling method utilized. The fuel supplier provides an adequate supply of fuel sample containers with packaging and pre-addressed return labels to each Sequence VE laboratory. Upon receipt of fuel samples from the laboratory, the fuel supplier performs the

²⁴ Phillips "J" reference gasoline is a product of Phillips Chemical Co. Speciality Chemicals, P.O. Box 968, Borger, TX 79008. For general information about Phillips "J" fuel, see X2.6.

following analyses, tabulates the results, and reports results to the submitting laboratory.

Test Method D 323 Reid Vapor PressureTest Method D 381 Washed GumsTest Method D 287 API GravityTest Method D 381 Unwashed GumsTest Method D 86 DistillationTest Method D 3606 BenzeneTest Method D 3237 Lead(by way of Gas Chromatography)

(*a*) In instances where results from the tests listed previously appear to vary significantly from the expected results, a second sample shall be analyzed or the following tests shall be conducted, or both:

Test Method D 5134 Hydrocarbon Speciation (by way of Gas Chromatography) Test Method D 525 Oxidation Stability

Test Method D 873 Potential Gums

8.2.5.3 Forward the results of the analyses performed in 8.2.5.1 and 8.2.5.2 to the TMC for inclusion in the appropriate database.

8.2.6 *Fuel Batch Shipment and Storage*— Ship the fuel in containers with the minimum allowable venting as dictated by all safety and environmental regulations, especially when shipment times are anticipated to be longer than one week. Store the fuel following all applicable safety and environmental regulations.

8.3 Engine Oil:

8.3.1 Non-Reference Test Oil Description:

8.3.1.1 The non-reference sample shall be uncontaminated and representative of the lubricant formulation being evaluated.

8.3.1.2 A minimum of 2.0 gal (7.6 L) of new oil is required to complete the test. A 5-gal (18.9-L) sample of new oil is normally provided to allow for inadvertent losses.

8.3.2 System Description:

8.3.2.1 Design the oil system to minimize stand-to-stand variations that could influence test severity. Control the oil flow rate and pressure drop by specifying the volume, plumbing configuration, and orientation of the heat exchanger. Specify the location of the heat exchanger only in a vertical plane. Configure the heat exchanger so that the process water and engine oil are in counter-flow. The lengths of the lines are not specified, but the line length and diameter have a large influence on the volume of the external system. The internal volume of the entire external system shall be 18 ± 3 oz (530 \pm 90 mL).

8.3.2.2 Configure the external oil system in accordance with the schematic diagrams and photographs shown in Figs. A3.10 through A3.13. Use the remote filter adapter OHTA-007-1 for tests started on or after September 1, 1996. This adapter is available from the supplier shown in X2.1.28. Typical volumes of the various components of the external oil system are detailed in Fig. A3.10. Install a new Motorcraft FL-300²⁵ oil filter on the external oil filter adapter as shown in Fig. A3.12. Be sure all hoses and fittings on the oil heat exchanger are properly connected and secure. *Do not use brass and copper fittings in the external oil system since they may influence used oil (wear metals) analysis.*

8.3.3 *Heat Exchanger*—The heat exchanger has been chosen to minimize the volume of the external system. The heat exchanger has adequate but not excessive capacity to control the oil temperature. The system requires a high level of maintenance to provide adequate cooling, especially when process water temperatures are high. An effective, wellmaintained process water control system is necessary to achieve the specified oil temperatures. Carefully choose the controller and the system configuration to achieve proper response time, achieve the cyclic ramping specifications, and provide stable control at the three set points.

8.3.4 System Cleaning:

8.3.4.1 Clean the external oil cooling system thoroughly before each test. An acceptable technique for cleaning the oil heat exchanger is detailed in Annex A2. Flush and rinse the external lines before each test. The specific technique used (removed from or flushed on the stand, and so forth) is left to the discretion of the laboratory.

8.3.4.2 Regardless of the flushing technique employed, use an organic solvent (see 7.7.3) for the final flushing followed by separate rinses with hot (> 60° C) water and aliphatic naphtha before air-drying the components. (**Warning**—Incomplete cleaning of the external oil system may allow debris to dislodge and circulate throughout the engine during subsequent tests. Incomplete cleaning may also cause oil temperature control problems and contaminate subsequent test oils.)

8.3.5 *Engine Oil Pre-Lube Device*—Turn the engine oil pump with a portable power source to fill the galleries with oil after engine reassembly. A modified distributor, without a cap, rotor, or gear, and a ³/₈-in. (9.5-mm) electric drill are recommended to drive the oil pump.

8.3.6 *Control Specifications*—The oil inlet temperature and allowable oil pressure differential are specified in Table 2. Additional information concerning the oil pressure differentials is found in 12.5.6.2. Cyclic ramping specifications are detailed in Table 4.

8.4 Coolants:

8.4.1 *Description*—The engine and RAC coolant is a solution of demineralized (less than 2 grains/lb [0.034 g/kg]) or distilled water and an additive treatment—16 oz (470 mL) (fluid) of Pencool 2000^{26} per 4 gal (15 L) of water.

8.4.2 *General*—The following guidelines are common to both the engine and RAC coolant systems:

8.4.2.1 A transparent section is required to permit visual inspection of the coolant. Provide air blends to allow removal of entrained air. Provide a drain at the low point of the system to allow complete draining of the system.

8.4.2.2 Carefully choose the controller and the system configuration to achieve proper response time, achieve the cyclic ramping specifications, and provide stable control of the set points. An effective, well-maintained process water control system is necessary to achieve the specified coolant temperatures.

²⁵ Motorcraft FL-300 oil filter is a product of Ford Distribution Center, 11871 Middleton Belt, Levonia, MI 48150.

²⁶ The sole source of supply of Pencool 2000 known to the committee at this time is The Penray Companies, Inc., 1801 Estes Ave., Elk Grove, IL 60007. If you aware of alternative suppliers, please provide this information to ASTM Headquarters. Your comments will receive careful consideration at a meeting of the responsible technical committee,¹ which you may attend.

🖽 D 5302 – 01a

TABLE 4 Sequence VE Test Ramping Requirements^A

TABLE 4 (M) Sequence VE Test Ramping Requirements^A (continued)

Stage 3 to 1			
Engine speed	Attain 1600 r/min during 25–40 s Attain 2490 r/min during 55–75 s		
Engine torque	7–20 lbf-ft at 50 s		
	Attain 45 lbf-ft during 70–90 s		
22	Attain 65 lbf-ft during 85–105 s		
СО	<10 % CO at 10 s 10–15 % CO at 30 s		
	Attain 10 % CO during 60–85 s		
	12.7–15.3 % CO at peak of CO curve		
	<5 % CO at 90 s		
Oil inlet temperature	127–143°F at 5 min		
	153°F within 10 \pm 2 min		
Coolant outlet temperature	123°F within 4 min		
Stag	e 1 to 2		
Oil inlet temperature	175–190°F at 6.5 min		
	208°F within 13 \pm 2 min		
Coolant outlet temperature	146–167°F at 4.5 min		
Rocker arm cover inlet temperature	183°F within 9 ± 2 min 125–155°F at 6.5 min		
	183°F within 13 \pm 2 min		
Stag	e 2 to 3		
Engine speed	1500–1750 r/min at 55 s		
	Attain 775 r/min during 105–125 s		
Engine torque	Attain 45 lbf-ft during 10–30 s		
CO	7–20 lbf-ft at 55 s <1 % CO at – 30 s		
0	10.0–15.5 % CO at 35 s		
	Attain 10.0 % CO during 50–75 s		
	13.0–15.5 % CO at peak of CO curve		
Oil inlet temperature	149–177°F at 6.5 min		
	117°F within 13 \pm 2 min		
Coolant outlet temperature	131–169°F at 3.5 min		
Desker arm sover inlet temperature	117°F within 7 \pm 2 min		
Rocker arm cover inlet temperature	125–145°F at 6.5 min >95°F at 10 min		
	87° F within 13 ± 2 min		
4.0 1 1 1 1 1 1 1			

Stage	e 3 to 1			
Engine speed	Attain 1600 r/min during 25–40 s Attain 2490 r/min during 55–75 s			
Engine torque	7–20 lbf-ft at 50 s Attain 45 lbf-ft during 70–90 s			
со	Attain 65 lbf-ft during 85–105 s <10 % CO at 10 s 10–15 % CO at 30 s Attain 10 % CO during 60–85 s 12.7–15.3 % CO at peak of CO curve <5 % CO at 90 s			
Oil inlet temperature	127–143°F at 5 min 153°F within 10 ± 2 min			
Coolant outlet temperature	123°F within 4 min			
Stage 1 to 2				
Oil inlet temperature	175–190°F at 6.5 min 208°F within 13 ± 2 min			
Coolant outlet temperature	146–167°F at 4.5 min 183°F within 9 ± 2 min			
Rocker arm cover inlet temperature	125–155°F at 6.5 min 183°F within 13 \pm 2 min			
Stage	e 2 to 3			
Engine speed	1500–1750 r/min at 55 s Attain 775 r/min during 105–125 s			
Engine torque	Attain 45 lbf-ft during 10–30 s 7–20 lbf-ft at 55 s			
со	<1 % CO at – 30 s 10.0–15.5 % CO at 35 s Attain 10.0 % CO during 50–75 s 13.0–15.5 % CO at peak of CO curve			
Oil inlet temperature	149–177°F at 6.5 min 117°F within 13 ± 2 min			
Coolant outlet temperature	131–169°F at 3.5 min 117°F within 7 ± 2 min			
Rocker arm cover inlet temperature	125–145°F at 6.5 min >95°F at 10 min 87°F within 13 ± 2 min			

^A Speed and torque should be linear between setpoint specifications. Fuel flow not to exceed 18 lb/h. Exhaust gas oxygen not to exceed 4 %. Fuel flow and exhaust gas oxygen are not required to be plotted.

8.4.2.3 The system shall allow precise calibration of the flowmeters, after installation in the test stand. Avoid turbulence near the measurement meters and the flowmeters used for calibration.

8.4.3 Engine Coolant System:

8.4.3.1 Configure the engine cooling system according to the schematic diagram shown in Fig. A3.7, and the photographs in Fig. A3.8 and Fig. A3.9. Install a thermocouple into a modified thermostat housing and install the thermostat housing. *Do not install the thermostat*. Inspect the water pump drive V-belt for defects before installation. Install and tension the water pump drive V-belt.

8.4.3.2 A radiator cap is used to limit system pressure, although the coolant system pressure is not measured or controlled. The heat exchanger and control valve size are not specified, but the total coolant system capacity should be minimized.

8.4.3.3 The engine coolant flow rate and outlet temperature are controlled in accordance with the specifications listed in Table 2. Information concerning the coolant flow rate measurement device is detailed in 9.3.2. Cyclic ramping specifications are detailed in Table 4. The coolant flow rate is measured with a venturi flowmeter and varied with a control valve or a manual valve.

^A Speed and torque should be linear between setpoint specifications. Fuel flow not to exceed 18 lb/h. Exhaust gas oxygen not to exceed 4 %. Fuel flow and exhaust gas oxygen are not required to be plotted.

8.4.3.4 Maintain engine coolant system pressure at 10 ± 1 psig. Measure coolant system pressure at the top of the fluid level in the coolant reservoir.

8.4.4 RAC Coolant System:

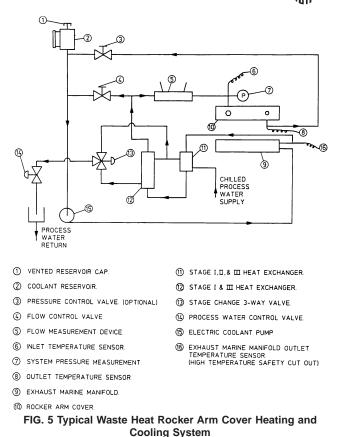
8.4.4.1 Periodically inspect and clean the complete RAC control system. If a high level of RAC coolant jacket deposit is found, flush the complete cooling system. A specific flushing technique is not specified. However, the technique should employ a commercial descaling cleaner.

8.4.4.2 Schematic diagrams of typical RAC coolant control systems are shown in Fig. 5 and Fig. 6. Heat for the control system can be derived from an external source, such as the marine manifold, an electric immersion heater, hot water, or steam.

8.4.4.3 Control the RAC coolant flow rate and inlet temperature in accordance with the specifications listed in Table 2. The coolant pressure is not specified, but design the system to minimize the pressure on the RAC and prevent distortion of the jacket. (**Warning**—Maintain the system pressure below 10 psig (69 kPa) to prevent distortion of the RAC jacket.)

8.4.4.4 Cyclic ramping specifications are detailed in Table 4.

8.4.5 Exhaust Manifold Coolant System:



8.4.5.1 Do not circulate coolant from the engine jacket through the exhaust marine manifold.

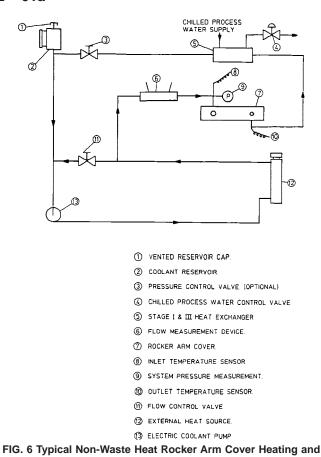
8.4.5.2 Exhaust system components downstream of the exhaust gas sampling and exhaust back pressure measurement fitting are not specified. Water-cooled exhaust plumbing downstream of the exhaust probes is a typical laboratory practice. The design should minimize buildup of corrosive materials in the exhaust system. The exhaust back pressure control equipment shall provide stable control within the limits specified. (Warning—Good engineering practices should be utilized to ensure safe operation of this system. The coolant outlet temperature of the marine manifold should be maintained below the boiling point of the coolant. High temperature, low water flow, and low water pressure alarms are recommended to prevent damage due to lack of cooling during engine operation.)

9. Measurement Instrumentation

9.1 Temperatures:

9.1.1 Equipment:

9.1.1.1 Temperature measurement equipment and locations for the seven required temperatures are specified. Alternative temperature measurement equipment shall be approved by the TMC. The accuracy and resolution of the temperature measurement sensors and the complete temperature measurement



Cooling System

system shall follow the guidelines detailed in ASTM Research Report RR: D02-1218.²⁷

9.1.1.2 If thermocouples are used, all thermocouples except the intake-air thermocouple shall be premium, sheathed, grounded types with premium wire. The intake-air thermocouple may be an open-tip type. Thermocouples of 0.125, 0.1875, or 0.25-in. (3.2, 4.8, or 6.4-mm) diameter may be used. However, 0.125-in. (3.2-mm) thermocouples are recommended at locations that require short immersion depths to prevent undesirable temperature gradients. Thermocouples, wires, and extension wires should be matched to perform in accordance with the special limits of error as defined in ANSI MC96.1. Either Type J (iron-constantan) or Type T (copper-constantan) thermocouples are acceptable (see ANSI MC96.1).

9.1.1.3 Resistance thermometer detectors (RTDs) are acceptable alternatives to thermocouples. However, if the TMC is not familiar with the particular RTDs used, the laboratory will be required to demonstrate equivalent system accuracy to the specifications noted in 9.1.1.2 (see ANSI MC96.1).

²⁷ ASTM Research Report RR: D02-1218, "Data Acquisition Guidelines— Instrumentation Accuracy and Calibration;" available from ASTM Headquarters. 100 Barr Harbor Drive, West Conshohocken, PA 19428.

9.1.2 *Engine Coolant Inlet*—Install the tip at the center of the flow stream in the perpendicular intersection of the tee fitting located 12 to 16-in. (300 to 410-mm) upstream from the water pump inlet (see Fig. A3.7). The recommended thermocouple diameter is 0.125 in. (3.2 mm).

9.1.3 *Engine Coolant Outlet*—Install the tip of the thermocouple at the center of the flow stream in the modified thermostat housing. The thermocouple tip shall be within 1 in. (25.4 mm) of the cylinder head. The inside diameter or inner orifice of the thermostat housing shall be no smaller than the Ford factory orifice and no larger than the cylinder head coolant inlet passage.

9.1.4 *Engine Oil Inlet*—Install the tip at the base of mounting face of the adapter block (see Fig. A3.10 and Fig. A3.13). The recommended thermocouple diameter is 0.125 in. (3.2 mm).

9.1.5 *Engine Oil Outlet*—Install the tip at the center of the cross fitting attached to the bottom of the heat exchanger (see Fig. A3.10). The recommended thermocouple diameter is 0.125 in. (3.2 mm).

9.1.6 *Intake Air*—Install the tip midstream in the intake-air horn adapter (see Fig. A3.4). A 0.25-in. (6.4-mm) thermo-couple is adequate if an open-tip thermocouple is used.

9.1.7 *RAC Coolant Inlet*—Install the tip at the center of the cross fitting attached to the RAC inlet fitting (see Fig. A3.14). The recommended thermocouple diameter is 0.125 in. (3.2 mm).

9.1.8 *Marine Manifold Coolant*—Install the tip at the perpendicular intersection of the tee fitting located on the outlet (upper) port of the marine manifold (see Fig. A3.8). Since the marine manifold coolant temperature is not a controlled parameter, a 0.25-in. (6.4-mm) thermocouple is adequate.

9.1.9 *Calibration*—Calibrate temperature sensors, including the engine oil in, engine coolant out, RAC and intake-air temperature measurement sensors prior to a reference oil test. The temperature measurement system shall indicate within $\pm 1^{\circ}$ F (0.6°C) of the laboratory calibration standard, which shall be traceable to national standards.

9.2 Pressures:

9.2.1 *Equipment*—Pressure measurement for each of the eight required parameters is detailed in the following sections. Specific measurement equipment is not specified. This allows reasonable opportunity for adaption of existing test stand instrumentation. However, the accuracy and resolution of the pressure measurement sensors and the complete pressure measurement system shall follow the guidelines detailed in ASTM Research Report RR: D02-1218.²⁷

NOTE 8—Tubing between the pressure tap locations and the final pressure sensors should incorporate condensate traps, as indicated by good engineering practice. This is particularly important in application where low air pressures are transmitted by means of lines that pass through low-lying trenches between the test stand and the instrument console.

9.2.2 *Intake Manifold Vacuum*—Measure the intake manifold vacuum at the top side of the upper intake manifold as detailed in Fig. A3.21.

9.2.3 *Oil Pressure*—Make oil pressure measurements at the oil filter adapter housing inlet, oil filter housing outlet, and the rear corner of the cylinder head, as shown in Fig. A3.13. A

multi-position selector valve can be utilized to connect the three oil pressure taps to a single pressure sensor. Individually dedicated pressure sensors may also be utilized.

9.2.4 *RAC Coolant Pressure*—Measure the RAC coolant pressure at the coolant inlet cross fitting, as detailed in Fig. A3.14.

9.2.5 *Fuel Pressure*—Measure the fuel pressure at the injector rail inlet as shown in Fig. 4. When utilizing a pressure gage mounted directly to the injector rail, the gage should be a damped, liquid-filled type.

9.2.6 *Intake-Air Pressure*—Measure the intake-air pressure at the air horn with the probe, as shown in Fig. 1 and detailed in Fig. A3.3, Fig. A3.4, and Fig. A3.5. If a manometer is used, install a liquid trap to prevent manometer fluid from entering the intake-air horn.

9.2.7 *Crankcase Pressure*—Measure the crankcase pressure at the dipstick tube. The sensor shall be capable of measuring positive and negative pressure. If a manometer is utilized, install a liquid trap to prevent manometer fluid from entering the crankcase.

9.2.8 *Exhaust Back Pressure*—Measure exhaust back pressure with the exhaust sampling probe downstream of the marine manifold (see Fig. 3 and Fig. A3.5). Install the exhaust gas sample and exhaust back pressure probe assembly against the marine manifold. A sensor capable of absolute measurement or a gage measurement corrected with a barometric pressure reading is acceptable. Install a condensate trap between the probe and sensor to accumulate water present in the exhaust gas.

9.2.9 Engine Coolant Pressure—Measure the coolant pressure above the fluid level in the engine reservoir, as shown in Fig. A3.7. Maintain engine coolant system pressure at 10 ± 1 psig (69 \pm 6.9 kPa).

9.2.10 *Calibration*—Calibrate all pressure sensors, including the RAC and engine coolant flow differential pressure and coolant pressure measurement sensors prior to each reference oil test. The calibration standard shall be traceable to national standards.

9.3 Flow Rates:

9.3.1 *Equipment*—Flow rate measurement procedures for each of the four required parameters are detailed in the following subsections. With the exception of the engine coolant and blowby flow rates, measurement equipment is not specified for a given parameter. This allows reasonable opportunity for adaption of existing test stand instrumentation. The accuracy and resolution of the flow rate measurement system shall follow the guidelines detailed in ASTM Research Report RR: D02-1218.²⁷

9.3.2 Engine Coolant—Determine the engine coolant flow rate by measuring the differential pressure drop across the specified venturi flowmeter (see Fig. A3.7). The pressure drop is approximately 18 in. (4.5 kPa) H_2O in the controlled flow range. Take precautions to prevent air pockets from forming in the lines to the pressure sensor. Transparent lines are beneficial in this application.

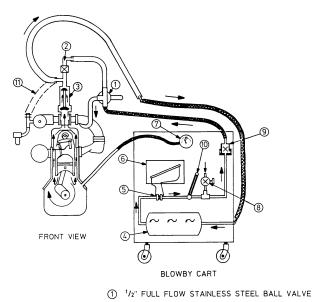
9.3.3 *RAC Coolant*—Measure the total volumetric coolant flow rate through the RAC.

9.3.3.1 *Calibration*—Calibrate the flowmeters used in the measurement of both the engine coolant flow rate and RAC coolant flow rate before every reference oil test. Calibrate the flowmeters as installed in the system at the test stand. Alternatively, the flowmeters may be detached from the test stand and calibrated, providing the adjacent upstream and downstream plumbing is left intact during the calibration process. Calibrate the flowmeters with a turbine flowmeter or by a volume/time method, at Stage 1 operating conditions.

9.3.4 Blowby:

9.3.4.1 Measure the blowby flow rate using the apparatus shown in Fig. 7. The measurement system routes the blowby through an external, sharp-edged orifice and into the engine intake manifold by means of an auxiliary (*dummy*) PCV valve. Mount the orifice plate with the flow in a horizontal position. Maintain crankcase pressure at 0.0 ± 0.1 in. (0.0 ± 25 Pa) during operation of the system to minimize the potential for crankcase leakage. *Mount the dummy PCV valve in a vertical position.*

9.3.4.2 Determine the blowby flow rate by measuring the differential pressure drop across the sharp-edged orifice; an inclined manometer or differential pressure sensor is required for measurement of the differential pressure drop. The crank-case pressure sensor shall have a range from 0 to 4 in. (0 to 1 kPa) H_2O and be adequately damped to indicate a zero gage pressure.



- (2) PCV VALVE.
- (3) OIL SEPARATOR
- (4) SURGE CHAMBER.
- (5) ORIFICE METER.
- 6 MANOMETER.
- (7) CRANKCASE PRESSURE GAGE
- 8 BLEED VALVE.
- DUMMY PCV VALVE.
- ① THERMOMETER OR TEMPERATURE SENSOR.
- (1) EXCESS BLOWBY VENT PIPE. (DURING NORMAL OPERATION).

FIG. 7 Blowby Measurement Apparatus

9.3.4.3 The sharp-edged orifice assembly is specifically designed for blowby flow rate measurement and shall be fabricated in strict compliance with the specifications, that are available from the TMC. The assembly contains five orifices. The 0.375-in. (9.53-mm) orifice is generally satisfactory for the range of blowby flow rate encountered. The complete orifice assembly can also be purchased from the supplier listed in Appendix X2. Additional details concerning fabrication of the flow rate measurement orifice assembly and measurement of blowby flow rate can be found in Footnote 14.

9.3.4.4 *Maintenance*—Clean the blowby measurement apparatus at least once every seven days. Replace the O-ring with each cleaning. Exercise particular care when cleaning the orifice meter assembly. Clean the three-way valve by soaking the valve in agitated organic solvent (see 7.7.3) until clean, followed by hot [>140°F (>60°C)] water rinse and spray rinse with aliphatic naphtha. Use compressed air to force air-drying. Inspect the port passages and remove any carbonaceous deposits by scraping. If the valve is disassembled for cleaning, make sure the core is properly seated upon reassembly. (Warning—Internal leakage within the three-way valve may cause some of the blowby gas to pass directly to the intake manifold from the test PCV valve and result in erroneous blowby flow rate measurements (see Fig. 7)).

9.3.4.5 *Calibration*—Calibrate the blowby orifice meters used for laboratory measurements every six months. Calibrate laboratory blowby measurements standards, used for the calibration of other meters, yearly. The calibration standard shall be traceable to national standards. Calibrate the temperature measuring devices in the blowby system every six months.

9.4 *Fuel Consumption*—Determine the fuel consumption rate by measuring the amount of make-up fuel flowing from the external fuel tank. The measurement point is upstream of the return flow from the fuel rail (see Fig. 4).

9.4.1 *Calibration*—Calibrate mass flow systems or gravimetric systems before every reference oil test. Volumetric systems shall be temperature-compensated and calibrated against a mass flow device. Values obtained with the test stand measuring device shall be within 1 % of the calibration standard values.

9.5 Speed and Load:

9.5.1 Required Capabilities—The dynamometer speed and load control systems shall be capable of maintaining the limits specified in Table 2 and meet the ramping requirements specified in Table 4 and Table 5. These limits require control within ± 0.4 % for operation during Stages 1 and 2. Because the dynamometer and driveline frictional losses may approach the Stage 3 load, manage the control input and system response during Stage 3 carefully to maintain engine operation within the specified tolerances. These tolerances are necessary to maintain a stable air-fuel ratio during Stage 3. Hydraulic dynamometers have a high residual load and may not be suitable for operation during Stage 3.

9.5.2 *Suitable Systems*—Both partial and full closed-loop systems have been successfully utilized. A typical partial closed-loop system maintains speed by varying dynamometer excitation and maintains load by a fixed throttle. A typical

TABLE 5	Sequence	VE	Break-in	Test	Condition	Ramping
		R	equireme	nts		

Break in Dipstick Calibra	tion Engine Start-Up to Step B			
Engine speed	Must reach 2490 rpm within 0.5 to 1.5 min			
Engine power	Must reach 33.5 \pm 0.5 bhp within 0.5 to 2.0 min			
Oil inlet temperature	127 to 143°F at 7 min			
	Must reach 208°F within 25 min			
Coolant outlet temperature	143 to 167°F at 8.5 min			
	Must reach 183°F in 13 \pm 2 min			
Rocker arm cover inlet temperature	>95°F at 2 min			
	125 to 145°F at 6.5 min			
	Must reach 183°F in 13 \pm 2 min			
Break li	n Step B to C			
Engine speed	Must reach 775 rpm within 0.5 to 2.0 min			
Engine power	Must reach 1 \pm 0.5 hp within 0.5 to 2.0 min			
Oil inlet temperature	149 to 177°F at 6.5 min			
	Must reach $117^{\circ}F$ in 13 ± 2 min			
Coolant outlet temperature	131 to 169°F at 3.5 min			
	Must reach 117° F in 7 ± 2 min			
Rocker arm cover inlet temperature	125 to 145°F at 6.5 min			
	>95°F at 10 min			
	Must reach 87°F in 13 \pm 2 min			

TABLE 5 (M) Sequence VE Break-in Test Condition Ramping Requirements (continued)

Break in Dipstick Calibration Engine Start-Up to Step B		
Engine speed	Must reach 2490 r/min within 0.5 to 1.5 min	
Engine power	Must reach 25.0 \pm 0.37 kW within 0.5 to 2.0 min	
Oil inlet temperature	52.8 to 61.7°C at 7 min Must reach 97.8°C within 25 min	
Coolant outlet temperature	61.7 to 75.0°C at 8.5 min Must reach 83.9°C in 13 ± 2 min	
Rocker arm cover inlet temperature	>35.0°C at 2 min 51.7 to 62.8°C at 6.5 min	
	Must reach to 83.9° C in 13 ± 2 min	
Break In Step B to C		
Engine speed	Must reach 775 r/min within 0.5 to 2.0 min	
Engine power	Must reach 0.75 \pm 0.37 within 0.5 to 2.0 min	
Oil inlet temperature	65.0 to 80.6°C at 6.5 min Must reach 47.2°C in 13 \pm 2 min	
Coolant outlet temperature	55.0 to 76.1°C at 3.5 min Must reach 47.2°C in 7 \pm 2 min	
Rocker arm cover inlet temperature	51.7 to 62.8°C at 6.5 min >35.0°C at 10 min Must reach 30.6°C in 13 ± 2 min	

closed-loop system maintains engine speed by varying dynamometer load and maintains engine load by varying the engine throttle position.

9.5.3 *Calibration*—Calibrate the load measurement and readout system with deadweights at least once per test. Calibration of the zero scale readout is recommended once per day when the test stand is in use. Calibrate the speed measurement system prior to each reference oil test.

9.6 Exhaust Gas:

9.6.1 Equipment:

9.6.1.1 Precision instruments for measurement of O_2 , CO, and NO_x are required. Equipment suitable for automobile

emission measurements is recommended. Precision nondispersive infrared instrumentation for CO and polarographic instrumentation for O₂ are suggested (see SAE J254). Response time is an important consideration in the performance of this instrumentation. A CO meter capable of measuring CO from 0 to 20 % is also required, for determining CO values during Stage 2 to 3 and 3 to 1 transitions. An extended range air-fuel ratio (UGEO) sensor can also be used to determine CO values during transitions. To decrease the response time of the meter, it is recommended that the CO meter or UGEO sensor be mounted as close to the stand as possible or in a portable installation.

9.6.1.2 A typical exhaust gas analysis system is shown in Fig. 3. The required instrument calibration gases are listed in Note of Fig. A3.6, and system design details are specified in Fig. A3.5 and Fig. A3.6. (Warning—Safety precautions are necessary concerning venting carbon monoxide, nitrous oxide, and ozone gases from the analyzer instruments.) (Warning—Long sample lines and unnecessary fitting connections or valves between the sample probe and the analyzer increase the potential for measurement error.)

9.6.2 *Calibration—General*—Calibrate the exhaust gas analysis equipment before each set of measurements taken during the test. The calibration technique should compensate for the flow rate sensitivity of the exhaust gas analysis meters.

9.6.3 *CO Meter Calibration for Transitions*—For ramping transitions, calibrate the CO meter from 0 to 15 % CO, as well as response time to ensure accurate readings during stage transitions. A zero and span calibration is required every day that the CO meter is used. No more than seven stage transition recordings or a maximum of one week is to transpire between a full stabilized and response calibration.

9.6.3.1 Conduct *stabilized* calibrations with the following nominal calibration gases: 0, 0.5, 6.5, and 15 % CO. The same trace recorder used for the transition traces (see Fig. A7.18) is used for the calibration. Zero and span the CO meter to the 15 % CO bottle. The calibration of the indicated and actual CO measurements of 0.5, 6.5, and 15 % CO shall be within 0.5 % and the zero and span within 0.2 % CO.

9.6.3.2 Conduct *response* calibrations with the 0.5 and 15 % nominal CO calibration gases. Use a solenoid valve to switch between the calibration sample bottles (see Fig. A7.19). Connect the solenoid valve directly to the longest sample line in the CO measurement system. Make the connection where the sample line would normally go onto the exhaust pipe. Use the same sample flow rate for the calibration as that used for transition trace recordings. The same trace recorder used for the transition trace recordings (see Fig. A7.18) is used for calibration. Show the solenoid switching voltage on one of the trace recorder channels. The solenoid valve switches the sample gas from the 0.5 to 15.0 to 0.5 % CO bottles at 40-s intervals. For satisfactory calibration, the indicated CO % shall duplicate the actual 15 % CO bottle value \pm 0.3 % CO (see Fig. A7.18).

9.6.3.3 If an UEGO sensor is used to determine CO values, correlate the sensor voltage output every six months or after 1000 h of use, whichever occurs first, to CO levels at the following conditions:

🕼 D 5302 – 01a

Speed	
Power	
Engine coolant outlet temperature	
CO levels at	

 $\begin{array}{l} 1500 \pm 10 \text{ r/min} (157.1 \pm 1.05 \text{ rad/s}) \\ 6 \pm 0.5 \text{ hp} (4.5 \pm 0.4 \text{ kW}) \\ 150 \pm 2^\circ\text{F} (65.6 \pm 1^\circ\text{C}) \\ 4, 6.5, 10, \text{ and } 15 \% \end{array}$

Install the UEGO sensor to avoid obstruction of the exhaust backpressure and air-fuel ratio probes. Where continuous air purge is maintained on the air-fuel ratio probe, install the sensor upstream of the exhaust backpressure and air fuel ratio probes.

9.7 *Humidity*—Measure humidity with the laboratory's primary humidity stream. Correct each reading for non-standard barometric conditions, using the following equation:

Humidity (corrected) =
$$4354 \times (Psat/(Pbar-Psat))$$
 (3)

where:

Psat = saturation pressure, in. Hg, and

Pbar = barometric pressure, in. Hg.

Metric Units:

Humidity (corrected) = $621.98 \times (Psat/Pbar-Psat)$) (4)

where:

Psat = saturation pressure, mm Hg, and

Pbar = barometric pressure, mm Hg.

9.7.1 Calibration—Calibrate the primary laboratory measurment system at each stand on a semiannual basis using a hygrometer with a minimum dew point accuracy of $\pm 1^{\circ}$ F at 60°F ($\pm 0.55^{\circ}$ C at 16°C). Locate the sample tap on the air supply line to the engine, between the main duct and 2 ft upstream of the intake air horn. The calibration consists of a series of paired humidity measurments comparing the laboratory system with the calibration hygrometer. The comparison period lasts from 20 min to 2 h with measurements taken at 1 to 6 min intervals, for a total of 20 paired measurements. The measurement interval shall be appropriate for the time constant of the humidity measuring instruments.

9.7.1.1 Verify that the flow rate is within the equipment manufacturer's specification, and that the sample lines are non-hygroscopic. Correct dew point hygrometer measurements to standard conditions (29.92 in. Hg [101.12 kPa]) using the appropriate equation (see 9.7). Compute the difference between each pair of readings and calculate the mean and standard deviation of the 20 paired readings, using equations Eq A13.1 and Eq A13.2 in Annex A13. The absolute value of the mean difference shall not exceed 10 grains/lb (1.43 g/kg) and the standard deviation shall not be greater than 5 grains/lb (0.714 g/kg). If these conditions are not met, investigate the cause, make repairs, and recalibrate. Maintain calibration records for two years.

10. Miscellaneous Laboratory Equipment

10.1 *Timing Light*—Use an inductive pickup timing light to measure ignition timing.

10.2 *Volumetric Graduates*—Measure the test oil quantity, fresh and used, by volume. A16-oz (500-mL) (fluid) graduate is recommended for the small samples (2 to 10 oz [60 to 300 mL]); this graduate shall have a resolution of 0.25 oz (7 mL) (fluid). A large graduate, capable of transferring 124 oz (4 L)

(fluid), is required, and it shall have a resolution of 1.0 oz (30 mL) (fluid). All graduates shall have an accuracy of 2 % of full scale.

11. Test Standard Calibration

11.1 Verification:

11.1.1 Verify the calibration of test stands with reference oils supplied by the TMC. Stand calibration tests are normally conducted upon expiration of the 180-day calibration time period. However, calibration time periods may be adjusted by the TMC. Any deviation from the standard calibration time frequency shall be approved by the TMC and reported on a supplemental operational data sheet of the final test report. Any non-reference test started within 168 days of the completion date of the previous calibration test is considered within the calibration time period.

11.1.2 A 3.0-gal (11.4-L) sample of reference oil is provided by the TMC for each stand calibration test.

11.2 Unacceptable Calibration Results:

11.2.1 It is recognized that a certain percentage of calibration tests will fall outside the acceptance limits because of the application of statistics in the development of the acceptance limits. Failure of a reference oil test to meet Shewhart or exponentially weighted moving average (EWMA) precision control chart limits can be indicative of a false alarm or a stand, laboratory, or industry problem. When this occurs, the laboratory in conjunction, with the TMC, shall attempt to determine the problem source. The TMC may solicit input from industry expertise (other testing laboratories, the test sponsor, ASTM Technical Guidance Committee, Sequence VE Surveillance Panel, Sequence VE Operations and Hardware Subpanel, and so forth) to help determine the cause and extent of a problem. Industry problems shall be adjudicated by the Sequence VE Surveillance Panel.

11.2.2 If the TMC determines the problem is a false alarm, and is stand related, there is no impact on non-reference tests running in other stands within the laboratory. If the TMC determines the problem is laboratory related, non-reference tests run during the problem period shall be considered invalid, unless there is specific evidence to the contrary for each individual test.

11.2.3 The TMC will reschedule a calibration test once it is satisfied that no particular problem exists or the problem has been resolved. The laboratory shall provide adequate documentation to support the conclusions reached during this process. This documentation shall be attached to the acceptable calibration test report. It shall provide sufficient information to show how the problem related to other tests operated during the same period of time.

11.3 *Test Stand Modifications*—Report modification of test stand control systems or completion of any nonstandard test on a calibrated stand to the TMC immediately. A nonstandard test includes any test completed under a modified procedure requiring hardware or controller tuning modifications to the test stand. The TMC will determine whether another calibration test is necessary after the modifications have been completed.

11.4 Reference Oil Accountability:

11.4.1 Laboratories conducting calibration tests are required to provide a full accounting of the identification and quantities of all reference oils used.

11.4.2 With the exception of new oil analysis required in 13.7.1, no physical or chemical analysis of new reference oils shall be performed without the permission of the TMC. Retain engine parts from reference oil tests for six months.

11.5 Test Numbering System:

11.5.1 Acceptable Tests—The test number shall follow the format AAA-BB-CCC. AAA represents the stand number. BB represents the number of tests since the last reference. CCC represents the total number of tests on the stand. As an example, 6-10-175 represents the 175th test on Stand 6 and the 10th test since the last reference. Consecutively number all tests on a given stand.

11.5.2 Unacceptable or Aborted Tests- If a calibration test is aborted or the results are outside the acceptance limits, the CCC portion of the test number for subsequent calibration test(s) shall include a letter suffix. The suffix shall begin with the letter A and continue alphabetically until a calibration test is completed within the acceptance limits. For example, if three consecutive unacceptable calibration tests are completed on the same test stand, and the test number of the first test is 6-0-175, the next two test numbers would be 6-0-175A and 6-0-175B. If the results of the next calibration test are acceptable, the test number 6-0-175C would permanently identify the test and appear on future correspondence. The completion of any amount of operational time on tests other than calibration tests will cause the test number to be increased by one. No letter suffix will be added to the test number of tests other than calibration tests.

12. Procedure

12.1 Pre-Test Procedure:

12.1.1 Engine Cooling System Flushing:

12.1.1.1 Use the following detailed procedure to flush the engine cooling system (use a flushing cart similar to the design shown in Fig. 2). The circulating pump shall be capable of flowing cleanser neutralizer and water at a rate of 25 ± 5 gal/min [95 \pm 19 L/min]. Calibrate the temperature, flow rate, and volume measuring equipment for the engine coolant flush cart at least every six months at the operating temperature of 150° F [65.6°C]).

12.1.1.2 Isolate the hoses leading from the venturi meter to the differential pressure sensor by closing the valves or removing the hoses at the venturi.

12.1.1.3 Remove the engine oil drain plug. This allows any accumulated build-up oil to drain. It also allows a way to determine the presence of any internal leaks while completing the flushing procedure.

12.1.1.4 Close the valve above the heat exchanger and isolate the coolant reservoir. This is necessary to prevent a loss of fluid during the flushing procedure and to avoid accumulation of flushing material in the coolant reservoir.

12.1.1.5 Isolate the water pump bypass by disconnecting the return hose from the intake manifold and routing it to the flush cart tank (see Fig. A3.41). Plug the water pump bypass port. (**Warning**—The flow rate through the intake manifold return hose should be choked down to prevent excessive flow rates

through the intake manifold during flushing. This is a safety precaution and will not prevent adequate flushing of the intake manifold.)

12.1.1.6 Connect the flush cart outlet hose to the water pump inlet. Connect the flush cart inlet hose to the engine coolant outlet from the heat exchanger.

12.1.1.7 Fill the engine and flush cart with tap water (demineralized or distilled water if the tap water has a high mineral content). Circulate the water through the engine and flush cart to heat the water. Continue circulation and heating until the water temperature reaches $150 \pm 10^{\circ}$ F ($66 \pm 6^{\circ}$ C).

12.1.1.8 Isolate the flush cart from the engine cooling system by opening the flush cart recirculation valve and closing the inlet and outlet valves.

12.1.1.9 Add the oxalic acid (see 7.7.6) at the rate of 3 oz/gal (23 g/L) of water in the flush cart system; add Petro Dispersant 425 (see 7.7.6) at the rate of 0.15 oz/gal (1 g/L). Add the cleanser directly to the flushing water.

12.1.1.10 Allow the oxalic acid to mix for 2 min in the flush cart. Open the inlet and outlet valves to the engine, and close the recirculation valve. Circulate the oxalic acid through the cooling system at a flow rate of 25 ± 5 gal/min (1.6 ± 0.3 L/s) for 45 min while maintaining a temperature of $150 \pm 10^{\circ}$ F ($66 \pm 6^{\circ}$ C).

12.1.1.11 Drain the cleanser from the engine cooling system and the flush cart. Flush the system with tap water (25 ± 5 gal/min, < 100° F [95 \pm 19 L/min, <43.7°C]) until the cooling system and flush cart drains run clear.

12.1.1.12 Close the cooling system and flush cart drains. Fill the engine and flush cart with tap water and circulate the water throughout the engine until the water temperature reaches $150 \pm 10^{\circ}$ F (66 \pm 6°C).

12.1.1.13 Isolate the flushing cart from the engine cooling system by opening the flush cart recirculation valve and closing the inlet and outlet valves.

12.1.1.14 Add the sodium carbonate at the rate of 0.05 oz/gal (3.8 g/L) of water in the flush cart system. Add the neutralizer directly to the flushing water.

12.1.1.15 Allow the neutralizer to mix for 2 min in the flush cart. Open the inlet and outlet valves to the engine and close the recirculation valve. Circulate the neutralizer through the cooling system at 25 ± 5 gal/min (1.6 \pm 0.3 L/s) for 45 min while maintaining a temperature of $150 \pm 10^{\circ}$ F (66 \pm 6°C).

12.1.1.16 Drain the neutralizer from the engine cooling system and the flush cart. Flush the engine cooling system and the flush cart with tap water at 25 ± 5 gal/min (95 ± 19 L/min) until the pH of the incoming and the outgoing water is the same. The temperature of the tap water shall be below 100° F (43.7°C).

12.1.1.17 Drain the cooling system, remove one or more freeze plugs, and inspect the coolant jacket. The cylinder walls should be clean and free of deposits when wiped. Install new freeze plugs quickly to help prevent the coolant jackets from rusting. If deposits are still present in the coolant jacket, repeat 12.1.1.6-12.1.1.17.

12.1.1.18 Disconnect the flush cart and reconnect the intake manifold coolant return line to the water pump bypass port. Reconnect the heat exchanger outlet to the water pump inlet.

Charge the engine and RAC with premixed coolant (described in 8.4.1) immediately to prevent rusting of the coolant jacket.

12.1.2 *Initial Test Oil Charge*—Reinstall the oil pan drain plug and charge 124 ± 1 oz $(3.67 \pm 0.03 \text{ L})$ (fluid) of the candidate test oil to the crankcase.

12.1.3 *Engine Pre-Lubrication*—Rotate the oil pump clockwise until oil pressure registers in the cylinder head gallery. Turn the crankshaft a minimum of two full revolutions during the pre-lubrication procedure to ensure that the camshaft is wetted with test oil during the initial engine start-up (see 8.3.5).

12.1.4 Engine Break-In Procedure:

12.1.4.1 The laboratory ambient atmosphere shall be reasonably free of contaminants. The temperature and humidity level of the operating area are not specified. Divert air from fans or ventilation systems away from the test engine.

12.1.4.2 Prior to Step A of the break-in procedure, start the engine and adjust timing to 28° BTDC at 750 r/min.

12.1.4.3 During Step A (see Table 6 and Table 7), set the ignition-timing at 28° BTDC and allow the oil and coolant temperatures to reach 115°F (46°C). Adjust the fuel injector pulse width to provide 6.5 % CO maximum in the exhaust. Bleed the air from the engine and RAC coolant systems.

12.1.4.4 During Steps B and C (see Table 6 and Table 7), all parameters shall be controlled and recorded as normal test Stage 2 and Stage 3 conditions. Record all *normal* parameters in Steps B and C after operation for 35 min. In addition, record fuel pressure and RAC coolant pressure during Steps B and C.

NOTE 9-The fuel pressure and the RAC coolant pressure are not recorded after the break-in.

12.1.4.5 At the earliest opportunity in each step, adjust the fuel injector pulse width to provide the specified values for CO and O_2 in the exhaust. (**Warning**—Prolonged operation at a rich air-fuel ratio can cause excessive fuel dilution and alter test severity.)

NOTE 10—The engine will normally require approximately 15 min to reach steady-state conditions after a stage change.

12.1.5 *Dipstick Calibration*—After the completion of Step A (see Table 6), shut the engine down (see 12.2.2). Disconnect the intake-air supply immediately after shutdown to cause the inlet air pressure to fall to atmospheric. After 20 min, calibrate

TABLE 6	Sequence	VE Break-in	Schedule
---------	----------	-------------	----------

		Step	
Conditions	A (Timing Run)	В	С
Time per step (minutes)	10	40	40
Cumulative time (minutes)	10	50	90
Engine speed (rpm)	1500 ± 25	2500 ± 10	750 ± 25
Engine load (bhp)	2 ± 0.5	33.5 ± 0.5	1 ± 0.5
Water out (°F)	115 ^A	185 ± 2	115 ± 2
Oil into engine (°F)	115 ^A	210 ± 2	115 ± 2
RAC inlet (°F)	85 ^A	185 ± 2	85 ± 2
RAC flow (gpm)	2 ± 0.2		2 ± 0.2
RAC pressure (psig)		10 max	10 max
Fuel pressure (psig) ^B		27 to 40	27 to 40
Exhaust gas CO (%)	6.5 max	0.4 max	6.5 ± 0.5
Exhaust gas O ₂ (%)		1.1 to 0.3, + 0.1	0.7 max

^A Target value.

^B Fuel pressure should be constant at steady state but within the given range.

TABLE 7 (M)Sequence VE Break-in Schedule

		Step	
Conditions	A (Timing Run)	В	С
Time per step (minutes)	10	40	40
Cumulative time (minutes)	10	50	90
Engine speed (rpm)	1500 ± 25	2500 ± 10	750 ± 25
Engine load kW	1.50 ± 0.37	25.00 ± 0.37	0.75 ± 0.37
Water out °C	46.1 ^{<i>A</i>}	85.0 ± 1.1	46.1 ± 1.1
Oil into engine °C	46.1 ^{<i>A</i>}	98.9 ± 1.1	46.1 ± 1.1
RAC inlet °C	29.4 ^A	85.0 ± 1.1	29.4 ± 1.1
RAC flow L/min	7.6 ± 0.7		7.6 ± 0.7
RAC pressure kPa		68.9 max	68.9 max
Fuel pressure kPa ^B		186 to 276	186 to 276
Exhaust gas CO (%)	6.5 max	0.4 max	6.5 ± 0.5
Exhaust gas O ₂ (%)		1.1 to 0.3,	0.7 max
		+ 0.1	

^A Target value.

^B Fuel pressure should be constant at steady state but within the given range.

the dipstick *full* mark at the existing sump level and set the lock nuts on the adjustable dipstick.

12.1.6 Oil Filter Removal and Filter Dummy Installation— After completion of Step C (see Table 6 and Table 7), shut the engine down (see 12.2.2). Remove the oil filter 20 min after stopping the engine. Install the oil filter *dummy* cap to the remote filter housing. Check and record the oil level. *Do not adjust the dipstick.* This oil level check is provided to establish the level of fuel dilution present after the break-in and should not be used for computing the overall test oil consumption.

12.2 Engine Operating Procedure:

12.2.1 *Engine Start-up*—Use the following detailed procedure each time the engine is started. (**Warning**—Before starting the engine, be sure there is an adequate supply of cooling water to the exhaust marine manifold. Without coolant flow, the marine manifold will overheat and sustain serious damage.)

12.2.1.1 Turn on the ignition, safety circuits, fuel management system, fuel pump, and the RAC coolant pump.

12.2.1.2 Set the fuel injector pulse width and connect the intake-air supply duct.

12.2.1.3 Crank the engine. After the engine has started, the pulse width should be returned to the Stage 3 setting at the earliest opportunity.

12.2.1.4 The fuel management, cranking, and throttle control systems should be set up to allow the engine to start within 10 s. Since spark ignition engines require a rich air-fuel ratio during startup, a Stage 1 or Stage 2 pulse width is normally necessary for hot and cold starts.

12.2.1.5 If starting difficulties are encountered, the laboratory should not continue to crank the engine excessively. Perform diagnostics to determine the reason the engine will not start (ignition problems, insufficient or excess fuel, and so forth). (**Warning**—Excessive cranking times can promote additional fuel dilution of the test oil and increased engine wear.) (**Warning**—In addition to other precautions, do not attempt to pour gasoline into the intake-air horn.)

12.2.2 Engine Shutdown:

12.2.2.1 *Scheduled Shutdown Procedure*—Follow the procedure detailed as follows each time a scheduled shutdown is performed. Scheduled shutdowns include shutdowns that occur during engine break-in and oil leveling:

- (a) Bring the engine speed to 750 r/min.
- (b) Switch the ignition off.
- (c) Reduce the intake-air pressure to atmospheric.

12.2.2.2 *Unscheduled Engine Shutdown*—Follow the procedure detailed as follows each time an unscheduled engine shutdown is performed:

- (a) Bring the engine speed to 750 r/min.
- (b) Switch the ignition and RAC coolant pump off.
- (c) Reduce the intake-air pressure to atmospheric.

12.2.3 Cyclic Schedule, General Description:

12.2.3.1 The test is composed of three stages as shown in Table 2. Together, the three stages comprise one cycle. Each cycle lasts 4 h and is repeated 72 times for a total of 288 h. Six consecutive cycles are completed each 24-h period. Every sixth cycle is modified to provide time for oil leveling (see A7.9).

12.2.3.2 Ramping requirements specifying parameter change rates are shown in Table 4. The rate of speed, temperature, and load changes, as well as the amount of enrichment between stages, can influence sludge and wear severity. Therefore, ramping rates are very important. Plot speed, load (torque), exhaust gas, CO, and O₂ levels during Transitions 2 to 3 and 3 to 1, as shown in Fig. A7.17. Record these parameters once during test hours 12 through 48 for all tests. If a stand calibration test is conducted, also record these parameters once during test hours 240 to 288. Format and scaling shall conform to Fig. A7.17.

12.2.4 Unscheduled Downtime—The 30-min oil leveling periods are the only scheduled shutdown allowed during the test and are counted as test time. However, the test can be interrupted to perform necessary maintenance (see 12.3.6). All unscheduled downtime shall be noted in the final test report.

12.2.5 Lost Time—This is defined as time lost with respect to a master clock in an automatic control system. As an example, if the engine stalls but the master clock continues to run, the time that elapses between the engine stall and resumption under operating conditions is *lost time*. The test clock is restarted after the engine has reached operating conditions. Lost time shall not exceed 60 min total throughout the entire test. Make up *lost time* in the correct stages at some time during the test, and report it in the final test report.

12.2.6 *Resumption of Test Time After Unscheduled Shutdown*—After an unscheduled shutdown, test time does not begin until the engine has reached operating conditions.

12.3 Periodic Measurements and Functions:

12.3.1 *Data Logging*—Measure and record all data, except the specific parameters noted in this section, immediately prior to the midpoint and conclusion of Stage 1, the conclusion of Stage 2, and the conclusion of Stage 3. The format for the data logs is shown in Annex A5. Special considerations concerning computerized data acquisition are detailed in Annex A9.

12.3.2 Blowby Flow Rate Measurement—Measure and record the blowby flow rate during the second hour of Stage 1 of each cycle. The engine shall be stable at normal Stage 1 operating conditions. Measure blowby when the gas temperature is at least 90°F (32°C). Blowby gas temperature shall not differ from the laboratory average by more than \pm 10°F (5.6°C). The installation of the blowby flow rate measurement apparatus is shown in Fig. 7. The procedure for measuring

blowby flow rate is detailed in 12.3.2.1. Normally, only one blowby flow rate measurement should be completed during each cycle. Additional blowby flow rate measurements are occasionally performed to determine or verify a problem with the flow rate measurement apparatus or the engine. Record additional blowby flow rate measurements and an explanation of the reason for the additional measurements. Include these data in the supplemental operational data in the final test report.

12.3.2.1 Measurement Procedure:

(a) Open the bleeder valve completely.

(b) Connect the bleeder line to the three-way valve.

(c) Connect the hose from the blowby meter surge chamber to the 0.625-in. (15.9-mm) inside-diameter air vent hose on the intake horn.

(*d*) Position the three-way valve to divert intake manifold vacuum from the engine PCV valve to the *dummy* PCV valve in the blowby measurement apparatus.

(e) Connect the apparatus pressure sensor to the dipstick tube.

(f) Adjust the bleeder valve to maintain crankcase pressure at 0.0 \pm 0.1 in. H₂O (0 \pm 25 Pa).

(g) Record the differential pressure, blowby gas temperature, and the barometric pressure.

(*h*) Disconnect the apparatus pressure sensor and reconnect the engine crankcase pressure sensor to the dipstick tube.

(i) Disconnect the surge chamber hose from the air vent hose.

(*j*) Position the three-way valve to divert intake manifold vacuum to the engine PCV valve and disconnect the bleeder line from the three-way valve.

(k) Connect the air vent hose to the intake-air horn.

(*l*) Calculate the blowby flow rate using the calibration data for the orifice. Correct the value to standard conditions using the following equation:

Corrected blowby flow rate = blowby flow rate

$$\times (18.844 (P/459.67 + T))^{0.5}$$
 (5)

where:

P = barometric pressure, in. Hg, and

T = blowby gas temperature, °F.

12.3.3 *Ignition Timing Measurement*—Measure and record the ignition timing at least once per two cycles in Stage 1, 2, or 3.

12.3.4 Exhaust Gas Analysis:

12.3.4.1 The recorded readings for exhaust gas CO and O_2 content are taken in accordance with the instructions in 12.3.1. However, the exhaust gas CO and O_2 content should be measured immediately after the engine oil and coolant temperatures have reached steady-state conditions in each stage (approximately 15 min into each stage). These data do not have to be recorded. The data are used to adjust the fuel management system to help ensure the engine is operated at the correct air-fuel ratio throughout each stage.

12.3.4.2 Measure and record the NO_x content of the exhaust gas during Stage 2 once every 24 h. The intent is to take subsequent readings after 24 h of run time have elapsed.

12.3.5 *PCV Valve Replacement*—Replace the PCV valve at 48, 96, 144, 192, and 240 h during the oil-leveling procedure

(see 12.3.6.3). Retain the used PCV valves for possible after-test clogging measurements.

12.3.6 Oil Additions and Used Oil Sampling:

12.3.6.1 The test procedure provides a forced rate of oil consumption. Make oil additions at 12-h intervals, and take used oil samples at 24-h intervals. No other new oil additions are permitted during the test. Used oil additions are permitted only during engine reassembly for maintenance (see 12.4.2.2). The new oil addition and sampling procedure is detailed as follows. The procedure is shown in a flowchart format on the Oil Sampling, Addition and Leveling Worksheet in Fig. A5.1. This form serves as the oil sampling and oil addition data sheet.

12.3.6.2 Oil Sampling Procedure:

(*a*) Pour 5 oz (150 mL) (fluid) of new oil in a beaker. Fifteen minutes after the engine has been switched to Stage 3, remove a 5-oz (150-mL) (fluid) purge sample into the beaker containing the new oil.

(*b*) Remove a 2-oz (60-mL) (fluid) analysis sample from the engine. Label the sample bottle for identification of test number, date, test hour, and oil code (see 13.7).

(c) Return the 10-oz (300-mL) (fluid) composite of new oil and purge oil to the engine at all sample hours except hour 288. Take precautions to prevent any oil loss.

(d) Do not add new oil to the engine at hour 288. This allows the final drain to be used as a backup to the 288-h sample.

12.3.6.3 Oil Leveling Procedure:

(a) After the engine is switched to Stage 3, add 5 oz (150 mL) (fluid) of new oil to the engine.

(b) Run the engine for 10 min at Stage 3 conditions.

(c) Switch the ignition off but leave the RAC coolant pump on.

(d) Bring the intake-air pressure to atmospheric.

(e) Measure and record the oil level 20 min after the engine is shut down.

(*f*) If the oil level is more than 12 fluid oz (360-mL) low during the first 192 h, invalidate the test because of excessive oil consumption. Use engineering judgment concerning test invalidation and termination if the oil level is more than 12 fluid oz (360-mL) low during the remainder of the test.

(g) If the oil level is above full mark, remove a sufficient quantity to bring the sump level to full.

(h) If the oil level is at or below the full mark, no leveling sample can be taken.

(*i*) Record all appropriate entries on the data log.

(*j*) Service the PCV valve in accordance with 12.3.5.

(*k*) Restart the engine 5 min before the scheduled beginning of Stage 1 and run the engine at Stage 3 speed and load.

(*l*) Resume the test at the scheduled beginning of the next cycle.

12.3.7 *General Maintenance*—The 30-min scheduled shutdown periods during oil leveling allow limited opportunity for engine and stand maintenance. In addition, the test can be shut down at any convenient time to perform unscheduled maintenance. However, the duration of a shutdown should be minimized. Report any unscheduled shutdown on the Special Maintenance Record. Report any attempts to restart the engine after stalling when resuming test conditions from a scheduled or unscheduled shutdown as additional *start attempts* on the Supplemental Operational Data Form, Fig A7.5.

12.4 *Special Maintenance Procedures*— Functions that require special maintenance procedures are listed in this section. These maintenance procedures are specifically detailed because of the effect on test validity or because they require special care while being completed.

12.4.1 Blowby Flow Rate Adjustment-The blowby flow rate can be adjusted if the cumulative average is projected to drift outside the specified limits. Adjust the blowby flow rate by increasing or decreasing the size of the compression ring gaps. Only two ring gap adjustments (one rering and one regap, or two regaps) may be performed during one test. Measure and record the existing ring gap dimensions before enlarging the gaps to calculate ring gap increase due to wear at the end of the test. Install a new set of torque-to-yield cylinder head bolts when reassembling the engine after the blowby flow rate adjustment. Specific limitations concerning high and low blowby flow rate adjustments are noted as follows. If stuck rings are discovered during a rework, any corrective action such as freeing the stuck ring(s) shall be noted in the Special Maintenance section of the final test report. Note whether the ring(s) are cold or hot stuck.

12.4.1.1 *High Blowby Flow Rate Adjustment*— An adjustment for high blowby flow rate can be performed only once and shall be completed during the first 48 h of the test. Reduce high blowby flow rate by replacing the compression rings with new rings that have smaller ring gaps.

12.4.1.2 Low Blowby Flow Rate Adjustment— Make adjustments for low blowby flow rate during the first 120 h of the test. A maximum of two adjustments for low blowby flow rate can be performed. Increase low blowby flow rate by increasing the ring gaps of the compression rings. Do not remove any deposits from the ring or ring land except those located in the ring gap.

12.4.2 Engine Disassembly and Reassembly for Maintenance (Before EOT):

12.4.2.1 When the engine is disassembled for maintenance, drain as much test oil as possible from the oil pan, and retain the oil for installation in the engine after reassembly. Take precautions to ensure the oil is not contaminated. Take precautions to ensure the deposits are not disturbed on any parts that are used to determine the final test results.

12.4.2.2 During reassembly, up to 6 oz (180 mL) (fluid) of the most recent, available oil-leveling sample may be used to lubricate the engine parts. Do not use EF-411 oil or new test oil during engine reassembly. After the engine has been reassembled, charge the oil pan with the oil removed from the oil pan during reassembly. Record all excess oil additions and report them in the Supplemental Operational Data.

12.4.3 *Engine Pre-Lubrication Prior to Restart*—Prelubricate the engine in accordance with 12.1.3 each time the oil charge has been removed from the engine or when the length of the shutdown exceeds 8 h.

12.4.4 *Engine Parts Replacement*—Parts that are rated to determine the final test results cannot be replaced during the test. The following parts can be replaced if necessary (record

the circumstances involved in the replacement): engine camshaft drive belt, ignition system, PCV valve (see 12.5.5), seals and gaskets, valves or valve springs, fuel injectors, oil separator (PCV system), piston rings, and spark plugs.

12.5 *Diagnostic Data Review*—This section outlines significant characteristics of specific engine operating parameters. The parameters can directly influence the test or may be used to indicate normalcy of other parameters.

12.5.1 Intake Manifold Vacuum:

12.5.1.1 Intake manifold vacuum is affected by several factors, including engine load, air-fuel ratio, ignition timing, and engine wear. As a result, it is not a specifically controlled parameter but is used to monitor the condition of the engine.

12.5.1.2 The intake manifold vacuum normally decreases from Stage 1 to Stage 2, even though the engine speed, load, and lambda are the same. This is caused by the increased air temperature in the intake manifold reducing the density of the intake-air charge. The intake manifold vacuum generally increases during the first few cycles of the test, as the engine breaks-in. It may decrease toward the end of the test if the engine wears significantly. The target range offers adequate flexibility to account for differences in engine builds and barometric pressure. Operation outside of the target range suggests possible errors in load, ignition timing, exhaust gas composition, or combinations thereof.

12.5.2 Fuel Consumption Rate:

12.5.2.1 The fuel consumption rate should remain relatively constant throughout the test. Like intake manifold vacuum, fuel consumption rate is not a specifically controlled parameter but is used as a diagnostics tool. In addition, fuel consumption rate and intake manifold vacuum relate to similar operating parameters.

(a) High fuel consumption rate can promote excessive cylinder bore, camshaft, and rocker arm wear. Maintain the fuel consumption rate below 18 lb/h (8.2 kg/h) throughout the entire test.

12.5.2.2 The fuel consumption rate is normally greater in Stage 1 than in Stage 2 even though the engine speed, load, and air-fuel ratio are the same. This is caused by the increased operating temperatures in Stage 2 that allow an increase in engine efficiency and a decrease in brake specific fuel consumption (bsfc). Excessive fluctuation of the fuel consumption rate at steady-state speed and load may indicate problems with the fuel management or fuel delivery systems.

12.5.3 *Spark Knock*—Spark knock does not normally occur in the VE test. The octane rating of the fuel, ignition timing, engine speed and load, and operating temperatures do not promote spark knock. Spark knock indicates abnormal combustion is occurring and can cause extensive engine damage. Take corrective action immediately if spark knock is noted. Errors in the measurement and control of engine load, ignition timing, operating temperatures, and air-fuel ratio may result in spark knock.

12.5.4 Exhaust Gas Component Levels:

12.5.4.1 The CO, O_2 , and NO_x levels in the exhaust gas are used to determine the characteristics of combustion that occur during the test. Use these three parameters in conjunction to

determine the normalcy of combustion and any significant changes in combustion that occur throughout a particular test.

12.5.4.2 Exhaust gas content measurements of CO and O 2 are used to ensure the engine is being operated at a specific lambda during a specific stage. Measurements of CO and O₂ are used because a specific lambda measurement has not been developed, and a numeric value for each lambda has not been determined. The lambda for each stage was chosen to produce specific effects. During Stages 1 and 2, the engine is operated at a lambda value greater than one (leaner than stoichiometric). This causes the engine to be operated at a relatively flat portion of the NO_x versus O_2 curve, near the point of maximum NO_x. During Stage 3, the engine is operated at a lambda value less than one (richer than stoichiometric). This helps dilute the crankcase oil with fuel. The presence of components other than the component that is measured by a particular instrument can influence the accuracy of the measurement. The laboratory shall ensure the accuracy of the measurement of each specific component.

12.5.4.3 Exhaust gas CO and O_2 content are useful indicators of various combustion and exhaust gas sampling problems. Content (CO) increases relatively linearly with decreasing lambda (increasing richness), for lambda values less than one (richer than stoichiometric). Content (O_2) increases relatively linearly with increasing lambdas for lambda values greater than one.

12.5.4.4 High CO levels indicate excess fuel is being introduced into the engine, while low CO levels indicate too little fuel is being introduced into the engine. High or erratic CO levels can be caused by a variety of problems, including an air leak in the sampling system (The high CO results from excessive reduction of lambda to reduce the O_2 level.); PCV valve that is excessively clogged; or a problem with the fuel management system.

12.5.4.5 The presence of high O_2 levels and normal CO levels is usually caused by a slight misfire or a leak in the exhaust gas sampling or measurement system. The misfire can be caused by a plugged fuel injector or an ignition problem.

12.5.4.6 The VE test has been set up to produce significantly elevated exhaust gas NO_x levels when compared to a production engine. This raises the NO_x levels in the blowby gas and increases test severity. The NO_x levels should be consistent throughout a test and relatively consistent from test-to-test. Exhaust gas NO_x levels are primarily a function of combustion temperatures. As a result, the NO_x levels are influenced by several factors. The cylinder head, camshaft, ignition timing, and lambda have strong effects on NO_x levels. Load and engine operating temperatures have a smaller effect. Accurate NO_x measurement is very sensitive to sample flow rate.

12.5.5 *Crankcase Pressure*—Crankcase pressure is a function of blowby flow rate and PCV valve flow. High crankcase pressure is usually caused by high blowby flow rate or a significant loss of PCV valve flow. Incorrect three-way valve plumbing or port plugging also promotes high crankcase pressure. Low or negative crankcase pressure may be caused by low blowby flow rate or a restriction of vent air to the PCV valve.

12.5.6 Oil Pressure:

12.5.6.1 The oil pressure is a function of oil viscosity and operating temperature. The oil pressure will normally be highest in Stage 1 and lowest in Stage 3. The oil pressure should remain consistent throughout the test, unless the oil exhibits a significant increase in viscosity.

12.5.6.2 An excessive oil pressure differential between the pump output and engine gallery can indicate the presence of a restriction in the external system or a large bearing clearance. An excessive oil pressure differential between the engine gallery and head gallery indicates the presence of a gallery restriction at the head gasket or an increased oil flow rate to the valve deck. When the engine gallery-to-head gallery differential pressure is lower than normal, camshaft lobe orifice plugging is indicated, causing a reduction in oil flow rate through the camshaft.

12.5.7 *Oil Temperature Differential*—The oil temperature differential is primarily a function of oil flow rate and oil viscosity and is normally stable throughout the test. The differential can increase if the oil viscosity increases significantly during the test.

12.5.8 *Coolant Temperature Differential*— The coolant temperature differential is primarily a function of the coolant flow rate and is normally stable throughout the test. Large variations in the differential can be caused by coolant flow rate or temperature measurement errors. Coolant flow rate measurement errors can be caused by foreign objects in or near the venturi flowmeter.

12.6 End of Test Procedure:

12.6.1 *Engine Removal*—Drain the engine and RAC coolant immediately after the completion of the last test cycle. Do not drain the engine oil. Remove the engine from the test stand and allow the engine to stand for 6 h in the same attitude in which it was positioned on the test stand. This allows the oil to drain completely into the oil pan.

12.6.2 *Final Oil Drain and Engine Disassembly*—Drain the oil completely and disassemble the engine. During disassembly, use extreme care to ensure the original location of the parts can be identified with respect to the cylinder number or valve location, or both.

12.6.3 Parts Layout for Rating:

12.6.3.1 Arrange the following parts in the parts rating area in accordance with the layouts detailed in this section. A photograph of the parts layout is shown in Fig. A6.12. After the parts have been arranged, allow the parts to drain a minimum of 4 h before rating. Do not attempt to accelerate or force the oil draining. Any fixtures can be used to support the parts as long as they orient the parts in the specified configurations.

12.6.3.2 *RAC*—Position the RAC vertically (upper jacket surface perpendicular to the ground) with the front of the RAC pointing down.

12.6.3.3 *Camshaft Baffle*—Position the camshaft baffle vertically (top baffle surface perpendicular to the ground) with the rear of the camshaft baffle pointing down.

12.6.3.4 *Cylinder Head*—Position the cylinder head with exhaust port surface pointing down. The exhaust manifold studs should be left in the head to raise the head off of the supporting surface and allow drainage.

12.6.3.5 *Front Seal Housing*—Position the front seal housing in the same orientation as it is installed in the engine.

12.6.3.6 *Oil Pan*—Position the oil pan upside down, at a 45° angle, with sump end pointing down.

12.6.3.7 *Oil Screen and Pickup Tube*—Position the oil screen and pickup tube in the same orientation as it is installed in the engine. The screen should be raised off of the supporting surface to allow drainage. A fixture is necessary to support the oil screen and pickup tube.

12.6.3.8 *Engine Block*—Position the engine block with the valve deck in a horizontal plane and facing the ceiling.

13. Interpretation of Results

13.1 Parts Rating Area—Environment:

13.1.1 Ensure that the ambient atmosphere of the parts rating area is reasonably free of contaminants, and that the temperature is maintained at $75 \pm 5^{\circ}$ F ($24 \pm 3^{\circ}$ C). When a rater seeks advice from another rater, report the rating as a consensus rating in the final report. Do not use raters from outside sources (other laboratories) for consensus ratings and do not average these ratings. Include independent ratings in the supplemental pages as information only.

13.1.2 Rate all engine parts except the pistons, RAC, and cylinder block under cool white fluorescent lighting exhibiting approximately 4500 K color temperature, and an illumination level of 350 to 500 fc (3800 to 5400 lx). All background and adjacent surfaces shall be flat white.

13.1.3 Rate pistons and RACs against a white background using white fluorescent bulbs and a 100 % white deflector. Maintain the illumination level between 350 and 600 fc (3800 to 6500 lx), and measure the illumination level 14 in. (360 mm) from the desk top. Provide a 15-W bore light with a cool white fluorescent tube for the cylinder wall varnish rating.

13.2 Sludge Ratings:

13.2.1 Rate the following parts for sludge deposits: RAC, valve deck, cylinder block, camshaft baffle, front seal housing, and oil pan. Use the rating locations identified on the rating worksheets (see Annex A6). Determine the ratings using the techniques detailed in CRC Manual No. 12. Adjust RAC sludge on candidate tests for laboratory severity in accordance with Annex A12.

13.2.2 Average Sludge (Unweighted Average of All Six Parts):

13.2.2.1 Use the self-weighting procedure detailed as follows to determine the sludge rating for each part. Adjust the average sludge on non-reference tests for laboratory severity in accordance with Annex A12.

(*a*) Determine the sludge depth at each of the sites shown on rating worksheets. A site is defined as a 0.75-in. (19.1-mm) diameter circle.

(b) Determine an interpolated sludge value for a designated site that exhibits more than one level of sludge depth within this area. This value is generated by multiplying each rated value's volume factor by the percentage of area covered, totaling these volume factor percentages, and comparing the total to the values given in Table 8 to determine the rating for the site. As an example, if a site was determined to be 50 % A and 50 % E, the calculation would be: 50 % of the average sludge depth factor for A (or 1.0), plus 50 % of the average

TABLE 8 Average Sludge Site Ratings

Total	Site Ratings	Total	Site Ratings
<0.125	Clean	≥3.500<6.000	С
≥0.125 < 0.375	1/4 A	≥6.000 < 12.00	D
≥0.375 < 0.625	1/2 A	≥12.00 < 24.00	E
≥0.625 < 0.875	3⁄4 A	≥24.00 < 48.00	F
≥0.875 < 1.250	А	≥48.00 < 96.00	G
≥1.250 < 1.750	AB	≥96.00 < 192.0	Н
≥1.750 < 2.500	В	≥192.0 < 384.0	I
≥2.500 < 3.500	BC	≥384.0	J

sludge depth factor for *E* (or 16.0); that is: $([0.5 \times 1.0 = 0.5] + [0.5 \times 16.0 = 8] = 8.5)$, and the calculated site sludge depth would be a *D*. In the event that there are areas where it is apparent that deposits had formed, but are no longer adhering to the part, the rating site is the closest point to the designated point that will eliminate the voided area from the 0.75-in. (19.1-mm) rating area.

(c) Add the total rating checks made for each line on the appropriate worksheets. These shall equal 10 or 20, depending on the part that is rated.

(d) Multiply the total rating checks made on each line by ten or five, respectively (refer to (c)), to obtain the percent covered by the rated sludge depth. The grand total shall equal 100 %.

(e) Convert the percent covered by the rated sludge depth at each location to a volume factor using the procedure shown in CRC Manual No. 12.

(f) Add the volume factors on each line to determine the total volume factor. Use CRC Manual No. 12 to convert the total volume factor to the sludge merit rating.

13.2.2.2 *Flaky, Bubbly Sludge Deposits*— Since the occurrence of flaky, bubbly sludge deposits is thought to have a possible detrimental effect on long-term engine lubrication system performance, document the occurrence of this type of deposit in the Supplemental Operational Data section of the Final Test Report. The type of information to be recorded is the engine part(s) where this type of deposit was observed. Suggested wording is as follows: Approximately 6 % of the RAC was found to exhibit flaky, bubbly sludge deposits.

13.3 Varnish Ratings:

13.3.1 *Preparation of Parts*—Rate the following parts for varnish deposits: piston skirts, camshaft baffle, oil pan, rocker arm cover, and cylinder walls. Perform the varnish ratings after the sludge ratings are completed. The rating locations and dimensions shall conform with the locations and dimensions detailed on the rating worksheets (see Annex A6). Avoid disturbing sludge deposits when the parts are being prepared for varnish ratings. Heavy sludge can be removed from a varnish rating area with a 1-in. (25-mm) rubber spatula prior to wiping. Wipe all parts firmly with wiping materials specified in CRC Manual No. 14. Firmly rub all wiping areas in the same direction until the surface is dry and free of sludge (until no more deposit is present on the wiping material after wiping).

13.3.2 Average Varnish (Unweighted Average of the First Five Parts)—Use the procedure detailed as follows to determine the varnish rating of each part:

13.3.2.1 Rate any areas where varnish deposits have been altered during disassembly or sludge removal in accordance

with deposits on the surrounding non-altered areas. Do not rate altered areas as *clean*.

13.3.2.2 Determine varnish ratings of all parts except pistons by comparison of the deposit on the rating location with color chips. The piston photographs shown in CRC Manual No. 14 shall be used to determine the varnish ratings on the pistons.

13.3.2.3 Determine the average varnish rating over each area specified in the rating worksheets. Adjust piston skirt varnish ratings on non-reference tests for laboratory severity in accordance with Annex A12.

13.3.2.4 Determine the average varnish rating of the entire part.

13.4 Clogging:

13.4.1 *Oil Screen Clogging*—Determine the percentage of the total screen opening that is obstructed with sludge and debris.

13.4.1.1 Flexible, transparent rating aids can be made for different surface areas so that when compared to the test screen's surface, a more accurate determination of surface clogging is possible. These are made of thin, colored transparent plastic cut into circles equivalent in area to 20, 10, and 5 % (1.5, 1.125, and 0.75 in. [38.1, 28.6, and 19.1 mm] in diameter, respectively) of the 9.5 in.²(6130 mm²) flow surface.

13.4.1.2 If there is any question concerning whether the screen is covered with oil or sludge, blow lightly on the screen (see CRC Manual No. 12). Note an analysis of deposits identified as debris in the Supplemental Operational Data.

13.4.2 *Oil Ring Clogging*—Determine the percentage of slot clogging for each oil ring in accordance with the procedure detailed in CRC Manual No. 12. Calculate and record the average clogging for all four rings.

13.4.3 *PCV Valve Clogging*—Determine the percent clogging of the PCV valve that is used during the last 48 h of the test, in accordance with the following procedure:

13.4.3.1 Measure the PCV valve flow rate at differential pressures from 8 and 18 in. Hg (13.5 and 60.8 kPa). Calculate the percent clogging in accordance with the following equation:

PCV valve clogging,
$$\% = \frac{I - F}{I} \times 100$$
 (6)

where:

I = initial flow rate, and

F =final flow rate.

13.4.3.2 If any of the PCV valves have been replaced before the scheduled replacement interval, calculate and report the PCV valve clogging for all of the PCV valves. Report the percent clogging of the last valve on the Ratings and Measurements Page.

13.4.4 *Camshaft Lobe Orifice Clogging*— Determine the camshaft lobe orifice clogging with the camshaft lobe orifice flow rate device (see 7.5.3 and Figs. A3.32 and A3.33) in accordance with the following procedure:

13.4.4.1 Connect the apparatus to a source of compressed air.

13.4.4.2 Adjust the two pressure regulators until the probe pressure is 10 in. water (2.5 kPa).

13.4.4.3 Calibrate the apparatus by measuring the flow rate when the probe pressure is 10.0 ± 0.2 in. water $(2.49 \pm 0.05 \text{ kPa})$ and the probe is firmly seated in the calibration lobe orifice (clean orifice) with the probe pressure adjusted to 10.0 ± 0.2 in. water $(2.49 \pm 0.05 \text{ kPa})$. Record the flow rate as *Baseline Flow Rate* on the rating worksheet.

13.4.4.4 Firmly seat the probe in each lobe hole, adjust the probe pressure to 10 ± 0.2 in. water (2.49 \pm 0.05 kPa), and record the flow rate.

13.4.4.5 Calculate and record the percent flow rate reduction in accordance with the following equation:

camshaft lobe orifice flow rate reduction,
$$\% = \frac{I - F}{I} \times 100$$
 (7)

where:

I =initial flow rate, and

F = final flow rate.

13.4.4.6 Record the number of clogged camshaft lobe orifices. Clogged orifices are defined as orifices having a flow rate reduction greater than 80 %.

13.5 Sticking:

13.5.1 *Compression Rings*—Record the number of stuck compression rings. Definitions to classify the degree and type of sticking are detailed in Section 3. List only hot stuck compression rings on the Ratings and Measurements Summary page. Denote hot or cold stuck rings on the Supplementary Operational Data page and include the ring location (top or second) and the piston number.

13.5.2 *Hydraulic Lifter Plungers*—Record the number of hydraulic lifter plungers that are completely stuck in the lash adjuster bodies. Note partially stuck plungers on the final engine inspection sheet.

13.6 Wear Measurements:

13.6.1 *Camshaft Lobe Wear*—Measure and record the heelto-toe measurement of each camshaft lobe at the maximum lift point using a device reading directly in micrometres, with an accuracy of at least $\pm 3 \mu m$. Calculate the wear based on the before-test measurements.

Note 11—The + 0.14 adjustment replaces the + 0.24 adjustment. Do not apply both adjustments.

13.6.1.1 For reference oil tests completed on or after May 16, 1996, add 1.451 to the transformed average cam wear result and 1.693 to the transformed maximum cam wear result. Until a reference oil test is conducted on or after May 16, 1996, do not add 1.451 and 1.693 to average and maximum cam wear non-reference oil test results. Adjust wear results for laboratory severity in accordance with Annex A12.

13.6.2 *Rocker Arm Wear*—Wash the rocker arms with aliphatic naphtha. Wash the rocker arms with ethyl acetate and allow the rocker arms to air dry. Weigh the rocker arms and calculate the weight loss based on the before-test weights.

13.6.3 *Connecting Rod Bearing Wear*—Wash the bearings in aliphatic naphtha. Dip the bearings in pentane and allow the bearings to air dry. Measure the bearing weights. Calculate the weight loss based on the before-test measurements. The bearing weight loss is the total loss for two halves of each bearing.

13.6.4 *Piston Ring Gap Increase*—Clean the ring ends thoroughly and measure the ring gap after the rings have been installed in the block (see 7.8, 11.2). Calculate the ring gap increase. Compensate for any ring gap adjustments made during the test.

13.6.5 *Cylinder Bore Wear*—Measure the transverse bore wear at the top, middle, and bottom of the second compression ring travel, using the technique specified in 7.8.9.5 (*a*). Measure the bores with both stress plates installed. Calculate wear based on the before-test versus after-test measurements.

13.7 Used Oil Analysis:

13.7.1 Perform the following analyses on the used oil samples from hours 0 (new oil), 12, 108, 204, and 288: Iron content, ppm; copper content, ppm; silicon content, ppm; and kinematic viscosity at 40°C, cSt (in accordance with Test Method D 445).

13.7.2 *Fuel Dilution*—Determine the fuel dilution, percent mass, by gas chromatography (see Test Method D 3525, with the following modifications) on the used oil samples from hours 12, 108, 204, and 288:

13.7.2.1 Use C16 in place of C14 for the internal standard (1-L injector volume).

13.7.2.2 Presume that all components lighter than C16 are fuel.

13.7.2.3 The integrator should establish a horizontal baseline under the output curve until the leading edge of C16 is reached. Establish a second baseline extending horizontally from the output curve, at the intersection of the output curve, and the leading edge of the C16 peak.

13.7.2.4 Column details are 10 ft by 0.125 in. (305 cm \times 3.2 mm) SS; and the packing material is 5 % OV-1 on Chromosorb W HP.

13.7.2.5 Increase the oven temperature from 60 to 320° C, with the rate of change of temperature controlled at 8° C/min. Hold the temperature at 320° C for 16 min to elude oil.

13.7.3 *Insolubles*—Perform the following analysis on the used oil samples from Hours 108, 204, and 288: pentane insolubles. (See Test Method D 893B, pentane only.)

13.7.4 *Substitute Samples*—Samples from hours 36, 132, and 228 can be substituted if the sample from the previous sampling period has been misplaced (see Fig. A7.9).

13.8 Other Deposits:

13.8.1 *Piston Undercrown*—Determine the deposit rating of the piston undercrown by comparison of the deposit on the rating location with CRC Induction System Rating Technique contained in CRC Manual No. 16. Rate any areas where deposits have been altered by disassembly in accordance with deposits on the adjacent non-altered areas. Do not rate altered areas as clean. Identify the percentage of altered area on Fig. A7.1.5.

13.8.2 *Ring Land Deposits*—Determine the deposit rating of the ring land faces on all four pistons for each land type: crown land, second land (area between the two compression rings), and the oil ring land (area directly above the oil ring) by comparison of the deposit on the rating location with color chips contained in CRC Manual No. 14 (non-rubbing scale, Appendix A.51). Average the results for each land type from each of the four pistons. Rate deposits that are less than 1.0

merits using two levels of carbon, heavy carbon, and light carbon. Heavy carbon is carbon that has more buildup when compared with adjacent surfaces and has a polished or rubbed finish. Assign a merit value of 0.0 to areas of heavy carbon. Rate all other deposits as light carbon and assign a merit value of 0.75 to those areas. Rate any areas where deposits have been altered by disassembly in accordance with deposits on the adjacent non-altered areas. Do not rate altered areas as clean. Identify the percentage of altered area on Fig. A7.1.5.

14. Assessment of Test Validity

14.1 The testing laboratory shall assess the validity of tests that have deviations from the procedure. The TMC will assist the laboratory in the determination of calibration test validity, if requested by the laboratory. Use the following guidelines as a basis for determining test validity:

14.2 Average Exhaust Gas NO_x Levels—The average NO_x level during a test should be within 600 ppm of the overall cumulative calibration test average for the laboratory. The cumulative calibration test average is the average NO_x level of the last ten calibration tests. Significantly lower NO_x levels could cause a reduction in sludge severity.

14.3 Used Oil Analyses—Interpretation:

14.3.1 *Iron and Copper Content*—The iron and copper content of the used oil samples can indicate the level of wear that occurs during a test. Increased camshaft, rocker arm, and cylinder bore wear causes the iron content to increase. High levels of iron may also increase sludge severity. The rate of change in iron levels indicates the rate of change of wear levels, although wear tends to be linear throughout the test. Copper levels tend to be relatively stable unless excessively high levels of connecting-rod bearing weight loss are experienced.

14.3.2 *Silicon Content*—Silicon content indicates the level of contamination of the oil from external sources and silicone-based sealers. Silicone-based sealers are used near the rear crankshaft seal housing to seal the oil pan. Investigate the presence of silicone-based sealers, the cleanliness of engine parts during build-up, the cleanliness of the intake-air, and the cleanliness of containers used for dispensing and sampling the oil if high levels of silicone are experienced.

14.3.3 *Fuel Dilution*—Fuel dilution indicates the degree to which the crankcase oil has been diluted with fuel. Fuel dilution of the crankcase oil is necessary to achieve adequate test severity. However, excessive fuel dilution may promote increased wear and sludge severity. The reproducibility of fuel dilution tends to be poor. Therefore, fuel dilution levels should only be compared to other results obtained from the same analytical laboratory, using the same analytical equipment and technique. Investigate a higher level of fuel dilution than is normally noted.

14.4 Blowby Flow Rate:

14.4.1 The corrected average blowby flow rate shall fall within the range from 1.75 to 2.25 cfm (0.83 to 1.06 L/s). In addition, the requirements listed in 14.3.2 shall also be met.

14.4.2 Between 48 and 192 h, a maximum of twelve corrected blowby flow rate readings may be greater than 2.25 cfm (1.06 L/s). Between 48 and 192 h, the summation of the deltas (delta = [corrected reading in cfm] – 2.25) of blowby

flow rate readings above 2.25 cfm are not to exceed 1.2. No more than 16 of the corrected blowby flow rate readings can be less than 1.75 cfm (0.83 L/s) during the first 192 h. The summation of the deltas (delta = 1.75 - [corrected cfm]) of blowby flow rate readings below 1.75 cfm (0.83 L/s) are not to exceed 1.6 during the first 192 h.

14.5 Intake Manifold Vacuum—Depending upon an individual laboratory, intake manifold vacuum will vary in accordance with the altitude of the laboratory, ambient barometric pressure, and the mechanical efficiency of the engine. Laboratories should investigate intake manifold vacuum readings that are abnormal for their laboratory. Abnormal readings can be due to measurement errors in manifold vacuum, engine build, exhaust back pressure, engine speed, engine load, and air-fuel ratio.

14.6 *Fuel Consumption Rate*—The fuel consumption rate should not exceed 18 lb/h (8.2 kg/s) at any time during the test.

14.7 *Oil Consumption*—The final oil level cannot be lower than 12 oz (360 mL) (fluid) during the first 192 h of the test. Use engineering judgment concerning test invalidation if the oil level is lower than 12 oz (360 mL) during the remainder of the test.

14.8 *Shutdowns*—The number of shutdowns are to be minimized. Under no circumstances is the number of shutdowns to exceed twelve, exclusive of shutdowns for blowby flow rate adjustments and scheduled oil level shutdowns. Length of downtime is to be minimized, and under no circumstances is the total length of downtime to exceed 100 h.

14.9 *Deviation Percentage*—Calculate the deviation percentage in accordance with the following equation:

$$DP = \sum_{i=1}^{i=n} \left[\frac{M_i}{0.5R} \times \frac{T_i}{D} \right] \times 100$$
(8)

where:

- DP = deviation, %,
- M_i = magnitude of test-parameter deviation from specification limit at occurrence *i*,
- R = test parameter specification range,
- T_i = length of time that test parameter was outside of specification range at occurrence *i*,
- *n* = number of times that a test parameter deviated from test specification limits, and
- D = length of test-phase duration in same units as T_i .

Note $12-T_i$ is assumed to be no less than the recorded-data-acquisition frequency unless supplemental readings are documented.

14.10 Invalidate any tests exceeding the following deviation percentages:

Primary test parameter	2.5 %
Secondary test parameter	5.0 %

14.10.1 Primary Parameters:

Primary Parameters	Divisor (D)
Oil inlet temperature	288
Coolant outlet temperature	288
Timing	288
Exhaust gas oxygen (Stages I and II)	234
Exhaust gas CO (Stage III)	48

14.10.2 Secondary Parameters:

Secondary Parameters	Divisor (D)
Engine Speed	288
Power	288
Rocker arm cover coolant flow rate (Stages I and III)	198
Exhaust back pressure (Stages I and II)	234

14.10.3 *Special*—For tests started prior to September 1, 1992, invalidate the test if the deviation percentage for either engine coolant flow rate or rocker arm cover inlet coolant temperatures exceeds 5.0 %. For tests started on or after September 1, 1992, invalidate the test if the deviation percentage for either parameter exceeds 2.5 %.

14.10.4 Special Parameters:

Special Parameters	Divisor (D)
Coolant flow rate	144
Rocker arm cover inlet	288

14.11 *Mid Specification Operation*—For all tests started on or after September 1, 1992, invalidate the test if the average of steady state operating data for a controlled, primary parameter is offset from the target mean by more than 12.5% of the allowable control range. For tests started on or after March 30, 1992, list in the test report any parameters that fall outside of these limits.

Example: Temperature specification is $100 \pm 2^{\circ}$

Primary: Average must be within $100 \pm 0.5^{\circ}$

Secondary: Average must be within 100 \pm 1.0°

14.11.1 Primary Parameters:

Primary Parameters	Stage/Range
Oil inlet temperature (°F)	Stage 1 155 \pm 0.5
	Stage 2 210 ± 0.5
	Stage 3 115 \pm 0.5
Coolant outlet temperature (°F)	Stage 1 125 \pm 0.5
	Stage 2 185 \pm 0.5
	Stage 3 115 ± 0.5

14.11.2 Secondary Parameters:

Secondary Parameters	Stage/Range	
Engine speed (r/min)	Stage 1 and 2	2500 ± 5
	Stage 3	750 ± 12.5
Power (bhp)	Stage 1 and 2	33.5 ± 0.25
	Stage 3	1.0 ± 0.25
Rocker arm cover coolant flow rate (gpm)	Stage 1 and 3	2.0 ± 0.1

14.12 *Stage Transitions*—Invalidate any test that does not meet the speed, torque, CO, and O_2 values listed in Table 4 during transition from Stage 2 to 3 and Stage 3 to 1.

15. Report

15.1 *Report Format*—The various required sections and specific details concerning the format of the report are outlined in this section. Examples of each section are shown in Annex A5 and Annex A7. Deviations in the format are generally not permitted. However, deviations in nomenclature and other details may be permitted. Any desired deviations shall be approved by the TMC. Calibration data for particular reference oils are available from the TMC.

15.2 Special Forms for Automated Data Acquisition— Automated data acquisition systems may require the use of forms that differ from those shown in Annex A5 and Annex A7. Variations in forms for automated data acquisition systems shall be approved by the TMC.

15.3 *Standard Report*—Include the following sections in the standard test report. Begin each section on a new page and insert the sections in the following order:

15.3.1 Title page (includes validity statement—Fig. A7.1),

15.3.2 Ratings and measurements summary (Fig. A7.2),

15.3.3 Calibration test ratings and measurements summary (Fig. A7.3),

15.3.4 Operational summary (Fig. A7.4),

15.3.5 Supplemental operational data (Fig. A7.5),

15.3.6 Special maintenance record (Fig. A7.6),

15.3.7 Oil analyses (Fig. A7.7),

15.3.8 Valve train inspection detail (Fig. A7.8),

15.3.9 Piston skirt varnish rating breakdown (Fig. A7.9),

15.3.10 Corrected blowby flow rate plot (Fig. A7.10),

15.3.11 Oil addition record (Fig. A7.11),

15.3.12 Origin of significant engine parts (Fig. A7.12),

15.3.13 Characteristics of the data acquisition system (Fig. A7.13),

15.3.14 RAC and camshaft baffle photographs (Fig. A7.14),

15.3.15 Oil pan and screen photographs (Fig. A7.15),

15.3.16 Piston skirt photographs (Fig. A7.16),

15.3.17 Camshaft lobe photographs (Fig. A7.17),

15.3.18 Front seal housing photographs (Fig. A7.18),

15.3.19 Report speed, torque, and exhaust gas CO output during transition, including limits in Table 4 (in accordance with 12.2.3.2) in the format of Fig. A7.19. Take data during test hours 12 through 48. Include copies of the transition plots from the previous stand reference test with each candidate test run. If adjustments are needed to bring the transition into conformance with the requirements of Table 4, include a plot conducted after the adjustment, showing conformance to Table 4 criteria. Where additional plots are required, the last plot may be taken through test hour 64.

15.3.20 Ring adjustment summary, includes cylinder bore wear (Fig. A7.20),

15.3.21 Deviation percentage summary (Fig. A7.21),

15.3.22 Mid limit specification summary (Fig. A7.22),

15.3.23 Engine operational and special maintenance data log (Fig. A7.4).

15.4 *Calibration Test Report*—Include the following sections in the calibration test report. Begin each section on a new page and insert the sections in the following order:

15.4.1 Title page (includes validity statement—Fig. A7.1), 15.4.2 Calibration test ratings and measurements summary (Fig. A7.3),

15.4.3 Operational summary (Fig. A7.4),

15.4.4 Supplemental operational data (Fig. A7.5),

15.4.5 Special maintenance record (Fig. A7.6),

15.4.6 Oil analyses (Fig. A7.7),

15.4.7 Valve train inspection detail (Fig. A7.8),

15.4.8 Piston skirt varnish rating breakdown (Fig. A7.9),

15.4.9 Corrected blowby flow rate plot (Fig. A7.10),

15.4.10 Oil addition record (Fig. A7.11),

15.4.11 Origin of significant engine parts (Fig. A7.12),

15.4.12 Characteristics of the data acquisition system (Fig. A7.13),

15.4.13 Rocker arm cover and camshaft baffle photographs (Fig. A7.14),

15.4.14 Oil pan and screen photographs (Fig. A7.15),

15.4.15 Piston skirt photographs (Fig. A7.16),

15.4.16 Camshaft lobe photographs (Fig. A7.17),

15.4.17 Front seal housing photographs (Fig. A7.18),

15.4.18 Report speed, torque, and exhaust gas CO output during transition, including limits in Table 4 (in accordance with 12.2.3.2) in the format of Fig. A7.19. Take data during test hours 12 through 48. Include copies of the transition plots from the previous stand reference test with each candidate test run. If adjustments are needed to bring the transition into conformance with the requirements of Table 4, include a plot conducted after the adjustment, showing conformance to Table 4 criteria. Where additional plots are required, the last plot may be taken through test hour 64.

15.4.19 Ring adjustment summary, includes cylinder bore wear (Fig. A7.20),

15.4.20 Deviation percentage summary (Fig. A7.21),

15.4.21 Mid limit specification summary (Fig. A7.22),

15.4.22 Engine operational and special maintenance data log (Fig. A7.4, Fig. A7.5, and Fig. A7.6).

15.5 *Photographs*—The required photographs are listed in this section and shown in Fig. A7.14 through Fig. A7.18. These figures depict the headings and the recommended orientation of the photographs. Digital images, comparable in pictorial quality to photographs, may be used instead of photographs.

15.5.1 RAC,

15.5.2 Front seal housing,

15.5.3 Camshaft baffle,

15.5.4 Oil pick-up screen,

15.5.5 Oil pan,

15.5.6 Average and worst piston, anti-thrust sides,

15.5.7 Average and worst piston, thrust sides,

15.5.8 Best camshaft lobe (lowest wear),

15.5.9 Worst camshaft lobe (highest wear), and

15.5.10 Best and worst camshaft follower (lowest and highest wear).

16. Precision and Bias

16.1 Precision:

16.1.1 *Non-Reference Oils*—To aid the potential user of this test method to assess the variability that can be expected between test results when the test method is used in one or more reasonably competent laboratories, the precision information in Table 9 has been developed:

16.1.1.1 The non-reference oil precision for this test method is determined by the statistical analysis of sets of data submitted by users of the test method. Each set of data consists of two or more engine test results, obtained on the same engine oil formulation. Each data set may be from tests run in the same laboratory or different laboratories, and with any approved fuel batch. The data are collected and analyzed by industry volunteers, and reported to the ASTM Sequence VE Test Method Surveillance Panel.

16.1.1.2 The statistical criteria summarized in Table 9 represents the estimate of *Reproducibility*, based on 250 df obtained over the period of April 1, 1992, through September

TABLE 9 Non-reference Oil Precision Statistics^A

Measured Units			
Variable -	Reproducibility ^B		
	S_R^{C}	R ^D	
Average engine varnish, merits	0.35	0.98	
Average piston varnish, merits	0.25	0.70	
Transformed Units			
N/ 111	Reproducibility ^B		
Variable	S _R ^C	R ^D	
Average engine slude, – In (9.65-merits)	0.56	1.57	
Rocker cover sludge, – In (9.65-merits)	0.51	1.43	
Average cam wear, - (mils)	0.71	1.60	
Maximum cam wear, In (mils)	1.02	2.86	

^A These statistics are based on non-reference oil test results obtained over the period of April 1992 through September 1994.

^B Reproducibility values refer to tests run on the same oil in different laboratories.

 ^{C}s = Standard deviation.

 $^{\it D}$ On the basis of test error alone, the difference, in absolute value, between two test results will be expected to exceed this value about 5 % of the time. This value is obtained by multiplying the reproducibility standard deviation ($S_{\it R}$) by 2.8.

30, 1994, for the various required engine deposit and wear performance parameters. This reasonably long time period was selected as being representative of a consistent period of test operation. (If a potential user has a desire to know what the most recent precision data are for non-reference oils, the TMC can provide this information by consulting the appropriate Sequence VE Surveillance Panel meeting minutes.)

16.1.1.3 *Reproducibility* (R)—The difference between two single and independent results obtained on the same oil by different operators working in different laboratories would, in the long run, in the normal and correct conduct of the test method, exceed the values in Table 9 in only one case in twenty.

16.1.1.4 No similar estimate of non-reference oil *Repeat-ability* is available with this test method since the number of non-reference oil tests repeated in the same engine, test stand, and laboratory are limited. An estimate of current *Repeatability* is contained in the statistical reference oil data base maintained by the TMC.

16.1.2 *Reference Oils*—The TMC uses the precision information developed from frequent tests on several reference oils to help those laboratories conducting the test method maintain consistent test severity. These precision data are updated regularly and are available upon request from the TMC. The reference oil precision from operationally valid tests from April 1, 1992, through September 30, 1994, is summarized in Annex A11.

16.2 *Bias*—Bias is determined by applying an accepted statistical technique to reference oil test results, and when a significant bias is determined, a severity adjustment is permitted for non-reference oil test results (see 13.2.1, 13.3.2.3, and Annex A12).

17. Keywords

17.1 lubricating oils; Sequence VE; sludge and varnish; spark-ignition automotive engine; stop-and-go service; valve train wear

ANNEXES

(Mandatory Information)

A1. SPECIAL SERVICE TOOLS FOR THE TEST ENGINE

A1.1 See Table A1.1.

TABLE A1.1 Special Service Tools for the Test Engine

	opecial bervice roois for the rest Engine
Tool Number	Description
T74P-6000LA	2.3 L service tool kit
T74P-6015-A	engine plug replacer
T74P-6019-B	front cover alignment tool
T68P-6135-A	piston pin remover/replacer
T74P-6136-A	crankshaft timing gear remover/adapter
T74P-6150-A	camshaft and auxiliary shaft seal replacer
T71P-6250-A	camshaft bearing replacer
T74P-6254-A	camshaft belt tension adjusting tool
T74P-6256-B	camshaft and auxiliary shaft glide hammer sprocket tool
T59L-100-B	slide hammer
T74P-6375-A	flyway holding tool
T74P-6565-A	valve spring compressor
T73P-6571-A	valve seal installer
T79P-6634-A	pressure diagnostic gage
T74P-6700-B	front cover seal remover
T82L-6701-A	crankshaft seal installer
D79L-6731-B	oil filter wrench
T71P-7137-C	pilot bearing replacer
T71P-7137-H	clutch aligner
T77L-9533-B	jet plug remover/small
T58L-101-A	puller attachment
T74P-6306-A	crankshaft timing gear tool
T74P-6312-A	crankshaft damper remover

A2. EXTERNAL OIL HEAT EXCHANGER CLEANING TECHNIQUE

A2.1 Remove and completely disassemble the heat exchanger including the end caps.

A2.2 Flush the shell side of the heat exchanger with an organic solvent for a minimum of 12 h to ensure the removal of all deposits. This can be accomplished by using a positive displacement pump to flow the solvent through the heat exchanger at a flow rate of 10 ± 2 g/min (0.63 \pm 0.126 L/s). Several heat exchangers can be flushed at the same time by connecting them in series. Flushing can be required for longer than 12 h if the deposits are particularly heavy or the solvent is aged. Filtration of the solvent can extend the useful life of the solvent.

- A2.3 Rinse the heat exchanger with hot water.
- A2.4 Rinse the heat exchanger with aliphatic naphtha.
- A2.5 Air dry the heat exchanger.

NOTE A2.1—The tube side of the heat exchanger should be periodically cleaned to ensure adequate heat transfer. A rifle bore cleaner attached to an electric drill has been found to be an effective way to remove process water deposits. Ultrasonic cleaning is also effective in this application.

A3. DETAILED SPECIFICATIONS AND PHOTOGRAPHS OF APPARATUS

A3.1 See Figs. A3.1-A3.48.

∰ D 5302 – 01a

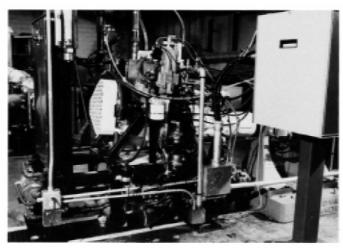


FIG. A3.1 Typical Test Stand, Left Forward View

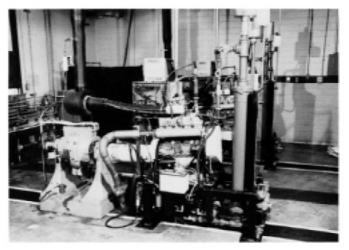


FIG. A3.2 Typical Test Stand, Right Forward View

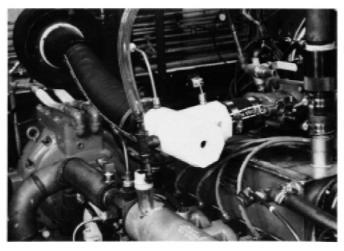


FIG. A3.3 Intake Air Supply System, Air Horn

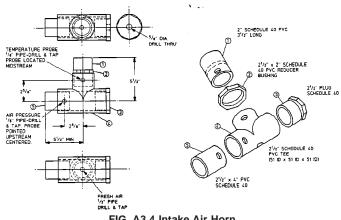
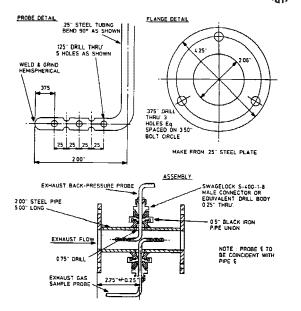
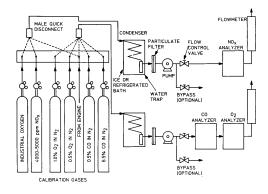


FIG. A3.4 Intake Air Horn



NOTE 1—Bend external tubing segment after assembly parallel with probe axis as shown. This permits visual alignment check during operation.





NOTE 1—A typical exhaust gas analysis system is shown in Fig. A3.6. The condenser can use an ice bath or mechanical refrigeration. Ice bath condensers should use a coil fabricated from 0.25-in. (0.64-cm) stainless steel tubing and provide sufficient cooling to condense moisture at a dew point of 34° F (1°C). Mechanical refrigeration should provide control of the bath temperature to $34 \pm 2^{\circ}$ F (1 $\pm 1^{\circ}$ C). The flow rate of engine exhaust and calibration gases should be identical and within the specifications of the instrumentation. An air-conditioned chamber for instrumentation is required if ambient temperatures are above the maximum recommended for the particular instrumentation used.

Stainless steel fittings are preferred throughout the analysis system. Aluminum fittings may cause erroneous NO, and ultimately NO_x readings. Brass fittings should not be used in the analysis system. The porosity of the particulate filter should be between 2 and 10 μ m. A diaphragm-type pump is recommended to reduce pump "hang up." **Warning**—Safety precautions are necessary concerning venting CO, NO, and ozone gases from the analyzer instruments.

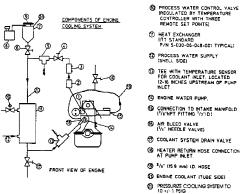
NOTE 2—Require Calibration Gases:

Nominal 5000 ppm NO_x, balance N₂ Nominal 6.5 % CO, balance N₂ Nominal 0.5 % CO, balance N₂ Nominal 1.0 % O₂, balance N₂ Nominal 0.5 % O₂, balance N₂ *Optional Zero Standard Gas:* N₂ for NO_x, O₂, and CO analyzers

NOTE 3—If the optional zero standard gases are not used to zero the analyzers, the CO calibration gases can be used to zero the O_2 analyzer, the O_2 calibration gases can be used to zero the CO analyzer, and bottled air can be used to zero the NO_x analyzer.

FIG. A3.6 Exhaust Gas Analysis System

- THERMOSTAT HOUSING WITH TEMPERATURE SENSOR SEE A312 FOR DETAILS
- (2) SIGHT GLASS
- () VENTURI FLOWMETER MODEL BR12705-16-31 WITH PRESSURE TAPS ISEE NOTE 21
- FLOW CONTROL VALVE
- S OPTIONAL TEMPERATURE CONTROL SENSOR
- FLUSHING ISOLATION VALVE
- PABRICATED COOLANT RESERVOR
- ③ CONSTANT FULL EXPANSION TANK
- PRESSURE RADIATOR CAP (MOTORCRAFT RS40 P/N D2YY-8100-A)



Note 1—Observe temperature sensor locations in thermostat housing and at water pump inlet.

NOTE 2—See X2.

FIG. A3.7 Typical Engine Cooling System Schematic

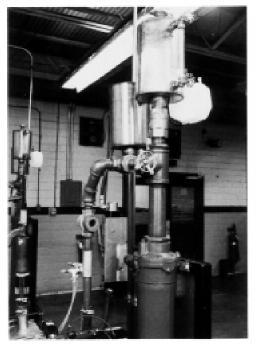


FIG. A3.9 Typical Engine Cooling System, Flow Control Valve, Venturi Flowmeter, Expansion Tank, and Sight Glass

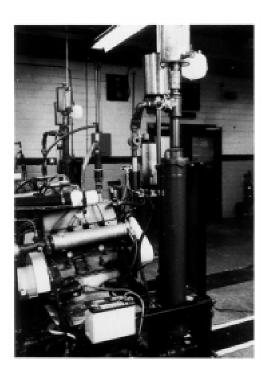
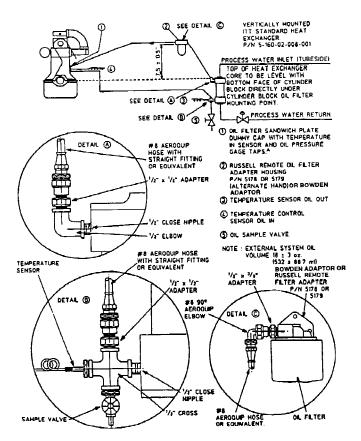


FIG. A3.8 Typical Engine Cooling System, Venturi Flowmeter



ABowden Adapter BX-570-2 may be used in lieu of Sandwich Plated. FIG. A3.10 Oil Cooling System Specification

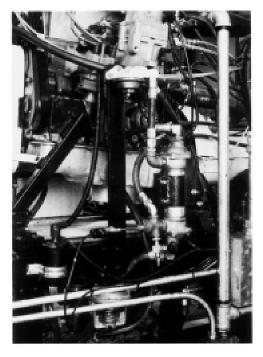


FIG. A3.11 Oil Cooling System Required Heat Exchanger Mounting, Remote Oil Filter Mounting and Typical Hoses and Fittings

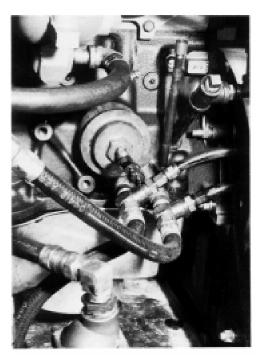


FIG. A3.13 Oil Cooling System, Adapter Housing at Engine with Typical Fittings for Temperature Sensor, Pressure Taps, Dipstick Tube, and Oil Fill Tube

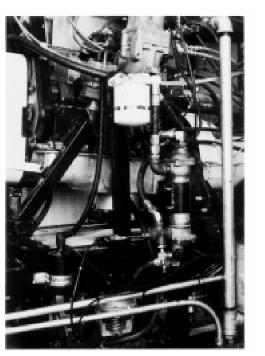
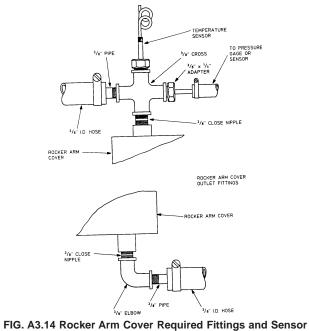


FIG. A3.12 Oil Cooling System Remote Oil Filter Fitted for Break-In



4 Rocker Arm Cover Required Fittings and Locations

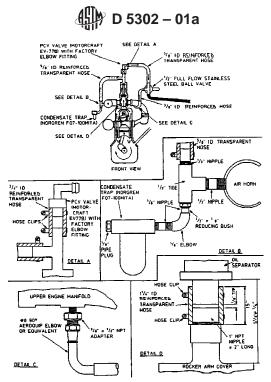
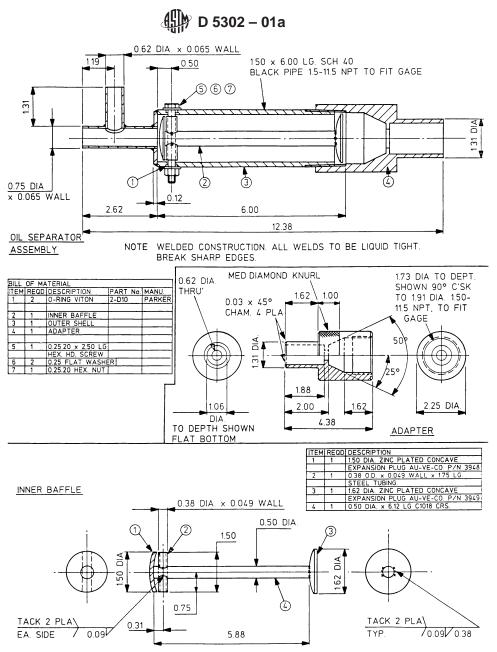


FIG. A3.15 Closed Crankcase Ventilation System, Required Fittings and Parts Configuration





🕼 D 5302 – 01a

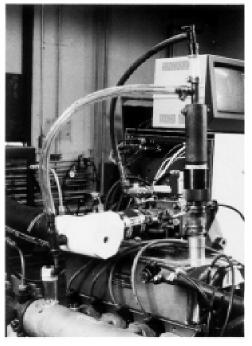


FIG. A3.17 Engine Ventilation System

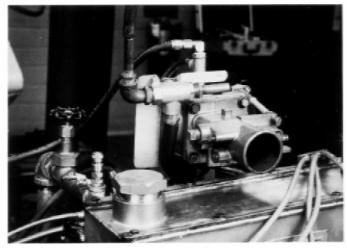


FIG. A3.18 Engine Ventilation System, Three-Way Valve

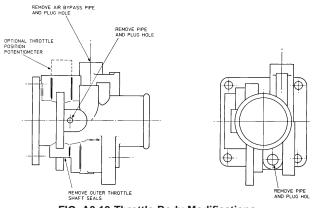


FIG. A3.19 Throttle Body Modifications

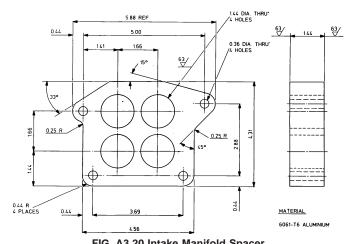


FIG. A3.20 Intake Manifold Spacer

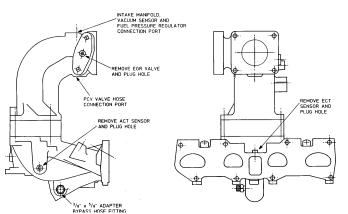


FIG. A3.21 Intake Manifold Modifications

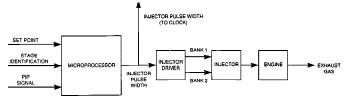


FIG. A3.22 Schematic Drawing of Open-Loop Fuel Management System

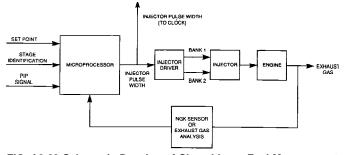
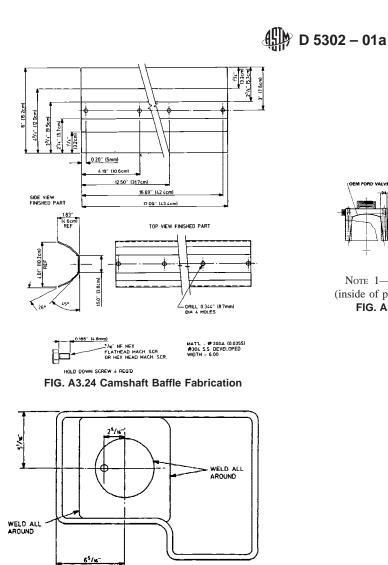
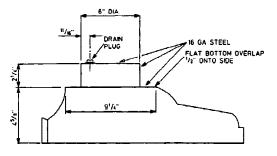


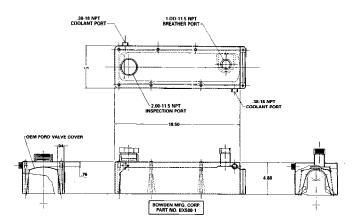
FIG. A3.23 Schematic Drawing of Closed-Loop Fuel Management System





Note 1—Tin plate all over by electroplating to a thickness of 0.0005 in. (0.013 mm).

FIG. A3.25 Modified Engine Oil Pan



Note 1—Dimensions: inches (outside of parentheses)—millimetres (inside of parentheses).

FIG. A3.26 Heated and Cooled Rocker Cover Modification

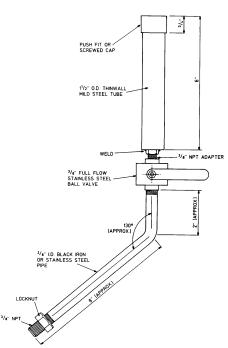


FIG. A3.27 Typical Crankcase Oil Fill Tube and Cap

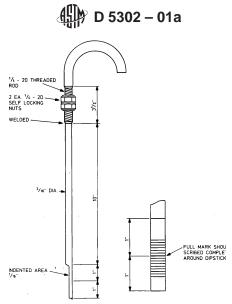


FIG. A3.28 Typical Adjustable Dipstick Fabrication

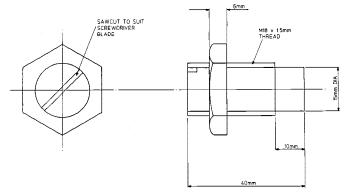
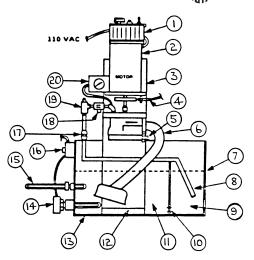


FIG. A3.29 Oil Pressure Relief Valve Adjuster



- NOTE 1 DAYTON DC MOTOR MODEL 22846 IS SUGGESTED. (%hp @ 2500 rpm, rpm 0-2500, FRAME 56/56C, DAYTON ELECTRIC MFG. CO. CHICAGO, ILL. 60648)
- NOTE 2 · INVALCO MODEL W3/0750 IS SUGGESTED. 1-13 gpm, ss, C-E INVALCO, COMBUSTION ENGINEER-ING INC., P. O. BOX 556, TULSA, OKLAHOMA, 74101.
- NOTE 3 ORIFICE IS 0.180 ± 0.0005 in. (4.572 ± 0.013mm) dia., ¼ in. (6.4 mm) DEEP, MACHINED FROM BRASS.
- NOTE 4 TO ELIMINATE AERATION THE OIL IS RETURNED TO A SEPARATE COMPARTMENT VIA THE % IN. ID TUBING. THE OIL RETURNS TO THE PICK-UP COMPARTMENT VIA THE HOLES INDICATED. A TRANSPARENT SECTION IN THE DISCHARGE TUBE IS HELPFUL IN IDENTIFYING EXCESSIVE AERATION
- NOTE 5 OPERATING CONDITIONS: SPEED 1250 ± 10 rpm. TEMPERATURE 125 ± 5°F (51.7 ± 2.8°C), FLOW RATE 6.6 ± 0.1 gpm (0.416 ± 0.006 dm³/8 AT 60 ± 1 psi (413.7 ± 6.9 kPa).
- NOTE 6 PURCHASE AVAILABILITY:

MOTOR CONTROLLER WITH SPEED ADJUSTMENT (\cdot)

- PERMANENT MAGNET VARIABLE SPEED DC MOTOR (2) (SEE NOTE 1).
- (3) MOTOR AND PUMP MOUNTING PEDESTAL.
- 60 TOOTH SPEED PICK-UP ASSEMBLY. (4)
- (5) RELIEF VALVE ADJUSTER
- 6 OIL PICK-UP TUBE MOUNTED TO THE OIL PUMP.
- OIL LEVEL.
- 8) ½ in. (12.7 mm) ID DISCHARGE TUBE.
- (9) SUMP DISCHARGE COMPARTMENT.
- (10) OIL RETURN HOLE 1 in. (25 4 mm) DIA 2 PLACES.
- (11) PICK-UP COMPARTMENT OF SUMP ASSEMBLY.
- (12) BASE OF MOTOR AND PUMP MOUNTING PEDESTAL.
- (13) SECTIONED VIEW OF SUMP ASSEMBLY.
- (14) 110 IMMERSION HEATER
- (15) THERMOMETER
- (16) TEMPERATURE CONTROL FOR HEATER.
- (17) %" PIPE TO %" TUBING UNION
- (18) TURBINE FLOW METER (OPTIONAL) (SEE NOTE 2)
- (19) %" TEE FITTING WITH 0 180 IN. ORIFICE THREADED IN LOWER BRANCH, WITH PLUG IN UPPER BRANCH (SEE NOTE 3)
- (20) OIL PRESSURE GAGE1% ACCURACY, 1 pri RESOLUTION (6.9 kPa).

SEE X 2.5, X 2.14

Schedule	40 A.S.A	. Pipe	Dimensions
----------	----------	--------	------------

C	D	ID			
in	mm	in	mm		
.540 .840	13.72 21.34	.364 .622	9.24 15.80		
	in .540	.540 13.72	in mm in .640 13.72 .364		



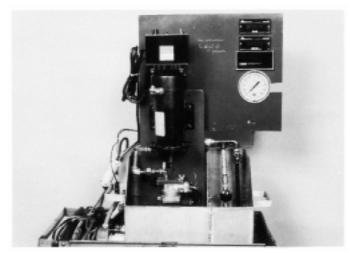
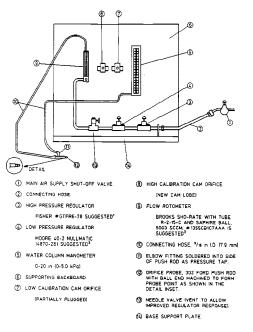


FIG. A3.31 Oil Pump Calibration Apparatus



NOTE 1—Available from Fisher Controls, Marshalltown, Iowa 50158. NOTE 2— Available from Moore Products Co., Springhouse, PA 19477. NOTE 3—Available from Brooks Instrument, Div. Emerson Electric Co., Hatfield, PA 19440.

FIG. A3.32 Camshaft Lobe Flow Rating Apparatus

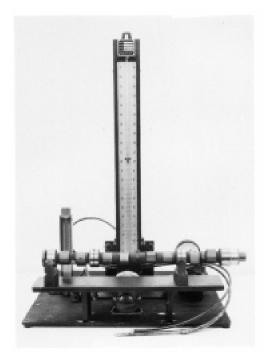


FIG. A3.33 Typical Camshaft Lobe Flow Rating Apparatus

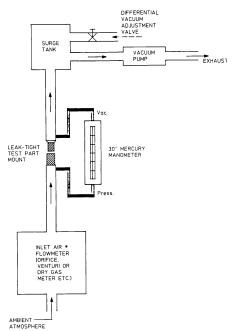


FIG. A3.34 PCV Valve Flow Test Apparatus

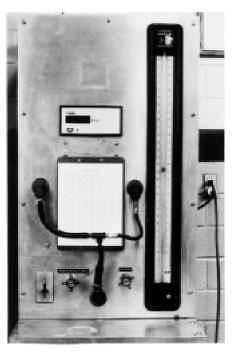
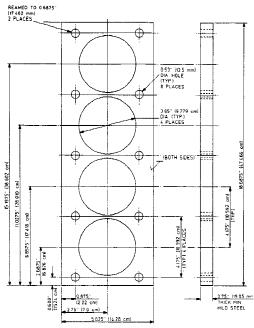
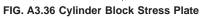


FIG. A3.35 Typical PCV Valve Flow Test Stand





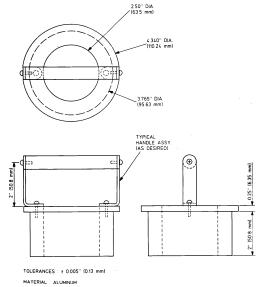


FIG. A3.37 Piston Ring Positioner

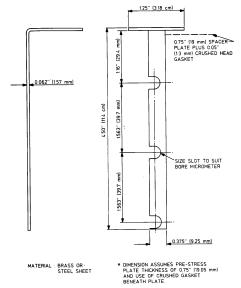


FIG. A3.38 Positioning Ladder for Bore Micrometer

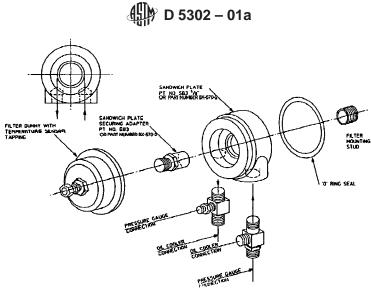
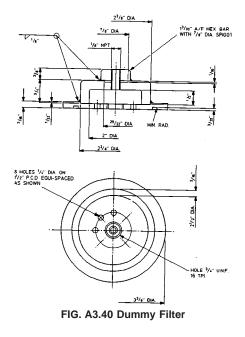


FIG. A3.39 Modified Oil Filter Sandwich Plate With Dummy Filter



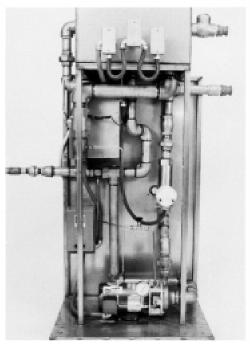


FIG. A3.41 Engine Cooling System Flushing Cart

Tool Number	Description	Tool Number	Description
T74P-6000LA	2.3 litre Service Tool Kit	T74P-6375-A	Flywheel Holding Tool
T74P-6015-A	Engine Plug Replacer	T74P-6565-A	Valve Spring Compressor
T74P-6019-B	Front Cover Alignment Tool	T73P-6571-A	Valve Seal Installer
T68P-6135-A	Piston Pin Remover/Replacer	T79P-6634-A	Pressure Diagnostic Gauge
T74P-6136-A	Crankshaft Timing Gear	Т74Р-6700-В	Front Cover Seal Remover
	Remover/Adapter	T82L-6701-A	Crankshaft Seal Installer
T74P-6150-A	Camshaft and Aux. Shaft		
	Seal Replacer	D79L-6731-B	Oil Filter Wrench
T71P-6250-A	Camshaft Bearing Replacer	T71P-7137-C	Pilot Bearing Replacer
T74P-6254-A	Camshaft Belt Tension	Т71Р-7137-Н	Clutch Aligner
	Adjusting Tool	T77L-9533-B	Jet Plug Remover/Small
T74P-6256-B	Camshaft and Aux. Shaft	T58L-101-A	Puller Attachment
	Glide Hammer Sprocket Tool	T74P-6306-A	Crankshaft Timing Gear To
T59L-100-B	Slide Hammer	T74P-6312-A	Crankshaft Damper Remove

FIG. A3.42 Special Service Tools for the Ford 2.3-L Engine

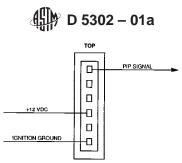


FIG. A3.43 Schematic Diagram of Ignition Module Connector

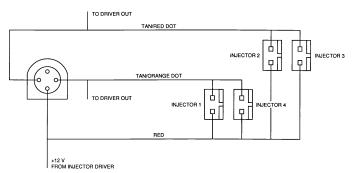


FIG. A3.44 Schematic Diagram of Injector Wiring Harness

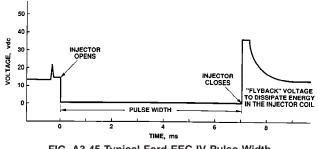


FIG. A3.45 Typical Ford EEC-IV Pulse Width

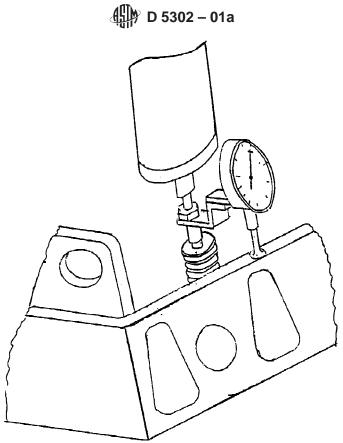
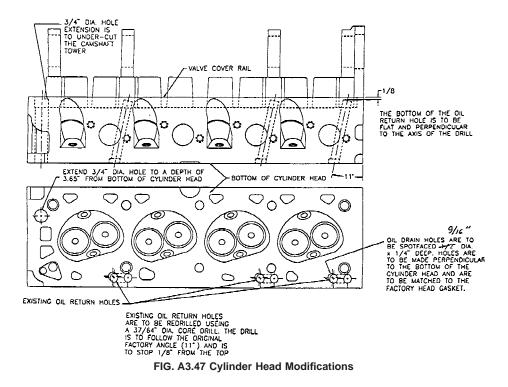


FIG. A3.46 Locating the Dial Indicator for Cylinder Head Calibration



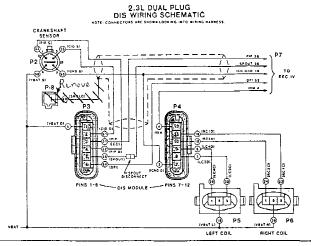


FIG. A3.48 Modifications to Distributorless Ignition System Wiring Harness

A4. ENGINE PART NUMBER LISTING

A4.1 See Table A4.1 and Table A4.2.

Number Per Kit	Part Name	Engineering Part Number			
1	Short Block Assembly	F17E-6009-BA			
1	Bare Block Assembly	E97E-6010-BA			
4	Gasket—Front Cover	E89E-6020-AB			
4	Cylinder Head Assembly	F27E-6049-A11A			
8	Gasket—Cylinder Head	E89E-6059-D5A			
1	Cover Front	E89E-6059-AC			
2	Kit Piston Ring 0.15 mm oversize	E9JL-6148-AB			
2	Kit Piston Ring 0.45 mm oversize	E9JL-6148-BB			
32	Connecting Rod Bearing	D42E-6211-AA			
4	Camshaft Assembly	E59E-6251-DA			
4	Belt Timing	E5RE-6268-A2A			
1	Washer-Camshaft Sprocket	D42E-6278-AA			
28	Main Bearings	E97E-6333-AA			
1	Crankshaft	E89E-6303-AB			
12	Main Bearings—Blue	E97E-6333-BA			
4	Upper Thrust Bearing	E97E-6333-BA			
20	Valve—Exhaust	E43E-6505-B2A			
20	Valve—Intake	E79E-6507-A2A			
8	Spring—Valve	E7JL-6513-AA			
8	Retainer—Valve Spring	E43E-6514-AC			
32	Key—Valve Spring Retainer	D0AE-6518-B3			
32	Rocker Arm	D8EE-6564-A1A			
8	Gasket—Rocker Arm Cover	F0ZE-6584-ACA			
4	Pump Assembly—Oil	E59E-6621-BC			
4	Pickup Tube Screen and Cover	E59E-6622-BA			
4	Gasket—Oil Pump Tube	E8ZE-6625-AA			
16	Seal-Crankshaft Front	E6ZE-6700-A2A			
4	Seal—Crankshaft Rear	E4ZE-6701-A2A			
8	Gasket—Oil Pan	E6ZE-6710-A6B			
4	Filter Oil	E9TE-6714-A1A			
4	Gasket—Oil Drain Plug	E8DE-6734-AA			
2	Insert—Oil Filter	E85E-6890-AA			
4	Gasket—Thermostat Housing	E87E-8255-AA			
4	Gasket—Low Man. Head	E89E-9439-A5C			
4	Gasket—Exhaust Manifold	D9EE-9448-BA			
	Clamp—Distributor	D9ZE-12270-AA			
1	Ignition Wires—R.H.	F17E-12280-BA			
1	0				
1	Ignition Wires—L.H.	F17E-12280-AA			

TABLE A4.1 1987 ASTM Sequence VE Engine Kit

TABLE A4.1 Continued

Number Per Kit	Part Name	Engineering Part Number
40	Spark Plugs—AWSF32F	E7DE-12405-BA
1	Washer, Crankshaft, Damper	D8EE-6378-AA
1	Plug, Oil Pan Drain	E9DE-6730-AA
8	Dowel-Head-Block	D42E-6A008-A2A
2	Piston Kit 0.5 mm oversize	E9JL-6A108-AB
2	Piston Kit 0.2 mm oversize	E9JL-6A108-BB
2	Plate-Cam/Aux Shaft	D42E-6A222-A2B
20	Bearing—Main Lower	E97E-6A338-BA
48	Seal Intake Valve	E7ZE-6A517-AA
1	Shaft and Hsg. Assembly-Oil Pump Drive	E97E-6C642-AA
1	Guide-Timing Belt-Cam	D42E-6B260-AA
4	PCV Valve (EV-17B) and Elbow Assembly	D8DE-6A666-BA
1	Guide-Timing Belt-Crank	D42E-6C312-AB
32	Tappet	EOEE-6C501-AA
1	Cover-Timing Belt Inner	F07E-6E006-AA
1	Cover-Timing Belt Outer	F27E-5E004-AA
1		E7TE-6K230-AA
	Shaft Assembly Auxiliary	
1	Bolt-Tensioner Assembly	E97E-6K282-BA
1	Spring-Tensioner Assembly	D52-6L273-AA
4	Gasket-Up. ManThrottle	E59E-9E936-AA
4	Fuel Injector	F03E-9F593-A2B
8	Seal-Throttle Body Insert	E57E-9F791-AA
8	Gasket-Upper Manifold-Lower Manifold	E89E-9H486-AC
1	Seal, Oil Pump Shaft	E89E-12A118-AA
4	Plug-Cup 2 ⁵ / ₆₄ in. (mm)	376047-S100
16	Plug-Cup $1\frac{1}{2}$ in. (mm)	376053-S36
1	Plug-Cup 3/4 Oil Pump Shaft	376099-S
1	Washer-Auxiliary Sprocket	385803-S
1	Dowel-Auxiliary Shaft	385803-S
8	,	
-	Plug-Oil Gallery-Head	87837-S100
16	Plug-Oil Gallery-Block	87837-S100
2	S and W-Fuel Rail to Intake Manifold	N602734-S100
1	Nut-Oil Screen to Bearing Cap	N620484-S2
1	Bolt-Auxiliary Shaft Sprocket	N600313-S2
1	Bolt-Camshaft Sprocket	N600414-S100
4	S and W Front/Auxiliary Cover-Block	N602529-S2
4	S and W Auxshaft Cover to Block	N800110-S
8	S and W-Exhaust Manifold to Head	N802884-S100
-	Key, C/S, Hub and Sprocket	N806700-S
2	S and W-Thermostat Housing to Head	N800025-S
8	S and W-Intake Manifold to Head	N805731-S100
2		
	S and W-Oil Pump-Block	N800032-S
2	Dowel-Oil Pump-Block	N800059-S
4	S and W-Auxiliary Shaft Plate	N604465-S
2	S and W-Pickup Tube to Oil Pump	N602550-S
1	Bolt-Distributor Clamp	N800088-S2
1	Screw-Timing Belt Cover	N800167-S2
5	S and W-Inner Timing Belt Cover to Front Cover	N621906-S2
40	Bolt-Heat to Block	N807013-S
16	S and W-Oil Pan to Block	N806378-S141
2	Stud-Tensioner to Inner Timing Belt Cover	N804964-S100
2 4	Bolt-Oil Pan to Front Cover	N806379-S141
4		
1	Stud-Inner Timing Belt Cover to Front Cover	N805595-S2

|--|

Number Per Kit	Part Name	Engineering Part Number
2	Sprocket—Camshaft/Auxshaft	D6EE-6256-A1A
1	Sprocket—Crankshaft	E87E-6306-AA
1	Conn—W/O	F1ZE-8594-AA
1	Intake Manifold, Upper	E79E-9425-BA
2	Cover, Auxiliary Shaft	D42E-6E007-AB
1	Throttle Body	E77E-9E926-AC
1	Intake Manifold, Lower	E7ZE-9K461-BA
1	Manifold Assembly, Fuel Injector	F12E-9S441-AA
1	Tensioner Assembly	D42E-6K254-AD
1	Flywheel, Manual Transmission	E27R-6375-AB

A5. OPERATIONAL DATA LOG SHEETS

A5.1 See Figs. A5.1-A5.3.

							OIL S			EQUEN		EVELING	RECO	RD										
				Test N	umber:					_ Lat	ooratory	Oil Co	de:											
											Clien	t Oil Co	de:											
Cycle	3	6	9	12	15	18	21	24	27	30	33	36	39	42	45	48	51	54	67	60	63	66	69	72
Test Hour, hr	12	24	36	48	60	72	84	96	108	120	132	144	156	168	180	192	204	216	228	240	252	264	276	288
Specific Action Time, hr:min	11:25	23:25	35:24	47:25	59:25	71:25	83:25	95:25	107:25	119:25	131:25	143:25	155:25	167:25	179:25	191:25	203:25	215:25	227:25	239:25	251:25	263:25	275:25	287:25
1. Remove a 5-oz purge sample.																							—	
2. Replace the 5-oz purge sample.	_														—								—	—
3. Remove a 2-oz analysis sample.																							—	—
4. Add 5 oz of new oil (except at 288 hours).		—								—	—								<u> </u>			—		
5. Shut down the engine.										—		—				—				—		—		—
 Allow 20 min. to elapse and remove the dipstick. 								—				—										—		
 DIPSTICK OIL LEVEL Record the dipstick oil level in oz. "O" indicates fully, while "H" indicates high and "L" indicates low. "H" and "L" must be pre- ceeded by the number of oz high or low. 																								
 Restart the engine 5 min. before the official beginning of Stage I. 		—		—		—						<u> </u>		—		—		—		—				
 AMOUNT OF OIL DRAINED If the oil level is high in Step 7, record the amount of oil drained in oz to reduce the level to full. If no oil is drained, enter "zero." 			I																					
10. FINAL OIL LEVEL Enter "O" for full or the number of oz tow indicated after comple- tion of Step 9.																								
11. OPERATOR'S INITIALS																								
Note: If the FINAL OIL LEVEL (Step 10) is mor								validated bec	cause of ex	cessive oil c	onsumption													
OIL CONSUMPTION - Calculate the oil consum	nption pe	r 24-hour pi	eriod accore	ding to the a	quations sl	nown below.																		

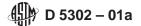
OiL CONSUMPTION (or per 24-hour period Cycles (1-69) = 8 - x - y - z

OIL CONSUMPTION loz per 24-hour period Cycles 69-721 = 1 - x - y - 2

where x - the FINAL CIL LEVEL of the previous period

$$\label{eq:generalized} \begin{split} \gamma & \sim \mbox{the amount of oil removed in Step 9 of the current period} \\ z & ~\mbox{the FINAL OIL LEVEL of the current period} \end{split}$$

Note 1-SI Equivalents-2 oz (54 mL) 5 oz (148 mL). FIG. A5.1 Oil Sampling, Addition, and Leveling Data Record



SEQUENCE VE

MANUAL DATA RECORD

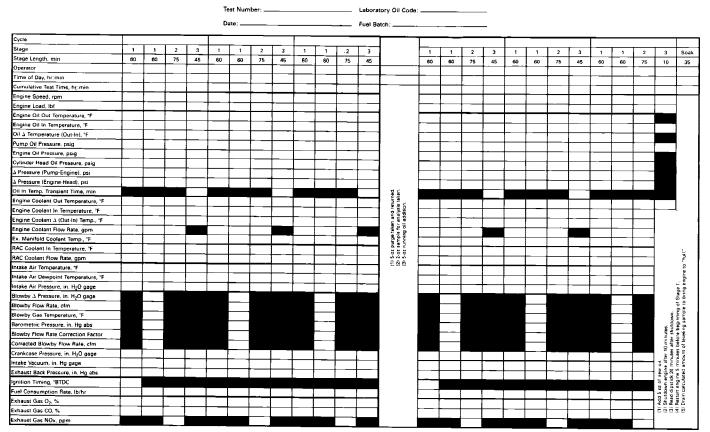


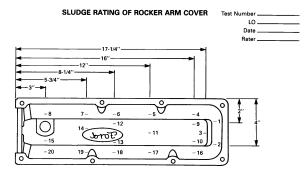
FIG. A5.2 Routine Engine Operation Data Log and Special Maintenance Data Log

item		Test				IF SH	UTDOWN		r
No.	Stg.	Hrs.	Time	Complete Description of the Proble	m and Actions Taken	Authorized	Shutdown	Restart	1
	ŧ		· · ·			Ву	Time		Observer
		1				· · · · ·			
						<u> </u>			
						+		L	
	1			· · · · · · · · · · · · · · · · · · ·		· · · · · · · · · · · · · · · · · · ·			
		1							1
						1			
						1			
		· .				·			
						· · · ·			
						I		_	
						1			
			1						
1						<u>├</u>			
1			- (┟╼╼╴╼┨			
1						l	<u> </u>		
			ł			L			
			ł						
			1						
			L						
	1	- 1	- [
	1		t						
	- 1		ŀ						
			-						
1				NOTE: THE FOLLOWING MUST BE RECORDED:					
					All Stand Maintenance All Parts Replacements				
- 1					All Instrument Calibration Adjustments				
			- (Test Operation Out of Limits	⊨∔			
						 			
							1		

FIG. A5.3 Special Maintenance Record, Trouble Record, and Unschedule Shutdown Record

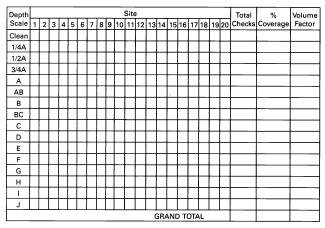
A6. RATING WORKSHEETS

A6.1 See Figs. A6.1-A6.19. See 13.2 for sludge rating details and 13.3 for varnish rating details.



Rating sites on the top surface of the RAC are located 0.75 in. from the side or in the center.

Rating sites on vertical surfaces are located 2.75 in. from the bottom of the cover.

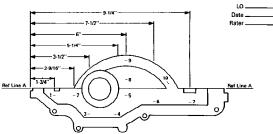


The depth gauge should be oriented as indicated by the (-) and the dimensions shown on the drawing.

Rocker Arm Cover Sludge Rating ____

FIG. A6.1 Sludge Rating of Rocker Arm Cover

SLUDGE RATING OF FRONT SEAL HOUSING Test Number



Rating sites 1, 2, and 5 are located 0.25 in. below Reference Line A.

Rating sites 6 and 7 are located 0.5 in. below Reference Line A.

Rating sites 3 and 4 are located 1.5 in. below Reference Line A. Rating site 9 is located 1.5 in. above Reference Line A.

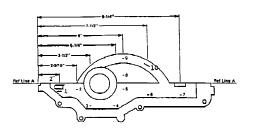
Rating site 9 is located 1.5 in. above Reference Line A.

Rating site 10 is located 0.75 in. above Reference Line A.

The depth gauge should be oriented as indicated by the (-) and the dimensions shown on the drawing.

					Total	%	Volume					
1	2	3	4	5	6	7	8	9	10	Checks	Coverage	Factor
									ſ			
									Γ			
							Gra	nd Tot	al			
	1			1 2 3 4 - - - - - - - - - - - - - - - - - - - - - - - - - - - - - - - - - - - - - - - - - - - - - - - - - - - - - -		Site 1 2 3 4 5 6 - - - - - - - - - - -			1 2 3 4 5 6 7 8 9 1 2 3 4 5 6 7 8 9 1 2 3 4 5 6 7 8 9 1 2 3 4 5 6 7 8 9 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1		1 2 3 4 5 6 7 8 9 10 Checks 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 <t< td=""><td>1 2 3 4 5 6 7 8 9 10 Checks Coverage 1 2 3 4 5 6 7 8 9 10 Checks Coverage 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1</td></t<>	1 2 3 4 5 6 7 8 9 10 Checks Coverage 1 2 3 4 5 6 7 8 9 10 Checks Coverage 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1

Front Seal Housing Sludge Rating ______ FIG. A6.2 Sludge Rating of Front Seal Housing



Rating site 1 is located 0.5 in. below Reference Line A.

Rating sites 2 and 5 are located 0.25 in. below Reference Line A.

Rating sites 6 and 7 are located 0.5 in. below Reference Line A.

Rating sites 3 and 4 are located 1.5 in. below Reference Line A.

Rating site 8 is located 0.5 in. above Reference Line A.

Rating site 9 is located 1.5 in, above Reference Line A.

Rating site 10 is located 0.75 in. above Reference Line A.

The depth gauge should be oriented as indicated by the (-) and the dimensions shown on the drawing.

Depth				Total	%	Volume							
Scale	1	2	3	4	5	6	7	8	9	10	Checks	Coverage	Factor
Clean													
1/4A										T			
1/2A													
3/4A													
A													
AB													
В													
BC													
С													
D								1					
E													
F									1				
G													
Н								1					
I													
J								1					
·				• • • • •		•	GRA	ND TC	TAL	·			

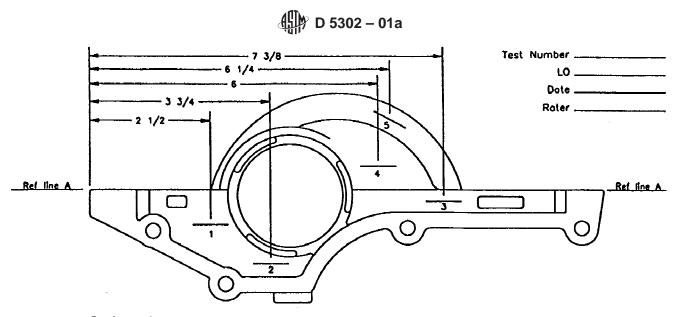
Front Seal Housing Sludge Rating_____

Test Number

LO_ Date

Rater

FIG. A6.3 Sludge Rating of Front Seal Housing (1989 and Later Kits)



Roting site 1 is located 0.75 in. below Reference Line A. Roting site 2 is located 1.5 in. below Reference Line A. Roting site 3 is located 0.25 in. below Reference Line A. Roting site 4 is located 0.5 in. above Reference Line A. Roting site 5 is located 1.5 in. above Reference Line A. The depth gauge should be oriented as indicated by the (----) and the dimensions shown on the drawing.

Depth			Site			Total	%	Volume
Scole	1	2	3	4	5	Checks	Coverage	Factor
Cleon								
1/4A							1	
1/2A								
3/4A								_
A								
AB								
B								
BC								
C								
D								
Ε								
F								
G								·····
н								
					1		1	
J							1	
			•	Gran	d Totol			
L					•••••••		L	

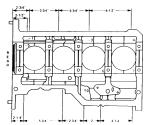
Front Seol Housing Sludge Rating _____

FIG. A6.4 Sludge Rating of Front Seal Housing (1992)

LO

Date Rater

SLUDGE RATING OF ENGINE BLOCK



Rating sites 1, 2, 3, and 4 are located 1.0 in. from the inside edge of the gasket mounting surface.

Rating sites 5, 6, and 7 are located 0.5 in. from the cyinder bore.

Rating sites 8 and 9 are located 1.125 in. from the inside edge of the gasket mounting surface.

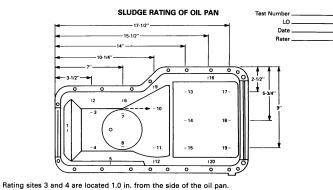
Rating site 10 is located 1.0 in. from the inside edge of the gasket mounting surface.

The depth gauge should be oriented as indicated by the $(\,-\,)$ and the dimensions shown on the drawing.

Depth						Total	%	Volume					
Scale	1	2	3	4	5	6	7	8	9	10	Checks	Coverage	Factor
Clean													
1/4A													
1/2A													
3/4A													
Α													
AB													
В													
BC													
С													
D													
Ε.													
F													
G													
н													
1													
J													
								Gra	nd Tot	al			

Engine Block Sludge Rating

FIG. A6.5 Sludge Rating of Engine of Block



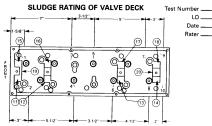
Rating sites 7 and 8 are located 0.75 in. from the side of the oil pan.

The depth gauge should be oriented as indicated by the $(\,-\,)$ and the dimensions shown on the drawing.

Depth											Site										Total	%	Volume
Scale	1	2	3	4	5	6	7	8	9	10	11	12	13	14	15	16	17	18	19	20	Checks	Coverage	Factor
Clean																							
1/4A																							
1/2A																							
3/4A																							
Α																							
AB																							
В																							
BC																							
С																							
D																							
Е																							
F																							
G																							
н																							
I																							
J																							
		GRAND TOTAL																					

Oil Pan Sludge Rating _

FIG. A6.6 Sludge Rating of Oil Pan



Rating sites 1, 2, 3, 4, 5, 6, 7, and 9 are located 1.0 in. from the outer edge of the cylinder head.

Rating sites 8 and 10 are located 1.125 in. from the outer edge of the cylinder head.

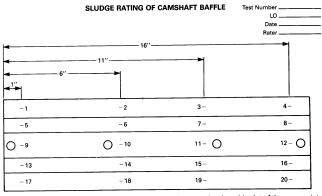
Rating sites 11, 12, 13, 14, 15, 16, 17, and 18 are located on the camshaft pedestal in line with the gasket mounting surface.

Rating sites 19 and 20 are located at the midpoint of the machined surface on the camshaft pedestal below the camshaft bearing.

The depth gauge should be oriented as indicated by the $(\,-\,)$ and the dimensions shown on the drawing.

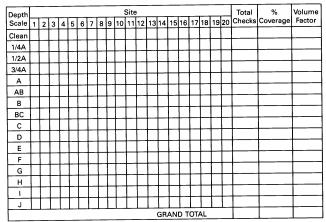
Depth Scale	1	2	3	4	5	6	7	8	9	_	Site	_	13	14	15	16	17	18	19	20	Total Checks	% Coverage	Volume Factor
Clean																							
1/4A																							
1/2A																							
3/4A																							
А																							
AB																							
в																							
BC																							
С																							
D																							
Е																							
F																							
G																							
н																							
1																							
J																							
													(GRA	NE	т	ЭТА	۱L					

Valve Deck Sludge Rating FIG. A6.7 Sludge Rating of Valve Deck



All rating sites are located according to the dimensions shown and at the midpoint of the appropriate plane.

The depth gauge should be oriented as indicated by the (-).



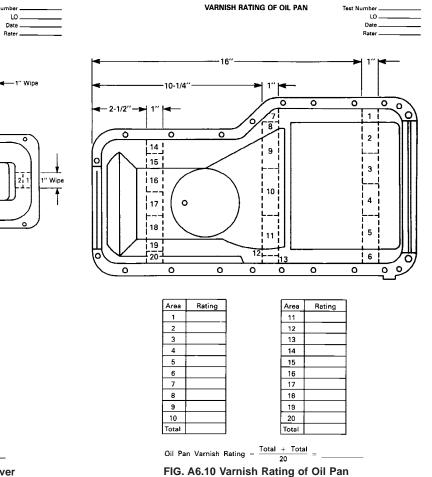
Camshaft Baffle Sludge Rating

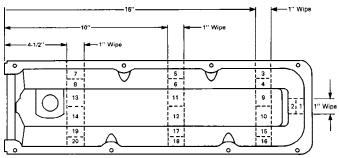
FIG. A6.8 Sludge Rating of Cam Cover Baffle

🕼 D 5302 – 01a

LO

VARNISH RATING OF ROCKER ARM COVER Test Numb





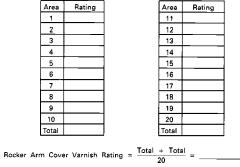
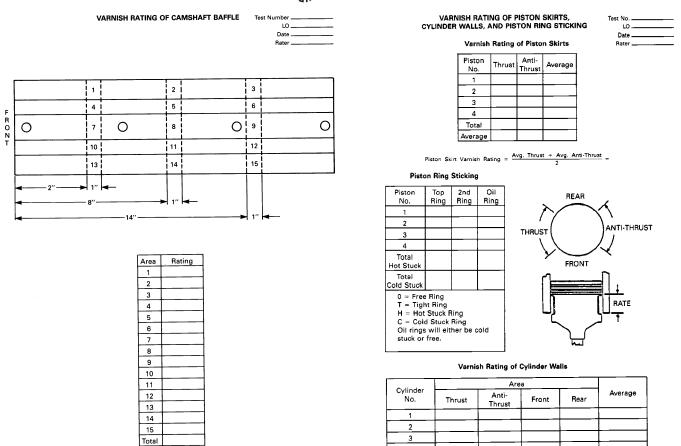


FIG. A6.9 Varnish Rating of Rocker Arm Cover



4

Camshaft Baffle Varnish Rating = $\frac{\text{Total}}{15}$ = _____ FIG. A6.11 Varnish Rating of Cam Cover Baffle

FIG. A6.12 Varnish Ratings of Piston Skirts and Cylinder Walls and Rating for Ring Sticking

Total



Pump Relief Valve & Intake Valves

CLOGGING OF OIL RINGS, OIL SCREEN, PCV VALVE, AND CAMSHAFT LOBE HOLES	Test No
STICKING OF HYDRAULIC LIFTERS	Rater

STICKING OF HYDRAULIC LIFTERS

VARNISH RATING OF OIL PUMP RELIEF VALVE,

			A	ND I	NTA	KE V	ALVE	DEPO	SITS						
Г	Piston N	No. CI	ogging.9	1				Г	Debr	is, %	Γ	Slud	ge, %		
Ī	1							Г	_						
	2	_						Γ			Т				
	з							Γ							
Γ	4							Γ						-7	
F	Averag	e													
-				;am	shaf	t Lot	e Ho	le Clog	ging						
	Lo	halp	aseline		Fina	. 1				w Bat	. 1		-		
	Num		ow Rate		ow F		Flow	/ Rate		uction,		Plug	iged		
	1	E													
		F													
	2	E													
	2	1													
	3	E													
	3	1													
	4	E													
	4	1													
								-						Relief	
	yaraulic	Lifter Sti	cking			intak	e van	ve Dep	OSITS			van	/e Va	misn	
Lifter	Stud		Partially			Valv		Rat	ina			ng		Varn	ish
Numbe	r Plung	ger Stu	ick Plung	er	H	Num	ber_			\downarrow		nber	-		
1E	-			_	⊢	1				$ \vdash$		1	-		
11	-			_	⊢	2				┥┝		2 3	-		
2E 21		_		-	H	3				+		3 4	-		
21 3E				_	H	4 Avera	_			+		4 rage			
31	-			-	Ľ.	Avera	aga				Ave	rage			
4E	+			-											
41															
5E															
51				-											
Total															
							e Clog	ging							
	Flow R	ate, cfm		1	Flo	w Ra	nte Di	fferenc	:0		С	loggi	ng, %		
18 in.			Hg ∆P	11	18 in. Hg ΔP 8 in. Hg ΔP					18 in				. Hg A	Р
Before Test	After Test	Before	After Test		(86	efore	– Af	ter, cfn	n)						
				1											

FIG. A6.13 Rating of Oil Rings, Oil Screen, PCV Valve, Cam Lobe Holes, Hydraulic Lifters, Oil Pump Relief Valve, and Intake Valves

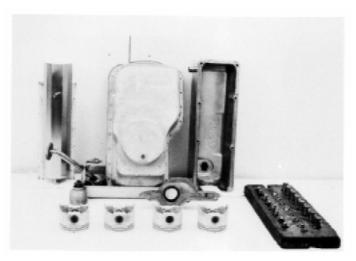
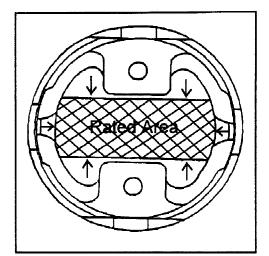


FIG. A6.14 Typical Parts Layout for Rating (see 12.6.3)

Rated by _	
Engine No.	

Date	
Test No.	
Oil Code	



Piston 1)	Area	Intensity	Merit	Rating
2)				
3)				
,				
·				-
4)				
		-		
			· · · · · · · · · · · · · · · · · · ·	
			Average	

FIG. A6.15 Piston Undercrown Deposit Breakdown Sheet

Rated by	
Engine No.	
Oil Code	

Date	
Test No.	

Piston 1)	Area	Intensity	Merit	Rating
		· · · · · · · · · · · · · · · · · · ·		
		<u> </u>		
	<u> </u>			
2)				· <u>-</u> ···
2)		·		
3)				
, ,		<u> </u>	<u>-</u>	
			· · · · · · · · · · · · · · · · · · ·	
			· · · · · · · · · · · · · · · · · · ·	
4)				
			······	
			·	
				·
			Average	

FIG. A6.16 Second Land Deposit Breakdown Sheet

Rated by	
Engine No.	
Oil Code	

Date	
Test No.	

Piston 1)	Area	Intensity	Merit	Rating
2)				
2)			·	
3)				
	••••••••••••••••••••••••••••••••••••••			
4)				
		· ····································		
	<u></u>			
		17 Oil Ring Deposit Breakdow	Average	

FIG. A6.17 Oil Ring Deposit Breakdown Sheet

Rated by	
Engine No.	
Oil Code	

Date	
Test No.	

Piston Area Intensity Merit Rating 1)
2)
2)
2)
2)
2)
2)
2)
2)
2)
3)
4)
Average

FIG. A6.18 Crown Land Deposit Breakdown Sheet

(D	5302	_	01	a
---	---	------	---	----	---

Rated by	Date
Engine No.	Test No
Oil Code	

Cylinder	Before	After	Difference (bore wear, mils)	
1) Top				Cylinder
Middle	<u></u>			Average (mils)
Bottom	, 			Average (mins)
Douoin				
2)				
Top				Cylinder
Middle				Average (mils)
Bottom				
3)				
Тор				Cylinder
Middle				Average (mils)
Bottom				
4				
4) Top				Cylinder
Middle				-
Bottom		· · · · · · · · · · · · · · · · · · ·		Average (mils)
Douom		•		
			Average Cylinder Bore Wear	
				(mils)
				(11113)
			Max Cylinder Bore Wear	
				(mils)
		FIG. A6.19 Cylinder B	Sore Wear	()

A7. FINAL REPORT FORMS AND PHOTOGRAPHS

A7.1 Report Forms—See Figs. A7.1-A7.22.



TEST METHOD D5302 (SEQUENCE VE) VERSION 19970902

CONDUCTED FOR

TSTSPON1

TSTSPON2

	V =VALID
LABVALID	I = INVALID
	N - RESULTS CAN NOT BE INTERPRETED AS REPRESENTATIVE OF OIL PERFORMANCE (NON-REFERENCE OIL) AND SHALL NOT BE USED FOR MULTIPLE TEST ACCEPTANCE

			Fest Number	
Test Stand:	STAND	Runs Between Calibr RSTRUN/S	ation Tests: STRUN	Total Runs on Test Stand: RTOTSRUN/TOTSTRUN
Date Comple	ted: RD	TCOMP/DTCOMP	End of Tes	t Time: REOTTIME/EOTTIME
Oil Code ^A :	CMIR/0	DILCODE		
Formulation/	Stand Co	de: FORM		
Alternate Co	des ALT	CODE1	ALTCODE2	ALTCODE3

In my opinion this test OPVALID been conducted in a valid manner in accordance with the Test Method D 5302 and the appropriate amendments through the Information Letter system. The remarks included in the report describe the anomalies associated with this test.

The test stand and laboratory have been calibrated in accordance with the requirements specified in Test Method D 5302 and the appropriate amendments through the Information Letter system.

ACMIR/Oil Code

SUBMITTED BY:

SUBLAB

Testing Laboratory

SUBSIGIM

Signature

SUBNAME

Typed Name

SUBTITLE

FIG. A7.1 Test Report Cover

Title

(SEQUENCE VE) RATING AND MEASUREMENTS SUMMARY

Laboratory LAS	Oil Code OILCODE	SAE Viscosity Grade
Calibration A stand - ASTRUN * ATOTSRUN	TMC Oil Code IND	Date/Time Calibrated RDTCOMP / REOTTIME
Test Number STAND - STRUN- TOTSTRUN	Test Longth TESTLEN	Date/Time Completed DTCOMP / EOTTIME
Engine Number BIGINE	Nominal Piston Oversize NOMPISOV	Fuel Batch FUELBTID
Date/Time Started DTSTRT / STRTTIME	Internal Oil Code LABOCODE	
Formulation/Stand Code: FORM		

_		Rating (Reported Units)	Rating (Trans.)	Industry Correction Factor (Transformed)	Corrected Rating (Transformed)	Severity Adjustment (Transformed)	Final Rating Value (Transformed)	Final Rating Value (Reported Units)
S	Rocker Arm Cover	RACSRT	TRANRACS	RACCF	TRACSCOR	RACSSA	TRACSPNL	RACSIENL
L	Cemehaft Baffie	CAMBESRT						
U	Valve Deck	VLVDCKRT						
Ď	Front Seal Housing	FSLHSGRT						
G	Cylinder Block	CYLBLKRT						
E	Oil Pan	OILPNSRT						
	Average Engine Sludge	AES	TRANAES	AESCF	TAESCOR	AESSA	TAESFNL	AESFNL

V	Piston Skirts, merits	PISKVRT	PISKVCF	PSVSA	PISKVFNL
A	Rocker Arm Cover, merits	RACVRT			
R	Cemshaft Baffle, merits	CAMBEVET			
ni I	Cylinder Wells, merits	CYLWLVRT			
S	Oil Pan, merits	OILPNVRT			
Ĥ	Average Engine Varnish, merits	AEV	AEVCF	AEVSA	AEVFNL

	ACW, micrometers	ACW	TRANACW	ACWCF	TACWCOR	ACWSA	TACWFNL	ACWFNL
W	MCW, micrometers	MCW	TRANMCW	MCWCF	TMCWCOR	MCWSA	TMCWFNL	MCWFNL
E	Avg. Rocker Arm, mg	ARAW						
A	Max. Rocker Arm, mg	MAXRAW						
R	Avg. Top Ring Inc., mils	ATRGINC						
	Max. Top Ring Inc., mils	MXTRGINC						
	Avg. Rod Bearing, mg	ARBWL						
1	Max. Rod Bearing, mg	MAXBWL						

c	Oil Screen Sludge, %	OSCANSLG	0	n	Average Crown Land Deposit, merits	
Ĭĭ					Average Second Land Deposit, merits	AVGSLD
10	Oil Screen Debris, %	OSCRNDEB	HP		Average Oil Ring Land Deposits, merits	AVGORLD
G	Oil Ring, %	OILRING	Ε	Ō	Average Piston Under Crown Deposit, merits	AVGPUCD
G	PCV Valve at 18 in. Hg., %	PCV18	R	S	Intake Valve Deposits, merits	INVLVDEP
1				I	Number of Stuck Lifter Plungers	NSLFTPLG
Ν	PCV Valve at 8 in. Hg., %	PCVB			Number of Hot Stuck Compression Rings	NHSCMPRG
G	Number of Camshaft Lobe Holes	NOCAMLOB		S	Number of Stuck Oil Rings	NSTKORNG

ADDITIONAL INFORMATION

ACBLWRT1

Oil Consumption, fl. oz.	OILCON	Average Blowby Flow Rate, cfm

A Test number consists of the stand number, number of runs between calibration tests and total number of runs on the stand. FIG. A7.2 Rating and Measurement Summary, Non-Reference Oil Test

(SEQUENCE VE) Rating and Measurement Summary, Calibration Test Results

Laboratory LA	B	Oil Code CMIR	SAE Viscosity Grade RSAEVISC
Test Number ^A ST	AND - RSTRUN - RTOTORU	Test Length RTESTLEN	Dete/Time Completed RDTCOMP / REOTTIME
Engine Number	RENGINE	Nominal Piston Oversize RNOMPISO	Fuel Batch RFUELBID
Date/Time Started	RDTSTRT / RSTRTIME	Internal Oil Code RLABCODE	A

s		Rating (Reported Units)	Reting (Transformed)	Industry Correction Factor (Transformed)	Corrected Rating (Transformed)	Final Rating (Reported Units)
6	Rocker Arm Cover	RRACSRT	RTRNRACS	RACCF	RTRCSCOR	RRACSPNL
	Camshaft Baffle	RCAMBSRT				
2	Valve Deck	RVLVDKRT				
D	Front Seal Housing	RESLUSET				
G	Cylinder Block	RCYLBLRT				
E	Oil Pan	ROILPNRT				
	Average Engine Sludge	RAES	RTRANAES	AESCF	RTAESCOR	RAESFNL

V	Piston Skirts, merits	RPISKVRT		PISKVCF		RPSKVFNL
A	Rocker Arm Cover, merits	RRACVRT				
R	Camshaft Baffle, merits	RCAMBVRT				
N I	Cylinder Walls, merits	RCYLWVRT				
S	Oil Pen, merits	ROILPNVR				
Ĥ	Average Engine Varnish, merits	RAEV		AEVCF		RAEVFNL
	ACW, micrometers	RACW	RTACW	ACWCF	IRTACWCOR	RACWFNL
	MCW, micrometers	RMCW	RTMCW	MCWCF	RTMCWCOR	RMCWFNL
W	Avg. Rocker Arm, mg	RARAW				
E	Max. Rocker Arm, mg	RMAXRAW				
A	Avg. Top Ring Inc., mils	RATEGINC				
R	Max. Top Ring Inc., mils	RXTRGINC				
	Avg. Rod Bearing, mg	RARBWL				
	Max. Rod Bearing, mg	RMAXBWL				

C	Oil Screen Sludge, %	ROSCRSLG		D	D	Average Crown Land Deposit, merits	RAVGCLD
L	Oil Screen Debris, %	ROSCRDEB	-	Т	Ε	Average Second Land Deposit, merits	RAVGSLD
0	Ul Screen Debris, 76		_	H	Ρ	Average Oil Ring Land Deposit, merits	RAVGORLD
G	Oil Ring, %	ROILRING		E	0	Average Piston UnderCrown Deposit, merits	RAVGPUCD
G	PCV Valve at 18 in. Hg., %	RPCV18	Τ	R	S	Intake Valve Deposit, merits	RINVDEP
		RPCVB	-		I	Number of Stuck Lifter Plungers	RSLFTPLG
N	PCV Valve at 8 in. Hg., %		4		Т	Number of Hot Stuck Compression Rings	RHSCMPRG
G	Number of Camshaft Lobe Holes	RNCAMLOB		L	S	Number of Stuck Oil Rings	RSTKORNG

	Additional Info	rmation	
Oil Consumption, fl. oz.	ROILCON	Average Blowby Flow Rate, cfm	RABLEYRT

Reference Oil		IND	Statistics Date	STATDATE
Parameter	Target	<u>s</u>	Delta	Deita/s
AES	TARGAES	SAES	DELAES	DELSAES
RACS	TARRACS	SRACS	DELRACS	DELSRACS
APV	TARAPV	SAPV	DELAPV	DELSAPV
AEV	TARAEV	SAEV	DELAEV	DELSAEV
ACW	TARACW	SACW	DELACW	DELSACW
MCW	TARMCW	SMCW	DELMOW	DELSMCW
	· · · · · · · · · · · · · · · · · · ·			

A Test number consists of the stand number, number of runs between calibration tests, and total number of runs on the stand. FIG. A7.3 Ratings and Measurement Summary, Calibration Test



(SEQUENCE VE) OPERATIONAL SUMMARY

LABORATORY : LAB	LABORATORY : LAB OIL CODE: CMIR/OILCODE								
TEST NUMBER ^A : STAND - RSTRUN/STRU	N - RTOTE	RUN/TOTS1		TE CON	APLETED); RD [*]	TCOMP	DTCOM	IP
Formulation/Stand Code: FORM	Formulation/Stand Code: FORM								
PARAMETER		STAGE	1		STAGE	11		STAGE	
	MAX	MIN	AVG	MAX	MIN	AVG	MAX	MIN	AVG
Engine speed, r/min	XRPM 1	IRPM1	ARPM1	XRPM2	IRPM2	ARPM2	ХЛРМЗ	нрмз	АЯРМ3
Engine Power, bhp	XPWR1	IPWR1	APWR1	XPWR2	IFWR2	APWR2	хрула	IFWR3	APWR3
Engine Oil In Temperature, ^o F	XENGOIN1	IENGOIN 1	AENGOIN1	XENGOIN2	IENGOIN2	AENGOIN2	KENGOIN3	IENGOIN3	AENGOIN3
Engine Oil Delta Temp. (Out-In), °F	XENGODT1	IENGODT1	AENGODTI	XENGODT2	IENGODT2	ABNGODT2	XENGODT	IENGODTS	AENGODTS
Pump Oil Pressure, psig	XPMPOPR1	IPMPOPR1		XPMPOPR2	IPMPOPR2	APMPOPR2	XPMPOPRS	IPMPOPR3	APMPOPR3
Engine Oil Pressure, psig	XENGOPR1	IENGOPRI	AENGOPRI	KENGOPR2	IENGOPR2	AENGOPR2	XENGOPRS	IENGOPRS	AENGOPR3
Cylinder Head Oil Pressure, psig	XCYLOPR1	ICYLOPHI	ACYLOPRI	XCYLOPR2	ICYLOPR2	ACYLOPR2	XCYLOPRS	ICYLOPR3	ACYLOPRS
Delta Pressure (Pump-Engine), psi	XDPPE1	IDPPE1	ADPPE1	XDPPE2	IDPPE2	ADPPE2	XDPPE3	IDPPE3	ADPPE3
Delta Pressure (Engine-Head), psi	XDPEH1	IDPEH 1	ADPEH 1	XDPEH2	IDPEH2	ADPEH2	XDPEH3	IDPEH3	ADPEH3
Engine Coolant Out Temperature, ^o F	XCOLOUT1	COLOUTI	ACOLOUTI	KCOLOUT2	ICOLOUT2	ACOLOUT2	хсоголта	ICOLOUTS	ACOLOUTS
Eng. Coolant Delta Temp. (Out-In),°F	XCOLDT1	ICOLDT1	ACOLDT1	XCOLDT2	ICOLDT2	ACOLDT2	XCOLDT3	ICOLDT3	ACOLDT3
Engine Coolant Flow Rate, gal/min	XCOLFRT1	ICOLFRT1	ACOLITIT1	XCOLFRT2	ICOLFRT2	ACOLIFRT2		•••	
Engine Coolant Pressure, psig	XCOLPRET	ICOLPRE1	ACOLPRET	XCOLPRE2	ICOLPRE2	ACOLPRE2	XCOLPRES	ICOLPRES	ACOLPRES
Exhaust Manifold Coolant Temp., °F	XEXCOOL1	EXCOOL1	AEXCOOLI	KEXCOOL2	IEXCOOL2	AEXCOOL2	XEXCOOLS	IEXCOOLS	AEXCOOLS
RAC Coolant In Temperature, °F	XRACCTP1	RACCTP1	ARACCTPI	XRACCTP2	IRACCTP2	ARACCTP2	KRACCTP3	IRACCTP3	ARACCTP3
RAC Coolant Flow Rate, gal/min	XRACCFR1	IRACCERI	ARACCERI				KRACCIRS	IRACCERS	ARACCERS
Intake Air Temperature, °F	XINAIRT1	IINAIRT1	ANAIRT1	XINAIRT2	HNAIRT2	AINAIRT2	XINAIRTS	IINAIRTS	AINAIRTS
Intake Air Specific Humidity, grains/lb	XAIRHUM1	IAIRHUM1	AAIRHUM1	XAIRHUM2	IAIRHUM2	AAIRHUM2	CAIRHUMS	IAIRHUMS	AAIRHUM3
Intake Air Pressure, in. H ₂ O gage	XINAIRP1	IINAIRP1	ANAIRP1	KINAIRP2	IINAIRP2	aiñairp2	XINAIRP3	HNAJRP3	AINAIRP3
Corrected Blowby Flow Rate, ft ³ /min	XCBLWRT1	ICBLWRT1	ACBLWRT1				•••		•••
Crankcase Pressure, in. H ₂ O gage	XCCASEP1	ICCASEP1	ACCASEP1	KCCASEP2	ICCASEP2	ACCASEP2	KCCASEPS	ICCASEP3	ACCASEPS
Intake Manifold Vacuum, in. Hg gage	XIMNVAC1	IMNVAC1	AIMNVAC1	XIMNVAC2	IIMNVAC2	AIMNVAC2	XIMNVACI	IIMNVAC3	AIMNVAC3
Exhaust Back Pressure, in. Hg abs.	XEXBKPR1	IEXBKPR1	AEXBKPR1	KEXBKPR2	IEXBKPR2	AEXBKPR2	XEXBKPRS	IEXBKPR3	AEXBKPR3
Ignition Timing, °BTDC	XIGNTIM1	IIGNTIM1	AIGNTIM1	•••	• • •	•••			
Fuel Consumption Rate, Ib/hr	XFUELRT1	IFUELRT1	AFUELRT1	XFUELRT2	IFUELRT2	AFUELRT2	XFUELRTS	IFUELRTS	AFUELRTS
Exhaust Gas O ₂ , %	XEXHGO1	IEXHGO1	AEXHGO1	KEXHGO2	IEXHGO2	AEXHGO2	KEXHGO3	IEXHGO3	AEXHGO3
Exhaust Gas CO, %	XEXH CO1	IEXHGCO1	AEXHGCO1	KEXHGC02	IEXHGC02	AEXHGC02	XEXHGCO3	IEXHGCO3	AEXHGCO3
Exhaust Gas NO _x , ppm				XEXHGN02	IEXHGN02	AEXHGN02			

ATest number consists of the stand number, number of runs between celibration tests, and total number of runs on the stand.

FIG. A7.4 Operational Summary

(SEQUENCE VE) SUPPLEMENTAL OPERATIONAL DATA

Laboratory: LAB	Oil Code: CMIR/OIL	CODE
Test Number: STAND - RSTRUN/STRUN - RTOTSRUN/TOTSTRUN	Date Completed:	RDTCOMP/DTCOMP
Number of Remarks or Deviations: OPROCR		
Formulation/Stand Code: FORM		

ItemRemark or DeviationITEMH001REMKH001

NOTE 1—Test number consists of the stand number, number of runs between calibration tests, and total number of runs on the stand. FIG. A7.5 Supplemental Operational Data

	Laboratory: LAB Oil Code: CMIR/OILCODE	RDTCOMP/DTCOMP				Problems/Actions/Comments																
		Date Completed:					LSTMHOO1 DREAHOO1															
			nces: DWNOCR			Lost Time	LSTMH001															TOTLLOST
		TOTS				Down Time	DTIMH001															TOTLDOWN TOTLLOST
						Stage	STGEH001															
				Stand Code:		Test Hours	TESTH001															Total Downtime/Lost Time
			Number of D	Formulation/Stand Code:		Type of Shutdown A	TYPEH001															Total Do
					1	L	1	1	L	l	L	I	I	l	L	L	I	L	I	I	i	1

72

(SEQUENCE VE) SPECIAL MAINTENANCE RECORD

U if unscheduled shutdown FIG. A7.6 Special Maintenance Record

A P if parts replacement S if stand maintenance



(SEQUENCE VE) OIL ANALYSES

Laboratory: LAB	Oil Code: CMIR/OILCODE	
Test Number: STAND - RSTRUN/STRUN - RTOTSRUN/TOTSTRUN	Date Completed: RDTCOMP/DTCOMP	
Formulation/Stand Code: FORM		

Test Hour	Fe ppm	Cu ppm	Si ppm	Viscosity @ 40°C cSt Test Method D445	Fuel Dilution % mass Test Method D3525 (modified)	Pentane Insolubles Test Method D893B
New Oil	FE_HNEW	CU_HNEW	SI_HNEW	VS40HNEW		
12	FEH012	CU_H012	SI_H012	VS40H012	FUELH012	
108	FEH108	CU_H108	SI_H108	VS40H108	FUELH108	PENTH108
204	FEH204	CU_H204	SI_H204	VS40H204	FUELH204	PENTH204
288	FEH288	CU_H288	SI_H288	VS40H288	FUELH288	PENTH288

FIG. A7.7 Oil Analyses



VALVE TRAIN INSPECTION DETAIL

Laboratory:	boratory: LAB			Oil Code: CMIR/OILCODE			
Sest Number: STAN	t Number: STAND - RSTRUN/STRUN - RTOTSRUN/TOTSTRUN Date Completed: RDTCOMP/DTCOMP						
ormulation/Stand	I Code: FORM						
Position	Cam Lobe Wear (micrometers)	Lobe C Plugging Flow	3 % Air	Rocker Arm Wear mg	Valve Spring Load, Ibf		
1 (1E)	CAMW01	LO	BPLG01	ROCKW01	VSPGLD01		
2 (11)	CAMW02	LO	BPLG02	ROCKW02	VSPGLD02		
3 (2E)	CAMW03	LO	BPLG03	ROCKW03	VSPGLD03		
4 (21)	CAMW04	LC)BPLG04	ROCKW04	VSPGLD04		
5 (3E)	CAMW05		BPLG05	ROCKW05	VSPGLD05		
6 (31)	CAMW06	LC	BPLG06	ROCKW06	VSPGLD06		
7 (4E)	CAMW07	LC	BPLG07	ROCKW07	VSPGLD07		
8 (41)	CAMW08	LO	BPLG08	ROCKW08	VSPGLD08		
Average	RACW/ACW	A	/GLOBPL	ARAW	AVGSPGLO		

Lobe	Lobe Hardness (> 50 Rockwell C) ^A	Rocker Arm Hardness (> 57 Rockwell C)
1E	LOBHRD01	RCKHRD01
11	LOBHRD02	RCKHRD02
2E	LOBHRD03	RCKHRD03
21	LOBHRD04	RCKHRD04
3E	LOBHRD05	RCKHRD05
31	LOBHRD06	RCKHRD06
4E	LOBHRD07	RCKHRD07
41	LOBHRD08	RCKHRD08

^ACamshafts with individual lobe hardness less than 50 Rockwell C may be used. Record lobe location(s) and hardness value(s) on the Supplemental Operational Data Form, Fig A7.5.

FIG. A7.8 Valve Train Inspection Detail

(SEQUENCE VE) VARNISH RATING OF PISTON SKIRTS RATING OF RING LAND AND PISTON UNDER CROWN DEPOSITS

Laboratory: LAB		Oil Code:	CMIR/OILCODE	
Test Number: STAND - RETRUN/STRUN	- RTOTSRUN/TOTSTRUN	Date Completed:	RDTCOMP/DTCOMP	

FORM

Formulation/Stand Code:

Piston Number	Thrust (Merits)	Antithrust (Merits)
1	PSVTH1	PSVAT1
2	PSVTH2	PSVAT2
3	PSVTH3	PSVAT3
4	PSVTH4	PSVAT4
Average	PSVTHAVG	PSVATAVG

Piston Skirt Varnish = <u>Average Thrust + Average Antithrust</u> = RPISKVRT/PISKVRT 2

Piston Number	Crown Land (Merits)	2nd Land (Merits)	Oil Ring Land (Merits)	Piston Undercrown (Merits)
1	CLD1	SECLD1	ORLD1	PISUC1
2	CLD2	SECLD2	ORLD2	PISUC2
3	CLD3	SECLD3	ORLD3	PISUC3
4	CLD4	SECLD4	ORLD4	PISUC4
Average	RAVGPUCD/AVGPUCD	RAVGSLD/AVGSLD	RAVGORLD/AVGORLD	RAVGCLD/AVGCLD

FIG. A7.9 Varnish Ratings of Piston Skirts

SEQUENCE VE CORRECTED BLOWBY FLOWRATE PLOT

Formulation/Stand Code:

FORM

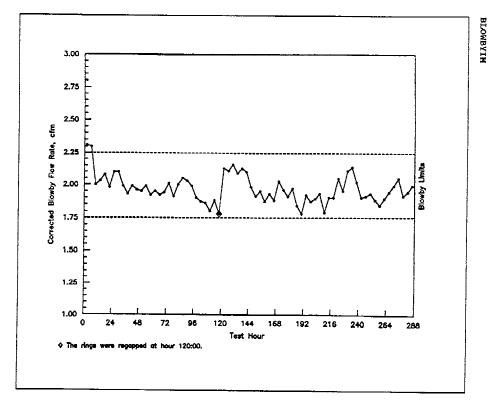


FIG. A7.10 Corrected Blowby Plot



OIL ADDITION RECORD

Laboratory: LAB	Oil Code:	CMIR/OILCODE
Test Number: STAND - RETRUN/STRUN - RTOTERUN/TOTETRUN	Date Completed:	RDTCOMP/DTCOMP
Formulation/Stand Code: FORM		

Cycle	Test Hour	Oil Consumption fl. oz.	Oil Level fl. oz.
6	23 h 35 min.	OILCH006	OILLH006
12	47 h 35 min.	OILCH012	OILLH012
18	71 h 35 min.	OILCH018	OILLH018
24	95 h 35 min.	OILCH024	OILLH024
30	119 h 35 min.	OILCH030	OILLH030
36	143 h 35 min.	OILCH036	OILLH036
42	167 h 35 min.	OILCH042	OILLH042
48	191 h 35 min.	OILCH048	OILLH048
54	215 h 35 min.	OILCH054	OILLH054
60	239 h 35 min.	OILCH060	OILLH060
66	263 h 35 min.	OILCH066	OILLH066
72	287 h 35 min.	OILCH072	OILLH072
otal Oil Con	sumption, fl. oz.:	ROILC	ON/OILCON

FIG. A7.11 Oil Addition Record



ORIGIN OF SIGNIFICANT ENGINE PARTS

Laboratory: L	AB	Oil Code:	CMIR/OILCODE
Test Number: STAND -	RSTRUN/STRUN - RTOTSRUN/TOTSTRUN	Date Completed:	RDTCOMP/DTCOMP
Formulation/Stand Cod	de: FORM		

ORIGIN OF SIGNIFICANT ENGINE PARTS

ENGINE PART	ORIGIN		
Camshaft	CAMOGN		
Connecting Rod Bearings	CRODBOGN		
Cylinder Block	CYLBKOGN		
Cylinder Head	CYLHDOGN		
Pistons	PISTOGN		
Piston Rings	PISRGOGN		
Rocker Arms	RCKRAOGN		

FIG. A7.12 Origin of Significant Engine Parts



CHARACTERISTICS OF THE DATA ACQUISITION SYSTEM

· · · · · ·				
Leboratory:	LAB	Test Stand:	STAND	

PARAMETER (1)	SENSING DEVICE (2)	CALIBRATION FREQUENCY (3)	RECORD DEVICE (4)	OBSERVATION FREQUENCY (5)	RECORD FREQUENCY (6)	LOG FREQUENCY (7)	SYSTEM RESPONSE (8)
Secondary							_
Oil Inlet	OILISENS	OILICALF	OILIRECD	OILIOBSF	OILIRECF	OILILOGF	
RAC Inlet	RCCISENS	RCCICALF	RCCIRECD	RCCIOBSF	RCCIRECF	RCCILOGF	
Coolant Outlet	COTSENS	COTCALF	COTRECD	COTOBSF	COTRECF	COTLOGF	
Criter							
Speed	RPMSENS	RPMCALF	RPMRECD	RPMOBSF	RPMRECF	RPMLOGF	RPMSYSR
Power	PWRSENS	PWRCALF	PWRRECD	PWROBSF	PWRRECF	PWRLOGF	PWRSYSR
CO	COSENS	COCALF	CORECD	COOBSF	CORECF	COLOGF	COSYSR
0,2	O2SENS	O2CALF	02RECD	020BSF	O2RECF	O2LOGF	02SYSR
Timing	TIMGSENS	TIMGCALF	TIMGRECD	TIMGOBSF	TIMGRECF	TIMGLOGF	
Blowby	BLWGSENS	BLWGCALF	BLWGRECD	BLWGOBSF	BLWGRECF	BLWGLOGF	
Exhaust Backpressure X	EXPRSENS	EXPRCALF	EXPRRECD	EXPROBSE	EXPRRECF	EXPRLOGF	
Engine Coolant Flow	CFLWSENS	CFLWCALF	CFLWRECD	CFLWOBSF	CFLWRECF	CFLWLOGF	
Rocker Arm Cover	RCCFSENS	RCCFCALF	RCCFRECD	RCCFOBSF	RCCFRECF	RCCFLOGF	

LEGEND:

- (1) OPERATING PARAMETER
- (2) THE TYPE OF DEVICE USED TO MEASURE TEMPERATURE, PRESSURE OR FLOW
- (3) FREQUENCY AT WHICH THE MEASUREMENT SYSTEM IS CALIBRATED
- (4) THE TYPE OF DEVICE WHERE DATA IS RECORDED
 - LG HANDLOG SHEET
 - DL AUTOMATIC DATA LOGGER
 - SC STRIP CHART RECORDER
 - C/M COMPUTER, USING MANUAL DATA ENTRY
 - C/D COMPUTER, USING DIRECT I/O ENTRY
- (5) DATA ARE OBSERVED BUT ONLY RECORDED IF OFF SPEC.
- (6) DATA ARE RECORDED BUT ARE NOT RETAINED AT EOT
- (7) DATA ARE LOGGED AS PERMANENT RECORD, NOTE SPECIFY IF:
 - SS SNAPSHOT TAKEN AT SPECIFIED FREQUENCY

AG/X AVERAGE OF X DATA POINTS AT SPECIFIED FREQUENCY

(8) TIME FOR THE OUTPUT TO REACH 63.2% OF FINAL VALUE FOR STEP CHANGE AT INPUT

FIG. A7.13 Characteristics of the Data Acquisition System



Rocker Cover and Camshaft Baffle Photographs

Laboratory:	LAB	Oil Code: CMIR/OILCODE
Test Number:	STAND - RSTRUN/STRUN - RTOTSRUN/TOTSTRUN	Date Completed: RDTCOMP/DTCOMP
Formulation/Sta	Ind Code: FORM	

RC_CSBIM



FIG. A7.14 Rocker Arm Cover and Camshaft Baffle Photographs



Oil Pan and Oil Screen Photographs

Laboratory:	ratory: LAB		Oil Code: C			
Test Number:	STAND - RSTRUN	/STRUN *	RTOTSRUN/TOTSTRUN	Date Completed:	RDTCOMP/DTCOMP	
Formulation/St	and Code:	FORM				

OP_OSIM



FIG. A7.15 Oil Pan and Oil Screen Photographs



Piston Skirt Photographs

Laboratory: LAB	Oil Code:	CMIR/OILCODE
Test Number: stand - RSTRUN/STRUN - RTOTSRUN/TOTSTRUN	Date Completed:	RDTCOMP/DTCOMP
Formulation/Stand Code: FORM		

PISTONIM



FIG. A7.16 Piston Skirt Photographs



Camshaft Lobe Photographs

Laboratory:	LAB			Oil Code:	CMIR/OILCODE	
Test Number:	STAND .	RSTRUN/STRUN	* RTOTSRUN/TOTSTRUN	Date Completed:	RDTCOMP/DTCOMP	
Formulation/S	tand Co	de: FORM				

CAMLOBIM



FIG. A7.17 Camshaft Lobe Photographs



Front Seal Housing and Rocker Arm Photographs

Laboratory:	LAB			Oil Code:	CMIR/OILCODE	
Test Number:	STAND .	RSTRUN/STRUN	* ATOTSRUN/TOTSTRUN	Date Completed:	RDTCOMP/DTCOMP	
Formulation/S	tand Co	de: FORM				

FSH_RAIM



FIG. A7.18 Front Seal Housing and Rocker Arm Photographs



Sequence VE Sample Transition Plot

Laboratory:	LAB		Oil Code: CMIR/OILCODE			
Test Number:	STAND - RSTRUN/STRUN	* RTOTSRUN/TOTSTRUN	Date Completed: RDTCOMP/DTCOMP			
Formulation/Stand Code: FORM						

TRANSIIM

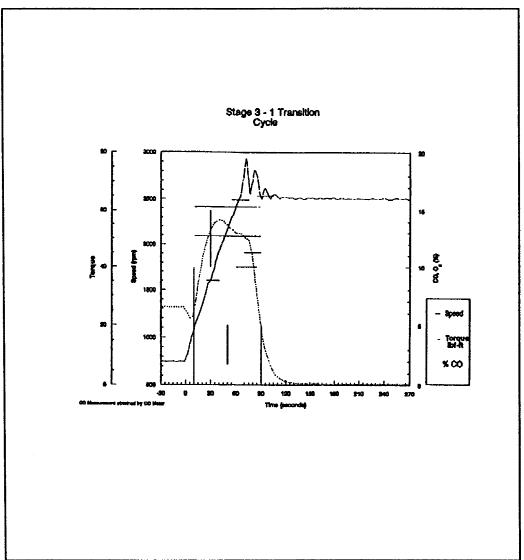


FIG. A7.19 Sequence VE Sample Transition Plot



Ring Adjustment Summary

Laboratory: LA	.8	Oil Code:	CMIR/OILCODE
Test Number: STAND *	RSTRUN/STRUN - RTOTSRUN/TOTSTRUN	Date Completed:	RDTCOMP/DTCOMP
Formulation/Stand Code:	FORM		

Compression Ring Gap Measurements

Cylinder Number	Top Ring Gap, in.	Second Ring Gap, In.	Top Ring-Land Side Clearance, in.	Second Ring-Land Side Clearance, in.
1	TPRNGGP1	N2RNGGP1	TPRLCLR1	N2RLCLR1
2	TPRNGGP2	N2RNGGP2	TPRLCLR2	N2RLCLR2
3	TPRNGGP3	N2RNGGP3	TPRLCLR3	N2RLCLR3
4	TPRNGGP4	N2RNGGP4	TPRLCLR4	N2RLCLR4

Compression Ring Regap History and Wear Data

Cylinder Number	Ring Location	Ring Gap O GAPHRAhrs, in.	Ring Gap After Rework, in.	Ring Gap @gapung, hrs, in.	Ring Gap After Rework, in.	Ring Gap Increase Due to Wear, in.
1	Тор	TPRGGP1A	TPRGRW1A	TPRGGP1B	TPRGRW1B	TPRGINC1
2	Тор	TPRGGP2A	TPRGRW2A	TPRGGP2B	TPRGRW28	TPRGINC2
3	Тор	TPRGGP3A	TPRGRW3A	TPRGGP38	TPRGRW3B	TPRGINC3
4	Тор	TPRGGP4A	TPRGRW4A	TPRGGP4B	TPRGRW4B	TPRGINC4

BORE WEAR

POSITION	· · · · · · · · · · · · · · · · ·	Wear (mils)	Cylinder Average
Cylinder 1	lop	BWCYL1T	
	Middle Bottom	BWCYLIM	BWCYL1A
	Bottom	BWCYL1B	
Cylinder 2	lop	BWCYL2T	
	Middle	BWCYL2M	BWCYL2A
I	Bottom	BWCYL2B	
Cylinder 3 Top	Тор	BWCYL3T	
	Middle	BWCYL3M	BWCYL3A
	Bottom	BWCYL3B	
Cylinder 4	Тор	BWCYL4T	
	Middle	BWCYL4M	BWCYL4A
	Bottom	BWCYL4B	
Average Bore	Wéar		CYLBWAVG
Maximum Bore Weat		CYLBWMAX	

FIG. A7.20 Ring Adjustment Summary

SEQUENCE VE

SEQUENCE VE DEVIATION PERCENTAGE SUMMARY

Laboratory:	LAB		Oil Code: CMIR/OILCODE
Test Number:	STAND - RSTRUN/STRUN	- RTOTSRUN/TOTSTRUN	Date Completed: RDTCOMP/DTCOMP
Formulation/Stand Code:		FORM	

Primary Parameter	Maximum Permitted Deviation Percentage	Calculated Deviation Percentage
Engine Oil Inlet Temperture, °F	2.5%	OILINDP
Engine Coolant Outlet Temperature, °F	2.5%	COLOUTDP
Timing, BTDC	2.5%	TIMNGDP
Exhaust Gas Oxygen (Stages 1 and 2), %	2.5%	EXO212DP
Exhaust Gas CO (Stage 3), %	2.5%	EXCO3DP
Engine Coolant Flow, gal/min	2.5%	COLFLODP
Rocker Arm Cover Inlet Temperature, °F	2.5%	RACINDP
Secondary Parameters		
Engine Speed, r/min	5%	RPMDP
Power, bhp	5%	PWRDP
Rocker Arm Cover Coolant Flow Rate, gal/min	5%	RACCOLDP
Exhaust Backpressure (Stages 1 & 2), in. Hg abs	5%	EXBK12DP

FIG. A7.21 Deviation Percentage Summary



Mid Limit Operation Specification Summary

Laboratory:	LAB		Oil Code:	CMIR/OILCODE
Test Number:	STAND - RSTRUN/STRUN -	RTOTSRUN/TOTSTRUN	Date Completed:	RDTCOMP/DTCOMP
Formulation/Stand	d Code: FORM			

Primary Parameter	Mid Limit Operation Specification Range	Test Average Value
Engine Oil Inlet Temperature, Stage 1 (°F)	155.0 ± 0.5	AENGOIN1
Engine Oil Inlet Temperature, Stage 2 (°F)	210.0 ± 0.5	AENGOIN2
Engine Oil Inlet Temperature, Stage 3 (°F)	115.0 ± 0.5	AENGOIN3
Coolant Outlet Temperature, Stage 1 (°F)	125.0 ± 0.5	ACOLOUT1
Coolant Outlet Temperature, Stage 2 (°F)	185.0 ± 0.5	ACOLOUT2
Coolant Outlet Temperature, Stage 3 (°F)	115.0 ± 0.5	ACOLOUT3
Secondary Parameters		
Engine Speed, Stage 1 (r/min)	2500 ± 5	ARPM1
Engine Speed, Stage 2 (r/min)	2500 ± 5	ARPM2
Engine Speed, Stage 3 (r/min)	750.0 ± 12.5	ARPM3
Power, Stage 1 (bhp)	33.50 ± 0.25	APWR1
Power, Stage 2 (bhp)	33.50 ± 0.25	APWR2
Power, Stage 3 (bhp)	1.00 ± 0.25	APWR3
Rocker Arm Coolant Flow Rate, Stage 1 (gal/min)	2.0 ± 0.1	ARACCFR1
Rocker Arm Coolant Flow Rate, Stage 3 (gal/min)	2.0 ± 0.1	ARACCFR3

FIG. A7.22 Mid Limit Operation Specification Summary

A8. SAFETY PRECAUTIONS

A8.1 General Information

A8.1.1 The operation of this procedure *can* expose personnel to hazardous materials, operations, and equipment. Personnel who are involved in the design, installation, and operation should be thoroughly trained and experienced. Personnel should be provided with safety glasses, hearing protection, and proper tools. All loose clothing should be removed or secured.

A8.1.2 The laboratory facilities should be inspected and approved by the laboratory's safety department. All laboratory areas should be kept clean and free of oil and fuel spills. The laboratory should also be kept free of tripping hazards. Containers of fuel and oil should not be allowed to accumulate excessively. A fixed fire protection system and adequate fire extinguishers should be available in all parts of the laboratory. Emergency showers should be provided throughout the laboratory. A8.1.3 The test stands should be equipped with a fuel shut-off valve that is designed to automatically interrupt the fuel supply when the engine is not running. The engine should also be automatically shutdown if any of the following events occur: dynamometer loses field current, engine overspeeds, exhaust system fails, room ventilation fails, or the fire protection system activates. Guards should be installed around all external rotating parts and hot surfaces. All fuel lines, oil lines, steam lines, process water lines, and electrical wiring should be properly routed, protected, and kept in good working order.

A8.1.4 The Sequence VE test exposes personnel to physical hazards and various hazardous chemicals to prepare parts for the test. These chemicals and a summary of specific precautions concerning each chemical are listed as follows. Emergency showers and eye-rinse facilities should be provided in parts preparation areas.

A8.2 Physical Hazards

A8.2.1 Electrical shock,

A8.2.2 High-speed rotating equipment,

A8.2.3 High-temperature surfaces, and

A8.2.4 Noise.

A8.3 Hazardous Chemicals and Materials

A8.3.1 Aliphatic Naphtha (Stoddard Solvent):

A8.3.1.1 Before opening the container, relieve pressure. Keep the container tightly closed when not in use.

A8.3.1.2 Store at moderate temperatures and keep away from heat, sparks, open flame, oxidizing agents, acids, and bases.

A8.3.1.3 Use dry chemical, foam, or $\rm CO_2$ as extinguishing media.

A8.3.1.4 In case of spillage, cover with absorbent material and sweep up. Alternatively, flush with water into a retaining area or container.

A8.3.1.5 Use safety glasses and impervious gloves when handling.

A8.3.1.6 Use cartridge or air-line respirators in enclosed areas.

A8.3.1.7 Use only if adequate ventilation is available.

A8.3.1.8 Avoid contact with eyes, skin, and clothing.

A8.3.1.9 Flush eyes with water for 15 min after contact. Wash skin thoroughly with soap and water.

A8.3.2 Cooling System Cleanser, No. 7 (DuPont Formulation):

A8.3.2.1 Store at moderate temperatures. Keep container closed until use.

A8.3.2.2 Use water spray, dry chemical, foam, or CO $_2$ as extinguishing media.

A8.3.2.3 In case of spillage, sweep up. Prevent entry into natural bodies of water.

A8.3.2.4 Use safety glasses and impervious gloves when handling.

A8.3.2.5 Use respiratory protection in absence of proper environmental control.

A8.3.2.6 Use only if adequate ventilation is available.

A8.3.2.7 Avoid contact with eyes, skin, and clothing.

A8.3.2.8 Flush eyes with water for 15 min after contact. Wash skin thoroughly with soap and water.

A8.3.3 *Ethyl Acetate*:

A8.3.3.1 Before opening the container, relieve pressure. Keep the container tightly closed when not in use.

A8.3.3.2 Store at moderate temperatures and keep away from heat, sparks, open flame, oxidizing agents, acids, and bases.

A8.3.3.3 Use dry chemical, foam, or CO_2 as extinguishing media.

A8.3.3.4 In case of spillage, cover with absorbent material and sweep up. Alternatively, flush with water into a retaining area or container.

A8.3.3.5 Use safety glasses and impervious gloves when handling.

A8.3.3.6 Use cartridge or air-line respirators in enclosed areas.

A8.3.3.7 Use only if adequate ventilation is available.

A8.3.3.8 Avoid contact with eyes, skin, and clothing.

A8.3.3.9 Flush eyes with water for 15 min after contact. Wash skin thoroughly with soap and water.

A8.3.4 Unleaded Gasoline (Phillips J Fuel):

A8.3.4.1 Before opening the container, relieve pressure. Keep the container tightly closed when not in use.

A8.3.4.2 Store at moderate temperatures and keep away from heat, sparks, open flame, and oxidizing agents.

A8.3.4.3 Use dry chemical, foam, or $\rm CO_2$ as extinguishing media.

A8.3.4.4 In case of spillage, cover with absorbent material and sweep up. Alternatively, flush with water into a retaining area or container.

A8.3.4.5 Use safety glasses and impervious gloves when handling.

A8.3.4.6 Use respiratory hydrocarbon vapor canister in enclosed areas.

A8.3.4.7 Use only if adequate ventilation is available.

A8.3.4.8 Avoid contact with eyes, skin, and clothing.

A8.3.4.9 Flush eyes with water for 15 min after contact. Wash skin thoroughly with soap and water.

A8.3.5 New and Used Oil Samples:

A8.3.5.1 Store at moderate temperatures and keep away from extreme heat, sparks, open flame, and oxidizing agents.

(a) Use dry chemical, foam, or CO_2 as extinguishing media.

(b) In case of spillage, cover with absorbent material and sweep up.

(c) Use safety glasses and impervious gloves when handling.

(d) Avoid contact with eyes, skin, and clothing.

(*e*) Flush eyes with water for 15 min after contact. Wash skin thoroughly with soap and water.

A8.3.5.2 Used Oil Samples Only—Since used oils contain compounds that were not originally present in the new oil, follow the most stringent Material Safety Data Sheets guidelines for all components present. (Warning—In addition to other precautions, note that continuous contact with used motor oils has caused skin cancer in laboratory mice.)

A8.3.6 Organic Solvent (Oakite 811):

A8.3.6.1 Before opening the container, relieve pressure. Keep the container tightly closed when not in use.

A8.3.6.2 Store at moderate temperatures and keep away from direct sunlight, heat, sparks, open flame, and oxidizing agents.

A8.3.6.3 Use dry chemical, foam, or CO_2 as extinguishing media.

A8.3.6.4 In case of spillage, cover with absorbent material, sweep up, and haul away. Alternatively, flush with water into a retaining area or container.

A8.3.6.5 Use safety glasses and impervious gloves when handling.

A8.3.6.6 Use respiratory hydrocarbon vapor canister in enclosed areas.

A8.3.6.7 Use only if adequate ventilation is available.

A8.3.6.8 Avoid contact with eyes, skin, and clothing.

A8.3.6.9 Flush eyes with water for 15 min after contact. Wash skin thoroughly with soap and water.

A8.3.7 Organic Solvent (Penetone ECS):

A8.3.7.1 Before opening the container, relieve pressure. Keep the container tightly closed when not in use.

A8.3.7.2 Store at moderate temperatures and keep away from heat, sparks, open flame, and strong oxidizing agents.

A8.3.7.3 Use dry chemical, foam, or CO_2 as extinguishing media.

A8.3.7.4 In case of spillage, cover with absorbent material, and sweep up. Dispose in accordance with RCRA procedures.

A8.3.7.5 Use safety glasses and impervious gloves when handling.

A8.3.7.6 Use respiratory hydrocarbon vapor canister in enclosed areas.

A8.3.7.7 Use only if adequate ventilation is available.

A8.3.7.8 Avoid contact with eyes, skin, and clothing.

A8.3.7.9 Flush eyes with water for 15 min after contact. Wash skin thoroughly with soap and water.

A8.3.8 n-Pentane:

A8.3.8.1 Before opening the container, relieve pressure. Keep the container tightly closed when not in use.

A8.3.8.2 Store at moderate temperatures and keep away from heat, sparks, and open flame.

A8.3.8.3 Use dry chemical, foam, or CO_2 as extinguishing media.

A8.3.8.4 In case of spillage, cover with absorbent material, sweep up, and haul away.

A8.3.8.5 Use safety glasses and impervious gloves when handling.

A8.3.8.6 Use respiratory hydrocarbon vapor canister in enclosed areas.

A8.3.8.7 Use only if adequate ventilation is available.

A8.3.8.8 Avoid contact with eyes, skin, and clothing.

A8.3.8.9 Flush eyes with water for 15 min after contact. Wash skin thoroughly with soap and water.

A9. AUTOMATIC DATA ACQUISITION

A9.1 Three levels of performance for data acquisition systems have been outline by the Data Acquisition Task Force. Guidelines for the three systems are outlined as follows. Additional information concerning data acquisition systems can be found in 2.6.

A9.2 *Manual*—Manual systems may utilize hand logging of operating data or automatic recording of one or more measured parameters. These systems record data only at the intervals prescribed by the procedure. Reports generated from manual systems should be formatted as prescribed by the procedure.

A9.3 *Enhanced*—Some or all of the data is recorded at a higher frequency than is prescribed by the procedure. Enhanced systems usually utilize automatic data acquisition and

utilize some technique to reduce the volume of data reported, that is, averaging. Reports generated from enhanced systems should be formatted as prescribed in the procedure. However, a statement is added that defines data that is enhanced and the method used for enhancement.

A9.4 *Automated*—Alarms for parameter excursions must be at a frequency compatible with process control requirements. The frequency of data recording must be compatible with sound engineering and statistical practice for trend analysis and review of operation. The reporting capability must be consistent with data user requirements, including reporting out-of-limit data. The system must be capable of accepting manual entry of data not available from full-time sensors in a data base that is common with the other data.

A10. OAKITE 811 MONITORING PROGRAM

A10.1 If the laboratory utilizes Oakite 811, the level of butyl cellosolve must be monitored on a weekly basis. The Oakite 811 should be replaced when the butyl cellosolve content falls below 5 % by volume.

A10.2 Two test methods have been found to be effective to determine the level of butyl cellosolve: determination by capillary column gas chromatography (GC) and determination by GC. The two test methods are detailed as follows:

A10.3 Determination of Butyl Cellosolve Content by Capillary Column GC:

A10.3.1 *Scope*—This test method utilizes a fused capillary column GC method.

A10.3.2 Summary of Test Method—Samples of Oakite 811 are injected using the split injection mode on a crosslinked, 5 % phenylmethyl silicone-fused silica column. An internal

standard is added to each sample prior to injection. Percent volume butyl cellosolve is determined by standard addition.

A10.3.3 *Significance*— If the butyl cellosolve level of Oakite 811 falls below 5 % by volume, a gel-like substance forms on parts when they are rinsed with water. The cleaning solution should be replaced when this occurs.

A10.3.4 Apparatus:

A10.3.4.1 *Gas Chromatograph*—The gas chromatograph should be equipped with a flame ionization detector (FID). The capillary must have an inlet and detector system designed for fused-silica capillary chromatography. The recorder must have a range from 0 to 1 mV and a response time of 1 s or less.

A10.3.4.2 *Data Analysis System*—The data analysis system must provide a means to perform peak integration.

A10.3.4.3 *Column*—The column must be a crosslinked, 5 % phenylmethyl silicone-fused silica column. The dimensions of the column are 25 m by 0.32 mm.

A10.3.4.4 *Syringe*—All syringes must be a 10-L syringe graduated in 0.1-L increments.

A10.3.5 Conditions of Analysis:

A10.3.5.1 *Temperatures*— Sample inlet system is 250°C; detector is 320°C; initial column temperature is 40°C; and final column temperature is 300°C.

A10.3.5.2 Temperature Change Rate, 4°C/min for 15 min; 14°C/min after 15 min.

A10.3.5.3 *Total Time for Temperature Change*, 20 min (variable).

A10.3.5.4 *Electrometer Attenuation*, 32/range 10^{-11} .

A10.3.5.5 Column Head Pressure, 5 psig (34 kPa).

A10.3.5.6 Injection Mode, split.

A10.3.5.7 Inlet Split Vent Flow Rate, 480 mL/min.

A10.3.5.8 Column Makeup Gas Flow Rate, 30 mL/min.

A10.3.5.9 Recorder Chart Speed, 1 cm/min.

A10.3.6 Reagents and Materials:

A10.3.6.1 *Gases*—Helium (carrier gas), regular grade \geq 99.995 % pure; hydrogen (detector gas), prepurified; and air (detector gas), breathing grade.

A10.3.6.2 *Solvents*—Methanol, HPLC grade; hexonal, HPLC grade; and butyl cellosolve, ≥ 99 % pure.

A10.3.7 Preparation of Standards and Samples:

A10.3.7.1 Preparation of Standards-Standards are prepared using the standard addition technique. Prepare a stock solution of 20 % butyl cellosolve by volume in Oakite 811. From the stock solution, prepare the following solutions in 10-mL flasks: (a) 5-mL stock solution + 5 mL Oakite 811 = 10 %, (b) 2-mL stock solution + 8 mL Oakite 811 = 4 %, and (c) 7-mL stock solution + 3 mL Oakite 811 = 14 %. Pipette 10 mL of the stock solution into each of the 10-mL flasks and fill the flasks to the 10-mL mark with Oakite 811. Add 1 mL of hexonal into three additional 10-mL flasks that are labeled a-I, b-I, and c-I. Hexonal is a slightly viscous liquid. When dispensing it with a pipette, a thin film of liquid can be seen trailing the slug of the solvent. This film should travel to the tip of the pipette approximately 1 min before the pipette is removed from the flask. Withdraw 9 mL of the standard solution from flasks a, b, and c, and dispense into the respective 10-mL flasks containing hexonal. Cap the flasks, mix well, and seal the flasks with parafilm until ready for use.

A10.3.7.2 *Preparation of Samples*—Add 1 mL of hexonal to a labeled 10-mL flask for each sample to be analyzed. Fill each flask to the 10-mL mark with a sample. Cap the flasks, mix well, and seal the flasks with parafilm until ready for use. Prepare all samples and standards in a 9:1 ratio:sample: internal standard or standard:internal standard.

A10.3.8 Procedure:

A10.3.8.1 Prepare the gas chromatograph for the operating conditions stated in 6.1.2.

A10.3.8.2 Sample Injection Technique—A solvent flush method is utilized. Draw methanol into the syringe to the 0.5-L mark. Raise the methanol slug to the 3-L mark. Draw the analysis sample to the 1-L mark in one continuous motion. Raise the sample slug to the 3-L mark to produce a slug of air in the syringe needle and lower barrel. Inject the sample and hold the syringe in the injection port for 5 s before removing it.

A10.3.8.3 Start the gas chromatograph and all recording devices.

A10.3.8.4 Complete the analysis in accordance with the temperature change specifications shown in A10.2.5.2 and A10.2.5.3.

A10.3.9 Calculation:

A10.3.9.1 *Calibration Data*—Determine the peak area ratio for each standard in accordance with the following equation:

$$\frac{area of butyl cellosolve in V}{area of hexonal} = W$$
(A10.1)

where:

V =Oakite 811 blend used to make up the standard blends, and

 $S = W \times \%$ volume of Oakite 811 in the standard blend.

Example—A 14 % butyl cellosolve standard solution made up in Oakite 811 would be 86 % Oakite 811; $W \times 0.86 = S$. The *actual* peak area ratio of standard is calculated using the following formula:

$$\frac{area of butyl cellosolve in V}{area of hexonal} - S$$
(A10.2)

where:

 $S = W \times \%$ volume of Oakite 811 in the standard blend.

Example—A 14 % butyl cellosolve standard solution made up in Oakite 811 would be 86 % Oakite 811; $W \times 0.86 = S$.

where:

X = total area of butyl cellosolve in the sample to be quantitated.

A10.3.9.2 *Percent Volume Butyl Cellosolve*—The data from the following calibration curve is used to determine the percent volume butyl cellosolve in the sample after the *actual* peak area ratio is calculated for the sample. The correlation coefficient of the data is 0.99998:

Actual Peak Area Ratio	% Volume
0.267	4
0.690	10
0.996	14

A10.3.10 *Comments*—The injection port liner becomes contaminated after a series of analyses and should be used prior to use in another analytical project.

A10.4 Determination of Butyl Cellosolve Content by Gas Chromatography:

A10.4.1 *Scope*—This test method utilizes gas chromatography to determine the volume percentage of butyl cellosolve in Oakite 811 cleaning compound.

A10.4.2 *Summary of Test Method*—The percent volume of butyl cellosolve in Oakite 811 is determined by the technique of standard addition using gas chromatography.

A10.4.3 *Significance and Use*—If the butyl cellosolve level of Oakite 811 falls below 5 % by volume, a gel-like substance forms on parts when they are rinsed with water. The cleaning solution should be replaced when this occurs.

A10.4.4 Apparatus:

A10.4.4.1 *Gas Chromatograph*—The gas chromatograph should be equipped with a thermal conductivity detector (the

suitability of a FID has not been determined). The recording potentiometer must have a full-scale response time of 1 s or less.

A10.4.4.2 *Data Analysis System*—The data analysis system must provide a means to perform area integration.

A10.4.4.3 *Column*—The column must be a OV-101 stationary phase with 10 % on 100/200 and Gas Chrom Q packing. The dimensions of the column are 6-ft by 0.125-in. (1.83-m by 3.18-mm) outside diameter.

A10.4.4.4 *Syringe*—All syringes must be a 10-L syringe graduated in 0.1-L increments.

A10.4.4.5 *Flow Controllers*—The gas chromatograph must be equipped with constant flow controllers.

A10.4.5 Reagents and Materials:

A10.4.5.1 Helium (Carrier Gas), zero grade.

A10.4.5.2 *Butyl Cellosolve*, purity ≥ 99 %.

A10.4.6 *Preparation of Sample*—Prepare four solutions containing 10, 15, 20, and 25 % volume butyl cellosolve in Oakite 811. Mix the solutions thoroughly for 2 min.

A10.4.7 *Procedure*:

A10.4.7.1 Set the injection port temperature to 300° C. Adjust the flow rate to 40 mL/min. Set the detector temperature to 350° C. Adjust the oven temperature to 80° C. Inject exactly 5 L of sample.

A10.4.7.2 Turn on the chart recorder and integrator immediately after sample injection. Maximize the butyl cellosolve peak area of the sample containing the highest level of butyl cellosolve. Do not adjust the sensitivity for subsequent samples.

A10.4.7.3 After the butyl cellosolve peak has eluded (approximately 5 min), rapidly raise the oven temperature to 300° C to elute the high boiling components of the sample. When the baseline returns to its initial level, lower the oven temperature to 80° C.

A10.4.7.4 Record the butyl cellosolve peak areas for all four samples.

A10.4.8 *Interpretation of Results*—Plot the area of the butyl cellosolve peaks versus the percent volume of the butyl cellosolve added to the Oakite 811. The negative of the *X*-axis intercept is the value of the volume percentage of butyl cellosolve in the Oakite 811. The negative of the *X*-axis intercept can also be determined by least squares analysis.

A10.4.9 Precision:

A10.4.9.1 *Repeatability*— The difference between successive test results obtained by the same operator with the same apparatus under constant operating conditions on identical test material should not exceed ± 0.3 % volume in one case out of twenty.

A11. TEST PRECISION—REFERENCE OILS

Note A11.1—This is for operationally valid tests through June 30, 1991.

A11.1 *Repeatability* (*r*) in Table A11.1 is the difference between successive results obtained on the same test oil by the same operators on the same test stand, and would, in the long run, in the normal and correct conduct of the test method, exceed the values shown in only one case in twenty. It should be noted that these successive tests are not run in the same engines; that is, each engine is completely rebuilt before each test, and the engine is believed to be an important variable affecting the precision of the test.

A11.2 *Reproducibility* (R) is as defined previously in 16.1.1.3.

A11.3 In Table A11.1, X is the average value of two or more tests on the same oil.

A11.4 The reference oil test operating conditions are more carefully scrutinized than those for the candidate oil tests, and a significantly higher percentage of reference oil tests are rejected as a result. This could represent a source of bias between the two precision estimates.

TABLE A11.1 Reference Oil Precision Statistics^A

Measured Units										
Variable -	Repea	tability ^{<i>B</i>}	Reproducibility ^C							
Variable –	s, D	r ^E	s _R ^D	R ^E						
Average Engine Varnish, merits	0.26	0.73	0.28	0.78						
Average Piston Varnish, merits	0.25	0.70	0.24	0.67						
Transformed Units										
Variable	Repea	tability ^B	Reproducibility ^C							
variable	s, ^D	r E	s _R ^D	R ^E						
Average Engine Sludge, – In (9.65- merits)	0.48	1.34	0.48	1.34						
Rocker Cover Sludge, – In (9.65-merits)	0.50	1.40	0.52	1.46						
Average Cam Wear, $\sqrt{(mils)}$	0.56	1.57	0.56	1.57						
Maximum Cam Wear, $\sqrt{(mils)}$	0.83	2.32	0.83	2.32						

^A These statistics are based on results obtained on Test Monitoring Center Reference Oils 926-1, 925-2, 927-1, 925, and 1002 over the period from April 1992 through September 1994.

^B Repeatability values refer to tests run on the same oil in the same laboratory. ^C Reproducibility values refer to tests run on the same oil in different laboratories.

 ^{D}s = Standard deviation.

^E On the basis of test error alone, the difference, in absolute value, between two test results will be expected to exceed this value about 5 % of the time. This value is obtained by multiplying the standard deviation by 2.8.

A12. CONTROL CHART TECHNIQUE FOR SEVERITY ADJUSTMENT (SA)

A12.1 Average Engine Sludge (AES) SA—An EWMA technique is applied to standardized calibration test AES results. Results are converted to transformed units by the formula $-1 \times (\ln (9.65 - \text{AES}))$. Transformed values are standardized using delta/s (Δ /s) ((result – target)/standard deviation). The targets and standard deviations for current reference oils are published by the ASTM TMC.

A12.1.1 Include all operationally valid reference tests in a laboratory control chart. Chart tests in order of completion date and time. A minimum of two tests is required to initialize a control chart. Calculate EWMA values using Eq A12.1.

$$Z_i = 0.2 (Y_i) + 0.8 (Z_{i-1})$$
(A12.1)

where:

 $Z_0 = 0$ and Y_i = standardized test result, and

 Z_i = EWMA of the standardized test result at test order *i*.

If the absolute value of the EWMA, rounded to three places after the decimal, exceeds 0.653, then apply an SA to subsequent non-reference results. The following example illustrates the application of Eq A12.1 for determining the application of AES SA.

$$Z_1 = 0.694$$
 and $Y_2 = 1.247$ (A12.2)
EWMA = 0.2 (1.247) + 0.8 (0.694) = 0.805

A12.1.2 Since |0.805| > 0.653, an SA must be applied. Multiply 0.805 by the AES standard deviation for oil 926-1. Multiply this value by -1 and round to three places after the decimal. Record this value on the test results summary of the test report in the space for AES applied SA. Add this value to the transformed non-reference result. Multiply this sum by -1. Find the antilog. Subtract the antilog from 9.65 and round to two places after the decimal. Enter this number on the test results summary of the test report in the space for the AES Final Result. An SA will remain in effect until the next reference test. At that time, calculate a new EWMA.

A12.2 Rocker Arm Cover Sludge (RACS) SA—An EWMA technique is applied to standardized calibration test RACS results. Results are converted to transformed units by the formula – $1 \times (\ln (9.65 - RACS))$. Transformed values are standardized using delta/s (Δ /s) ((result – target)/standard deviation). The targets and standard deviations for current reference oils are published by the ASTM TMC.

A12.2.1 Include all operationally valid reference tests in a laboratory control chart. Chart tests in order of completion date and time. A minimum of two tests is required to initialize a control chart. Calculate EWMA values using Eq A12.1. If the absolute value of the EWMA, rounded to three places after the decimal, exceeds 0.653, then apply an SA to subsequent non-reference results. The following example illustrates the use of Eq A12.1 for determining the application of a RACS SA.

$$Z_1 = 0.570 \text{ and } Y_2 = 1.195$$
 (A12.3)

EWMA = 0.2 (1.195) + 0.8 (0.570) = 0.695

A12.2.2 Since |0.695| > 0.653, an SA must be applied. Multiply 0.695 by the RACS standard deviation for oil 926-1. Multiply this number by -1 and round to three places after the decimal. Record this value on the test results summary page of the test report in the space for RACS applied SA. Add this value to the transformed non-reference result. Multiply this sum by -1. Find the antilog. Subtract the antilog from 9.65 and round to two places after the decimal. Enter this number in the RACS final result space on the test results summary page of the test report. An SA will remain in effect until the next reference test. At that time, calculate a new EWMA.

A12.3 Average Engine Varnish (AEV) SA—An EWMA technique is applied to standardized calibration test AEV results. For reference tests starting after January 15, 1992, and before May 1, 1993, adjust each average engine varnish result by adding 0.18 to the AEV rating. Enter the corrected rating in the corrected results space in the final test report for tests completed after May 1, 1993. Values are standardized using delta/s (Δ /s) ((result – target)/standard deviation). The targets and standard deviations for current reference oils are published by the ASTM TMC.

A12.3.1 Include all operationally valid reference tests in a laboratory control chart. Chart tests in order of completion date and time. A minimum of two tests is required to initialize a control chart. Calculate EWMA values using A12.1. If the absolute value of the EWMA, rounded to three places after the decimal, exceeds 0.653, then apply an SA to subsequent non-reference results. The following example illustrates the use of Eq A12.1 for determining the application of an AEV SA.

$$Z_1 = -0.572$$
 and $Y_2 = 1.469$ (A12.4)

$$EWMA = 0.2 (-1.469) + 0.8 (-0.572) = -0.751$$

A12.3.2 Since |-0.751| > 0.653, an SA must be applied. Multiply -0.751 by the AEV standard deviation for oil 926-1. Multiply this value by -1 and round to three places after the decimal. Record this value on the test results summary of the test report in the space for AEV applied SA. Add this value to the non-reference result. Enter this number in the AEV final result space on the test results summary page of the test report. An SA will remain in effect until the next reference test. At that time, calculate a new EWMA.

A12.4 Piston Skirt Varnish (PSV) SA—An EWMA technique is applied to standardized calibration test PSV results. Values are standardized using delta/s (Δ /s) ((result – target)/ standard deviation). The targets and standard deviations for current reference oils are published by the ASTM TMC.

A12.4.1 Include all operationally valid reference tests in a laboratory control chart. Chart tests in order of completion date and time. A minimum of two tests is required to initialize a control chart. Calculate EWMA values using Eq A12.1. If the absolute value of the EWMA, rounded to three places after the decimal, exceeds 0.653, then apply an SA to subsequent non-reference results. The following example illustrates the use of Eq A12.1 for determining the application of a PSV SA.

$$Z_1 = 0.667$$
 and $Y_2 = -1.062$ (A12.5)
EWMA = 0.2 (1.062) + 0.8 (0.667) = 0.746

A12.4.2 Since |0.746| > 0.653, an SA must be applied. Multiply 0.746 by the PSV standard deviation for oil 926-1. Multiply this value by -1 and round to three places after the decimal. Record this value on the test results summary of the test report in the space for PSV applied SA. Add this value to the non-reference result. Enter this number in the PSV final result space on the test results summary page of the test report. An SA will remain in effect until the next reference test. At that time, calculate a new EWMA.

A12.5 Average Cam Wear (ACW) SA—Apply an EWMA technique to standardized calibration test ACW results. Convert tests completed prior to March 10, 1997, to micrometres by multiplying the result by 25.4. Convert results to transformed units by taking the square root of the ACW result. Add 1.451 to transformed reference oil test results completed after August 31, 1995, and before May 16, 1996. Standardize values using delta/s (Δ /s) ((result – target)/standard deviation). The targets and standard deviations for current reference oils are published by the ASTM TMC.

A12.5.1 Include all operationally valid reference tests in a laboratory control chart. Chart tests in order of completion date and time. A minimum of two tests is required to initialize a control chart. Calculate EWMA values using Eq A12.1. If the absolute value of the EWMA, rounded to three places after the decimal, exceeds 0.653, then apply an SA to subsequent non-reference results. The following example illustrates the use of Eq A12.1 for determining the application of an ACW SA.

$$Z_1 = -0.541$$
 and $Y_2 = -1.197$ (A12.6)
EWMA = 0.2 (-1.197) + 0.8 (-0.541) = -0.672

A12.5.2 Since |-0.672| > 0.653, an SA must be applied. Multiply -0.672 by the ACW standard deviation for oil 926-1. Multiply this value by -1 and round to three places after the decimal. Record this value on the test results summary of the test report in the space for ACW applied SA. Add this value to the transformed non-reference result. If the adjusted transformed non-reference result is less than zero, enter zero. Otherwise, square the result and round to two places after the decimal. Enter this number in the ACW final result space on the test results summary page of the test report. An SA will remain in effect until the next reference test. At that time, calculate a new EWMA.

A12.6 Maximum Cam Wear (MCW) SA—Apply an EWMA technique to standardized calibration test MCW results. Convert tests completed prior to March 10, 1997, to micrometres by multiplying the result by 25.4. Convert results to transformed units by taking the square root of the MCW result. Add 1.693 to transformed reference oil test results completed after August 31, 1995, and before May 16, 1996. Standardize transformed values using delta/s (Δ /s) ((result – target)/ standard deviation). The targets and standard deviations for current reference oils are published by the ASTM TMC.

A12.6.1 Include all operationally valid reference tests in a laboratory control chart. Chart tests in order of completion date and time. A minimum of two tests is required to initialize a control chart. Calculate EWMA values using Eq A12.7.

$$Z_i = 0.2 (Y_i) + 0.8 (Z_{i-1})$$
(A12.7)

where:

 $Z_0 = 0$

 Y_i = standardized test result, and

 Z_i = EWMA of the standardized test result at test order *i*. If the absolute value of the EWMA, rounded to three places after the decimal, exceeds 0.653, then apply an SA to subsequent non-reference results. The following example illustrates this process:

$$Z_1 = 0.634$$
 and $Y_2 = 1.369$ (A12.8)
EWMA = 0.2 (1.369) + 0.8 (0.634) = 0.781

A12.6.2 Since |-0.781| > 0.653, an SA must be applied. Multiply 0.781 by the MCW standard deviation for oil 926-1. Multiply this value by -1 and round to three places after the decimal. Record this value on the Test Results Summary of the test report in the space for MCW Applied SA. Add this value to the transformed non-reference result. Square result and round to one place after the decimal. Enter this number in the MCW Final Result space on the Test Results Summary page of the test report. An SA will remain in effect until the next reference test. At that time, calculate a new EWMA.

A13. STATISTICAL EQUATIONS FOR MEAN AND STANDARD DEVIATION

A13.1 See Eq A13.1 and Eq A13.2 for mean and standard deviation.

mean =
$$\frac{1}{n} \sum_{i=1}^{n} [Yi (standard) - Zi (reading)]$$
 (A13.1)

standard deviation =
$$\sqrt{\frac{\sum_{i=1}^{n} \left[(Yi - Zi) - mean \right]^2}{df}}$$
 (A13.2)

where:

n = total number of data pairs, and df = degrees of freedom = n-1.

A14. DATA DICTIONARY

A14.1 Figs. A14.1-A14.13 is the Data Dictionary, Figs. A14.14 and A14.15 is the Repeating Field Specifications, and Fig. A14.16 is the Header Data Dictionary.

2-sep-1997

Data Dictionary Test Field Field Decimal Data Sequence Form Area Length Size Type Units/Format Name Description 10 1 VERSION C YYYYMMDD **VE VERSION 19970902** VE 8 0 20 1 YÉ **TSTSPON1** 40 Ô C TEST PURCHASER, FIRST LINE 30 TSTSPON2 TEST PURCHASER, SECOND LINE 1 VE 40 ۵ C 40 1 VE LABVAL ID 1 D C V, I OR N TEST LAB VALIDATION (V, I OR N) 50 1 VĖ STAND 5 ٥ Ĉ STAND 60 1 VE RSTRUN 4 Ō Ĉ REFERENCE STAND RUN 70 1 VE CTRUN 4 n C RUNS BETWEEN CALIBRATION TESTS 80 1 VE RTOTSRUN 5 0 C REFERENCE TOTAL RUNS ON STAND 90 1 VE TOTSTRUN 5 0 C TOTAL RUNS-TEST STAND 100 VE 1 **BOTCOMP** 8 Ô Ċ YYYYMMDD REFERENCE DATE COMPLETED (YYYYHHDD) 110 1 VE DTCOMP 8 Q Ċ YYYYMMDD DATE COMPLETED (YYYYMDD) 120 1 VE REOTTIME 5 ٠ ¢ HH:MM REFERENCE END OF TEST TIME (HH:NM) 130 1 5 VE EOTTIME Ó Ċ HH:MM TIME COMPLETED (HH:MM) 140 1 VE OILCODE 38 ٥ C NON REFERENCE OIL CODE 150 1 VE CHIR 6 Ô C CHIR 160 1 VE FORM 38 0 C FORMULATION/STAND CODE 170 1 VE ALTCODE1 10 ۵ C ALTERNATE OIL CODE 1 180 1 VE ALTCODE2 10 ٥ Ç ALTERNATE OIL CODE 2 190 1 VÉ ALTCODE3 10 Ô Ĉ ALTERNATE OIL CODE 3 OPERATIONAL VALIDITY STATEMENT (HAS/HAS NOT) 200 1 VE **OPVALID** 8 Ô Ĉ HAS/HAS NOT 210 1 VĒ SURI AR 40 ۵ Ĉ SUBMITTED BY: TESTING LABORATORY 220 1 VE SUBSIGIN 70 Ô C SUBMITTED BY: SIGNATURE IMAGE 230 1 VË SUBNAME 40 Ō Ċ SUBMITTED BY: SIGNATURE TYPED NAME 240 1 VF SUBTITLE 40 Ω C SUBMITTED BY: TITLE 250 2 2 Q C LAB CODE VE LAB 260 2 VE **SAEVISC** 7 0 C SAE VISCOSITY GRADE Ć 270 2 6 ñ THE OIL CODE VE IND 280 2 TESTLEN 3 ۵ Z TEST LENGTH (HHH) VE HHH 290 2 VE ENGINE 6 0 C NON-REFERENCE ENGINE NOMINAL PISTON OVERSIZE 300 2 NOMP I SOV 4 Ż N VĒ 2 310 2 C FUEL BATCH IDENTIFIER VE. FUELBTID Ō 320 2 8 Ô C YYYYMMDD START DATE (YYYYMMDD) VE DISTRI 330 2 5 ٥ ¢ HH:MM START TIME (HH:MM) VE STRTTIME 2 VE 12 0 C LABORATORY INTERNAL DIL CODE 340 LABOCODE 2 RACSRT 5 2 N MERITS ROCKER ARN COVER SLUDGE RATING (MERITS) 350 **VE** 360 2 VE TRANRACS 6 3 N TRANS UNITS TRANSFORMED RACS RESULTS (TRANS UNITS) 2 3 N TRANS UNITS ROCKER ARM COVER SLUDGE CORRECTION FACTOR (TRANS UNITS) 370 VË RACCF 6 3 N MERITS ROCKER COVER SLUDGE CORRECTED RATING - TRANSFORMED (MERITS) 380 2 VE TRACSCOR 6 2 6 3 22 TRANS UNITS ROCKER ARM COVER SLUDGE -APPLIED SA (TRANS UNITS) 300 VF PACSSA N TRANS INITS TRANSFORMED RACS FINAL RESULTS (TRANS UNITS) 2 3 400 VE TRACSFNL 6 RACSFNL 5 2 M MERITS ROCKER ARM COVER SLUDGE FINAL RESULT (MERITS) 410 2 VE 420 2 CAMBFSRT 5 2 N MERITS CAMSHAFT BAFFLE SLUDGE RATING (MERITS) VE MERITS \$ 2 N VALVE DECK SLUDGE RATING (MERITS) 430 2 VE **VLVDCKRT** 2 MEDITS 440 2 VE FSLHSGRT 5 N FRONT SEAL HOUSING SLUDGE RATING (MERITS) CYLINDER BLOCK SLUDGE RATING (MERITS) 2 2 MERITS 450 2 VE CYLBLKRT 5 5 2 N MERITS OIL PAN SLUDGE RATING (MERITS) 460 2 VE OILPNSRT AVG ENGINE SLUDGE RATING (NERITS) 470 2 VE AES 5 2 M MERITS 480 2 VE TRANAES 6 3 ١ TRANS UNITS TRANSFORMED AES RESULTS (TRANS UNITS) AVG ENGINE SLUDGE CORRECTION FACTOR (TRANS UNITS) 400 2 VE AESCE 6 3 M TRANS UNITS 500 2 VË TAESCOR 6 3 M MERITS AVG ENGINE SLUDGE CORRECTED RATING - TRANSFORMED (MERITS) 510 2 VE AESSA 6 3 N TRANS UNITS AVG ENGINE SLUDGE -APPLIED SA (TRANS UNITS) TRANSFORMED AES FINAL RESULTS (TRANS UNITS) 6 3 N TRANS UNITS 2 TAESFNL 520 VE 5 2 AVG ENGINE SLUDGE FINAL RESULT (MERITS) AESFNL N MERITS 530 2 VE

FIG. A14.1 Data Dictionary

2-sep-199	•	V	PLAN	Plate	Basinal		•	Data Dictionary
	<u>Form</u>	Test <u>Area</u>	Field <u>Name</u>		Decimal <u>Size</u>		<u>Units/Format</u>	Description
5/0								
540	2	VE	PISKVRT	5	2	N	MERITS	PISTON SKIRTS VARNISH RATING (MERITS)
550	2	VE	PISKVCF	6	3		MERITS	PISTON SKIRTS VARNISH CORRECTION FACTOR (MERITS)
560 570	2	VE	PSVSA	6	3	N	MERITS	PISTON SKIRT VARNISH -APPLIED SA (MERITS)
570	2	VE	PISKVFNL	5	2	N	MERITS	PISTON SKIRTS VARHISH FINAL RESULT (MERITS)
580	2	VE	RACVRT	5	2	N	MERITS	ROCKER ARM COVER VARNISH RATING (MERITS)
590	2	VE	CAMBFVRT	5	2	N	MERITS	CAMSHAFT BAFFLE VARNISH RATING (MERITS)
600	2	VE	CYLWLVRT	5	2	Ň	MERITS	CYLINDER WALLS VARNISH RATING (MERITS)
610	2	VE	OILPNVRT	5	2		MERITS	OIL PAN VARNISH RATING (MERITS)
620	2	VE	AEV	5	2	N	MERITS	AVG ENGINE VARNISH RATING (MERITS)
630	2	VE	AEVCF	6	3		MERITS	AVG ENGINE VARNISH CORRECTION FACTOR (MERITS)
640	2	VE	AEVSA	6	3	N	MERITS	AVG ENGINE VARNISH -APPLIED SA (MERITS)
650	2	VE	AEVFNL	5	2	N.	MERITS	AVG ENGINE VARNISH FINAL RESULT (HERITS)
660	2	VE	ACW	6	1	N	MICRONETERS	AVG CANSHAFT WEAR (MICROMETERS)
670	2	VĒ	TRANACW	6	3	N	TRANS UNITS	TRANSFORMED ACH RESULTS (TRANS UNITS)
680	2	VE	ACWCF	6	3	1	TRANS UNITS	AVG CAMSHAFT WEAR CORRECTION FACTOR (TRANS UNITS)
690	2	VE	TACWCOR	6	3	N	TRANS UNITS	AVG CAMSHAFT WEAR CORRECTED RATING - TRANSFORMED (TRANS U
700	2	VE	ACUSA	6	3	N	TRANS UNITS	AVG CAN WEAR -APPLIED SA (TRANS UNITS)
710	2	VE	TACWFNL	6	3	N	TRANS UNITS	TRANSFORMED ACW FINAL RESULTS (TRANS UNITS)
720	2	VE	ACWFNL	6	1	N	MICROMETERS	AVG CAMSHAFT WEAR FINAL RESULT (NICROMETERS)
730	2	VE	NCW	6	D	N	MICROMETERS	MAX CAMSHAFT WEAR (NICROMETERS)
740	2	VE	TRANNCW	6	3	N	TRANS UNITS	TRANSFORMED MCW RESULTS (TRANS UNITS)
750	2	VE	MCWCF	6	3	N	TRANS UNITS	MAX CAMSHAFT WEAR CORRECTION FACTOR (TRANS UNITS)
760	2	VE	THEWCOR	6	3	N	TRANS UNITS	MAX CANSHAFT WEAR CORRECTED RATING - TRANSFORMED (TRANS U
770	2	VE	NCWSA	6	3	N	TRANS UNITS	MAX CAN WEAR -APPLIED SA (TRANS UNITS)
780	2	VE	THEWFIL	6	3	N	TRANS UNITS	TRANSFORMED NEW FINAL RESULTS (TRANS UNITS)
790	2	VE	NCWFNL	6	0	N	MICROMETERS	NAX CAMSHAFT WEAR FINAL RESULT (MICRONETERS)
800	2	VE	ARAW	6	1	N	MG	AVG ROCKER ARH WEAR (NG)
810	2	VE	MAXRAW	6	1	N	MG	NAX ROCKER ARN WEAR (NG)
820	2	VE	ATRGINC	5	1	N	NILS	AVG TOP RING GAP INCREASE WEAR (MILS)
830	2	VE	MXTRGINC	5	1	N	MILS	NAX TOP RING GAP INCREASE WEAR (NILS)
840	2	VE	ARBUL	6	1	N	MG	AVG ROD BEARING WEIGHT LOSS WEAR (MG)
850	2	VE	MAXBWL	6	1	N	MG	MAX ROD BEARING WEIGHT LOSS WEAR (NG)
860	2	VE	OSCRNSLG	4	Ō		X AREA	OIL SCREEN SLUDGE CLOGGING (X AREA)
870	2	VE	OSCRNDEB	4	0		X AREA	OIL SCREEN DEBRIS CLOGGING (X AREA)
880	2	VE	OILRING	4	ō		X AREA	OIL RING CLOGGING (X AREA)
890	,	VE	PCV18	ž	0		X ARFA	PCV VALVE AT 18 IN. HG. CLOGGING (X AREA)
900	2	VE	PCV8	4	0		X AREA	PCV VALVE AT 8 IN. HG. CLOGGING (X AREA)
910	2	VE	NOCAMLOB	1	0		A AAEA	NUMBER OF CAMSHAFT LOBE HOLES
	2	VE		5	2		MERITS	
920	2	VE	AVGCLD	5	2		MERITS	AVERAGE CROWN LAND DEPOSITS (MERITS)
930	-	VE	AVGSLD	5	2		NERITS	AVERAGE SECOND LAND DEPOSITS (MERITS) Average oil ring land deposits (merits)
940	2		AVGORLD	-	2			
950 940	2	VE	AVGPUCD	5			MERITS	AVERAGE PISTON UNDER CROWN DEPOSITS (MERITS)
960	Z	VE		4	1	N	MERITS	INTAKE VALVE DEPOSITS (MERITS)
970	2	VE	NSLFTPLG	1	0	N		NUMBER STUCK LIFTER PLUNGERS
980	2	VE	NHSCMPRG	1	0	N		NUMBER HOT STUCK COMPRESSION RINGS
990	2	VE	NSTKORNG	1	0	N	r. a -	NUMBER STUCK OIL RINGS
1000	2	VE	DILCON	3	0		FL. OZ.	OIL CONSUMPTION (FL. OZ.)
1010	2	VE	ACBLURT1	4	2		CFN	AVG CORRECTED BLOWBY FLOW RATE STAGE I (CFM)
1020	3	VE	RSAEVISC	7	0	C		REFERENCE SAE VISCOSITY GRADE
1030	3	VE	RTESTLEN	3	0	_	KHH	REFERENCE TEST LENGTH (HHH)
1040	3	VE	RENGINE	6	0	C		REFERENCE ENGINE
1050	3	VE	RNOMPISO	4	2	N		REFERENCE NOMINAL PISTON OVERSIZE
	_		BEHEI BTR	2	0	C		REFERENCE FUEL BATCH IDENTIFIER
1060	3	VĒ	RFUELBID	-	-	-		

								2 – 01a
2-sep-199	7						Report: ASTN D	ats Dictionary
		Test	field	Field	Decimal	Data		•
Sequence	Form	Area	Name	Length	Size	Type	<u>Units/format</u>	Description
1080	3	VE	RSTRTIME	5	0	C	NH:MM	REFERENCE STARTING TIME (HH:NM)
1090	3	VE	RLABCODE	12	0	C		REFERENCE LAB OIL CODE
1100	3	VE	RRACSRT	5	2	N	MERITS	REFERENCE ROCKER ARM COVER SLUDGE RATING (MERITS)
1110	3	VE	RTRNRACS	6	3	N	TRANS UNITS	REFERENCE TRANSFORMED RACS RESULTS (TRANS UNITS)
1120	3	VE	RTRCSCOR	6	3	N	TRANS UNITS	REFERENCE TRANSFORMED RACS CORRECTED RATING (TRANS UNITS)
1130	3	VE	RRACSFNL	5	2	N	MERITS	REFERENCE ROCKER ARN COVER FINAL RESULT (MERITS)
1140	3	VE	RCAMBSRT	5	2	N	MERITS	REFERENCE CAMSHAFT BAFFLE SLUDGE RATING (MERITS)
1150	3	VE	RVLVDKRT	5	2	N	MERITS	REFERENCE VALVE DECK SLUDGE RATING (MERITS)
1160	3	VE	RFSLHSRT	5	2	N	MERITS	REFERENCE FRONT SEAL HOUSING SLUDGE RATING (MERITS)
1170	3	VE	RCYLBLRT	5	2	N	MERITS	REFERENCE CYLINDER BLOCK SLUDGE RATING (MERITS)
1180	3	VE	ROILPART	5	2		MERITS	REFERENCE OIL PAN SLUDGE RATING (MERITS)
1190	3	VE	RAES	5	2	N	MERITS	REFERENCE AVERAGE ENGINE SLUDGE RATING (MERITS)
1200	3	VE	RTRANAES	6	3	N	TRANS UNITS	REFERENCE TRANSFORMED AES RESULTS (TRANS UNITS)
1210	3	VE	RTAESCOR	6	3	N	TRANS UNITS	REFERENCE TRANSFORMED AES CORRECTED RATING (TRANS UNITS)
1220	3	VE	RAESFNL	5	2	N	MERITS	REFERENCE AVERAGE ENGINE SLUDGE FINAL RESULT (MERITS)
1230	3	VE	RPISKVRT	5	2	N	MERITS	REFERENCE AVERAGE PISTON SKIRT VARNISH (MERITS)
1240	3	VE	RPSKVFNL	5	2	N	MERITS	REFERENCE FINAL RESULT PISTON SKIRT VARNISH (MERITS)
1250	3	VE	RRACVRT	5	2	N	MERITS	REFERENCE ROCKER ARM COVER VARNISH RATING (MERITS)
1260	3	VE	RCAMBVRT	5	2	N	MERITS	REFERENCE CAMSHAFT BAFFLE VARNISH RATING (MERITS)
1270	3	VE	RCYLWVRT	5	2	N	MERITS	REFERENCE CYLINDER WALLS VARNISH RATING (MERITS)
1280	3	VE	ROILPNVR	5	2	N	MERITS	REFERENCE OIL PAN VARNISH RATING (MERITS)
1290	3	VE	RAEV	5	2	Ħ	MERITS	REFERENCE OIL AVERAGE ENGINE VARNISH RATING (MERITS)
1300	3	VE	RAEVFNL	5	2	N	MERITS	REFERENCE AVERAGE ENGINE VARNISH FINAL RESULT (MERITS)
1310	3	VE	RACU	6	1	N	MICROMETERS	REFERENCE AVG CANSHAFT WEAR (NICROMETERS)
1320	3	VE	RTACW	6	3	N	TRANS UNITS	REFERENCE TRANSFORMED ACW RESULTS (TRANS UNITS)
1330	3	VE	RTACWCOR	6	3	N	TRANS UNITS	REFERENCE TRANSFORMED ACH CORRECTED RATING (TRANS UNITS)
1340	3	VE	RACWFNL	6	1	N	MICROMETERS	REFERENCE AVG CAMSHAFT WEAR FINAL RESULTS (MICROMETERS)
1350	3	VE	RMCW	6	0	N	MICROMETERS	REFERENCE MAX CAMSHAFT WEAR (NICROMETERS)
1360	3	VE	RTNCW	6	3	N	TRANS UNITS	REFERENCE TRANSFORMED NOW RESULTS (TRANS UNITS)
1370	3	VE	RTHCWCOR	6	3	N	TRANS UNITS	REFERENCE TRANSFORMED MCW CORRECTED RATING (TRANS UNITS)
1380	3	VE	RMCWFNL	6	0	N	MICROMETERS	REFERENCE MAX CAMSHAFT WEAR FINAL RESULT (MICROMETERS)
1390	3	VE	RARAW	6	1	N	NG	REFERENCE AVG ROCKER ARN WEAR (NG)
1400	3	VE	RMAXRAW	6	1	N	NG	REFERENCE MAX ROCKER ARN WEAR (NG)
1410	3	VE	RATRGINC	5	1	N	NILS	REFERENCE AVG TOP RING GAP INCREASE WEAR (NILS)
1420	3	VE	RXTRGINC	5	1	N	MILS	REFERENCE MAX TOP RING GAP INCREASE WEAR (MILS)
1430	3	VE	RARBWL	6	1	N	NG	REFERENCE AVG ROD BEARING WEIGHT LOSS WEAR (NG)
1440	3	VE	RMAXBWL	6	1	M	NG	REFERENCE MAX ROD BEARING WEIGHT LOSS WEAR (NG)
1450	3	VE	ROSCRSLG	4	0	N	X AREA	REFERENCE OIL SCREEN SLUDGE CLOGGING (X AREA)
1460	3	VE	ROSCRDEB	- 4	0	N	X AREA	REFERENCE OIL SCREEN DEBRIS CLOGGING (X AREA)
1470	3	VĖ	ROILRING	4	0	N	% AREA	REFERENCE OIL RING CLOGGING (X AREA)
1480	3	VE	RPCV18	4	0	M	X AREA	REFERENCE PCV VALVE AT 18 IN. HG. CLOGGING (X AREA)
1490	3	VE	RPCV8	4	0	M	X AREA	REFERENCE PCV VALVE AT 8 IN. HG. CLOGGING (X AREA)
1500	3	VE	RNCAMLOB	1	0	N		REFERENCE NUMBER OF CAMSHAFT LOBE HOLES CLOGGING
1510	3	VE	RAVGCLD	5	2	N	MERITS	REFERENCE AVG CROWN LAND DEPOSIT (NERITS)
1520	3	VE	RAVGSLD	5	2	N	MERITS	REFERENCE AVG SECOND LAND DEPOSIT (MERITS)
1530	3	VE	RAVGORLD	5	2		MERITS	REFERENCE AVG OIL RING LAND DEPOSIT (MERITS)
1540	3	VE	RAVGPUCD	5	2		MERITS	REFERENCE AVG PISTON UNDER CROWN DEPOSIT (MERITS)
1550	3	VE	RINVDEP	4	1		MERITS	REFERENCE INTAKE VALVE DEPOSITS (MERITS)
1560	3	VE	RSLFTPLG		0	N		REFERENCE NUMBER STUCK LIFTER PLUNGERS
1570	3	VE	RHSCMPRG		0	N		REFERENCE NUMBER NOT STUCK COMPRESSION RINGS
1580	3	VE	RSTKORNG	_	0	N		REFERENCE NUMBER STUCK OIL RINGS
1590	3	VE	ROILCON	3	0	N	FL. OZ.	REFERENCE OIL CONSUMPTION (FL. 02.)
1600	3	VE	RABLBYRT		2		CFN	REFERENCE AVG BLOWBY FLOW RATE (CFN)
1610	3	VE	STATDATE	8	0		YYYYNHDD IG A14 3 Data F	STAT DATE (YYYYMHDD)
							n - 414 S (1919	ni in nai V

FIG. A14.3 Data Dictionary

3	007							te Dietioneu
2-sep-1	777	Test	Field	Field	Decimal	Data	Report: ASTM Da	ice Discionary
Sequenc	<u>:e Form</u>		Name				<u>Units/format</u>	Description
					ميرين تر <u>ا</u> غيان	للبتان كبن		
1620	3	VE	TARGAES	6	3	N	TRANS UNITS	TRANSFORMED AES TARGET VALUE (TRANS UNITS)
1630	3	VE	SAES	6	3	N	STD DEV	TRANSFORMED AES STANDARD DEVIATION (STD DEV)
1640	3	VE	DELAES	6	3	N	TRANS UNITS	TRANSFORMED AES DELTA VALUE (TRANS UNITS)
1650	3	VE	DELSAES	7	3	N	TRANS UNITS	TRANSFORMED AES DELTA/S VALUE (TRANS UNITS)
1660	3	VE	TARRACS	6	3	N	TRANS UNITS	TRANSFORMED RACS TARGET VALUE (TRANS UNITS)
1670	3	VE	SRACS	6	3	N	STD DEV	TRANSFORMED RACS STANDARD DEVIATION (STD DEV)
1680	3	VE	DELRACS	6	3	N	TRANS UNITS	TRANSFORMED RACS DELTA VALUE (TRANS UNITS)
1690	3	VE	DELSRACS	7	3	N	TRANS UNITS	TRANSFORMED RACS DELTA/S VALUE (TRANS UNITS)
1700	3	VE	TARAPV	6	3	N	MERITS	APV TARGET VALUE (MERITS)
1710	3	VE	SAPV	6	3	N	STD DEV	APV STANDARD DEVIATION (STD DEV)
1720	3	VE	DELAPV	6	3	N	MERITS	APV DELTA VALUE (MERITS)
1730	3	VE	DELSAPV	7	3	N	MERITS	APV DELTA/S VALUE (MERITS)
1740	3	VE	TARAEV	6	3	N	MERITS	AEV TARGET VALUE (MERITS)
1750	3	VE	SAEV	6	3	N	STD DEV	AEV STANDARD DEVIATION (STD DEV)
1760	3	VE	DELAEV	6	3	N	MERITS	AEV DELTA VALUE (MERITS)
1770	3	VE	DELSAEV	7	3	N	MERITS	AEV DELTA/S VALUE (MERITS)
1780	3	VE	TARACU	6	3	N	TRANS UNITS	TRANSFORMED ACW TARGET VALUE (TRANS UNITS)
1790	3	VE	SACY	6	3	N	STD DEV	TRANSFORMED ACW STANDARD DEVIATION (STD DEV)
1800	3	VE	DELACW	7	4	N	TRANS UNITS	TRANSFORMED ACV DELTA VALUE (TRANS UNITS)
1810	3	VE	DELSACW	7	3	N	TRANS UNITS	TRANSFORMED ACW DELTA/S VALUE (TRANS UNITS)
1820	3	VE	TARMCW	6	3	N		TRANSFORMED NEW TARGET VALUE (TRANS UNITS)
1830	3	VE	SMCW	6	3	N	STD DEV	TRANSFORMED NEW STANDARD DEVIATION (STD DEV)
1840	3	VE	DELNCW	7	4	N	TRANS UNITS	TRANSFORMED NEW DELTA VALUE (TRANS UNITS)
1850	3	VE	DELSMCW	7	3	N	TRANS UNITS	TRANSFORMED NEW DELTA/S VALUE (TRANS UNITS)
1860	4	YE	XRPM1	4	0	N	R/MIN	MAX ENGINE SPEED STAGE I (R/MIN)
1870	4	VE	IRPN1	4	0	N	R/MIN	MIN ENGINE SPEED STAGE I (R/MIN) Avg Engine speed stage I (R/MIN)
1880	4	VE	ARPH1	4	0	N N	R/MIN	NAX ENGINE SPEED STAGE I (R/MIN)
1890	4	VE	XRPM2	4	0	N	R/HIN R/HIN	MAX ENGINE SPEED STAGE II (K/MIN) MIN ENGINE SPEED STAGE II (K/MIN)
1900	4	VE VE	IRPM2 ARPM2	4	0	N	R/MIN	AVG ENGINE SPEED STAGE II (R/MIN)
1910 1920	4	VE VE	XRPM3	4	0	N	R/NIK	MAX ENGINE SPEED STAGE III (R/MIN)
1920	4	VE	IRPM3	4	0	N	R/MIN	MIN ENGINE SPEED STAGE III (R/MIN)
1940	4	VË	ARPN3	4	Ö	Ň	R/MIN	AVG ENGINE SPEED STAGE III (R/MIN)
1950	4	VE	XPWR1	5	1	N	BHP	MAX ENGINE POWER STAGE I (BHP)
1950	4	VE	IPWR1	5	1	N	BHP	MIN ENGINE POWER STAGE I (BHP)
1970	4	VE	APWR1	5	1	N	BHP	AVG ENGINE POWER STAGE I (BHP)
1980	4	VE	XPWR2	5	1	N	BHP	MAX ENGINE POWER STAGE II (BHP)
1990	4	VE	IPWR2	5	1	N	BHP	MIN ENGINE POWER STAGE II (BHP)
2000	4	VE	APUR2	5	1	N	BHP	AVG ENGINE POWER STAGE II (BHP)
2010	4	VE	XPWR3	4	1	N	BHP	NAX ENGINE POWER STAGE III (BHP)
2020	4	VE	IPWR3	4	1	N	BHP	MIN ENGINE POWER STAGE III (BHP)
2030	4	VE	APUR3	4	1	N	SHP	AVG ENGINE POWER STAGE 111 (BHP)
2040	4	VĒ	XENGOIN1	4	0	N	*F	MAX ENGINE OIL IN TEMP STAGE [(°F)
2050	4	VE	IENGOIN1	4	0	N	*F	MIN ENGINE OIL IN TEMP STAGE I (*F)
2060	4	VE	AENGOIN1	5	1	N	*F	AVG ENGINE OIL IN TEMP STAGE I (*F)
2070	4	VE	XENGOIN2	4	0	N	*F	MAX ENGINE OIL IN TEMP STAGE II ("F)
2080	4	VE	1ENGOIN2	- 4	0	N	•F	MIN ENGINE OIL IN TEMP STAGE II (°F)
2090	4	VE	AENGOIN2	5	1	N	*F	AVG ENGINE OIL IN TEMP STAGE II ("F)
2100	4	VE	XENGOIN3		0	N	*F	MAX ENGINE OIL IN TEMP STAGE III (*F)
2110	4	VE	I ENGOIN3	4	0	Ņ	*F	MIN ENGINE OIL IN TEMP STAGE 111 (*F)
2120	4	VE	AENGOIN3	5	1	N	•F	AVG ENGINE OIL IN TEMP STAGE 111 (*F)
2130	4	VE	XENGODT1	4	0	N	*F	MAX ENGINE OIL DELTA TEMPOUT-IN STAGE I (°F)
2140	4	VE	IENGODT1	4	0	N	*F	MIN ENGINE DIL DELTA TEMPOUT-IN STAGE I (°F)
2150	4	VE	AENGODT1	4	0	N	*F	AVG ENGINE OIL DELTA TEMPOUT-IN STAGE I ("F)
							144 Data Distigr	

FIG. A14.4 Data Dictionary

2-sep-19	97						Report: ASTM [Data Dictionary
		Test	Field	Field	Decimal	Data	•	·····
<u>Sequence</u>	<u>form</u>	Area	Name	Length	<u>Size</u>	Ivpe	<u>Units/Format</u>	Description
					•		•-	
2160	4	VE	XENGODT2		0	N	*F	MAX ENGINE OIL DELTA TEMP-OUT-IN STAGE II (°F)
2170 2180	4	VE VE	IENGODT2		0	N	•F •F	MIN ENGINE OIL DELTA TEMPOUT-IN STAGE II (*F)
2180	4	VE	AENGODT2 XENGODT3		0	N N	-7 •F	AVG ENGINE OIL DELTA TEMPOUT-IN STAGE II (°F) NAX ENGINE OIL DELTA TEMPOUT-IN STAGE III (°F)
2200	4	VE	IENGODT3		0	N	• =	MAX ENGINE OIL DELTA TEMPDUT-IN STAGE III ("F) MIN ENGINE OIL DELTA TEMPDUT-IN STAGE III ("F)
2210	4	VE	AENGODT3	=	0	N	• • • •	AVG ENGINE OIL DELTA TEMPOUT-IN STAGE III ("F)
2220	4	VE	XPMPOPR1		1	N	PSIG	NAX PUMP OIL PRESSURE STAGE I (PSIG)
2230	4	VE	IPNPOPR1		1	Ň	PSIG	NIN PUMP OIL PRESSURE STAGE I (PSIG)
2240	4	VE	APHPOPR1		1	N	PSIG	AVG PUNP OIL PRESSURE STAGE I (PSIG)
2250	4	VE	XPMPOPR2	5	1	N	PS 1G	MAX PUMP OIL PRESSURE STAGE II (PSIG)
2260	4	VE	IPHPOPR2	5	1	N	PSIG	NIN PUNP OIL PRESSURE STAGE II (PSIG)
2270	4	VE	APHPOPRZ	5	1	N	PSIG	AVG PUMP OIL PRESSURE STAGE II (PSIG)
2280	4	VE	XPMPOPR3	5	1	N	PSIG	MAX PUMP OIL PRESSURE STAGE III (PSIG)
2290	4	VE	IPHPOPR3	5	1	M	PSIG	MIN PUMP OIL PRESSURE STAGE III (PSIG)
2300	4	VE	APHPOPR3	5	1	N	PSIG	AVG PUMP OIL PRESSURE STAGE III (PSIG)
2310	4	VE	XENGOPR1	5	1	N	PSIG	MAX ENGINE OIL PRESSURE STAGE I (PSIG)
2320	4	VE	IENGOPR1		1	N	PSIG	MIN ENGINE OIL PRESSURE STAGE I (PSIG)
2330	4	VE	AENGOPRI		1	N	PSIG	AVG ENGINE OIL PRESSURE STAGE I (PSIG)
2340	4	VE	XENGOPR2	_	1	N	PSIG	MAX ENGINE OIL PRESSURE STAGE II (PSIG)
2350	4	VE	IENGOPR2		1	N	PSIG	MIN ENGINE OIL PRESSURE STAGE II (PSIG)
2360	4	VE	AENGOPR2	_	1	N	PSIG	AVG ENGINE OIL PRESSURE STAGE II (PSIG)
2370	4	VE	XENGOPR3		1	N	PSIG	MAX ENGINE OIL PRESSURE STAGE III (PSIG)
2380	4	VE	IENGOPR3	_	1	N	PSIG	NIN ENGINE OIL PRESSURE STAGE III (PSIG)
2390 2400	4	VE	AENGOPR3	_	1	N	PSIG PSIG	AVG ENGINE OIL PRESSURE STAGE III (PSIG) NAX CYLINDER HEAD OIL PRESSURE STAGE I (PSIG)
2400 2410	4	VE VE	XCYLOPR1 ICYLOPR1	_	1	N N	PSIG	NIN CYLINDER HEAD OIL PRESSURE SINGE I (PSIG)
2410	4	VE	ACYLOPR1	-	1	N	PSIG	AVG CYLINDER HEAD OIL PRESSURE STAGE I (PSIG)
2420	4	VE	XCYLOPR2	_	1	N	PSIG	MAX CYLINDER HEAD OIL PRESSURE STAGE II (PSIG)
2440	4	VE	1CYLOPR2	_	i	N	PSIG	MIN CYLINDER HEAD OIL PRESSURE STAGE II (PSIG)
2450	4	VE	ACYLOPR2	_	1	N	PSIG	AVG CYLINDER HEAD OIL PRESSURE STAGE II (PSIG)
2460	4	VE	XCYLOPR3	_	1	N	PSIG	MAX CYLINDER HEAD OIL PRESSURE STAGE III (PSIG)
2470	4	VE	ICYLOPR3	-	1	N	PSIG	MIN CYLINDER HEAD OIL PRESSURE STAGE III (PSIG)
2480	4	VE	ACYLOPR3	_	1	N	PSIG	AVG CYLINDER HEAD OIL PRESSURE STAGE III (PSIG)
2490	4	VE	XDPPE1	5	1	N	PSI	NAX DELTA PRESSUREPUMP-ENGINE STAGE I (PSI)
2500	4	VE	IDPPE1	5	1	N	PSI	MIN DELTA PRESSUREPUMP-ENGINE STAGE I (PSI)
2510	4	VE	ADPPE1	5	1	N	PSI	AVG DELTA PRESSUREPUMP-ENGINE STAGE I (PSI)
2 520	4	VE	XDPPE2	5	1	N	PSI	NAX DELTA PRESSUREPUMP-ENGINE STAGE II (PSI)
2530	4	VE	IDPPE2	5	1	N	IZQ	MIN DELTA PRESSUREPUMP-ENGINE STAGE II (PSI)
2540	4	VE	ADPPE2	5	1	N	PSI	AVG DELTA PRESSUREPUMP-ENGINE STAGE II (PSI)
2550	4	VE	XDPPE3	5	1	N	PSI	MAX DELTA PRESSUREPUMP-ENGINE STAGE III (PSI)
2560	4	VE	IDPPE3	5	1	N	PSI	NIN DELTA PRESSUREPUMP-ENGINE STAGE 111 (PSI)
2570	4	VE	ADPPE3	5	1	N	PSI	AVG DELTA PRESSUREPUMP-ENGINE STAGE III (PSI)
2580	4	VE	XDPEH1	5	1	N	PSI	MAX DELTA PRESSUREENGINE-HEAD STAGE I (PSI)
2590	4	VE	IDPEH1	5	1	N	PSI	NIN DELTA PRESSUREENGINE-HEAD STAGE 1 (PSI)
2600	4	VE	ADPEH1	5	1	N	PSI	AVG DELTA PRESSUREENGINE-HEAD STAGE 1 (PSI)
2610	4	VE	XDPEH2	5	1		PSI	NAX DELTA PRESSURE - ENGINE - HEAD - STAGE II (PSI)
2620	4	VE	IDPEHZ	5	1	N	PSI	NIN DELTA PRESSUREENGINE-HEAD STAGE 11 (PSI) Ave delta describeencine-head stage 11 (PSI)
2630	4	VE	ADPEH2	5	1	N	PSI PSI	AVG DELTA PRESSUREENGINE-HEAD STAGE II (PSI) NAX DELTA PRESSUREENGINE-HEAD STAGE III (PSI)
2640	4	VE	XDPEH3	5 5	1	N N	PSI	NIN DELTA PRESSURE-"ENGINE-HEAD STAGE III (PSI)
2650	4	VE VE	IDPEH3 ADPEH3	5	1		PSI	AVG DELTA PRESSUREENGINE-HEAD STAGE III (PSI)
2660 2670	4	VE VE	XCOLOUT1		0	N	*F	NAX ENGINE COOLANT OUT TEMP STAGE I (*F)
2670	4	VE VE	ICOLOUTI	-	0	N	•F	MIN ENGINE COOLANT OUT TEMP STAGE I (°F)
2690	4	VE VE	ACOLOUT1	_	1	N	•F	AVG ENGINE COOLANT OUT TEMP STAGE I (°F)
2070	-	16	NAA74811	-	•		A14.5 Data Dict	· -

FIG. A14.5 Data Dictionary

							.412	
2-sep-199	77						Report: ASTN 1	Data Dictionary
		Test	Field	Field	Decimal	Data		
Sequence	Form	Area	<u>Name</u>	Length	<u>Size</u>	Type	<u>Units/Format</u>	Description
2700	4	VE	XCOLOUT2	- 4	0	N	*F	MAX ENGINE COOLANT OUT TEMP STAGE II ("F)
2710	4	VE	ICOLOUT2	4	0	N	•F	MIN ENGINE COOLANT OUT TEMP STAGE II (°F)
2720	4	VE	ACOLOUT2	5	1	N	•F	AVG ENGINE COOLANT OUT TEMP STAGE II (°F)
2730	4	VE	XCOLOUT3	4	0	N	•#	MAX ENGINE COOLANT OUT TEMP STAGE 111 (*F)
2740	4	VE	ICOLOUT3	4	Ó	N	•#	MIN ENGINE COOLANT OUT TEMP STAGE 111 ("F)
2750	4	VE	ACOLOUT3	_	1	N	•F	AVG ENGINE COOLANT OUT TEMP STAGE 111 ("F)
2760	4	VE	XCOLDT1	4	ò		•F	MAX ENGINE COOLANT DELTA TEMPOUT-INSTAGE I ("F)
2770	4	VE	1COLDT1	4	0	N	•F	NIN ENGINE COOLANT DELTA TEMPOUT-INSTAGE 1 (*F)
2780	4	VE	ACOLDT1	4	õ		•F	AVG ENGINE COOLANT DELTA TEMPOUT-INSTAGE 1 (*F)
2790	4	VE	XCOLDT2	4	0		• 5	NAX ENGINE COOLANT DELTA TENP-OUT-IN-STAGE II (*F)
	4			4	0		•F	NIN ENGINE COOLANT DELTA TEMP-OUT-IN-STAGE II (*F)
2800	-	VE	ICOLDT2	-	-		•	
2810	4	VE	ACOLDT2	4	0	*	•F	AVG ENGINE COOLANT DELTA TEMPOUT-INSTAGE II ("F)
2820	4	VE	XCOLDT3	4	0	N	*F	NAX ENGINE COOLANT DELTA TEMPOUT-INSTAGE III ("F)
2830	4	VE	ICOLDT3	4	0	N	•F	NIN ENGINE COOLANT DELTA TEMPOUT-INSTAGE III (*F)
2840	4	VE	ACOLDT3	4	0	N	•F	AVG ENGINE COOLANT DELTA TEMPOUT-INSTAGE III ("F)
2850	4	VË	XCOLFRT1	5	1	N	GPM	NAX ENGINE COOLANT FLOW RATE STAGE I (GPM)
2860	4	VE	ICOLFRT1	5	1	N	GPM	MIN ENGINE COOLANT FLOW RATE STAGE I (GPM)
2870	4	VE	ACOL FRT1	5	1	N	GPN	AVG ENGINE COOLANT FLOW RATE STAGE I (GPM)
2880	4	VE	XCOLFRT2	5	1	N	GPM	MAX ENGINE COOLANT FLOW RATE STAGE II (GPN)
2890	4	VE	ICOLFRT2	5	1	N	GPN	MIN ENGINE COOLANT FLOW RATE STAGE II (GPM)
2900	4	VE	ACOLFRT2	5	1	N	GPN	AVG ENGINE COOLANT FLOW RATE STAGE II (GPH)
2910	4	VE	XCOLPRE1	5	1	N	PSIG	NAX. ENGINE COOLANT PRESSURE, STAGE 1 (PSIG)
2920	4	VE	ICOLPRE1	5	1	N	PSIG	MIN. ENGINE COOLANT PRESSURE, STAGE I (PSIG)
2930	4	VE	ACOLPRE1	5	1	N	PSIG	AVG. ENGINE COOLANT PRESSURE, STAGE I (PSIG)
2940	4	VE	XCOLPRE2	5	1	N	PSIG	MAX. ENGINE COOLANT PRESSURE, STAGE II (PSIG)
2950	4	VE	ICOLPRE2	5	1	N	PSIG	MIN. ENGINE COOLANT PRESSURE, STAGE II (PSIG)
2960	4	VE	ACOLPRE2	5	1	N	PSIG	AVG. ENGINE COOLANT PRESSURE, STAGE II (PSIG)
2970	4	VE	XCOLPRE3	5	1	N	PSIG	NAX. ENGINE COOLANT PRESSURE, STAGE III (PSIG)
2980	4	VE	I COLPRE3	5	1	N	PSIG	NIN. ENGINE COOLANT PRESSURE, STAGE III (PSIG)
2990	4	VE	ACOLPRE3	5	1	N	PSIG	AVG. ENGINE COOLANT PRESSURE, STAGE III (PSIG)
3000	4	VE	XEXCOOL 1	3	0	M	*F	NAX EXHAUST NANIFOLD COOLANT TEMP STAGE 1 (*F)
3010	4	VE	IEXCOOL1	3	Ō	N	*F	MIN EXHAUST MANIFOLD COOLANT TEMP STAGE I (°F)
3020	4	VE	AEXCOOL 1	3	Ō	N	•F	AVG EXHAUST MANIFOLD COOLANT TEMP STAGE I (°F)
3030	4	VE	XEXCOOL2	3	0	N	•F	MAX EXHAUST MANIFOLD COOLANT TEMP STAGE II (*F)
3040	4	VE	IEXCOOL2	3	0	N	• 5	NIN EXHAUST MANIFOLD COOLANT TEMP STAGE II (°F)
	•			-	-		•F	AVG EXHAUST MANIFOLD COOLANT TEMP STAGE II (*F)
3050	4	VE	AEXCOOL2	3	0	14 N	-	
3060	4	VE	XEXCOOL3	3	0		*F	
3070	4	VE	IEXCOOL3	3	0		*F	MIN EXHAUST MANIFOLD COOLANT TEMP STAGE III ("F)
3080	4	VE	AEXCOOL3	3	0		*F	AVG EXHAUST NANIFOLD COOLANT TEMP STAGE III ("F)
3090	4	VE	XRACCTP1	4	0		*F	MAX RAC COOLANT IN TEMP STAGE I ("F)
3100	4	VE	IRACCTP1	4	0		•F	NIN RAC COOLANT IN TEMP STAGE 1 (°F)
3110	4	VE	ARACCTP1	5	1		• F	AVG RAC COOLANT IN TEMP STAGE 1 (*F)
3120	4	VE	XRACCTP2		0	N	•F	NAX RAC COOLANT IN TEMP STAGE II (°F)
3130	4	VE	IRACCTP2	4	0	¥	*F	MIN RAC COOLANT IN TEMP STAGE II (°F)
3140	4	VE	ARACCTP2	5	1	N	*F	AVG RAC COOLANT IN TEMP STAGE II (°F)
3150	4	VE	XRACCTP3	4	0	N	•F	NAX RAC COOLANT IN TEMP STAGE III (°F)
3160	4	VE	IRACCTP3	4	0	N	*F	NIN RAC COOLANT IN TEMP STAGE III ("F)
3170	4	VË	ARACCTP3	5	1	H	*F	AVG RAC COOLANT IN TEMP STAGE III (*F)
3180	4	VE	XRACCFR1	4	1	N	GPN	MAX RAC COOLANT FLOW RATE STAGE I (GPM)
3190	4	VE	IRACCFR1	4	1	N	GPN	HIN RAC COOLANT FLOW RATE STAGE 1 (GPN)
3200	4	VE	ARACCFR1	4	1	N	GPM	AVG RAC COOLANT FLOW RATE STAGE I (GPN)
3210	4	VE	XRACCFR3	4	1	N	GPM	NAX RAC COOLANT FLOW RATE STAGE III (GPM)
3220	4	VE	IRACCFR3	4	1	N	GPM	NIN RAC COOLANT FLOW RATE STAGE 111 (GPM)
3230	4	VE	ARACCERS	4	1		GPM	AVG RAC COOLANT FLOW RATE STAGE III (GPM)
~~~	~			-			G. A14.6 Data I	

FIG. A14.6 Data Dictionary

						ų		
2-sep-199	<b>7</b> 7				_		Report: ASTH D	ata Dictionary
	*	Test	Field	Field				
Sequence	rorie	ALLE	Name	Length	<u>5176</u>	TADE	<u>Units/Format</u>	Description
3240	4	VE	XINAIRT1	4	0	N	•F	MAX INTAKE AIR TEMP STAGE I (°F )
3250	4	VE	IINAIRT1	4	0	N	•F	NIN INTAKE AIR TEMP STAGE I ("F )
3260	4	VE	AINAIRT1	4	0	N	*F	AVG INTAKE AIR TEMP STAGE I (°F )
3270	4	VE	XINAIRT2	4	0	N	*F	MAX INTAKE AIR TEMP STAGE II (°F )
3280	4	VE	IINAIRTZ	4	Q	×.	*F	MIN INTAKE AIR TEMP STAGE II (°F )
3290	4	VE	AINAIRT2	4	0	N	*F	AVG INTAKE AIR TEMP STAGE II (°F )
3300	4	VE	XINAIRT3		0	N	•F	NAX INTAKE AIR TEMP STAGE III ("F )
3310	4	VE	IINAIRT3		0	N	•F	NIN INTAKE AIR TEMP STAGE III (*F )
3320	4	VE	AINAIRT3		0	N	•F	AVG INTAKE AIR TEMP STAGE III (*F )
3330	4	VE	XAIRHUNI	3	0	N	GRAINS/LB	MAX INTAKE AIR SPECIFIC HUNIDITY STAGE I (GRAINS/LB)
3340	4	VE	IAIRHUM1	3	0	N	GRAINS/LB	NIN INTAKE AIR SPECIFIC HUMIDITY STAGE I (GRAINS/LB)
3350 3360	4	VE VE	AAIRHUH1 XAIRHUH2		0	N	GRAINS/LB GRAINS/LB	AVG INTAKE AIR SPECIFIC HUMIDITY STAGE I (GRAINS/LB) NAX INTAKE AIR SPECIFIC HUMIDITY STAGE II (GRAINS/LB)
3370	4	VE	IAIRHUN2	_	0		GRAINS/LB	MIN INTAKE AIR SPECIFIC HUNIDITY STAGE II (GRAINS/LB)
3380	4	VE	AAIRHUH2	3	0		GRAINS/LB	AVG INTAKE AIR SPECIFIC HUNIDITY STAGE II (GRAINS/LB)
3390	4	VE	XAIRHUNS	3	Ō	N	GRAINS/LB	MAX INTAKE AIR SPECIFIC HUNIDITY STAGE 111 (GRAINS/LB)
3400	4	VE	IAIRHUM3	3	0	N	GRAINS/LB	WIN INTAKE AIR SPECIFIC HUNIDITY STAGE III (GRAINS/LB)
3410	4	VE	AAIRHUH3	3	0	N	GRAINS/LB	AVG INTAKE AIR SPECIFIC HUMIDITY STAGE III (GRAINS/LB)
3420	4	VE	XINAIRP1	5	2	N	in. H20 GAGE	MAX INTAKE AIR PRESSURE STAGE I (IN N20 GAGE)
3430	4	VE	11NAIRP1	5	2	N	in. H2O GAGE	NIN INTAKE AIR PRESSURE STAGE I (IN H2O GAGE)
3440	4	VE	AINAIRP1	5	2	N	in. N20 GAGE	AVG INTAKE AIR PRESSURE STAGE I (IN H20 GAGE)
3450	4	VE	XINAIRP2		2	N	in. N2O GAGE	MAX INTAKE AIR PRESSURE STAGE II (in. HZO GAGE)
3460	4	VE	IINAIRP2		2	N	in. H20 GAGE	MIN INTAKE AIR PRESSURE STAGE II (in. H2O GAGE)
3470	4	VE	AINAIRP2	5	2	N	in. H20 GAGE	AVG INTAKE AIR PRESSURE STAGE II (IN H20 GAGE)
3480	4	VE	XINAIRP3	5	2	N	In. N20 GAGE	NAX INTAKE AIR PRESSURE STAGE III (IN H20 GAGE)
3490	4	VE	IINAIRP3	5 5	2 2	N N	in. H2O GAGE in. H2O GAGE	MIN INTAKE AIR PRESSURE \$TAGE III (IN H2O GAGE) Avg intake air pressure stage III (in H2O Gage)
3500 3510	4	VE VE	AINAIRP3 XCBLWRT1	4	2		CFN	NAX CORRECTED BLOWBY FLOW RATE STAGE I (CFM)
3520	4	VE	ICBLWRT1	4	2		CFN	NIN CORRECTED BLOWBY FLOW RATE STAGE I (CFM)
3530	4	VE	XCCASEP1	5	1	Ň	in. H20 GAGE	NAX CRANKCASE PRESSURE STAGE I (IN HZO GAGE)
3540	4	VE	ICCASEP1	5	1	N	In. NZO GAGE	MIN CRANKCASE PRESSURE STAGE I (IN H20 GAGE)
3550	4	VE	ACCASEP1	5	1	N	in. H20 GAGE	AVG CRANKCASE PRESSURE STAGE I (IN H20 GAGE)
3560	4	VĒ	XCCASEP2	5	1	N	in. H2O GAGE	MAX CRANKCASE PRESSURE STAGE II (IN H20 GAGE)
3570	4	VE	ICCASEP2	5	1	N	in. H20 GAGE	NIN CRANKCASE PRESSURE STAGE II (IN H2O GAGE)
3580	4	VE	ACCASEP2	5	1	N	in. H2O GAGE	AVG CRANKCASE PRESSURE STAGE 11 (IN H2O GAGE)
3590	4	VE	XCCASEP3		1	N	in. H20 GAGE	NAX CRANKCASE PRESSURE STAGE III (IN H2O GAGE)
3600	4	VE	ICCASEP3	5	1	M	in. H2O GAGE	MIN CRANKCASE PRESSURE STAGE III (IN HZO GAGE)
3610	4	VE	ACCASEP3	5	1	N	in. H20 GAGE	AVG CRANKCASE PRESSURE STAGE III (IN H20 GAGE)
3620	4	VE	XIMNVAC1	5	1	N	in. Hg gage	NAX INTAKE MANIFOLD VACUUM STAGE I (IN HG GAGE)
3630	4	VE	IIMNVAC1	5	1	<b>H</b>	in. Hg gage	MIN INTAKE MANIFOLD VACUUM STAGE I (IN HG GAGE)
3640	4	VE	AIMNVAC1	5	1	×	in. He sage	AVG INTAKE MANIFOLD VACUUM STAGE I (IN HG GAGE) May intake manifold vacuum stage II (In Hg Gage)
3650	4	VE	XIMNVACZ	5 5	1	N N	in. Hg gage In. Hg gage	MAX INTAKE MANIFOLD VACUUN STAGE II (IN HG GAGE) Nin Intake Manifold Vacuun Stage II (In hg Gage)
3660 3670	4	VE VE	IIMNVAC2 AIMNVAC2	-	1		in. Hg gage	AVG INTAKE MANIFOLD VACUUM STAGE II (IN HG GAGE)
3680	4	VE VE	XIMNVACE		1	Ň	in. Hg gage	HAX INTAKE MANIFOLD VACUUM STAGE III (IN NG GAGE)
3690	4	VE	I IMNVAC3	5	1	N	in. Ng gage	MIN INTAKE MANIFOLD VACUUM STAGE III (IN HG GAGE)
3700	4	VE	AIMNVAC3	5	1	×	in. Ng gage	AVG INTAKE MANIFOLD VACUUM STAGE III (IN NG GAGE)
3710	4	VE	XEXBKPR1	5	1	Ň	in. Hg abs	MAX EXHAUST BACK PRESSURE STAGE I (IN HG ABS)
3720	4	VE	IEXBKPR1	5	1		in. Hg abs	NIN EXHAUST BACK PRESSURE STAGE I (IN NG ABS)
3730	4	VE	AEXBKPR1	5	1	N	in. Hg abs	AVG EXHAUST BACK PRESSURE STAGE I (IN HG ABS)
3740	4	VE	XEXBKPR2	5	1		in. Hg abs	MAX EXHAUST BACK PRESSURE \$TAGE II (IN HG ABS)
3750	4	VE	IEXBKPRZ	5	1	N	in. Hg abs	NIN EXHAUST BACK PRESSURE STAGE II (IN HG ABS)
3760	4	VE	AEXBKPR2	5	1	N	fn. Hg abs	AVG EXHAUST BACK PRESSURE STAGE II (IN HG ABS)
3770	4	VE	XEXBKPR3	5	1	N	in. Kg abs	MAX EXHAUST BACK PRESSURE STAGE III (IN HG ABS)
						FIG	6. A14.7 Data Dic	ctionary

FIG. A14.7 Data Dictionary

2-sep-199	7						Report: ASTM Da	sta Dictionary
		Test	Field	Field	Decimal	Data	•	
Sequence	Form	Area	Name	Length	<u>Size</u>	Туре	<u>Units/Format</u>	Description
3780	4	VE	I EXBKPR3	5	1	N	in. Hg abs	MIN EXHAUST BACK PRESSURE STAGE III (IN HG ABS)
3790	4	VE	AEXBKPR3	5	1	N	in. Hg abs	AVG EXHAUST BACK PRESSURE STAGE III (IN HG ABS)
3800	4	VE	XIGNTIN1	2	0	N	BTDC	MAX IGNITION TIMING STAGE I (BTDC)
3810	4	VE	IIGNTIM1	2	0	N	BTDC	MIN IGNITION TIMING STAGE I (BTDC)
3820	4	VE	AIGNTIM1	2	0	N	BTDC	AVG IGNITION TIMING STAGE I (BTDC)
3830	4	VE	XFUELRT1	5	1	N	lb/h	MAX FUEL CONSUMPTION RATE STAGE I (LS/HR)
3840	4	VĒ	IFUELRT1	5	1	N	lb/h	NIN FUEL CONSUMPTION RATE STAGE I (LB/HR)
3850	4	VE	AFUELRT1	5	1	N	Lb/h	AVG FUEL CONSUMPTION RATE STAGE I (LB/HR)
3860	4	VE	XFUELRT2	5	1	N	lb/h	MAX FUEL CONSUMPTION RATE STAGE II (LB/HR)
3870	4	VE	I FUELRT2	5	1		lb/h	MIN FUEL CONSUMPTION RATE STAGE II (LB/HR)
3880	4	VE	AFUELRT2	5	1		ib/h	AVG FUEL CONSUMPTION RATE STAGE II (LB/HR)
3890	4	VE	XFUELRT3	4	1	N	lb/h	NAX FUEL CONSUMPTION RATE STAGE III (LB/HR)
3900	4	VE	I FUELRT3		1	N	lb/h	MIN FUEL CONSUMPTION RATE STAGE III (LB/HR)
3910	4	VE	AFUELRT3		1		lb/h	AVG FUEL CONSUMPTION RATE STAGE III (LB/HR)
3920	4	VE	XEXHGO1	5	2	Ň	*	MAX EXHAUST GAS OXYGEN STAGE I (X)
3930	4	VE	IEXHGO1	5	2	N	X	MIN EXHAUST GAS OXYGEN STAGE I (X)
3940	4	VE	AEXHGO1	5	z	Ň	X	AVG EXHAUST GAS OXYGEN STAGE 1 (X)
3950	4	VE	XEXHGO2	5	2	N	x	MAX EXHAUST GAS OXYGEN STAGE II (X)
3960	4	VE	1EXHGO2	5	2	N	x .	MIN EXHAUST GAS OXYGEN STAGE II (%)
3970	4	VE	AEXHGOZ	5	2	N	X	AVG EXHAUST GAS OXYGEN STAGE II (X)
3980	4	VE	XEXHGO3	5	2	N	X	MAX EXHAUST GAS OXYGEN STAGE III (X)
3990	4	VE	IEXHGO3	5	2	Ň	ž	MIN EXHAUST GAS OXYGEN STAGE III (X)
4000	4	VE	AEXHGO3	5	2	N	x	AVG EXHAUST GAS OXYGEN STAGE III (%)
4010	4	VE	XEXHGCO1	5	2	N	x	MAX EXHAUST GAS CARBON MONOXIDE STAGE 1 (X)
4020	4	VE	IEXHGCO1	5	2		x x	MIN EXHAUST GAS CARBON NONOXIDE STAGE I (X)
4020	2	VE	AEXHGCO1	5	2	N	X	AVG EXHAUST GAS CARBON MONOXIDE STAGE I (%)
	•			-	2	N	x	MAX EXHAUST GAS CARBON NONOXIDE STAGE II (%)
4040	4	VE VE	XEXHGCO2 IEXHGCO2	-	2	N	x x	MIN EXHAUST GAS CARBON MONOXIDE STAGE II (%)
4050					2	N	x	AVG EXHAUST GAS CARBON MONOXIDE STAGE II (%)
4060	4	VE	AEXHGCO2	-	2	N N	× X	MAX EXHAUST GAS CARBON MONOXIDE STAGE II (X)
4070	4	VE	XEXHGCO3	-	2	N N	X	MIN EXHAUST GAS CARBON MONOXIDE STAGE III (%)
4080	4	VE	IEXHGCO3	-		N N	* *	AVG EXHAUST GAS CARBON MONOXIDE STAGE III (%)
4090	4	VE	AEXHGCO3	-	2 0	••	A PPN	NAX EXHAUST GAS NOX STAGE II (PPM)
4100	4	VE	XEXHGNO2	-	=	X		
4110	4	VE	IEXHGNO2		0	H	PPN	MIN EXHAUST GAS NOX STAGE II (PPM)
4120	4	VE	AEXHGNO2		0	N	PPM	AVG EXHAUST GAS NOX STAGE II (PPM)
4130	5	VE	OPROCR	3	0	Z		REMARKS OR DEVIATIONS, LINE COUNT
4140	5	VE	ITEMHXXX		0	C		OPERATIONAL DATA ITEM
4150	5	VE	REMKHXXX		0	. C	-	OPERATIONAL DATA REMARKS OR DEVIATIONS
4160	6	VE	DWNOCR	3	0	-	#	DOWNTINE OCCURRENCES (#)
4170	6	VE	TYPEHXXX		0	C		SHUTDOWN TYPE
4180	6	VE	TESTHXXX		0	C	HRH:MM	TEST HOURS (HKH:MM)
4190	6	VE	STGEHXXX		0	¢		STAGE
4200	6	VE	DTIMHXXX	6	Q	C	NKX:MM	DOWNTIME TIME (HHH:MM)
4210	6	VE	LSTMHXXX	. 6	0	C	KHH:MM	LOST TIME (HHH:MN)
4220	6	VE	DREAHXXX	60	0	C		DOLINTIME PROBLEMS/ACTIONS/COMMENTS
4230	6	VÉ	TOTLDOWN	6	0	C	HHH:MM	DOWN TIME, TOTAL (HHH:NN)
4240	6	VE	TOTLLOST	6	Ô	C	NHH : MM	LOST TIME, TOTAL (HHH:HM)
4250	7	VE	FE_HXXX	6	0	N	рри	IRON HOURS (PPH)
4260	7	VE	CU_HXXX	6	0	N	PPH	COPPER HOURS (PPN)
4270	7	VE	\$I_Hxxx	6	0	N	PPH	SILICON HOURS (PPH)
4280	7	VE	VS40Hxxx	7	2	N	CST	VISCOSITY 2 40 °C HOURS (CST)
4290	7	VE	FUELHXXX	5	1	N	X MASS	FUEL DILUTION HOURS (X MASS)
4300	7	VE	PENTHXXX	5	1	N	X VOLUME	PENTANE INSOLUBLES HOURS (X VOLUME)
4310	8	VE	CANNO1	6	0	N	MICROMETERS	CAN LOBE WEAR POSITION 1-1E (NICROMETERS)
=	-						14.9 Data Diction	

FIG. A14.8 Data Dictionary

	_						,	
2-sep-199	97					_	Report: ASTH D	ata Dictionary
	_	Test	Field		Decimal			
Sequence	Form	Area	Name	Length	<u>512e</u>	Ivpe	<u>Units/Format</u>	Description
4320	8	VE	LOBPLG01	3	0	M	X VOLUME	LOBE ORIFICE PLUGGING POSITION 1-1E (X VOLUME)
4330	8	VE	ROCKW01	6	1		MG	ROCKER ARN WEAR POSITION 1-1E (MG)
4340	8	VE	VSPGLD01	6	1		LBF	VALVE SPRING LOAD POSITION 1-1E (LBF)
4350	8	VE	CAMW02	6	0	*	MICROMETERS	CAN LOBE WEAR POSITION 2-11 (NICROMETERS)
4360	8	VE	LOBPLG02	3	0		X VOLUME	LOBE ORIFICE PLUGGING POSITION 2-11 (MICROMETERS)
4370	8	VE	ROCKV02	6	1		NG	ROCKER ARN WEAR POSITION 2-11 (NG)
4380	8	VE	VSPGLD02	6	1		LBF	VALVE SPRING LOAD POSITION 2-11 (AG)
4390	8	VE	CANIJOZ	6	0			
4400	8	VE	LOBPLG03	3	0	N	NICROMETERS	CAN LOBE WEAR POSITION 3-2E (NICROMETERS) LOBE ORIFICE PLUGGING POSITION 3-2E (X VOLUME)
4410	8	VE	ROCKV03	6	1		NG	ROCKER ARN WEAR POSITION 3-2E (NG)
4420	8	VE	VSPGLD03	6	1	N	LBF	VALVE SPRING LOAD POSITION 3-2E (LBF)
4430	8	VE	CAMU04	6	0		NICRONETERS	CAN LOBE WEAR POSITION 4-21 (NICROMETERS)
4440	8	VE	LOBPLG04	3	0		X VOLUNE	LOBE ORIFICE PLUGGING POSITION 4-21 (MICHAGETERS)
4440	8	VE	ROCKV04	6	1		MG	ROCKER ARN WEAR POSITION 4-21 (NG)
4450	8	VE	VSPGLD04	6	1		LBF	VALVE SPRING LOAD POSITION 4-21 (HG)
4470	8	VE	CAMU05		0		MICROMETERS	CAN LOBE WEAR POSITION 5-3E (NICROMETERS)
4470	8	VE	LOBPLG05	6 3	0	X	X VOLUNE	
4480	8	VE	ROCKU05	6	1	i i i i i i i i i i i i i i i i i i i	NG	LOBE ORIFICE PLUGGING POSITION 5-3E (% VOLUME) Rocker Arm Mear Position 5-3E (Mg)
4500	8	VE	VSPGLD05	-	1		LBF	VALVE SPRING LOAD POSITION 5-3E (HG)
4510	8	VE	CANU06	6	0	N	MICROMETERS	CAN LOBE WEAR POSITION 6-31 (NICRONETERS)
4520	8	VE	LOBPLGOG	3	0	N	X VOLUNE	LOBE ORIFICE PLUGGING POSITION 6-31 (X VOLUME)
4520	8	VE	ROCKW06	6	1		NG	
4540	8			=	-			ROCKER ARN WEAR POSITION 6-31 (NG)
	-	VE	VSPGLD06	6	1	N	LBF	VALVE SPRING LOAD POSITION 6-31 (LBF)
4550 4560	8	VE	CANW07	6	0 0	N N	NICROMETERS	CAN LOBE WEAR POSITION 7-4E (MICROMETERS)
4570	8	VE	LOBPLG07	3	-		X VOLUME	LOBE ORIFICE PLUGGING POSITION 7-4E (X VOLUME)
	8	VE	ROCKW07	6	1	N	MG	ROCKER ARN WEAR POSITION 7-4E (NG)
4580	8	VE	VSPGLD07	6	1	N	LBF	VALVE SPRING LOAD POSITION 7-4E (LBF)
4590	8	VE	CAMW08	6	0	N	NICROMETERS	CAN LOBE WEAR POSITION 8-41 (NICROMETERS)
4600	8	VE	LOBPLG08	3	0		X VOLUME	LOBE ORIFICE PLUGGING POSITION 8-41 (% VOLUME)
4610	8	VE	ROCKW08	6	1	N	MG	ROCKER ARN WEAR POSITION 8-41 (MG)
4620	8	VE	VSPGLD08	6	1	N	LBF	VALVE SPRING LOAD POSITION 8-41 (LBF)
4630	8	VE	AVGLOBPL	3	0		X VOLUME	LOBE ORIFICE PLUGGING POSITION AVG (% VOLUME)
4640	8	VE	AVGSPGLO	6	1		LBF	VALVE SPRING LOAD POSITION AVG (LBF)
4650	8	VE	LOBHRD01	5	1		ROCKWELL C	LOBE HARDNESS 1E->50 ROCKWELL C- (ROCKWELL C)
4660	8	VE	RCKHRD01	-	•			ROCKER ARN HARDNESS 18->57 ROCKWELL C- (ROCKWELL C)
4670	8	VE	LOBHRD02	5	1		ROCKWELL C	LOBE HARDNESS 11->50 ROCKWELL C- (ROCKWELL C)
4680	8	VĒ	RCKHRDOZ	5	1		ROCKWELL C	ROCKER ARN HARDNESS 11->57 ROCKWELL C- (ROCKWELL C)
4690	8	VĒ	LOBHRD03	5	1		ROCKWELL C	LOBE NARDNESS 2E->50 ROCKWELL C- (ROCKWELL C)
4700	8	VE	RCKHRD03	5	1		ROCKWELL C	ROCKER ARM HARDNESS 2E->57 ROCKWELL C- (ROCKWELL C)
4710	8	VE	LOBHRD04	5	1		ROCKWELL C	LOBE HARDNESS 21->50 ROCKWELL C- (ROCKWELL C)
4720	8	VE	RCKHRD04	5	1	N	ROCKWELL C	ROCKER ARN HARDNESS 21->57 ROCKWELL C- (ROCKWELL C)
4730	8	VE	LOBHRD05	5	1		ROCKWELL C	LOBE HARDNESS 3E->50 ROCKWELL C- (ROCKWELL C)
4740	8	VE	RCKHRD05	5	1		ROCKWELL C	ROCKER ARM HARDNESS 3E->57 ROCKWELL C- (ROCKWELL C)
4750	8	YE	LOBHRD06	5	1		ROCKVELL C	LOBE HARDNESS 31->50 ROCKWELL C- (ROCKWELL C)
4760	8	VE	RCKHRD06	5	1		ROCKWELL C	ROCKER ARN HARDNESS 31->57 ROCKWELL C- (ROCKWELL C)
4770	8	VE	LOBHRD07	5	1		ROCKWELL C	LOBE HARDNESS 4E->50 ROCKWELL C- (ROCKWELL C)
4780	8	VE	RCKHRD07	5	1		ROCKWELL C	ROCKER ARN HARDNESS 4E->57 ROCKWELL C- (ROCKWELL C)
4790	8	VE	LOBHRDDB	5	1		ROCKWELL C	LOBE HARDNESS 41->50 ROCKWELL C- (ROCKWELL C)
4800	8	VE	RCKHRD08	5	1		ROCKWELL C	ROCKER ARM HARDNESS 41->57 ROCKWELL C- (ROCKWELL C)
4810	9	VE	PSVTH1	4	1		MERITS	PISTON SKIRT VARNISH THRUST NUN 1 (MERITS)
4820	9	VE	PSVAT1	4	1		MERITS	PISTON SKIRT VARNISH ANTITHRUST NUM 1 (MERITS)
4830	9	VE	PSVTHŻ	4	1		MERITS	PISTON SKIRT VARNISH THRUST NUM 2 (MERITS)
4840	9	VE	PSVAT2	4	1		MERITS	PISTON SKIRT VARNISH ANTITHRUST NUM 2 (MERITS)
4850	9	VË	PSVTH3	4	1	N	MERITS	PISTON SKIRT VARNISH THRUST NUN 3 (MERITS)
						FIG.	A14.9 Data Dicti	onarv

FIG. A14.9 Data Dictionary

2-sep-199						Report: ASTN Data Dictionary			
•		Test	Field	Field	Decimal	Data	•		
Sequence	Form	Area	<u>Name</u>	Length	<u>Size</u>		<u>Units/Format</u>	Description	
4860	9	VE	PSVAT3	4	1	N	MERITS	PISTON SKIRT VARNISH ANTITHRUST NUM 3 (MERITS)	
4870	9	VE	PSVTH4	4	1	N	MERITS	PISTON SKIRT VARNISH THRUST NUM 4 (MERITS)	
4880	9	VE	PSVAT4	4	1	N	MERITS	PISTON SKIRT VARNISH ANTITHRUST NUM 4 (MERITS)	
4890	9	VE	PSVTHAVG	5	2	N	MERITS	PISTON SKIRT VARNISH THRUST AVG (MERITS)	
4900	9	VE	PSVATAVG	5	2	N	MERITS	PISTON SKIRT VARNISH ANTITHRUST AVG (MERITS)	
4910	9	VE	CLD1	5	2	N	MERITS	PISTON NUMBER 1 UNDERCROWN (MERITS)	
4920	9	VE	SECLD1	5	2	N	MERITS	PISTON NUMBER 1 2ND LAND (MERITS)	
4930	9	VE	ORLD1	5	2	N	MERITS	PISTON NUMBER 1 OIL RING LAND (MERITS)	
4940	9	VE	PISUCI	5	2	N	MERITS	PISTON NUMBER 1 UNDERCROWN (MERITS)	
4950	9	VE	CLD2	5	2	N	MERITS	PISTON NUMBER 2 CROWN LAND (MERITS)	
4960	9	VE	SECLD2	5	2	N	MERITS	PISTON NUMBER 2 2ND LAND (MERITS)	
4970	9	VE	ORLD2	5	2	N	MERITS	PISTON NUMBER 2 OIL RING LAND (MERITS)	
4980	9	VE	P1SUC2	5	2	N	MERITS	PISTON NUMBER 2 UNDERCROWN (MERITS)	
4990	9	VE	CLD3	5	2	N	MERITS	PISTON NUMBER 3 CROWN LAND (MERITS)	
5000	9	VE	SECLD3	5	2	N	MERITS	PISTON NUMBER 3 2ND LAND (MERITS)	
5010	9	VE	ORLD3	5	2	N	MERITS	PISTON NUMBER 3 OIL RING LAND (MERITS)	
5020	9	VE	PISUC3	5	2	N	MERITS	PISTON NUMBER 3 UNDERCROWN (MERITS)	
5030	9	VE	CLD4	5	2	N	MERITS	PISTON NUMBER 4 CROWN LAND (MERITS)	
5040	9	VE	SECLD4	5	2	N	MERITS	PISTON NUMBER 4 2ND LAND (MERITS)	
5050	9	VE	ORLD4	5	2	N	MERITS	PISTON NUMBER 4 OIL RING LAND (MERITS)	
5060	9	VE	PISUC4	5	2	N	MERITS	PISTON NUMBER 4 UNDERCROWN (MERITS)	
5070	10	VĒ	BLOWBYIN	70	0	C		CORRECTED BLOWBY FLOWRATE PLOT	
5080	11	VE	OILCHXXX	3	0	N	FL. OZ.	OIL CONSUMPTION CYCLE XXX (FL. 0Z.)	
5090	11	VĒ	OILLHXXX	3	Q	N	FL. 02.	OIL LEVEL CYCLE XXX (FL. QZ.)	
5100	12	VE	CANOGN	4	0	N	YYYY	CANSHAFT ORIGIN (YYYY)	
5110	12	VE	CRODBOGN	4	0	N	YYYY	CONNECTING ROD BEARINGS ORIGIN (YYYY)	
5120	12	VE	CYLBKOGN	4	0	N	YYYY	CYLINDER BLOCK ORIGIN (YYYY)	
5130	12	VE	CYLHDOGN	4	0	N	YYYY	CYLINDER HEAD ORIGIN (YYYY)	
5140	12	VE	PISTOGN	4	0		YYYY	PISTONS ORIGIN (YYYY)	
5150	12	VË	PISRGOGN	4	0	N	YYYY	PISTON RINGS ORIGIN (YYYY)	
5160	12	VE	RCKRAOGN	4	Ó	N	YYYY	ROCKER ARNS ORIGIN (YYYY)	
5170	13	VË	OILISENS	14	0	Ċ		OIL INLET SENSING DEVICE	
5180	13	VË	OILICALF	14	0	C		OIL INLET CALIBRATION FREQUENCY	
5190	13	VE	OILIRECD	3	Û	C		OIL INLET RECORD DEVICE	
5200	13	VE	OILIOBSF	12	0	Ċ		OIL INLET OBSERVATION FREQUENCY	
5210	13	VE	OILIRECF	12	0	C		OIL INLET RECORD FREQUENCY	
5220	13	VE	OILILOGF	12	0	C		OIL INLET LOG FREQUENCY	
5230	13	VE	RCCISENS	14	0	C		RAC INLET SENSING DEVICE	
5240	13	VE	RCCICALF		0	C		RAC INLET CALIBRATION FREQUENCY	
5250	13	VE	RCCIRECD	3	Ô	C		RAC INLET RECORD DEVICE	
5260	13	VE	RCCIOBSF		0	C		RAC INLET OBSERVATION FREQUENCY	
5270	13	VE	RCCIRECF		0	C		RAC INLET RECORD FREQUENCY	
5280	13	VE	RCCILOGF		0	C		RAC INLET LOG FREQUENCY	
5290	13	VE	COTSENS	14	0	C		COOLANT OUT TEMPERATURE SENSING DEVICE	
5300	13	VE	COTCALF	14	0	C		COOLANT OUT TEMPERATURE CALIBRATION FREQUENCY	
5310	13	VE	COTRECD	3	0	C		COOLANT OUT TEMPERTURE RECORD DEVICE	
5320	13	VE	COTOBSF	12	0	C		COOLANT OUT TEMPERATURE OBSERVATION FREQUENCY	
5330	13	VE	COTRECF	12	0	C		COOLANT OUT TEMPERATURE RECORD FREQUENCY	
5340	13	VE	COTLOGF	12	0	Ċ		COOLANT OUT TEMPERATURE LOG FREQUENCY	
\$350	13	VE	<b>RPMSENS</b>	14	Û	C		ENGINE SPEED SENSING DEVICE	
5360	13	VĒ	RPMCALF	14	0	C		ENGINE SPEED CALIBRATION FREQUENCY	
5370	13	VE	RPMRECD	3	0	C		ENGINE SPEED RECORD DEVICE	
5380	13	VE	RPHOBSF	12	0	C		ENGINE SPEED OBSERVATION FREQUENCY	
5390	13	VE	RPMRECF	12	0	C		ENGINE SPEED RECORD FREQUENCY	
					FI	G. A14	4.10 Data Diction	arv	

FIG. A14.10 Data Dictionary

					्रा	11× -	••••	
2-sep-199	7						Report: ASTM Dat	ta Dictionary
		Test	Field	Field	Decimal	Data		
Sequence	<u>Form</u>	Area	<u>Name</u>	Length	<u> șize</u>	Type	<u>Units/Format</u>	Description
5400	13	VE	RPHLOGF	12	0	C		ENGINE SPEED LOG FREQUENCY
5410	13	VE	RPMSYSR	8	0	¢		ENGINE SPEED SYSTEM RESPONSE
5420	13	VE	PWRSENS	14	0	C		POWER SENSING DEVICE
5430	13	VE	PWRCALF	14	0	C		POWER CALIBRATION FREQUENCY
5440	13	VE	PWRRECD	3	0	C		POWER RECORD DEVICE
5450	13	VE	PWROBSF	12	Ó	C		POWER OBSERVATION FREQUENCY
5460	13	VĒ	PURRECF	12	0	C		POWER RECORD FREQUENCY
5470	13	VE	PURLOGF	12	Ō	Ċ		POVER LOG FREQUENCY
5480	13	VE	PWRSYSR	8	Ō	C		POWER SYSTEM RESPONSE
5490	13	VE	COSENS	14	Ō	C		CARBON NONOXIDE SENSING DEVICE
5500	13	VE	COCALF	14	0	c		CARBON NONOXIDE CALIBRATION FREQUENCY
5510	13	VE	CORECD	3	0	c		CARBON NONOXIDE RECORD DEVICE
5520	13	VE	COORSE	12	0	č		CARBON MONOXIDE OBSERVATION FREQUENCY
5530	13	VE	CORECF	12	0	c		CARBON MONOXIDE RECORD FREQUENCY
5540	13	VE	COLOGF	12	0	č		CARBON MONOXIDE LOG FREQUENCY
5550	13	VE	COSYSR	8	ů ů	c		CARBON MONOXIDE SYSTEM RESPONSE
	13	VE VE	OZSENS	14	0	c		OXYGEN SENSING DEVICE
5560	13	. –			0	c		OXYGEN CALIBRATION FREQUENCY
5570		VE	O2CALF	14 3	0	с С		OXYGEN RECORD DEVICE
5580	13	VE	OZRECD	_	0			DXYGEN DESERVATION FREQUENCY
5590	13	VE	0208SF	12	-	C		OXYGEN RECORD FREqUENCY
5600	13	VE	OZRECF	12	0	C		
5610	13	VE	OZLOGF	12	0	C		OXYGEN LOG FREQUENCY
5620	13	VE	02SYSR	8	0	C		OXYGEN SYSTEM RESPONSE
5630	13	VE	TIMGSENS		0	C		TIMING SENSING DEVICE
5640	13	VE	TINGCALF		0	C		TINING CALIBRATION FREQUENCY
5650	13	VE	TINGRECD		0	C		TIMING RECORD DEVICE
5660	13	VE	TINGOBSF		0	C		TIMING OBSERVATION FREQUENCY
5670	13	VE	TIMGRECF		0	C		TINING RECORD FREQUENCY
5680	13	VĒ	TIMGLOGF		0	C		TINING LOG FREQUENCY
5690	13	VE	BLWGSENS	14	0	C		BLOWBY GAS SENSING DEVICE
5700	13	VE	BLVGCALF		0	C		BLOWBY GAS CALIBRATION FREQUENCY
5710	13	VE	BLWGRECD	3	0	C		BLOWBY GAS RECORD DEVICE
5720	13	VE	BLWGOBSF	12	0	C		BLOWBY GAS OBSERVATION FREQUENCY
5730	13	VE	BLWGRECF	12	0	C		BLOWBY GAS RECORD FREQUENCY
5740	13	VE	BLUGLOGF	12	0	C		BLOWBY GAS LOG FREQUENCY
5750	13	VE	EXPRSENS	14	0	C		EXHAUST PRESSURE SENSING DEVICE
5760	13	VE	EXPRCALF	14	0	C		EXHAUST PRESSURE CALIBRATION FREQUENCY
5770	13	VE	EXPRRECD	3	0	C		EXHAUST PRESSURE RECORD DEVICE
5780	13	VE	EXPROBSE	12	0	<b>C</b> -		EXHAUST PRESSURE OBSERVATION FREQUENCY
5790	13	VE	EXPRRECF	12	0	C		EXHAUST PRESSURE RECORD FREQUENCY
5800	13	VE	EXPRLOGE	12	0	C		EXHAUST PRESSURE LOG FREQUENCY
5810	13	VE	CFLWSENS	14	0	C		ENGINE COOLANT FLOW SENSING DEVICE
5820	13	VE	CFLUCALF	14	0	C		ENGINE COOLANT FLOW CALIBRATION FREQUENCY
5830	13	VE	CFLWRECD	3	0	C		ENGINE COOLANT FLOW RECORD DEVICE
5840	13	VE	CFLWOBSF	12	0	C		ENGINE COOLANT FLOW OBSERVATION FREQUENCY
5850	13	VE	CFLWRECF		Ó	C		ENGINE COOLANT FLOW RECORD FREQUENCY
5860	13	VE	CFLWLOGF		0	C		ENGINE COOLANT FLOW LOG FREQUENCY
5870	13	VE	RCCFSENS		0	C		ROCKER ARN FLOW SENSING DEVICE
5880	13	VE	RCCFCALF		0	c		ROCKER ARM FLOW CALIBRATION FREQUENCY
5890	13	VE	RCCFRECD	_	0	c		ROCKER ARM FLOW RECORD DEVICE
	13	VE	RCCFOBSF		ő	c		ROCKER ARM FLOW OBSERVATION FREQUENCY
5900 5010	13	VE	RCCFRECF		ŏ	c		ROCKER ARM FLOW RECORD FREquency
5910 5030					0	C		ROCKER ARM FLOW LOG FREQUENCY
5920	13	VE	RCCFLOGF		0	C		ROCKER COVER AND CAMSHAFT BAFFLE IMAGES
5930	14	VE	RC_CSBIM	~~	-	_	1 Data Dictionary	
					E1(4	<u>ы 14 1'</u>	L DATA DICTIONS/V	

FIG. A14.11 Data Dictionary

2-sep-1997				Report: ASTN Data Dictionary				
		Test	Field		Decimal			
Sequence	<u>form</u>	<u>Area</u>	Name	<u>Length</u>	<u>Size</u>	Туре	<u>Units/Format</u>	Description
5940	15	VE	OP_OSIN	70	0	C		OIL PAN AND OIL SCREEN IMAGES
5950	16	VE	PISTONIN		Ō	C		PISTON SKIRT INAGES
5960	17	VE	CANLOBIN	70	0	C		CAMSHAFT LOBE IMAGES
5970	18	VE	FSH_RAIM	70	0	C		FRONT SEAL HOUSING AND ROCKER ARN IMAGES
5980	19	VE	TRANSIIN	70	0	C		SAMPLE TRANSITION PLOT IMAGE
5990	20	VE	TPRNGGP1	5	3	N	IN	TOP RING GAP, CYLINDER 1 (IN)
6000	20	VE	N2RNGGP1	5	3	N	IN	SECOND RING GAP, CYLINDER 1 (IN)
6010	20	VE	TPRLCLR1	6	4	N	IN	TOP RING-LAND SIDE CLEARANCE, CYLINDER 1 (IN)
6020	20	VE	N2RLCLR1	6	4	N	IN	SECOND RING-LAND SIDE CLEARANCE, CYLINDER 1 (IN)
6030	20	VE	TPRNGGP2		3	N	IN	TOP RING GAP, CYLINDER 2 (IN)
6040	20	VE	N2RNGGP2		3	N	IN	SECOND RING GAP, CYLINDER 2 (IN)
6050	20	VE	TPRLCLR2		4	H	IN	TOP RING-LAND SIDE CLEARANCE, CYLINDER 2 (IN)
<b>6060</b>	20	VE	N2RLCLR2		4	N	IN	SECOND RING-LAND SIDE CLEARANCE, CYLINDER 2 (IN)
6070 6080	20 20	VE VE	TPRNGGP3	_	3 3	N M	IN IN	TOP RING GAP, CYLINDER 3 (IN) Second Ring GAP, Cylinder 3 (IN)
6090	20	VE	N2RNGGP3 TPRLCLR3		4	N	IN	TOP RING-LAND SIDE CLEARANCE, CYLINDER 3 (IN)
6100	20	VE	N2RLCLR3		4	N	IN	SECOND RING-LAND SIDE CLEARANCE, CYLINDER 3 (IN)
6110	20	VE	TPRNGGP4		3		1N	TOP RING GAP, CYLINDER 4 (IN)
6120	20	VE	N2RNGGP4	5	3	N	IN	SECOND RING GAP, CYLINDER 4 (IN)
6130	20	VE	TPRLCLR4	6	4	Ň	IN	TOP RING-LAND SIDE CLEARANCE, CYLINDER 4 (IN)
6140	20	VE	NZRLCLR4	6	4	N	11	SECOND RING-LAND SIDE CLEARANCE, CYLINDER 4 (IN)
6150	20	VE	GAPHRA	3	0	C	HOURS	TEST HOUR WHEN REGAP OCCURS A (HOURS)
6160	20	VE	GAPHRB	3	0	C	HOURS	TEST HOUR WHEN REGAP OCCURS B (HOURS)
6170	20	VE	TPRGGP1A	5	3	N	IN	TOP RING GAP & XXX HRS, CYLINDER 1A (IN)
6180	20	VE	TPRGRW1A	5	3	N	IN	TOP RING GAP AFTER REWORK, CYLINDER 1A (IN)
6190	20	VE	TPRGGP18	5	3	N	IN	TOP RING GAP @ XXX HRS, CYLINDER 1B (IN)
6200	20	VE	TPRGRW1B	5	3	N	IN	TOP RING GAP AFTER REWORK, CYLINDER 1B (IN)
6210	20	VE	TPRGINC1	5	3	N	IN	TOP RING GAP INCREASE DUE WEAR, CYLINDER 1 (IN)
6220	20	VĒ	TPRGGP2A	5	3	N	IN	TOP RING GAP & XXX HRS, CYLINDER 2A (IN)
6230	20	VE	TPRGRW2A	5	3	N	IN	TOP RING GAP AFTER REWORK, CYLINDER 2A (IN)
6240	20	VE	TPRGGP2B	5	3	N	IN	TOP RING GAP & XXX HRS, CYLINDER 2B (IN)
6250	20	VE	TPRGRW2B	5	3	N	IN	TOP RING GAP AFTER REWORK, CYLINDER 28 (IN)
6260	20	VE	TPRGINC2	5	3	N	IN	TOP RING GAP INCREASE DUE WEAR, CYLINDER 2 (IN)
6270	20	VE	TPRGGP3A	5 5	3 3	N N	IN	TOP RING GAP B XXX HRS, CYLINDER 3A (IN)
6280	20	VE	TPRGRW3A	=	3	N	IN IN	TOP RING GAP AFTER REWORK, CYLINDER 3A (IN) TOP RING GAP & XXX HRS, CYLINDER 3B (IN)
6290 6300	20 20	VE VE	TPRGGP38 TPRGRW38	5 5	3	N	IN	TOP RING GAP AFTER REWORK, CYLINDER 38 (IN)
6310	20	VE	TPRGRUSS TPRGINC3	-	3	N	IN	TOP RING GAP INCREASE DUE WEAR, CYLINDER 3 (IN)
6320	20	VE	TPRGGP4A		3	Ň	IN	TOP RING GAP & XXX HRS, CYLINDER 4A (IN)
6330	20	VE	TPRGRW4A	5	3	Ň	IN	TOP RING GAP AFTER REWORK, CYLINDER 4A (IN)
6340	20	VE	TPRGGP48	5	3	N	IN	TOP RING GAP & XXX HRS, CYLINDER 48 (IN)
6350	20	VE	TPRGRW4B	5	3	N	IN	TOP RING GAP AFTER REWORK, CYLINDER 48 (IN)
6360	20	VE	TPRGINC4	5	3	N	IN	TOP RING GAP INCREASE DUE WEAR, CYLINDER 4 (IN)
6370	20	VE	BWCYL1T	6	1	N	HILS	BORE WEAR CYLINDER 1 TOP (MILS)
6380	20	VE	BWCYL 1N	6	1	N	MILS	BORE WEAR CYLINDER 1 NIDDLE (NILS)
6390	20	VE	BWCYL1B	6	1	N	MILS	BORE WEAR CYLINDER 1 BOTTON (MILS)
6400	20	VE	BUCYL1A	6	1	N	NILS	BORE WEAR CYLINDER 1 AVERAGE (MILS)
6410	20	VË	BWCYL2T	6	1	N	MILS	BORE WEAR CYLINDER 2 TOP (NILS)
6420	20	VE	BWCYLŻN	6	1	N	NILS	BORE WEAR CYLINDER 2 MIDDLE (MILS)
6430	20	VE	BWCYL2B	6	1		MILS	BORE WEAR CYLINDER 2 BOTTON (HILS)
6440	20	VE	BWCYL2A	6	1		MILS	BORE WEAR CYLINDER 2 AVERAGE (NILS)
6450	20	VE	BWCYL3T	6	1		MILS	BORE WEAR CYLINDER 3 TOP (HILS)
6460	20	VE	BWCYL3M	6	1		MILS	BORE WEAR CYLINDER 3 MIDDLE (MILS)
6470	20	VE	BWCYL3B	6	1		NILS	BORE WEAR CYLINDER 3 BOTTON (MILS)
					F	·IG. A	14.12 Data Dictio	onary

FIG. A14.12 Data Dictionary

2-sep-1997							Report: ASTN Data Dictionary		
		Test	field	Field	Decimal	Data			
Sequence	<u>Form</u>	Area	Name	<u>Length</u>	Size		<u>Units/Format</u>	Description	
6480	20	VE	BWCYL3A	6	1	N	MILS	BORE WEAR CYLINDER 3 AVERAGE (MILS)	
6490	20	VE	BWCYL4T	6	1	N	MILS	BORE WEAR CYLINDER 4 TOP (MILS)	
6500	20	VE	BWCYL4M	6	1	N	MILS	BORE WEAR CYLINDER 4 MIDDLE (MILS)	
6510	20	VE	BWCYL4B	6	1	N	MILS	BORE WEAR CYLINDER 4 BOTTOM (MILS)	
6520	20	VE	SWCYL4A	6	1	N	MILS	BORE WEAR CYLINDER 4 AVERAGE (MILS)	
6530	20	VE	CYLBWAVG	6	1	N	MILS	BORE MEAR CYLINDER AVERAGE (MILS)	
6540	20	VE	CYLBUMAX	6	1	N	MILS	BORE WEAR CYLINDER MAXIMUM (MILS)	
6550	21	VE	OILINDP	5	1	N	X	ENGINE OIL INLET TEMP CAL DEV PERCENT (PERCENT)	
6560	21	VE	COLOUTDP	5	1	N	<b>X</b>	ENGINE COOLANT OUTLET TEMP CAL DEV (PERCENT)	
6570	21	VE	TIMNGDP	5	1	N	X	TINING CAL DEV (PERCENT)	
6580	21	VE	EX0212DP	5	1	N	X	EXHAUST GAS OXYGEN-STAGES 122- CAL DEV (PERCENT)	
6590	21	VE	EXCO3DP	5	1	N	X	EXHAUST GAS CARBON NONOXIDE-STAGE 3- (PERCENT)	
6600	21	VE	COLFLODP	5	1	N	X	ENGINE COOLANT FLOW CAL DEV (PERCENT)	
6610	21	VE	RACINDP	5	1	N	*	RAC INLET TEMP CAL DEV (PERCENT)	
6620	21	VE	RPMDP	5	1	N	x	ENGINE SPEED CAL DEV (PERCENT)	
6630	21	VE	PURDP	5	1	N	*	POWER CAL DEV (PERCENT)	
6640	21	VE	RACCOLDP	5	1	N	x	RAC COOLANT FLOW RATE CAL DEV (PERCENT)	
6650	21	VE	EXBK12DP	5	1	N	x	EXHAUST BACKPRESSURE-STAGES 122- CAL DEV (PERCENT)	
						FIG.	A14.13 Data Dicti	ionary	

FIG. A14.13 Data Dictionary

ŧ ŧ Data Dictionary Repeating Field Specifications ŧ ŧ ŧ ŧ ŧ ŧ # The following contains specifications and field groupings for fields in the # Data Dictionary that are REPEATING Fields. These fields can be identified # in the Data Dictionary by the Hxxx or Rxxx in the last four positions of the # field name. # Repeating fields are used to specify repeating measurements. # The format for a repeating field name is 4 descriptive characters followed by the letter H or R followed by 3 characters for the actual interval ŧ the measurement was taken. The field will always be a total of 8 characters. ŧ 1 Example ABCDHxxx. ŧ The following is the format of this specification: ŧ Repeating Field Name # Column 1 - 8: # Column 10 - 17: The Parent Field Name of the Group # Column 19 - 80: Comments about the Repeating Field Group. # # The lines following the Repeating Field Name Record will contain the required # measurements for the particular field. Multiple 80 characters lines # can be specified. A blank line marks the end of each specification. # The Field Name in Column 10-17 designates the the Group in which the field # belongs. The First field name in a group is the Parent of the grouping # and can be used to determine how fields should be grouped. # The changing of the Parent Field marks the end of a repeating group # specification. ŧ # Example: Ŧ # VIS Hxxx, DVISHxxx and PVISHxxx expanded for transmission (8 and 16 hours): ŧ ŧ VIS HOO8 ŧ DVISH008 ŧ PVISH008 VIS H016 **#** # DVISH016 ŧ PVISH016 Ŧ During electronic transmission, repeating field groups must be kept ŧ Note: together with in the specified group but the order with in the group ŧ ŧ does not have to be maintained. Start of Field Grouping Specifications **VE VERSION 19970902** ITEMHXXX ITEMHXXX OPERATIONAL DATA ITEM

REMKHXXX ITEMHXXX OPERATIONAL DATA REMARKS OR DEVIATIONS

FIG. A14.14 Repeating Field Specifications

	<b>↓</b> D 5302 – 01a
ТҮРЕНХХХ ТҮРЕНХХХ	SHUTDOWN TYPE
TESTHXXX TYPEHXXX	TEST HOURS (HHH:MM)
STGEHXXX TYPEHXXX	STAGE
DTIMHXXX TYPEHXXX	DOWNTIME TIME (HHH:MM)
LSTMHXXX TYPEHXXX	LOST TIME (HHH:MM)
DREAHXXX TYPEHXXX	DOWNTIME PROBLEMS/ACTIONS/COMMENTS
FE <u>HXXX</u> FE <u>HXXX</u> 012 108 204 288 NEW	IRON HOURS (PPM)
CU HXXX FE HXXX 012 108 204 288 NEW	COPPER HOURS (PPM)
SI_HXXX FE_HXXX 012 108 204 288 NEW	SILICON HOURS (PPM)
VS40Hxxx FE Hxxx 012 108 204 288 NEW	VISCOSITY @ 40 DEG C HOURS (CST)
FUELHXXX FE <u>H</u> XXX 012 108 204 288	FUEL DILUTION HOURS (* MASS)
PENTHXXX FEHXXX 108 204 288	PENTANE INSOLUBLES HOURS (% VOLUME)
	OIL CONSUMPTION CYCLE XXX (FL. OZ.) 036 042 048 054 060 066 072
	OIL LEVEL CYCLE XXX (FL. OZ.) O36 O42 O48 O54 O60 O66 O72 FIG. A14.15 Repeating Field Specifications

* preceed the data. The latest version of this Header Data Dictionary can	k k
* ************************************	R R

### A15. ONE-HALF MILLIMETRE OVERSIZE PISTON MODIFICATIONS

A15.1 See Fig. A15.1.

### Sequence VE Second Run Piston Modifications

Piston Lands and Rings Drawn to Scale

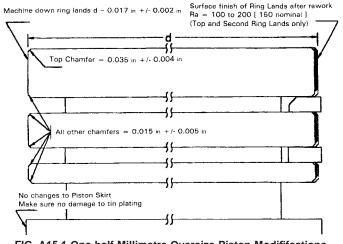


FIG. A15.1 One-half Millimetre Oversize Piston Modififcations

### A16. REQUIRED INSPECTIONS AND OPERATIONS FOR REUSE OF CYLINDER HEADS

A16.1 Cylinder heads may be reused more than once, provided the head remains within the specifications in Table A16.1. Replacement part numbers and part number for the suggested cam bearing tool are listed in Table A16.1.

### A16.2 Cleaning Process:

A16.2.1 Remove the cam bearing galley plugs.

A16.2.2 Soak the cylinder head in agitated organic solvent (see 7.7.3), or other approved solvent, until all sludge is dissolved. As an alternative, the head may be cleaned, using a caustic cleaning solution followed by a 1-h soak in agitated organic solvent.

A16.2.3 Rinse the cylinder head in tri-sodium phosphate/ hot water bath.

A16.2.4 Spray the cylinder head with a solution of aliphatic naphtha and 50 % Mobile EF-411.

A16.2.5 Dry the cylinder head with forced air.

A16.2.6 Check the cylinder head for flatness (no machining allowed). Scrap any cylinder head that does not meet the flatness requirements listed in Table A16.1.

A16.2.7 Measure the diameter of the valve guide bores. The valve guides may be reamed, or bored, to use 0.0015-in. diameter oversize Ford valves, if necessary, to maintain specified stem to guide clearance. Scrap any cylinder heads that do not meet the valve guide diameter clearances specified in Table A16.1 with standard or oversize valve stems.

### A16.3 *Rebuilding Method*:

A16.3.1 Remove any remaining combustion chamber deposits by using a rotary wire brush.

A16.3.2 Recondition the valve seats. Lapping the valve seats is the preferred method of valve seat reconditioning. Lapping results in an insignificant amount of metal being removed, thus having little if any effect on the installed valve

## TABLE A16.1 Sequence VE Cylinder Head Specifications and Required Part Numbers

Specifications	
Combustion chamber volume	59.8-62.8 cm ³
Valve guide bore diameter	0.3433-0.3443 in. (8.72-8.75 mm)
Valve stem to guide clearance:	
Intake	0.0014-0.0027 in. (0.035-0.069 mm)
Exhaust	0.0019-0.0032 in. (0.048-0.081 mm)
Valve springs:	
Free length	1.9-2.0 in. (48.3-50.8 mm)
Out-of-square	0.075-in. max (1.91 mm)
Compression force	167 $\pm$ 8 lbf at 1.16 $\pm$ 0.03 in. (743 $\pm$ 36 N at 29.5 $\pm$ 0.76 mm)
Installed height	1.53-1.59 in. (38.86-40.39 mm)
Intake and exhaust valve seat angle	45°
Intake valve seat width	0.060-0.080 in. (1.52-2.03 mm)
Exhaust valve seat width	0.070-0.090 in. (1.78-2.29 mm)
Seat runout limit (T.I.R. max)	0.0016 in. (0.041 mm)
Flatness, cylinder head deck (no machining allowed)	0.006 in. max (0.15 mm)
Ford part numbers:	
Cam bearings	D4FZ-6261-C
Camshaft oil seal	E8ZZ-6700-A
Intake valve	E8ZZ-6507-A
Intake valve (0.0015 in. oversize)	E8ZZ-6507-B
Exhaust valve	E43Z-6505-A
Exhaust valve (0.0015 in. oversize)	E43Z-6505-B
Cam bearing tool (suggested)	T71P-6250-A
Expander screw (not supplied with tool)	T65L-6250-A13

spring height (see 7.8.8.4). Seats may be ground, providing the installed valve spring height and combustion chamber volume comply with the values specified in Table A16.1.

A16.3.3 Clean with aliphatic naphtha spray, and use bristle brushes on the combustion chambers and valve deck and through the valve guides. Final spray with aliphatic naphtha containing 50 % Mobile EF-411.

A16.3.4 Dry with forced air.

A16.3.5 Install new cam bearings. Any method of installation may be used that ensures free movement of the camshaft.

A16.3.6 Install new camshaft oil seal.

A16.3.7 Assemble the cylinder head in accordance with 7.8.6 of this test method. Use new intake and exhaust valves, and lubricate all parts with Mobil EF-411.



## APPENDIXES

## (Nonmandatory Information)

## **X1. ENGINE MEASUREMENT SHEETS**

X1.1 See Figs. X1.1-X1.6.

pages are recommended to be utilized in accordance with the requirements of 7.8.6-7.8.15.

NOTE X1.1-The data sheets shown sequentially on the following

		Engine No Engine Build No Date Technician					
Cylinder Number	Location	Longitudinal Diameter (LD), in.	Transverse Diameter (TD), in.	Inder Bore Messuremer Average Bore Diameter (ABD), in, <u>TD # mid + TD # bot</u> 2	8ore Out of Round, in. [LD - TD] (0.0010) max)	Bore Taper, in. (max Δ) (0.001 max)	Microfinish, µin.
	Top						
1	Middle		L i				
	Bottom						
	Top						
2	Middle					l	
	Settom					1	1
	Тор						
3	Middle						
Botte	Bottern	Ī					1
	Tep						
	Middle				I.,		
	Bottom		T I				

#### Piston Measurements

Cylinder Number	Range of Acceptable Fiston Diameter, in. (ASD = 0.0014 to ASD = 0.0022 in.)	Diameter of Selected Piston, in,	Taper of Selected Piston, in.
1			
1			
3			
4			

#### Compression Ring Gap Measurements

Cylinder Number	Top Ring Gap, in.	Second Ring Gap, in.	Top Ring-Land Side Clearance, in.	Second Ring-Land Side Clearance, in.
1				
2				
3				
4				1

### Compression Ring Regap History and Wear Date

Cylinder   Number	Ring Location	Aing Gap , hm, in.	Ring Gap After Rework, in.	Ring Gap 	Ring Gap After Rework, in,	Ring Gap increase Due to Wear, in.
	Top					
	Second					1
-	Top					
4	Second					
	Тор		l			
3	Second					
	Tep					
•	Second					

FIG. X1.1 Bore Piston and Ring Measurement Data



## SEQUENCE VE

Engine No. _____ Engine Build No. _____ Date _____ Technician _____

## CYLINDER BORE, PISTON, AND COMPRESSION RING MEASUREMENTS

### **Cylinder Bore Messurements**

Cylinder Number	Location	Longitudinal Diameter (LD), mm	Transverse Diameter (TD), m m	Average Bore Diameter (ASD), 177 m TD e mid + TD e bot 2	Bore Out of Round, Mm [LD - TD] (0.03.5) max)	Bore Taper, mm [max 4] (0.0 3 8 mar.)	Microfinish, µ M
	Тор				<u> </u>		
1	Middle						
	Bottom						
	Top						
2	Middle						
	Bottom						
	Тор						
3	Middle						
	Bottom			i			
	Тор						
4	Middle						
	Bottom	I					

### Pleton Measurements

Cylinder Number	Range of Acceptable Piston Diameter, MM (ABD - 0.036)to(ABD - 0.056)	Diameter of Selected Piston, Mm	Taper of Selected Piston, Man
1			
2			
3			
4			

## **Compression Ring Gap Measurements**

Cylinder Number	Top Ring Gap, mm	Second Ring Gap, Iman	Top Ring-Land Side Clearance, yn m	Second Ring-Land Side Clearance, n1m
1				
2				
3				
4				

### Compression Ring Regap History and Weer Deta

Cylinder Number	Ring [—] Location	Ring Gep 	Ring Gep After Rework, mm	Ring Gap •, hrs, news	Ring Gap After Rework, mm	Ring Gap Increase Due to Wear, in in
	Top					
1	Second .					
	Тор					
2	Second		-			
	Тор		-			
3	Second					
	Тор					
4	Second					

FIG. X1.1 (M) Bore Piston and Ring Measurement Data (continued)

1

Engine N

Technician ____

#### SEQUENCE VE

Engine Build No. CRANKSHAFT, OIL PUMP, CONNECTING ROD, AND CAMSHAFT ENDPLAY MEASUREMENTS Camshaft No. Date .

**Crankshaft Measurements** 

		Main Bearin	ng Journais	
	Horizontal Diameter (HD)	Vertical Diameter (VD)	Out of Round	Fitted Bearing Clearance, in,
Journal Number	(HD), in.	(VD), in.	(VD - HD), in.	clearance, m.
	(2.3982 – 2.3990 in.) [60.91 – 60.94 mm]		(0.0006 in. max) [0.015 mm]	(0.0008 - 0.0026 in.) [0.020 - 0.066 mm]
1				
2	1			
3				
4				
5				

## Crankshaft Measurements Rod Rearing Journals

		nog beans	ig Journals	
	Horizontal Diameter (HD)	Vertical Diameter (VD)	Out of Round	Fitted Bearing Clearance, in,
Journal Number	(HD), in.	(VD), in.	(VD - HD), in.	Clearance, in.
		2.0472 in.) 52.00 mm]	(0.0006 in. max) [0.015 mm max]	(0.0008 - 0.0026 in.) [0.020 - 0.066 mm]
1				
2				
3		Γ		
4				

<b>Connecting Rod Oil Orifice Diameter</b>		Camshaft Endpla	iy
Connecting Rod Number	Oil Orifice Diameter (0.062 - 0.068 in.) [1.57 - 1.73 mm]	Endplay	
1		(0.001 - 0.007 in [0.03 - 0.18 mm	
2			
3			
4			

FIG. X1.2 Crankshaft Bearing Clearance, and Miscellaneous Measurement Data

	(1.4150	ft Lobe Height - 1.4228 in.) - 36.14 mm]	
Lobe Location	Height Before Test, (BH), in.	Height After Test, (AH), in.	Difference (BH - AH), in.
1É			
1)			
28			
2)			
3E			
31			
4E			
41			
	rage Differenc imum Differen		

Camshaft Lobe Oil Orifice Diameter (0.047 - 0.055 in.) [1.19 - 1.40 mm] Lobe Orifice Diameter 1E 11 2Ê 2) ЗE 31

> Camshaft Journal Oil Orifice Diameter (0.116 - 0.124 in.) [2.95 - 3.15 mm] Orifice Diameter Journal Front Rear

	Camshaft Oil D Groove Dimen	
		Groove Width (0.052 - 0.063 in.) [1.32 - 1.60 mm]
Front		
Rear		

FIG. X1.3 Camshaft Measurement Data

SEQUENCE VE CAMSHAFT LOBE AND ROCKER ARM HARDNESS

Engine No. __ Engine Build No. _ Camshaft No. ... Date Technician

Lobe Location	Lobe Hardness (>50 Rockwell C)	Rocker Arm Hardness (>57 Rockweil C)
1E		
11		
2E		
21		
3E		
31		
4E		
41		

Note 1-Camshafts with individual lobe hardness less than 50 Rockwell C may be used. Record lobe location(s) and hardness value(s) on the Supplemental Operational Data form, see Fig. A7.5.

FIG. X1.4 Hardness Measurement Data

### SEQUENCE VE

4E 41

CAMSHAFT DIMENSIONAL MEASUREMENTS

Camshaft No. Engine No. . Ne Build No. . Eng Date

Technician

115

Engine No. _____ Engine Build No. ____ Camshaft No. ____ Set No. ____ Date ____ Technician ____

### SEQUENCE VE

CONNECTING ROD BEARING AND ROCKER ARM WEIGHT

**Connecting Rod Bearing Weight** 

Connecting Rod Number	Bearing Location	Weight Before Test (WB), g	Weight After Test (WA), g	Weight Loss (WB – WA), mg	Total Weight Loss (Top + Bottom), mg
1	Тор				
•	Bottom				
,	Тор				
-	Bottom				
3	Тор				
	Bottom				
4	Тор				
-	Bottom				

### Rocker Arm Weight

Rocker Arm Location	Weight Before Test (BW), mg	Weight After Test (AW), mg	Difference (BW – AW), mg
16			
11			
2 <b>E</b>			
21			
38			
31			
4E			
41			
	Average Diffe Maximum Di	erence	_

FIG. X1.5 Weight Measurement Data

SEQUENCE VE

Engine No. ______ Engine Build No. _____ Head No. _____ Date _____ Technician _____

#### VALVE SPRING MEASUREMENTS

		Out of	Force	
	Square, in. (0.078 in. max) [1.98 mm max]	Uninstalled (167 ± 8 lbf @ 1.16 ± 0.03 in.) [75.7 ± 3.6 kgf @ 29.5 ± 0.8 mm]	installed (167 ± 8 lbf) [75.7 ± 3.6 kgf]	
16				
11				
2E				
21				
36				
31				
4E	1			
41				

### VALVE STEM TO GUIDE CLEARANCE

Valve Location	Valve Guide Diameter (GD), in.	Valve Stem Diameter (SD), in.	Stem to Guide Clearance (GD - SD), in. Exhaust (0.0019 - 0.0032 in.) [0.048 - 0.081 mm] Intake (0.0014 - 0.0027 in.) [0.035 - 0.088 mm]]
1E			
11			
2E			
21			
ЗE			
31			
46			
41			

FIG. X1.6 Cylinder Head Measurement Data



## **X2. SOURCES OF MATERIALS AND INFORMATION**

X2.1 The following sources are provided for convenience only. This does not represent an exclusive or complete listing of required materials or information sources.

X2.1.1 *ASTM Sequence VE Test Parts*—ASTM Sequence VE Test Parts Kits can be purchased through Ford Power Products distributors and Ford or Lincoln-Mercury dealers. The distributor or dealer should transmit orders to the address following:

Ford Parts and Service Division Power Products Operations 3000 Schaefer Road P.O. Box 6011 Dearborn, MI 48121

X2.1.2 *ASTM Test Monitoring Center*—All communications with the TMC should be directed as follows:

> ASTM Test Monitoring Center 4400 Fifth Ave. Pittsburgh, PA 15213

X2.1.3 *Test Sponsor*— All communications with the test sponsor (Ford Motor Company) should be directed as follows:

Ford Motor Co. EEE Bldg., Room D-145 (Box 44) 21500 Oakwood Blvd. Dearborn, MI 48121-2053

X2.1.4 *Aeroquip Hose and Fittings*—Aeroquip hose and fittings can be obtained from the following supplier:

Aeroquip Corp. 1225 W. Main Van Wert, OH 45891

X2.1.5 *Inspected Engine Parts*—Premeasured and calibrated Sequence VE engine parts and various component calibration devices are available from the following supplier:

Test Engineering Inc. 1530 Larkspur Drive San Antonio, TX 78213

X2.1.6 *Fuel Information*—General information concerning Phillips J fuel can be obtained from the following:

Phillips Petroleum Co. 3C3 Home Savings and Loan Building Bartlesville, OK 74004

X2.1.6.1 *Fuel Availability*—Phillips J Reference Gasoline is available from the following supplier:

Phillips Chemical Co. Speciality Chemicals P.O. Box 968 Borger, TX 79008-0968

X2.1.7 *Water-Cooled Exhaust Manifolds*—The exhaust manifold is available from the following supplier:

Edelbrock Inc. 2700 California St. Torrance, CA 90503

X2.1.8 *Engine Coolant Flowmeter*—Barco flowmeters for the engine coolant system (PN BR 12705-16-310) can be obtained from the following supplier:

Aeroquip Corp. 1225 W. Main Van Wert, OH 45891

X2.1.9 Intake-Air Humidity Instruments—The Alnor Dewpointer, EG & G, Foxboro, and Protimeter dewpoint meters (or equivalent) are suitable for measurement of the intake-air specific humidity.

X2.1.10 *Blowby Flow Rate Orifice*—Information concerning the blowby flow rate orifice meter is available from the following:

General Motors Research Laboratories Fuels and Lubricants Dept. 30500 Mound Road Warren, MI 48090-9055

X2.1.11 *Heat Exchangers*— ITT Standard and Ross Heat Exchangers can be obtained from the following supplier:

Kinetics Engineering Corp. 2055 Silber Road, Suite 101 Houston, TX 77055

X2.1.12 *Fuel Flow Measurement*—Mass fuel flowmeters are available from the following supplier:

Micro Motion Corp. 7070 Winchester Circle Boulder, CO 80301

X2.1.13 Various Materials—RAC kits, oil separators, camshaft baffles, oil filter adapters SB3 and EB3, and various other test stand parts and component calibration devices utilized in this test method are available from the following supplier:

> Bowden Manufacturing Corp. 4590 Beidler Road Willoughby, OH 44094

X2.1.14 *Fuel Management Systems*—Fuel management systems are available from the following supplier:

Intelligent Controls, Inc. 41000 Vincenti Court Novi, MI 48050

X2.1.15 *Exhaust Gas Analysis Instrumentation*—Exhaust gas analysis instrumentation can be obtained from the following suppliers:

X2.1.15.1 *CO Measurement (0 to 10 % CO)*—Beckman Model 864 or 868 is available from:

Rosemont 12001 W. Technology Drive Eden Prairie, MN 55344

X2.1.15.2 *CO Measurement (0 to 10 % CO)*—Horiba PIR 2000 is available from:

Horiba Inc. 1021 Duryea Irvine, CA 92714

X2.1.15.3  $O_2$  Measurement—Beckman Model 715 Process Oxygen Monitor is available from:

Rosemont 12001 W. Technology Drive Eden Prairie, MN 55344 X2.1.15.4  $O_2$  Measurement—Scott Oxygen Analyzer Model 250 is available from:

> Environmental Tectonics Corp. Countyline Industrial Park South Hampton, PA 18966

X2.1.15.5  $O_2$  Measurement—Teledyne 320 B/RC is available from:

Teledyne Analytical Instruments, Inc. 16830 Chestnut St. Industry, CA 91749

X2.1.15.6  $NO_x$  Measurement—Beckman Model 951 is available from:

Rosemont 12001 W. Technology Drive Eden Prairie, MN 55344

X2.1.15.7 UEGO Sensors are available from:

NGK Spark Plugs 27260 Haggerty Road Suite A-14 Farmington Hills, MI 41018

X2.1.16 *Exhaust Gas Analysis Calibration Gases*— Calibration gases for exhaust gas analysis equipment can be obtained from the following supplier:

Scott Environmental Technology, Inc. Route 611 Plumbsteadville, PA 18949

X2.1.17 *Crankcase and Intake-Air Pressure Gages*—Gages are available from the following supplier:

Dwyer Instrument Co. Junction of Indiana State Highway 212 and U.S. Highway 12 P.O. Box 373 Michigan City, IN 46360

X2.1.18 *Condensate Trap*— Meriam Model 932S condensate trap is available from the following supplier:

Meriam Instrument 10920 Madison Ave. Cleveland, OH 44102

X2.1.18.1 *Condensate Traps*—Norgren miniature filter/line traps have been found suitable and are available from the following supplier:

C. A. Norgren Co. 1 Norgren Plaza Littleton, CO 80120-1698

X2.1.19 *Cooling System Flushing Agent*—Formulation No. 7 Cooling System Cleanser—Heavy-Duty is available from a division of the following supplier:

Armorall Products Corp. Aliso Viejo, CA 92656

X2.1.20 *Lubricants*— EF-411 and Vacmul 3-D are available from local distributors of Mobil products.

X2.1.21 *Piston Ring Grinder*—A Sanford Piston Ring Grinder has been found satisfactory and is available from the following supplier (purchasers should specify the Ford 2.3 L engine application):

Sanford Manufacturing Co. 300 Cox Street P.O. Box 318 Roselle, NJ 07203

X2.1.22 *Hardness Tester*— A suitable hardness tester is available from the following supplier:

King Tester Corp. 510 Feheley Drive King of Prussia, PA 19406

X2.1.23 *Connecting Rod Heater*—The Sunnen Model CRH-50 connecting rod heater has been found suitable and is available from the following supplier:

Sunnen Inc. 7910 Manchester St. Louis, MO 63143

X2.1.24 *Tygon Hose*— Tygon hose is available through local Cadillac Plastic Company distributors or the following supplier:

The Norton Co. 12 East Avenue Tallmadge, OH 44278

X2.1.25 *Rating Lamps*— Suitable rating lamps are available from the following supplier:

Dazor Manufacturing Corp. 4455 Duncan Ave. St. Louis, MO 63110

X2.1.26 *Valve Springs*— Additional valve springs are available from the following supplier:

Associated Spring Co. 401 Stadium Drive Ann Arbor, MI 40810

X2.1.27 *Special Tools for the Test Engine*—Special tools to facilitate assembly and disassembly of the engine are available from the following supplier:

Owatonna Tool Co. 2013 4th St. NW Owatonna, MN 55060

X2.1.28 *Cylinder Head Stress Plate*—A stress plate, which in conjunction with the laboratory supplied stress plate (see Section 7.5.9 and Fig. A3.36) is designed to simulate the cylinder head, is available from the following supplier:

BHJ Products 37530 Enterprise Ct. Newark, CA 94560 (Reference No: 38413)

X2.1.29 *Remote Oil Filter Adapter*—A remote filter adapter, Part No. OHTA-007-1 is available from the following supplier:

OH Technologies P.O. Box 5039 Mentor, OH 44061-5039



## X3. ANALYSIS OF PHILLIPS J FUEL, BATCH 26

## X3.1 See Table X3.1.

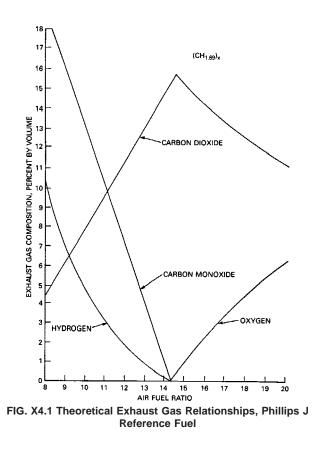
## TABLE X3.1 Analysis of Phillips J Fuel, Batch 26

API Gravity at 60°F (16°C)	54.6
Color	Undyed
Doctor test	Negative
Copper corrosion, 3 h at 212°F (100°C)	1-A
Reid vapor pressure, psig (kPa)	8.0 (55)
Research octane number	94.2
Motor octane number	83.0
Research + Motor/2[(R + M)/2]	88.6
Total sulfur, % weight	0.019
Gum, mg/100 mL	0.4
Oxidation Stability, min	1200 +
Lead, g/gal (mg/L)	<0.001 (<0.26)
Distillation, % Evaporation,° F (°C)	
IBP	98 (37)
10 %	130 (54)
50 %	225 (107)
90 %	335 (168)
End point	425 (218)
Recovery, %	98
PONA, % vol	
Paraffins plus Napthenes	43.5
Olefins	14.7
Aromatics	41.8



### X4. THEORETICAL EXHAUST GAS RELATIONSHIPS ) PHILLIPS J REFERENCE FUEL

X4.1 See Fig. X4.1.



### **X5. DESCRIPTION OF SCOTT QUARTERLY GAS AUDIT SERVICE**

X5.1 The Scott Quarterly Gas Audit Service is a subscription service to audit the performance of CO and  $O_2$  analysis instruments and is offered by Scott environmental Technology (see X2.1.16). The service is completed once every three months and utilizes span gases of CO and  $O_2$  in a balance of  $N_2$ . The exact concentration of the span gases is unknown to the customer, but falls within a specified range.

X5.2 The subscriber analyzes the span gas with the instrumentation used for VE testing. The gas is introduced through the calibration port and the sample inlet of the system. The laboratory submits the results to Scott, and Scott publishes a report detailing individual results and a statistical analysis

using data from all participating laboratories.

X5.3 Differences in results obtained at the sample port and the calibration inlet normally result from operating procedures, leaks in the sampling system, or improper design of the sampling system. Differences in results between a specific laboratory and the industry average usually indicate deficiencies in the analyzer or the quality of the calibration gases.

X5.4 Calibration gases traceable to national standards can also be utilized to verify the performance of exhaust gas analysis equipment.

ASTM International takes no position respecting the validity of any patent rights asserted in connection with any item mentioned in this standard. Users of this standard are expressly advised that determination of the validity of any such patent rights, and the risk of infringement of such rights, are entirely their own responsibility.

This standard is subject to revision at any time by the responsible technical committee and must be reviewed every five years and if not revised, either reapproved or withdrawn. Your comments are invited either for revision of this standard or for additional standards and should be addressed to ASTM International Headquarters. Your comments will receive careful consideration at a meeting of the responsible technical committee, which you may attend. If you feel that your comments have not received a fair hearing you should make your views known to the ASTM Committee on Standards, at the address shown below.

This standard is copyrighted by ASTM International, 100 Barr Harbor Drive, PO Box C700, West Conshohocken, PA 19428-2959, United States. Individual reprints (single or multiple copies) of this standard may be obtained by contacting ASTM at the above address or at 610-832-9585 (phone), 610-832-9555 (fax), or service@astm.org (e-mail); or through the ASTM website (www.astm.org).